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(54) **Automatic electronic wood cutting/chopping machine**

Automatische elektronische Holzschneid und -Spaltvorrichtung

Machine automatique électronique à couper et à fendre du bois

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**Description**

## FIELD OF THE INVENTION

**[0001]** This invention concerns an automatic electronic wood cutting/chopping machine as set forth in the preamble portion of claim 1. Such a device is known for example from US-A-4,373,564.

**[0002]** The machine according to an embodiment of the invention allows to automatically produce segments of firewood from tree trunks from which the branches have been removed.

**[0003]** To be more exact, the machine according to an embodiment of the invention automatically cuts the trunks first transversely so as to obtain sections of a desired length as a whole sub-multiple of the trunk, and then automatically chops the individual sections thus obtained in a lengthwise direction.

## BACKGROUND TO THE INVENTION

**[0004]** In the field of production of firewood it is well-known that an efficient and speedy machine is needed to produce firewood in segments the size of which depends on the use for which they are intended: for stoves, fireplaces, etc.

**[0005]** The present Applicants are not aware of efficient machines which carry out the steps of transverse cutting and lengthwise chopping in a completely automatic and sequential manner.

**[0006]** Machines such as are known in the state of the art have a limited production capacity because of the large number of manual operations required.

**[0007]** In a first step the trunks are cut transversely into sections of a desired length, normally with circular, strip or chain saws, and in a second step the sections are chopped lengthwise, usually in half or in quarters or eighths with a single stationary grid against which the section is thrust.

**[0008]** The wood produced is of extremely poor quality since the grid has no self-centering system with respect to the section to be chopped; moreover productivity is very low because of the long times required by the actuation cycles. As a consequence, the cost of the wood is very high inasmuch as it is considerably affected by the labour costs.

**[0009]** Moreover, some of the machines are not very safe for the worker because the trunks to be cut, or the sections to be chopped, have to be fed manually, or at most semiautomatically.

**[0010]** In order to increase productivity, reduce labour and the final cost of the wood, there have been proposals for semiautomatic machines; these, however, have not proved to be efficient and have shown limitations in their use and poor productivity. Machines known to the art, moreover, are complex to construct and to work, and therefore they are expensive, they need a great deal of maintenance and are subject to premature wear.

**[0011]** Furthermore, in such machines the cut sections are moved only partly automatically, and this necessarily requires the intervention of a worker.

**[0012]** In such machines, moreover, it is left entirely to the worker to decide the length of the sections, the number of parts obtained by chopping and the size of these parts; the worker uses his experience to divide the trunk into multiple sections of the overall length and then adapts the shape and the size of the chopping means according to the diameter of the sections.

**[0013]** For this reason, it is impossible to produce segments of a unified size, with a clean and well-defined profile; moreover, the final section often has to be discarded because it is too short.

**[0014]** The present Applicants have tested and embodied this invention to overcome all these shortcomings and to obtain further advantages.

## SUMMARY OF THE INVENTION

**[0015]** The invention is set forth in the characterising portion of claim 1, while the dependent claims describe variants of the idea of the main embodiment.

**[0016]** The purpose of the invention is to provide an entirely automatic wood cutting/chopping machine which is easy to use, highly efficient, productive, versatile and safe for the user.

**[0017]** Another purpose is to provide an automatic wood cutting/ chopping machine which is able to determine, automatically and in a standardised manner, the length of the sections into which the trunk is cut, and also the number and size of the parts obtainable by the chopping operation.

**[0018]** The machine according to an embodiment of the invention consists of an assembly to load the trunks, an assembly to feed the trunks, a gripper assembly, an assembly for cutting to size, a translation assembly to translate the sections and a chopping assembly.

**[0019]** According to a variant, downstream of the chopping assembly there is carrier assembly to separate the pieces of firewood produced.

**[0020]** The machine according to the invention is governed by an electronic control and drive unit which coordinates all these assemblies, the actuators and the auxiliary means such as the proximity sensors, end-of-travel means, accident prevention systems, optical means, detectors, etc.

**[0021]** In one embodiment, the actuators are oil dynamic so as to increase the versatility of the machine and to reduce the risks of accidents due to the presence of electric cables. The oil dynamic actuators are governed by a pressure control unit, controlled by the control and drive unit, with one or more pump means each feeding an autonomous circuit.

**[0022]** According to an embodiment of the invention, the loading assembly, for example, with a toothed chain, slide, sloping surfaces, rollers, etc., picks up the trunks one at a time from a stockpile zone and discharges

them, one by one, onto the feed assembly which moves them lengthwise towards the cutting assembly.

**[0023]** According to a variant, the loading assembly cooperates with at least a device suitable to temporarily retain the loaded trunk before it is discharged onto the feed assembly; it is thus possible to coordinate the movements of the two assemblies, and also to avoid downtimes while waiting for a new trunk to load. The feed assembly can be of any type whatsoever, with rollers, rails, conveyor belt, etc.;

**[0024]** According to a variant, the feed assembly has a sliding channel along which the trunks are thrust towards the cutting assembly by a thruster device; in another variant, the feed assembly cooperates with means to detect the presence of the trunk.

**[0025]** The machine according to the invention has means to automatically detect the length of the trunk at inlet. The value detected is processed by the control and drive unit, which coordinates the feed assembly and the cutting assembly to divide the trunk into a suitable number of sections each of a length which is a sub-multiple of the length of the trunk, so that no trailing end sections are fed which are too short to be used.

**[0026]** The cutting assembly performs a series of transverse cuts on the trunk fed by the feed assembly. In a preferential embodiment, the cutting assembly has at least a saw with a circular blade and cooperates with the gripper assembly which clamps the trunks in position during the cutting operation.

**[0027]** According to a variant, the circular blade cuts the trunks from the bottom to the top.

**[0028]** Each section of trunk produced by the cutting assembly is then translated by the translation assembly to a chopping assembly which has the function of chopping each of the sections into a suitable number of pieces or segments of firewood.

**[0029]** According to one embodiment, the translation assembly is a sloping surface; according to a variant it has thruster elements which at end-of-travel turn the section onto the chopping assembly.

**[0030]** According to another variant, the translation assembly comprises at least an element with a tray which can be moved and/or overturned. According to a further variant it has a plurality of tray elements which move in sequence into correspondence with the chopping assembly, carrying the cut sections; according to a variant, the tray elements are arranged on the circumference of a rotary disk.

**[0031]** In one embodiment, the chopping assembly has a thruster assembly suitable to progressively thrust the section of trunk against a chopping grid equipped with chopping blades.

**[0032]** The number and arrangement of the chopping blades determines the number and shape of the parts or segments into which each section is chopped.

**[0033]** According to the invention, the chopping grid is divided into several sectors, each of which is characterised by its own different arrangement, or pattern, of

the chopping blades.

**[0034]** The grid is movable so that one sector at a time is positioned in correspondence with the thruster assembly; this allows to vary the number and shape of the pieces of wood produced by every chopping operation, and also to choose the pattern of the blades most suitable for the diameter of the section of trunk.

**[0035]** According to a variant, the grid is governed by actuators which can be activated manually.

**[0036]** In one embodiment of the invention, the actuators are automatically driven by the control and drive unit according to the diameter of the section to be chopped; this allows the sector of the grid to be positioned in the most suitable position for the section to be chopped, and also allows the sector to be centered with respect to the section itself.

**[0037]** According to a variant, the means to detect the diameter are associated, or cooperate with, the gripper assembly.

**[0038]** According to an embodiment of the invention the grid is polygonal in shape and can be moved horizontally and/or vertically with respect to the thruster assembly.

**[0039]** According to a variant, the grid is circular, or similar in shape, and can rotate on its own center to position one sector at a time in correspondence with the thruster assembly.

#### BRIEF DESCRIPTION OF THE DRAWINGS

**[0040]** The attached Figures are given as a non-restrictive example and show a preferred embodiment of the invention as follows:

- 35 Fig. 1 shows in diagram form a part view from above of the wood cutting/chopping machine according to the invention;
- 40 Fig. 2a shows a section from A to A of Fig. 1;
- Fig. 2b shows an enlarged view from F of Fig. 1;
- Fig. 3a shows the section from B to B of Fig. 1;
- 45 Fig. 3b > shows the section from C to C of Fig. 1;
- Fig. 4a shows an enlarged view from D of Fig. 1;
- Fig. 4b is a side view of Fig. 4a;
- 50 Fig. 5a shows the section from E to E of Fig. 1;
- Fig. 5b is a front view of Fig. 5a;
- Fig. 5c shows the section from G to G of Fig. 1;
- 55 Fig. 6a shows the detail L of Fig. 4a;
- Fig. 6b shows a view from above of Fig. 6a;
- Fig. 7a shows a view from above of the

Fig. 7b enlarged detail H of Fig. 4a; shows the section from I to I of Fig. 7a;

Fig. 8a is a part view from above of a variant of Fig. 1;

Fig. 8b shows the section from L to L of Fig. 8a.

Figs. 9a and 9b show, respectively from above and below, a variant of Figs. 8a and 8b.

#### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

**[0041]** The wood cutting/chopping machine 10 according to an embodiment of the invention essentially consists of a loading assembly 11 for trunks 12, a feed assembly 13, a cutting assembly 14, a gripper assembly 29, a translation assembly 25 and a chopping assembly 15, arranged in sequence.

**[0042]** At outlet of the chopping assembly 15 there is a carrier assembly 16, in this case a conveyor belt, to transport the segments 49 produced.

**[0043]** In this case, (Figs. 2a-2b), the loading assembly 11 comprises a frame to support two pairs of toothed wheels 17 driven by respective chains 18; shaped brackets 19 are associated, at a pre-set distance, to the links of the chains 18, in this case six brackets 19 for each chain 18, to grip, raise and discharge the trunks 12.

**[0044]** In this case, the trunks 12 are conveyed from a stacking zone to the loading assembly 11 by means of a sloping surface 21.

**[0045]** The brackets 19 have a surface on which the trunks 12 rest consisting of a hook-shaped profile which has one part higher towards the chain 18; it is thus possible to prevent the trunks 12 knocking against the chain 18 as they roll on the sloping surface 21, and also to support trunks 12 of any diameter whatsoever.

**[0046]** The loading assembly 11 discharges the raised trunks 12 one by one onto the sliding channel 20 of the feed assembly 13.

**[0047]** In this case, between the loading assembly 11 and the feed assembly 13 there is an assembly 58 which temporarily retains the trunk 12 until the trolley 22 returns; the said trolley 22 feeds the trunks along the sliding channel 20; afterwards, the trunks 12 may be discharged onto the sliding channel 20 of the feed assembly 13. The cycle times of the feed assembly 13 are thus not bound to those of the loading assembly 11, which can begin to raise a trunk 12 irrespective of the position of the trolley 22.

**[0048]** The assembly 58 comprises a sloping surface 59 cooperating with a pair of arms 60 which move alternately from a position of interference 60a with the trunk 12 to a position of non-interference 60b which allows the trunk 12 to roll onto the sliding channel 20.

**[0049]** This happens when the trolley 22 which moves in the sliding channel 20 is in its return end-of-travel po-

sition.

