United States Patent

Keyser

[54] ADJUSTABLE STRIP CONDITIONER

- [72] Inventor: Naaman H. Keyser, Hinsdale, Ill.
- [73] Assignee: Interlake, Inc., Chicago, Ill.
- [22] Filed: July 27, 1970
- [21] Appl. No.: 58,275

[56] **References Cited**

UNITED STATES PATENTS

3,006,401	10/1961	Wognum et al	72/162
3,438,231	4/1969	Petzschke	72/160
2,323,768	7/1943	Hanna	72/165
3,459,027	8/1969	Brownstein	72/160
1,923,738	8/1933	McBain	72/165
2.963.071	12/1960	Krvnvtzky	

Primary Examiner-Milton S. Mehr

Attorney-Prangley, Clayton, Mullin, Dithmar & Vogel

[57] ABSTRACT

A metal strip conditioner includes a guide roll carriage for supporting a metal strip for longitudinal movement along a predetermined planar datum path and having a work section intermediate the ends thereof, a rectangular work platen mounted on the carriage above the work section and having four adjusting screws respectively mounted at the corner sections thereof for independently adjusting the positions thereof with respect to the datum path throughout a range of strip working positions, and a first assembly of edge and center working rolls carried by the platen and adjustable therewith toward and away from a like second assembly of working rolls carried by the work section of the guide carriage, the rolls of the first and second assemblies being so arranged that different combinations of edge and center working of the strip may be achieved by appropriate individual adjustment of the four drive assemblies; auxiliary guide rollers are provided adjacent to the working rolls for