**[0050]** The sliding channel 20 is V-shaped in section, so as to contain the trunks 12 laterally.

**[0051]** The driven trolley 22 travels along a rail 23, for example a rack rail, moving from a starting position 22a, at the beginning of the sliding channel 20, to an end position 22b at the end of the sliding channel 20.

**[0052]** In this case, the beginning and end of the sliding channel 20 coincide respectively with the end farthest from and nearest to the cutting assembly 14. In this case, moreover, the driven trolley 22 includes thrust-er means 24, an oil dynamic piston in this case, to vary the distance of the end of the trunk 12 from the body of the trolley 22.

**[0053]** As can be seen in Fig. 3b, the thruster means 24 may move to any position between a retracted position 24a and a position of maximum extension 24b; it is thus possible, as described later, to thrust, locate and center onto the translation assembly 25 also the last section 12 cut.

**[0054]** In the embodiment shown here, moreover, in order to facilitate the centering operation, there are auxiliary thrust means 47, in this case of the same type as the means 24 and opposite thereto (Fig. 1).

**[0055]** The cutting assembly 14, shown in detail in Figs. 4a-4b, has a trolley 55 to support at least one saw 27, in this case with a circular blade 28, driven by means of a transmission belt 62, by an electric motor 61. The belt 62 drives a pulley 63 which makes the blade 28 rotate by means of a rotary shaft 64.

**[0056]** The supporting trolley 55 is equipped with ascending/ descending motion, in this case driven by a vertical oil dynamic actuator 54, so that the circular blade 28 moves to at least two extreme positions, respectively the lowered position, or position of non-interference 28a, and the raised or cutting position 28b.

**[0057]** In this embodiment, as can be seen in Fig. 4a, the blade 28 cuts the trunks from the bottom upwards.

**[0058]** The supporting trolley 55 can be removed from the frame of the cutting assembly 14; this facilitates maintenance of the saw 27. The combined trolley 55-motor 61-saw 27 is assembled on another trolley 65 which is able to move transversely with respect to the structure which contains it, to facilitate removal.

**[0059]** The ascending/descending movements of the trolley 55 are correlated to the advance of the trolley 22.

**[0060]** To be more exact, the trolley 22 is made to advance at least from a pre-determined point of the sliding channel 20 onwards, according to a pitch which defines the length of the individual sections into which the trunk 12 is divided; the sections correspond to a whole sub-multiple of the overall length of the trunk 12.

**[0061]** The pitches are calculated by an electronic control and drive unit 26, for example with programmable logic or similar.

**[0062]** In this case, the machine 10 has monitoring means, for example optical monitoring means, to monitor the presence of the trunk 12 loaded onto the feed

assembly 13, and also means to measure the length of the said trunk 12.

**[0063]** In one formulation, the means to measure the length of the trunk 12 comprise an encoder which detects the space covered by the trolley 22 from the starting position 22a until it stops, which is when the leading end of the trunk 12 reaches the proximity of a sensor, such as a photocell, located in correspondence with the beginning of the cutting assembly 14 or the end of the sliding channel 20. The control and drive unit 26 processes the data supplied by the encoder and, knowing the length of the sliding channel 20, calculates the length of the trunk 12 to be cut.

**[0064]** The machine 10 thus makes possible to divide the trunk 12 into sections of equal length and above all to prevent sections of the trailing end from advancing which would have to be discarded.

**[0065]** In this case, during the transverse cutting operation, the trunks 12 are clamped in position by two gripper assemblies 29 consisting of pantograph arms comprising a pair of upper arms 30a pivoted in correspondence with the end facing upwards and constrained to the top of the frame of the cutting assembly 14.

**[0066]** The end of the upper arms 30a facing downwards pivots on pins 34 at the upper end of respective lower arms 30b.

**[0067]** The lower arms 30b in turn pivot on each other by means of a pin 31, substantially in correspondence with the center line.

**[0068]** The pin 31 is able to slide at one end in a vertical guide 32 along which it can slide upwards or downwards in response to, respectively, the actions of traction or of thrust exerted by an oil dynamic actuator 33 on one only of the two pins 34. When the oil dynamic actuator 33 is retracted, the lower ends of the lower arms 30b are lowered, gripping and clamping the trunk 12 in the cutting position.

**[0069]** When the oil dynamic actuator 33 is extended, the lower ends of the arms 30b are lifted into the position of non-interference with the trunk 12.

**[0070]** When the trunk 12 is clamped, whatever its diameter, the gripper assembly 29 has its lower arms 30b positioned at 120° with respect to the plane on which the trunk 12 rests.

**[0071]** This allows the control and drive unit 26 to determine the diameter of the trunk 12 by means of algorithms based only on the measurement of the travel made by the oil dynamic actuator 33.

**[0072]** Due to the fact that the gripper assembly 29 monitors the diameter of the trunk 12 before the trunk 12 is cut, it is possible to establish the travel of the oil dynamic actuator 54 to move the supporting trolley 55 of the saw 27. Since the diameter is monitored for every section of trunk 12 cut, it is possible, as described later, to position and center the grid 43 of the chopping assembly 15 correctly, in order to obtain a number of segments proportional to the diameter and to center the

center of the grid 43 with respect to the center of the section.

**[0073]** When the blade 28 has cut the trunk 12 in a transverse direction, a section of trunk 12 is positioned on the translation assembly 25, in this case comprising two tray elements 35 driven by respective oil dynamic actuators 36 and aligned with the sliding channel 20.

**[0074]** On a first lower face, facing towards the chopping assembly 15, the tray elements 35, in this case V-shaped, have wheels 37 running on guides 38, and on the opposite lower face they are associated with the piston of the oil dynamic actuator 36.

**[0075]** When the trunk 12 is in the cutting position the piston of the actuator 36 is retracted and the tray element 35 is in the cutting position 35a.

**[0076]** When cutting has been completed, the piston of the oil dynamic actuator 36 is extended so that the tray element 35, sliding on guides 38, progressively moves towards the chopping assembly 15 until it reaches an end-of-travel and turnover position 35b.

**[0077]** In this end-of-travel and turnover position 35b, the wheels 37 cooperate with a sloping surface 39 which causes the tray elements 35 to partly turn over forwards, and therefore the cut section of trunk 12 to turn over onto the tray element 40 of the chopping assembly 15.

**[0078]** By using two tray elements 35, it is possible to increase the productivity of the machine 10 particularly when producing pieces 49 of firewood of limited length, for example of the type for stoves 20÷25 cm long.

**[0079]** In this case, after the cutting operation, the section 12 occupies only the tray element 35 nearest the trolley 22, while the other tray element 35 is free; this allows the thruster means 24 to move to a condition of maximum extension 24b to push the already cut section 12 onto the tray element 35 farthest from the trolley 22.

**[0080]** At this point the farthest tray element 35 is activated to turn over the section of trunk 12 onto the tray element 40 of the chopping assembly 15 while a new section is cut onto the tray element 35 nearest the trolley 22.

**[0081]** When turnover is completed, the farthest tray element 35 returns in position to receive the new section of trunk 12, and the cycle is repeated.

**[0082]** In this case, the tray element 40 hinges on the side of the thruster element 42 of the chopping assembly 15 and oscillates downwards at the front part contrasted by a spring 41 which allows it to respond elastically to the stresses to which it is subjected when the section of trunk 12 is chopped; this oscillation, moreover, facilitates the section to open, rose-like, and the lower segments to run out along the grain caused by the chopping action.

**[0083]** The chopping assembly 15 consists of a frame to support the tray element 40, the thruster element 42 driven by an oil dynamic piston 56, associated by means of rods to the rear end of four cross pieces 57, of which the two upper ones act as guide means for the thruster

element 42, and at least a chopping grid 43 mounted on a grid-bearing frame.

**[0084]** In this case, the grid-bearing frame consists of two platbands, upper 66a and lower 66b, separated in height by three vertical bars which are aligned with the vertical chopping blades 48 of each chopping grid 43. At the side of each chopping blade 48 there are two tubular spacers 53 (the intermediate one is shared by two adjacent grids 43), also attached vertically to the two platbands 66a, 66b.

**[0085]** The tubular spacers 53 cooperate so that the grid 43 does not rotate during the chopping step.

**[0086]** The thrust exerted on the grid-bearing frame is contrasted by two small cross pieces, the upper 67a and the lower 67b, attached to the rods 68 which connect the two heads by means of bolts 69.

**[0087]** In this case, the chopping grid 43 is rectangular with its long sides parallel to the plane which supports the machine 10 and cooperates both above and below with wheels 44 located on tubular frames 70 located both above and below the relative platbands 66a, 66b.

**[0088]** The grid-bearing frame is associated with an oil dynamic actuator 45, governed by the control and drive unit 26, which allows the chopping grid 43 to be displaced to the left and to the right in a controlled manner. According to a variant, for grids which run vertically, the grid-bearing frame is moved by a driven pinion; in this variant, which is not shown here, the chopping grid 43 has its short sides parallel to the plane supporting the machine 10 and translates vertically.

**[0089]** According to a further variant, the chopping grid 43 is circular or similar, and can rotate around its center.

**[0090]** The translation or, possibly, rotation movements of the chopping grid 43 make possible to vary the size, number and/or shape of the segments 49 obtained according to necessity, and to center the grid with respect to the section of trunk 12.

**[0091]** In this case, the chopping grid 43 is divided into three sectors, respectively left 43a, central 43b and right 43c, each one characterised by a different arrangement of the chopping blades 48.

**[0092]** To be more exact, the right sector 43c is used to chop sections of trunk 12 of limited diameter, and the chopping blades 48 are arranged in a cross to divide the sections of trunk 12 into quarters.

**[0093]** In the right sector, moreover, the chopping blade 48 arranged horizontally is movable upwards or downwards, which allows to move it to a position of non-interference to divide the section of trunk 12 in half.

**[0094]** The central sector 43b is used to chop sections of trunk 12 with medium diameters, and the chopping blades 48 are arranged in a dial to allow the sections of trunk 12 to be divided into six parts.

**[0095]** The left sector 43c is used to chop sections of trunk 12 with a large diameter, and the chopping blades 48 are arranged in a dial to divide the sections into eight parts.

**[0096]** According to an embodiment of the invention, when one of the sectors 43a, 43b or 43c has been positioned opposite the tray element 40 supporting the section of trunk 12 to be chopped, the thruster element 42 moves progressively from a start-of-chopping position 42a to an end-of-chopping position 42b, progressively pushing the section of trunk 12 towards the chopping blades 48.

**[0097]** The positioning of a specific sector 43a, 43b or 43c opposite the tray element 40 is automatically managed by the control and drive unit 26; according to a variant, this positioning is commanded by the operator.

**[0098]** The automatic positioning of a specific sector 43a, 43b or 43c opposite the tray element 40, and the centering of the point of intersection of the chopping blades 48 with the center of the section to be chopped is made possible, as already described, by the means to measure the diameter, associated with the gripper assembly 29.

**[0099]** To be more exact, the data supplied by the diameter measuring means is processed by the control and drive unit 26 which activates the oil dynamic actuator 45, or the movement means otherwise provided, to locate the sector 43a, 43b or 43c most suitable for the diameter of the trunk 12 being worked.

**[0100]** Moreover, the control and drive unit 26 centers the center of the grid with respect to the center of the section to be chopped.

**[0101]** As can be seen from Figs. 5a, 5b, there is an oil dynamic actuator 50 associated, by means of a grooved bar 51 crossing above the grid 43, with the chopping blades 48 of the chopping grid 43; the chopping blades 48 are associated with the chopping grid 43 by means of holed fork means 52 which slide along the tubular spacers 53 of the chopping grid 43.

**[0102]** The grooved bar 51 can rotate by means of a pinion and rack 71 to drive the oil dynamic actuator 50, which allows to move vertically the grid-bearing frame so as to carry out this centering.

**[0103]** In this case, the pieces 49 of chopped firewood fall onto the carrier assembly 16 which conveys them to a storage zone or directly into the containing body of a truck.

**[0104]** The variant shown in Figs. 8a-8b, 9a, 9b is used in the machine 10 as an alternative to the translation assembly 25 to move the sections of trunks 12 cut to size from the cutting assembly 14 to the chopping assembly 15.

**[0105]** This embodiment, which considerably increases the productivity of the machine 10, includes a rotary disk 72 underneath the chopping assembly 15.

**[0106]** At the ends of two orthogonal diameters, the rotary disk 72 has respective tray elements 46, in this case four arranged at 90° to each other, cushioned like the tray element 40 as described above, which in this case is absent inasmuch as the trays 46 perform both the function of the tray 35 of the translation assembly 25 and also the function of the tray 40 of the chopping

assembly 15.

[0107] In this embodiment, in fact, the section of trunk 12 is supported, both during the cutting operation and during the chopping operation, by the individual tray elements 46 which, following the rotations of the disk 72, move one by one opposite the thruster element 42, transporting one section. With this variant, the feed assembly 13 and the cutting assembly 14 are rotated in a clockwise direction by 90° with respect to Fig. 1, which allows the section cut by the cutting assembly 14 to be arranged lengthwise to the tray element 46 and to move in correspondence with the thruster element 42 with the fibres orthogonal with respect to the chopping blades 48.

[0108] As can be seen in Fig. 9a, the trunk 12 is fed by the feed assembly 13 which is not shown here and arrives, moved at pitch, from the direction of the arrow A; it is positioned with its front end, calculated to size, on the tray 46 in position I.

[0109] Then, the saw 27 acts on the trunk 12 and cuts the section located on the tray 46; then the rotary disk 72 rotates by 90° and positions the cut section in correspondence with the position indicated as II.

[0110] This position II of the tray 46 is aligned with the axis of the thruster element 42 and with the chopping grids 43; when the thruster element 42 is activated, this causes the section to be chopped into pieces 49.

[0111] When one section has been chopped, the thruster element 42 retracts and a new section, cut by the saw 27 during the previous chopping cycle, is taken to the chopping position by the rotation of the disk 72.

[0112] The tray 46, in the chopping position II, cooperates at the lower part with a shock absorber element consisting of a support 75 associated with spring means 41.

[0113] The support 75, elastically yielding to the pressure exerted by the section of trunk 12 during the chopping action, facilitates and makes the chopping action qualitatively better, preventing the chopping blades of the grid 43 from jamming against the cutting surfaces, and ruining them.

[0114] In this case, as can be seen in Fig. 9a, the center of rotation 73 of the rotary disk 72 is displaced sideways with respect to the cutting axis 74 of the saw 27, which coincides with the axis of arrival of the trunk which is to be cut as fed by the feed assembly 13; this prevents interference, during the rotation of the disk 72, between the front end of the trays 46 and elements of the cutting assembly 14 or other structural elements of the machine 10.

## Claims

1. Automatic electronic wood cutting/chopping machine (10) for producing pieces or segments (49) of firewood from branchless trunks (12), being collected in turn from a stacking zone, said machine com-

prises, in coordinated cooperation, a loading assembly (11) to automatically pick up the trunks (12) from the stacking zone and to deposit them on a lengthwise feed assembly (13) with a sliding channel (20), a cutting assembly (14) located to follow the lengthwise feed assembly (13) to divide the trunks (12) into sections of pre-determined length, **characterised by** a gripper assembly (29) located in cooperation with the cutting assembly (14) to temporarily retain the trunks (12) during the cutting operation and to measure the diameter of the trunks (12), a translation assembly (25) located immediately downstream of the cutting assembly (14) to translate the cut sections of trunk (12), a chopping assembly (15), coplanar and substantially adjacent to the feed assembly (13), to divide the cut sections of trunk into a defined number of pieces or segments (49) and a carrier assembly (16) located immediately downstream of the chopping assembly (15) to discharge and/or separate the pieces or segments (49) produced, the assemblies (11,13,14,15,16,25, 29) being governed by an electronic control and drive unit (26) associated at least with means to automatically determine the length of the trunks (12).

2. Machine as in Claim 1, **characterised in that** the chopping assembly (15) comprises a thruster element (42) cooperating at the lower part with a tray element (40, 46) to support the section of trunk (12) to be chopped and at the front with a chopping grid (43) with multiple sectors (43a,43b,43c), each sector (43a,43b,43c) of the grid (43) having relative chopping blades (48) defining a different arrangement or pattern.
3. Machine as in Claim 2, **characterised in that** the grid (43) is associated with a grid-bearing frame movable on the plane on which the thruster element (42) lies.
4. Machine as in Claim 2, **characterised in that** the grid (43) is associated with a grid-bearing frame movable orthogonally to the plane on which the thruster element (42) lies.
5. Machine as in any claim hereinbefore, **characterised in that** the grid-bearing frame is associated with positioning means (45) governed by the electronic control and drive unit (26) according to the diameter of the section of trunk (12) to be chopped and/or of the sector (43a,43b,43c) to be positioned.
6. Machine as in any claim hereinbefore, **characterised in that** the grid-bearing frame is associated with position adjustment means (50, 51) according to the collimation of the center of the sector (43a, 43b,43c) with respect to the center of the section of

trunk (12).

7. Machine as in any claim hereinbefore, **characterised in that** the grid-bearing frame comprises two platbands (66a, 66b), parallel to the cutting plane, solidly constrained to the chopping blades (48) and to the tubular spacers (57) transverse thereto, the grid-bearing frame also comprising small cross pieces (67a, 67b) solid with the rods (68) which connect the heads of the machine.
8. Machine as in any claim hereinbefore, **characterised in that** the tray element (40, 46) is able to oscillate at the front according to the stresses imparted by the section of trunk (12) during the chopping operation.
9. Machine as in Claim 8, **characterised in that** the tray element (40, 46) is hinged at the lower part on the side facing the thruster element (42) and cooperates with elastic contrasting means (41) on the opposite side.
10. Machine as in Claim 1, **characterised in that** the feed assembly (13) is movable lengthwise at pitch, each pitch being functional to a sub-multiple of the length of the trunk (12).
11. Machine as in Claim 10, **characterised in that** the feed assembly (13) comprises a sliding channel (20) for the trunk (12) cooperating with thruster means (24) associated with a trolley (22), the trolley (22) including a start position (22a) cooperating with the beginning of the sliding channel (20) and an end position (22b) cooperating with the end of the sliding channel (20).
12. Machine as in Claim 11, **characterised in that** the sliding channel (20) comprises means to automatically determine the length of the trunk (12).
13. Machine as in Claim 11 or 12, **characterised in that** the thruster means (24) can be moved so as to center the cut sections of trunk (12) on the translation assembly (25).
14. Machine as in Claim 1, **characterised in that** the gripper assembly (29) includes means to measure the diameter of the cut section.
15. Machine as in Claim 14, **characterised in that** the gripper assembly (29) comprises pantograph arms (30a, 30b), a first upper pair (30a) of the arms having their upper ends pivoted at a same point at the top of the frame of the cutting assembly (14) and the lower ends pivoted with pins (34) at the upper end of respective lower arms, the lower arms (30b) being in turn pivoted together on a pin (31) substantially in correspondence with the center line, the pin (31) being able to slide along a vertical guide (32) in response to the actions of traction or thrust exerted by an actuator (33) acting on one only of the two pins (34).
16. Machine as in Claim 15, **characterised in that** the lower ends of the lower arms (30b) have a first lowered position wherein the trunk (12) is clamped in the cutting position associated with the retracted position of the actuator (33) and a second raised position of non-interference associated with the extended position of the actuator (33).
17. Machine as in any claim from 14 to 16 inclusive, **characterised in that** the means to measure the diameter of the section comprise an actuator to determine the travel made by the actuator (33).
18. Machine as in any claim from 14 to 17 inclusive, **characterised in that** the lower arms (30b), whatever the diameter of the trunk (12), are always arranged at 120° with respect to the plane on which the trunk (12) lies.
19. Machine as in Claim 1, **characterised in that** the cutting assembly (14) has at least a saw (27) with a circular blade (28) associated with a supporting trolley (55) movable vertically and defining, for the blade (28), a first lowered position (28a) of non-interference and a second, raised position (28a) for cutting.
20. Machine as in Claim 19, **characterised in that** the supporting trolley (55) is driven by actuators (54) which are able to move according to the diameter of the trunk (12).
21. Machine as in Claim 19, **characterised in that** the supporting trolley (55) can be removed from the bearing structure of the machine (10).
22. Machine as in Claim 1, **characterised in that** the translation assembly (25) comprises tray elements (35, 46).
23. Machine as in Claim 22, **characterised in that** the tray element (35) is associated at one end with an actuator (36) and has a cutting position (35a) cooperating with the cutting assembly (14) and an end-of-travel and turnover position (35b) wherein it discharges the cut section (12) onto the chopping assembly (15).
24. Machine as in Claim 22, **characterised in that** there are two tray elements (35), a first tray element (35) supporting one section of trunk (12) still to be cut and a second tray element (35) discharging the



section which has already been cut on the first tray element (35) and then thrust therefrom onto the second tray element (35).

25. Machine as in Claim 22, **characterised in that** the tray elements (46) are associated with a rotary disk (72). 5
26. Machine as in Claim 25, **characterised in that** the rotary disk (72) is suitable to move a tray (46) from a first position (I) aligned with the lengthwise feed assembly (13) wherein the trunk to be cut is received, to a second position (II) aligned with the axis of the chopping grid (43) wherein the section of cut trunk (12) is chopped. 10
27. Machine as in Claim 25 or 26, **characterised in that** the rotary disk (72) rotates around an axis of rotation (73) laterally displaced with respect to the axis of feed (74) of the feed assembly (13). 15
28. Machine as in Claim 1, **characterised in that** the loading assembly (11) has pairs of driven toothed wheels (17) driving chains (18) having links to which are associated bracket elements (19) to support and raise the trunks (12) to a desired height. 20
29. Machine as in Claim 28, **characterised in that** the bracket elements (19) have a hook-shaped profile. 25
30. Machine as in Claim 28, **characterised in that** upstream of the loading assembly (11) there is a sloping surface (21) over which the trunks (12) arriving from the stacking zone are rolled. 30
31. Machine as in Claim 28, **characterised in that** downstream of the loading assembly (11) there is an assembly (58) to temporarily retain the raised trunks (12) before they are discharged onto the feed assembly (13). 35

#### Patentansprüche

1. Automatischer elektronischer Holzschneider und -spalter (10) für die Herstellung von Brennholzstücken oder -scheiten (49) unter Anwendung astloser Holzstämmen (12), die aus einem Aufschichtungsbereich entnommen werden. Die betreffende Maschine setzt sich aus den folgenden funktionskoordinierten Einheiten zusammen und zwar im Einzelnen aus: einer Beschickungseinheit (11), durch die die Holzstämmen (12) aus dem Aufschichtungsbereich entnommen und auf eine in Längsrichtung angeordnete Vorschubeinheit (13) mit Gleitkanal (20) abgelegt werden; einer sich hinter der in Längsrichtung angeordneten Vorschubeinheit (13) befindlichen Schneideinheit (14) für das Trennen der Holz-

stämmen (12) in Segmente auf eine zuvor festgelegte Länge, die mit einer Zangeneinheit (29) versehen ist, die mit der Schneideinheit (14) für das vorübergehende Festhalten der Holzstämmen (12) während des Schnitts und für die Messung des Durchmessers der Holzstämmen (12) funktionskoordiniert ist; einer sich unmittelbar hinter der Schneideinheit (14) befindlichen Einheit (25) für die Beförderung der geschnittenen Holzstammsegmente (12); einer komplanaren und an die Vorschubeinheit (13) angrenzenden Spalteinheit (15) für die Verteilung der Holzstammsegmente, die in eine zuvor festgelegte Menge von Stücken oder Scheiten (49) geschnitten wurden und einer sich unmittelbar hinter der Spalteinheit (15) befindlichen Beförderungseinheit (16) für das Abführen u/o die Sortierung der auf diese Weise hergestellten Stücke oder Scheite, wobei diese Einheiten (11,13,14,15,16,25,29) von einer elektronischen Steuer- und Antriebseinheit (26) gesteuert werden, die wiederum die Einrichtungen für die automatische Bestimmung der Länge der Holzstämmen (12) steuert.

2. Maschine gemäß Anspruch 1, **dadurch gekennzeichnet, dass** die Spalteinheit (15) mit einem Schieber (42) ausgestattet ist, der im unteren Bereich mit einem wannenförmigen Behälter (40, 46) für die Abstützung des zu spaltenden Holzstammsegmentes (12) und im vorderen Bereich mit einem Spaltrost (43) mit Vielfachsektoren (43a,43b,43c) funktionskoordiniert ist, wobei ein jeder Sektor (43a,43b,43c) des Rostes (43) mit Trennmessern (48) versehen ist, die eine unterschiedliche Anordnung oder Formgebung haben. 30
3. Maschine gemäß Anspruch 2, **dadurch gekennzeichnet, dass** der Rost (43) an einem sich auf der Ablagefläche des Schiebers (42) montierten beweglichen Rosthalterrahmen befestigt ist. 35
4. Maschine gemäß Anspruch 2, **dadurch gekennzeichnet, dass** der Rost (43) an einem sich rechtwinklig zur Ablagefläche des Schiebers (42) befindlichen beweglichen Rosthalterrahmen befestigt ist. 40
5. Maschine gemäß einem der zuvor genannten Ansprüche, **dadurch gekennzeichnet, dass** der Rosthalterrahmen an Einrichtungen für Einstellung der Position (45) befestigt ist, die anhand des Durchmessers des zu spaltenden Holzstammes (12) u/o des zu positionierenden Sektors (43a,43b,43c) von einer elektronischen Steuer- und Antriebseinheit (26) gesteuert werden. 45
6. Maschine gemäß einem der zuvor genannten Ansprüche, **dadurch gekennzeichnet, dass** der Rosthalterrahmen in Abhängigkeit der Kollimation der Mitte des Sektors (43a,43b,43c) hinsichtlich der

Mitte des Holzstammsegments (12) an den Einrichtungen für Einstellung der Position (50, 51) befestigt ist.

7. Maschine gemäß einem der zuvor genannten Ansprüche, **dadurch gekennzeichnet, dass** der Rosthalterahmen mit zwei sich parallel zur Schneidebene befindlichen Gurtplatten (66a, 66b) versehen ist, die fest mit den Trennmessern (48) und mit den zu diesen quer angeordneten röhrenförmigen Distanzstücken (57) verbunden sind. Zudem ist der Rosthalterahmen mit Querstreben (67a, 67b) versehen, die fest mit den sich zwischen den Maschinenköpfen befindlichen Verbindungszugstangen (68) verbunden sind. 5
8. Maschine gemäß einem der zuvor genannten Ansprüche, **dadurch gekennzeichnet, dass** das wannenförmige Element (40, 46) in Abhängigkeit der vom Holzstammsegment (12) während des Spaltens erzeugten Beanspruchungen nach vorne schwingen kann. 10
9. Maschine gemäß Anspruch 8, **dadurch gekennzeichnet, dass** das wannenförmige Element (40, 46) im unteren Bereich auf der zum Schieber (42) hin zeigenden Seite drehbar gelagert ist und auf der gegenüberliegenden Seite mit elastischen Kontrasteinrichtungen (41) funktionskoordiniert ist. 15
10. Maschine gemäß Anspruch 1, **dadurch gekennzeichnet, dass** sich die Vorschubeinheit (13) auf der Längsseite schrittweise bewegt, wobei ein jeder Schritt in Abhängigkeit einer Teilung der Länge des Holzstamms (12) erfolgt. 20
11. Maschine gemäß Anspruch 10, **dadurch gekennzeichnet, dass** die Vorschubeinheit (13) mit einer für die Beförderung der Holzstämmen (12) bestimmten Rinne (20) versehen ist, die mit den Schiebern (24) funktionskoordiniert ist, die wiederum an einem Fahrgestell (22) befestigt sind, das sowohl über eine Ausgangsposition (22a), die mit dem Einlauf der Gleitrinne (20) funktionskoordiniert ist, als auch über eine Endposition (22b) verfügt, die mit dem Auslauf der Gleitrinne (20) funktionskoordiniert ist. 25
12. Maschine gemäß Anspruch 11, **dadurch gekennzeichnet, dass** die Gleitrinne (20) mit zwei Einrichtungen für die automatische Bestimmung der Länge des Holzstamms (12) ausgestattet ist. 30
13. Maschine gemäß Anspruch 11 oder 12, **dadurch gekennzeichnet, dass** die Schieber (24) für die Zentrierung der auf der Beförderungseinheit (25) geschnittenen Holzstammsegmente (12) beweglich sind. 35

14. Maschine gemäß Anspruch 1, **dadurch gekennzeichnet, dass** die Zangeneinheit (29) mit Einrichtungen für die Messung des Durchmessers des geschnittenen Segments ausgestattet sind. 40

15. Maschine gemäß Anspruch 14, **dadurch gekennzeichnet, dass** die Zangeneinheit (29) mit Gelenkarmen (30a, 30b) ausgestattet ist, wobei die oberen Enden der zwei oberen Gelenkarme (30a) an derselben Stelle am oberen Rahmen der Schneideinheit (14) verbolzt sind und die unteren Enden (30b) mit Zapfen (34) am oberen Ende der entsprechenden unteren Gelenkarme (30b) versehen sind, wobei die unteren Gelenkarme (30b) ihrerseits mit einem Zapfen (31) in Übereinstimmung mit der Mittellinie verbolzt sind und der Zapfen (31) sich längs einer vertikalen Führungsschiene (32) anhand des Antriebs oder des Schubs, der auf einen der zwei Zapfen (34) durch den Stellantrieb ausgeübt wird, bewegt. 45

16. Maschine gemäß Anspruch 15, **dadurch gekennzeichnet, dass** die unteren Gelenkarme (30b) im unteren Bereich über eine erste untere Position für das Festhalten des Holzstamms (12) in der Schneidposition verfügen, die der eingefahrenen Position des Stellantriebs (33) entspricht, und über eine zweite untere Position, zur Vermeidung von Kollisionen, verfügen, die der Ausfahrposition des Stellantriebs (33) entspricht. 50

17. Maschine gemäß einer der Ansprüche von 14 bis 16, **dadurch gekennzeichnet, dass** die Einrichtungen für die Messung des Segmentdurchmessers mit einem Stellantrieb für die Bestimmung des vom Stellantrieb (33) selbst durchgeführten Hubs ausgestattet sind. 55

18. Maschine gemäß einer der Ansprüche von 14 bis 17, **dadurch gekennzeichnet, dass** sich die unteren Gelenkarme (30b), gleich welchem Durchmesser der Holzstamm (12) aufweist, hinsichtlich der Ablagefläche des Holzstamms (12) stets bei einem Winkel von 120° positionieren. 60

19. Maschine gemäß Anspruch 1, **dadurch gekennzeichnet, dass** die Schneideinheit (14) mit mindestens einer Kreissäge (27/28) ausgestattet ist, die an einem vertikal verfahrbaren Schlitten (55) befestigt ist, und für das Sägeblatt (28) eine erste untere Position (28a) zur Vermeidung einer Kollision und eine zweite obere Position für den Schnitt (28b) bestimmt. 65

20. Maschine gemäß Anspruch 19, **dadurch gekennzeichnet, dass** der Schlitten (55) in Abhängigkeit des Durchmessers des Holzstammes (12) von verfahrbaren Stellantrieben (54) angetrieben wird. 70

21. Maschine gemäß Anspruch 19, **dadurch gekennzeichnet, dass** der Schlitten (55) aus dem Tragwerk der Maschine (10) entfernt werden kann.
22. Maschine gemäß Anspruch 1, **dadurch gekennzeichnet, dass** die Beförderungseinheit (25) mit wannenförmigen Elementen (35,46) ausgestattet ist.
23. Maschine gemäß Anspruch 22, **dadurch gekennzeichnet, dass** das wannenförmige Element (35) an einem Ende an einen Stellantrieb (36) gekoppelt ist und sowohl über eine Schneidposition (35a), die mit der Schneideinheit (14) funktionskoordiniert ist als auch über eine Endanschlag- und Kippposition (35b) für das Abladen des geschnittenen Segments (12) auf die Spalteinheit (15) verfügt.
24. Maschine gemäß Anspruch 22, **dadurch gekennzeichnet, dass** zwei wannenförmige Elemente (35) vorhanden sind, wobei das erste wannenförmige Element (35) für die Aufnahme des zu schneidenden Holzstammsegments (12) und das zweite wannenförmige Element (35) für das Abladen des geschnittenen Segments in das erste wannenförmige Element (35) bestimmt ist, welches danach auf das zuvor genannte zweite wannenförmige Element (35) geschoben wird.
25. Maschine gemäß Anspruch 22, **dadurch gekennzeichnet, dass** die wannenförmigen Elemente (46) an eine Drehscheibe (72) gekoppelt sind.
26. Maschine gemäß Anspruch 25, **dadurch gekennzeichnet, dass** die Drehscheibe (72) dazu geeignet ist, ein wannenförmiges Element (46) von einer ersten Position (I), die mit der in Längsrichtung angeordneten Vorschubeinheit (13) ausgerichtet ist, in der der zu schneidende Holzstamm abgeladen wird, zu einer zweiten Position (II), die mit der Achse des Spaltrastes (43) ausgerichtet ist, zu führen, in dem das Holzstammsegment (12) geschnitten wird.
27. Maschine gemäß Anspruch 25 oder 26, **dadurch gekennzeichnet, dass** sich die Drehscheibe (72) um eine Drehachse (73) dreht, die hinsichtlich der Vorschubachse (74) der Vorschubeinheit (13) seitlich versetzt ist.
28. Maschine gemäß Anspruch 1, **dadurch gekennzeichnet, dass** die Beschickungseinheit (11) mit zwei motorisch betriebenen Zahnrädern (17) ausgestattet ist, durch die die Ketten (18) angetrieben werden, an deren Gliedern konsolenförmige Elemente (19) für das Abstützen und das Anheben der Baumstämme (12) auf eine gewünschte Höhe befestigt sind.

29. Maschine gemäß Anspruch 28, **dadurch gekennzeichnet, dass** die konsolenförmigen Elemente (19) ein Hakenprofil aufweisen.

5 30. Maschine gemäß Anspruch 28, **dadurch gekennzeichnet, dass** am Einlauf der Beschickungseinheit (11) eine geneigte Ebene (21) vorhanden ist, auf der die vom Aufschichtungsbereich kommenden Holzstämme (12) durch einen Rolltisch zugeführt werden können.

10 31. Maschine gemäß Anspruch 28, **dadurch gekennzeichnet, dass** am Auslauf der Beschickungseinheit (11) eine Einheit (58) vorhanden ist, durch die die vor dem Abladen auf die Vorschubeinheit (13) angehobenen Holzstämme (12) vorübergehend festgehalten werden.

## 20 Revendications

1. Machine automatique électronique à couper / à fendre le bois (10) pour la production de tronçons ou de quartiers (49) de bois à brûler à partir de troncs (12) privés de leurs branches, eux-mêmes prélevés d'une zone d'empilement. Cette machine comprend, en coopération coordonnée: un groupe de chargement (11) pour le prélèvement automatique des troncs (12) de la zone d'empilement et leur dépôt sur un groupe d'avance longitudinale (13) avec un couloir de glissement (20) ; un groupe de coupe (14) placé à la suite du groupe d'avance longitudinale (13) pour la subdivision des troncs (12) en segments de longueur prédéterminée, **caractérisé par** un groupe de pincement (29) placé en coopération avec le groupe de coupe (14) pour le maintien temporaire des troncs (12) pendant l'opération de coupe et la mesure du diamètre des troncs (12) ; un groupe de translation (25) des segments de tronc (12) coupés placé tout de suite en aval du groupe de coupe (14) ; un groupe de fente (15), coplanaire et fondamentalement adjoint au groupe d'avance (13), pour la subdivision de ces segments de tronc coupés en nombre défini de tronçons ou de quartiers (49) et un groupe transporteur (16) placé tout de suite en aval du groupe de fente (15) pour l'évacuation et/ou le tri des tronçons ou des quartiers (49) produits. Ces groupes (11, 13, 14, 15, 16, 25, 29) sont asservis à un groupe électronique de contrôle et d'actionnement (26) associé au moins à des moyens pour la détermination automatique de la longueur des troncs (12).
2. Machine comme à la revendication 1, **caractérisée par le fait que** le groupe de fente (15) comprend un élément pousseur (42) coopérant au niveau inférieur avec un élément à plateau (40, 46) de soutien du segment de tronc (12) à fendre et de face

- avec une grille (43) de fente à secteurs multiples (43a, 43b, 43c), chaque secteur (43a, 43b, 43c) de la grille (43) présentant des lames coupantes (48) pour une disposition ou un dessin différents.
3. Machine comme à la revendication 2, **caractérisée par le fait que** la grille (43) est associée à un châssis porte-grille se mouvant sur le plan d'emplacement de l'élément pousseur (42).
4. Machine comme à la revendication 2, **caractérisée par le fait que** la grille (43) est associée à un châssis porte-grille se mouvant orthogonalement au plan d'emplacement de l'élément pousseur (42).
5. Machine comme à l'une ou l'autre des revendications précédentes, **caractérisée par le fait que** le châssis porte-grille est associé à des moyens de positionnement (45) asservis au groupe électronique de contrôle et d'actionnement (26) en fonction du diamètre du segment de tronc (12) à fendre et/ou du secteur (43a,43b,43c) à positionner.
6. Machine comme à l'une ou l'autre des revendications précédentes, **caractérisée par le fait que** le châssis porte-grille est associé à des moyens de réglage de la position (50, 51) en fonction de la collimation du centre du secteur (43a, 43b, 43c) par rapport au centre du segment de tronc (12).
7. Machine comme à l'une ou l'autre des revendications précédentes, **caractérisée par le fait que** le châssis porte-grille comprend deux plates-bandes (66a, 66b), parallèles au plan de coupe, liées solidairement aux lames coupantes (48) et à des pièces d'entretoise tubulaires (57) auxquelles elles sont transversales, le châssis porte-grille comprenant en outre des traverses (67a, 76b) solidaires aux tirants (68) de raccordement entre les têtes de la machine.
8. Machine comme à l'une ou l'autre des revendications précédentes, **caractérisée par le fait que** l'élément à plateau (40, 46) oscille vers l'avant en fonction des sollicitations occasionnées par le segment de tronc (12) pendant l'opération de fente.
9. Machine comme à la revendication 8, **caractérisée par le fait que** l'élément à plateau (40, 46) est fixé au niveau inférieur par des charnières sur le côté tourné vers l'élément pousseur (42) et qu'il coopère sur le côté opposé avec des moyens élastiques (41) de contraste.
10. Machine comme à la revendication 1, **caractérisée par le fait que** le groupe d'avance (13) se meut longitudinalement à pas, chaque pas étant en fonction d'un sous-multiple de la longueur du tronc (12).
11. Machine comme à la revendication 10, **caractérisée par le fait que** le groupe d'avance (13) comprend un couloir de glissement (20) du tronc (12) coopérant avec des moyens de poussée (24) associés à un chariot (22), le chariot (22) présentant une position de départ (22a) coopérant avec le début du couloir de glissement (20) et une position d'arrivée (22b) coopérant avec la fin du couloir de glissement (20).
12. Machine comme à la revendication 11, **caractérisée par le fait que** le couloir de glissement (20) comprend des moyens de détermination automatique de la longueur du tronc (12).
13. Machine comme à la revendication 11 ou 12, **caractérisée par le fait que** les moyens de poussée (24) sont mobiles pour le centrage des segments de tronc (12) coupés sur le groupe de translation (25).
14. Machine comme à la revendication 1, **caractérisée par le fait que** le groupe de pincement (29) présente des moyens pour la mesure du diamètre du segment coupé.
15. Machine comme à la revendication 14, **caractérisée par le fait que** le groupe de pincement (29) comprend des bras à pantographe (30a, 30b), la première paire supérieure (30a) des bras ayant ses extrémités supérieures montées sur pivots au même point au sommet du châssis du groupe de coupe (14) et ses extrémités inférieures montées sur pivots (34) à l'extrémité supérieure des bras inférieurs (30b) respectifs, les bras inférieurs (30b) étant à leur tour montés entre eux sur pivots (31) en correspondance de la ligne médiane, le pivot (31) glissant le long d'une coulisse verticale (32) en réponse aux actions de traction ou de poussée exercées par un actionneur (33) agissant sur un seul des deux pivots (34).
16. Machine comme à la revendication 15, **caractérisée par le fait que** les extrémités inférieures des bras inférieurs (30b) présentent une première position abaissée de blocage du tronc (12) en position de coupe associée à la position retirée de l'actionneur (33) et une seconde position rehaussée de non interférence associée à la position d'extension de l'actionneur (33).
17. Machine comme à l'une ou l'autre des revendications de 14 à 16, **caractérisée par le fait que** les moyens pour la mesure du diamètre du segment comprennent un actionneur pour la détermination de la course effectuée par l'actionneur (33).
18. Machine comme à l'une ou l'autre des revendica-

tions de 14 à 17, **caractérisée par le fait que** les bras inférieurs (30b), quel que soit le diamètre du tronc (12), sont toujours disposés à 120° par rapport au plan d'emplacement du tronc (12).

19. Machine comme à la revendication 1, **caractérisée par le fait que** le groupe de coupe (14) présente au moins une scie (27) à lame circulaire (28) associée à un chariot de support (55) se mouvant verticalement et définissant, pour la lame (28), une première position abaissée (28a) de non interférence et une seconde position rehaussée de coupe (28b).

20. Machine comme à la revendication 19, **caractérisée par le fait que** le chariot de support (55) est actionné par des actionneurs (54) se mouvant en fonction du diamètre du tronc (12).

21. Machine comme à la revendication 19, **caractérisée par le fait que** le chariot de support (55) peut être retiré de la structure portante de la machine (10).

22. Machine comme à la revendication 1, **caractérisée par le fait que** le groupe de translation (25) comprend des éléments à plateau (35, 46).

23. Machine comme à la revendication 22, **caractérisée par le fait que** l'une des extrémités de l'élément à plateau (35) est associée à un actionneur (36) et que l'élément à plateau (35) présente une position de coupe (35a) coopérant avec le groupe de coupe (14) et une position de fin de course et de basculement (35b) de déchargement du segment (12) coupé sur le groupe de fente (15).

24. Machine comme à la revendication 22, **caractérisée par le fait que** les éléments à plateau (35) sont au nombre de deux : un premier élément à plateau (35) soutenant un segment de tronc (12) à couper et un second élément à plateau (35) déchargeant le segment coupé sur le premier élément à plateau (35) et le poussant sur ce second élément à plateau (35).

25. Machine comme à la revendication 22, **caractérisée par le fait que** les éléments à plateau (46) sont associés à un disque rotatif (72).

26. Machine comme à la revendication 25, **caractérisée par le fait que** le disque rotatif (72) est apte à porter un plateau (46) d'une première position (I), alignée au groupe d'avance longitudinale (13), où est reçu le tronc à couper, à une seconde position (II), alignée à l'axe de la grille de fente (43), où le segment de tronc (12) est coupé.

27. Machine comme à la revendication 25 ou 26, ca-

**ractérisée par le fait que** le disque rotatif (72) tourne autour d'un axe de rotation (73) déplacé latéralement par rapport à l'axe d'avance (74) du groupe d'avance (13).

28. Machine comme à la revendication 1, **caractérisée par le fait que** le groupe de chargement (11) présente des paires de roues dentées (17) motorisées actionnant des chaînes (18) dont les mailles sont associées à des éléments à étagère (19) pour supporter et lever les troncs (12) à la hauteur souhaitée.

29. Machine comme à la revendication 28, **caractérisée par le fait que** les éléments à étagère (19) présentent un profil à crochet.

30. Machine comme à la revendication 28, **caractérisée par le fait qu'un** plan incliné (21) est situé en amont du groupe de chargement (11) pour le roulement des troncs (12) provenant de la zone d'empilement.

31. Machine comme à la revendication 28, **caractérisée par le fait qu'un** groupe (58) de maintien temporaire des troncs (12) soulevés avant le déchargement sur le groupe d'avance (13) est situé en aval du groupe de chargement (11).

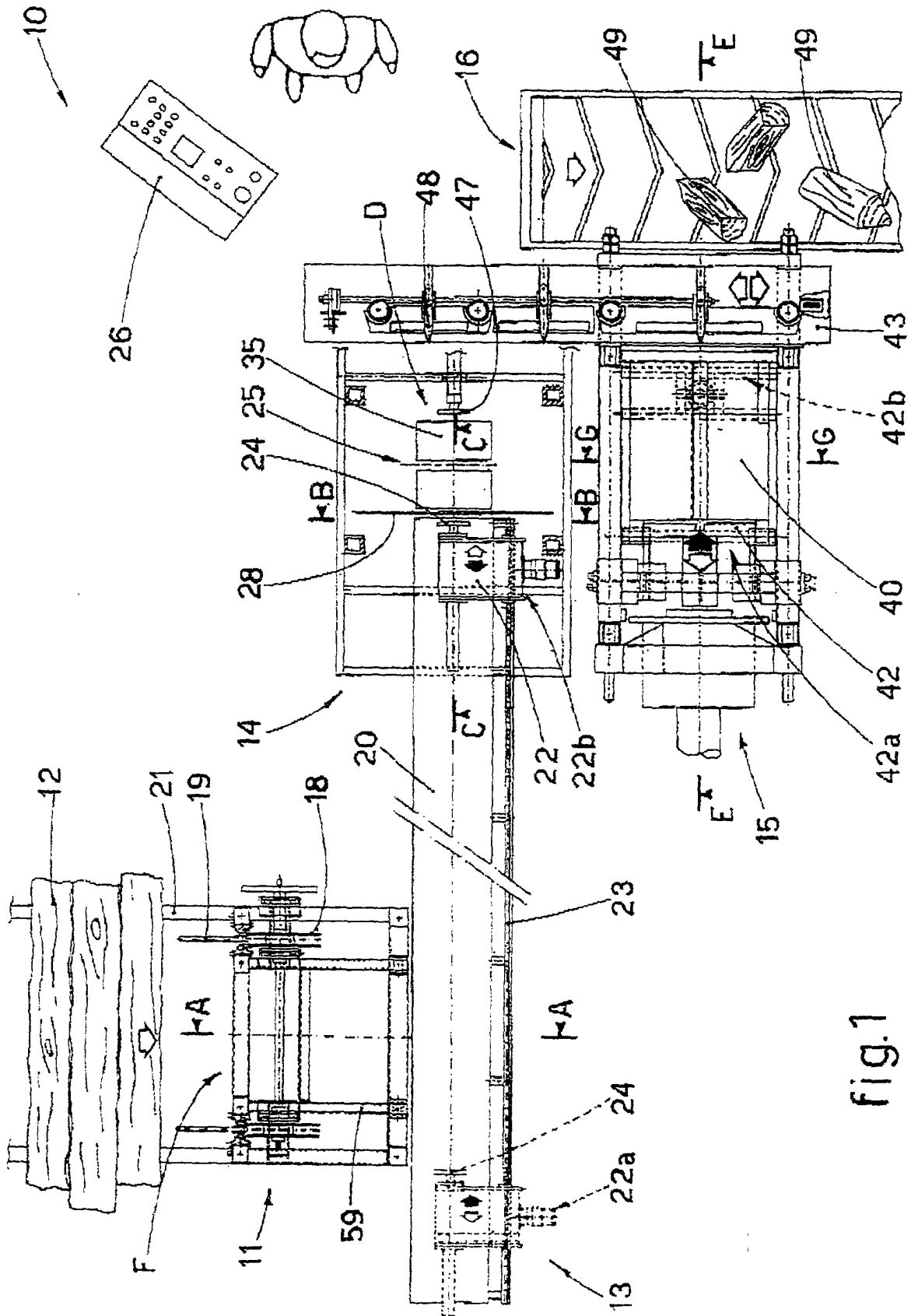


fig.1

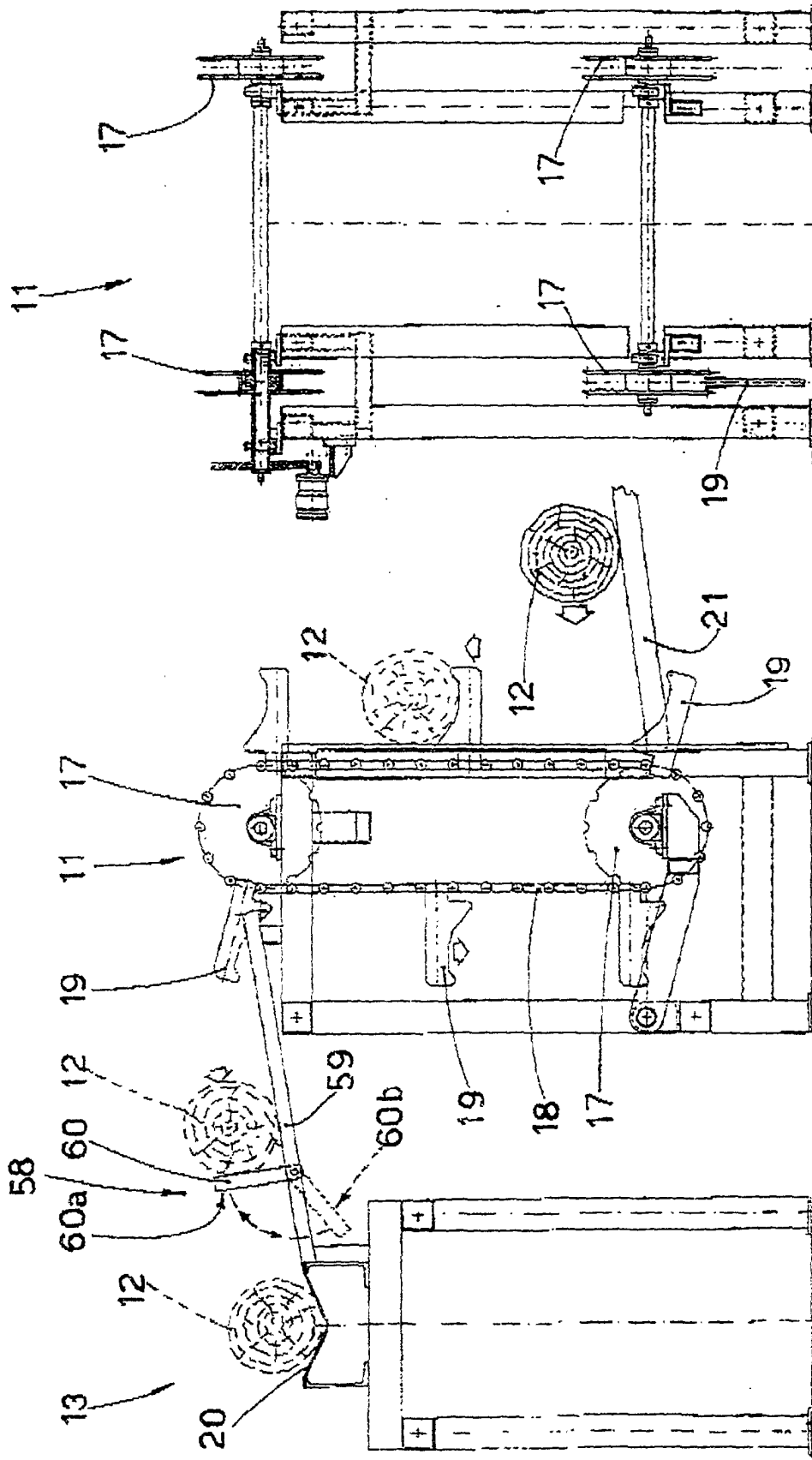
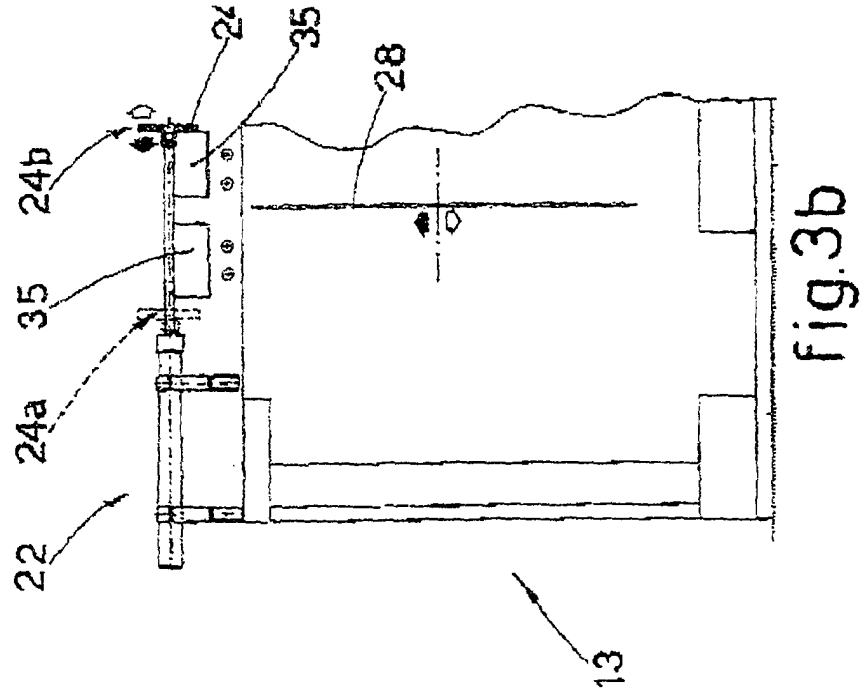
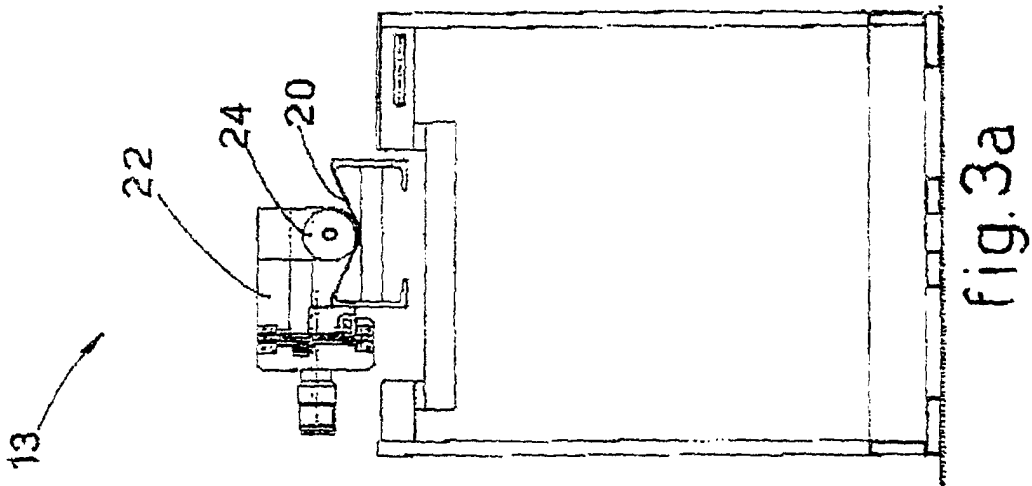


fig.2b

fig.2a







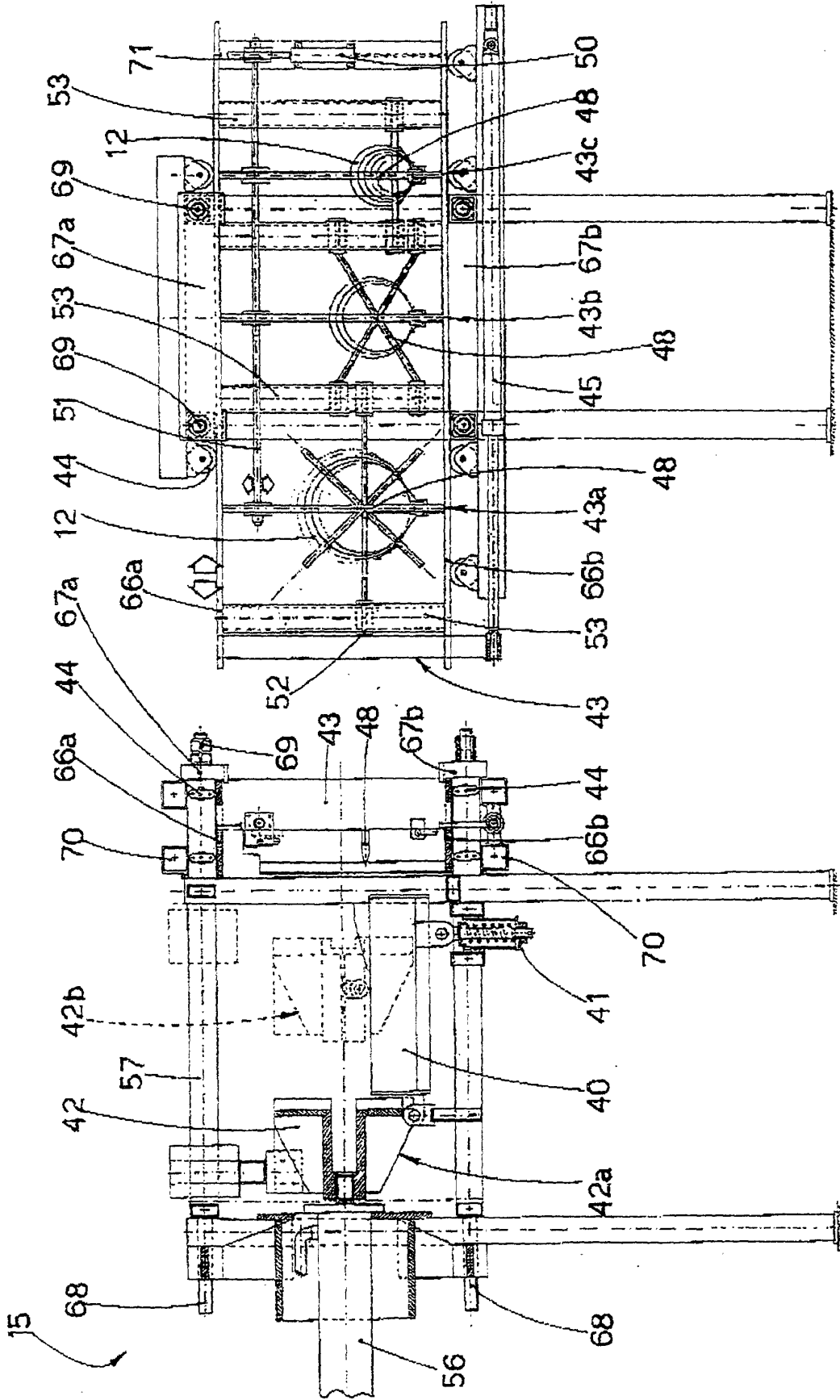


fig.5b

fig.5a

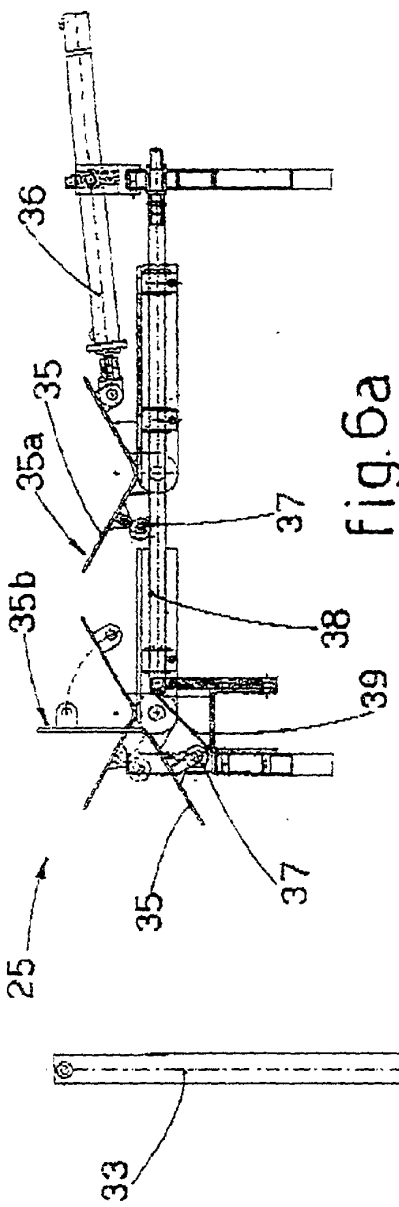


fig.6a

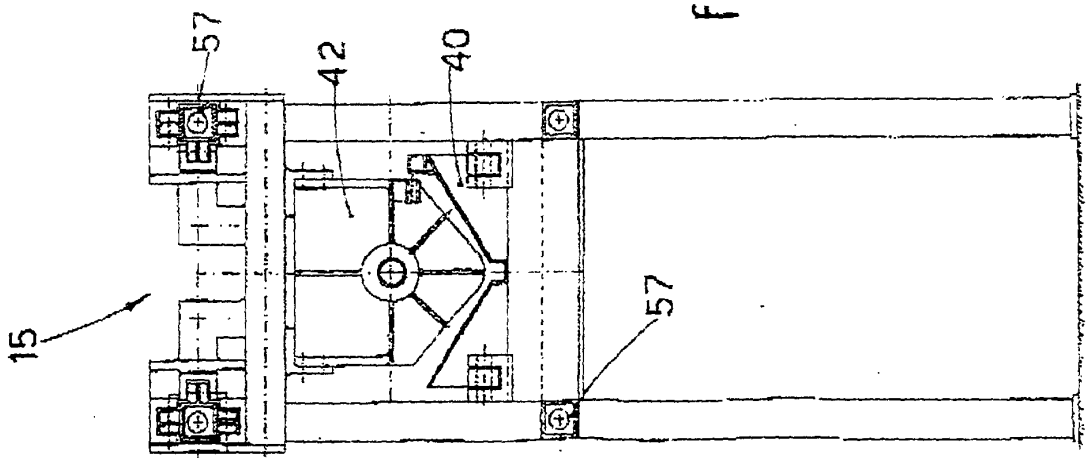


fig.5c

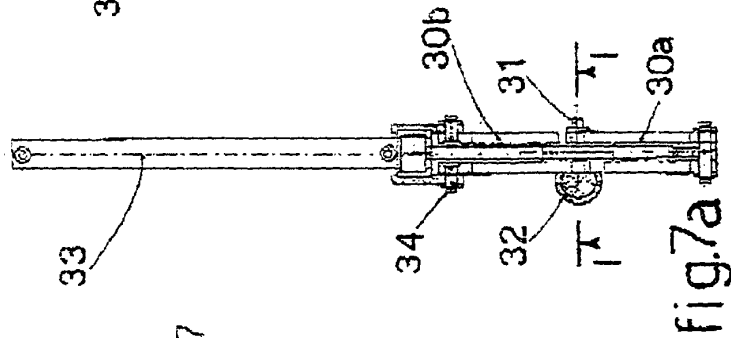


fig.7a

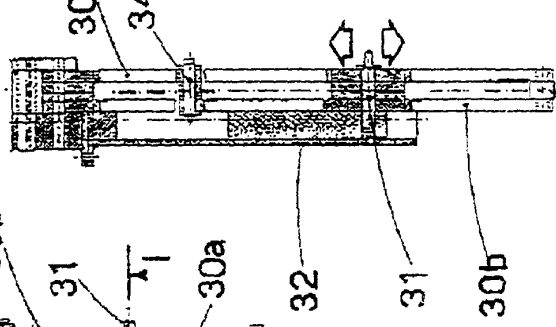


fig.7b

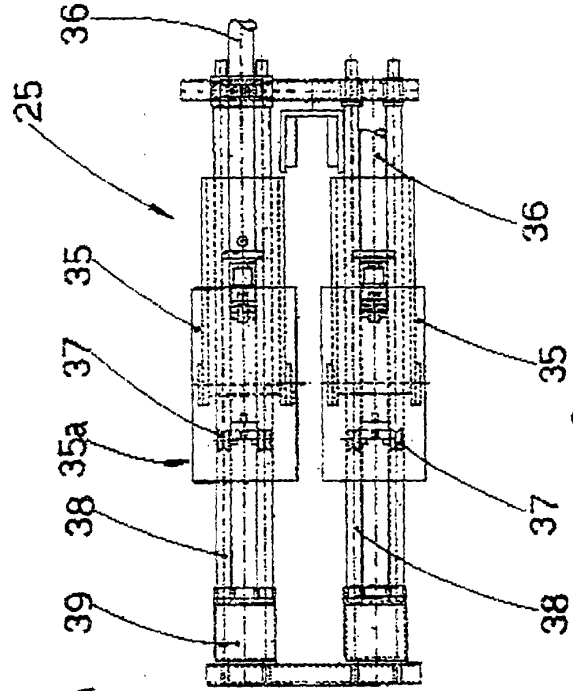
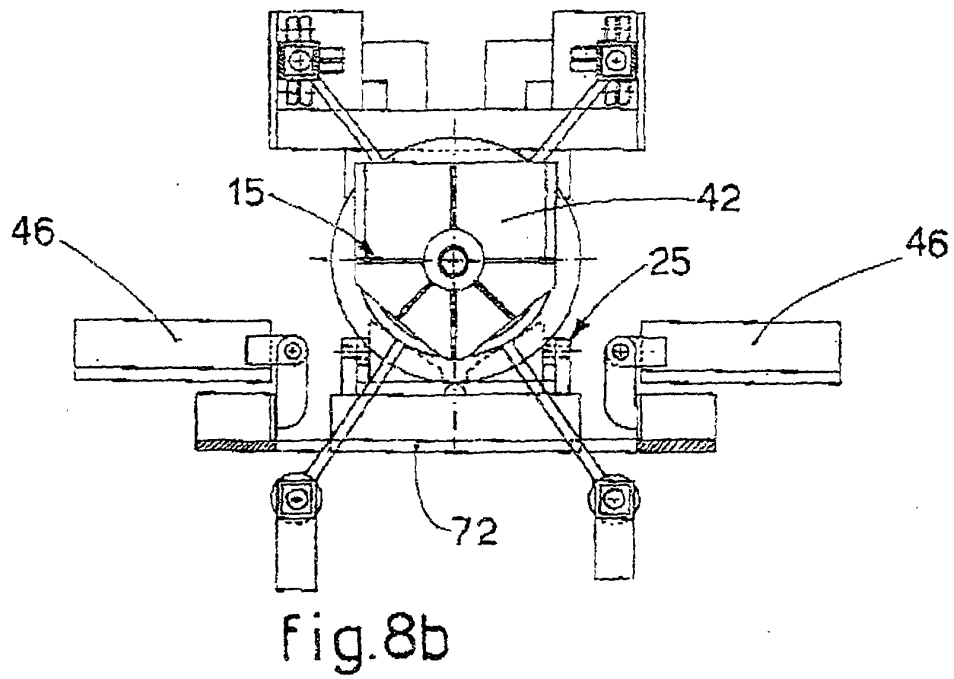
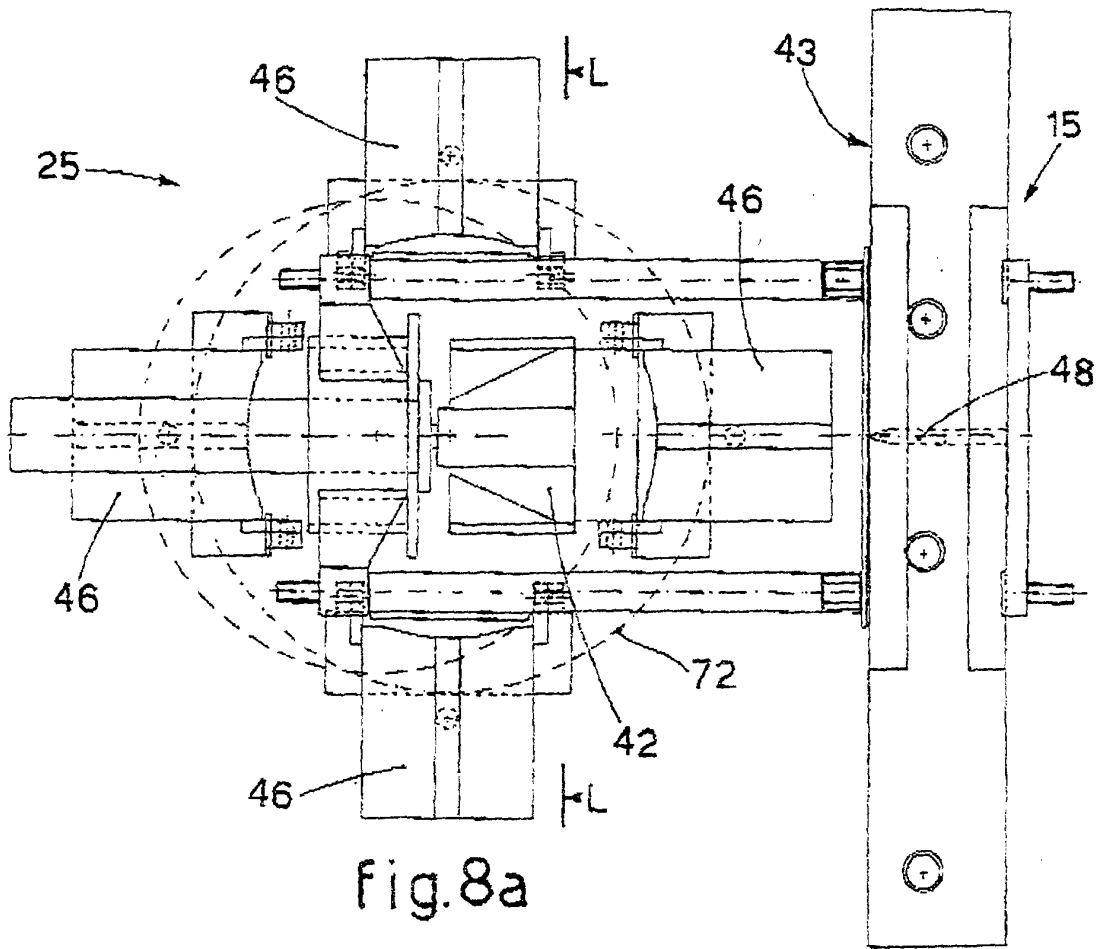


fig.6b



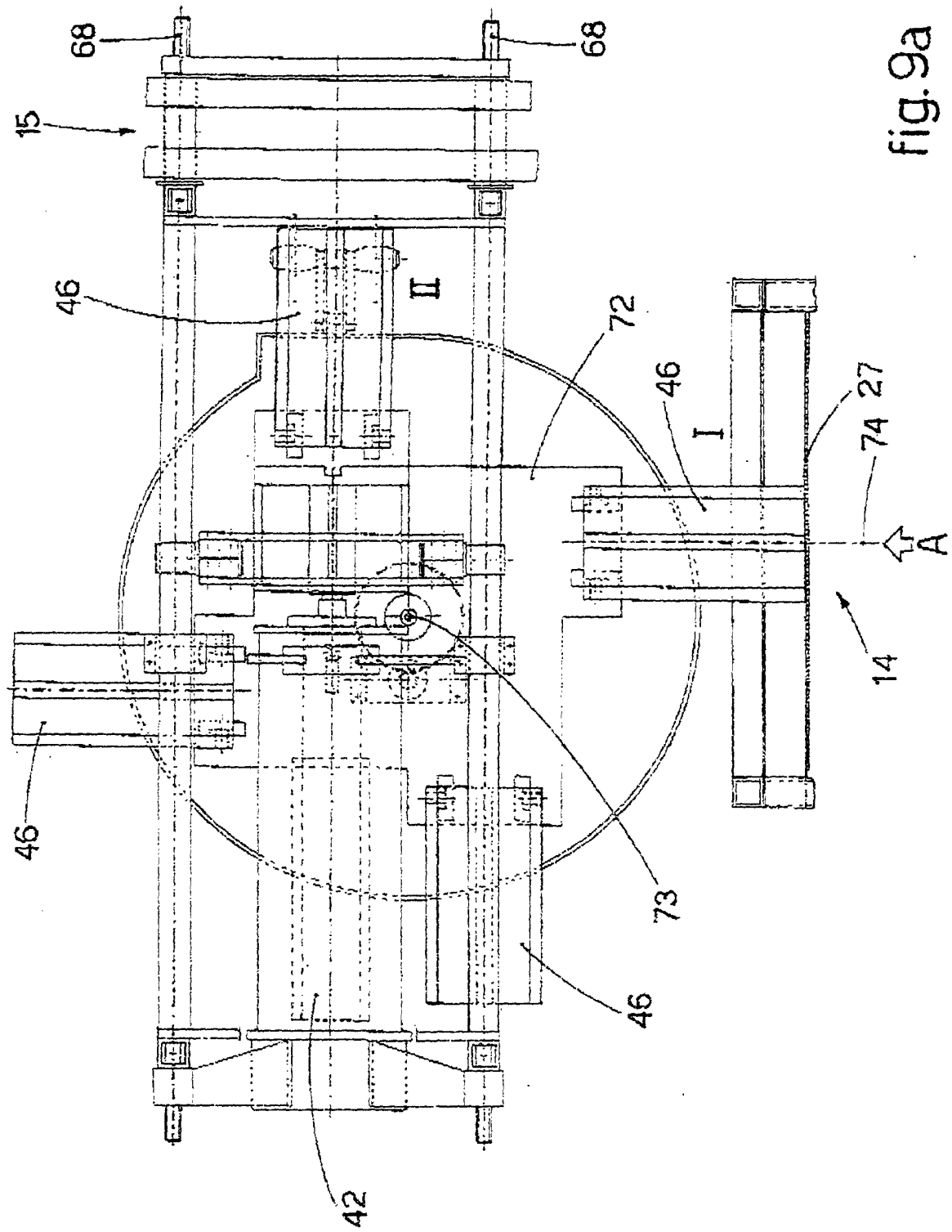


fig. 9a

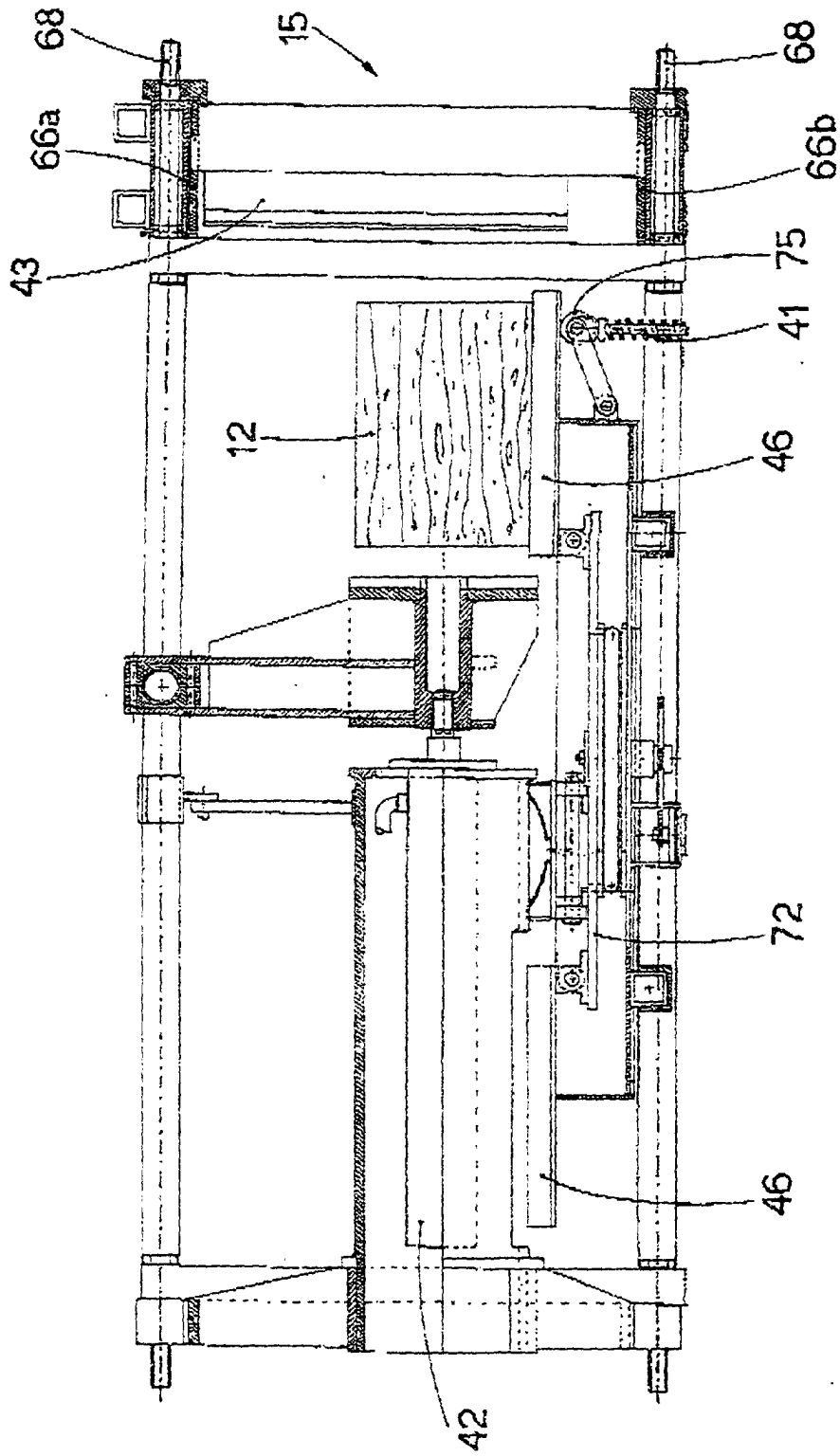


Fig. 9b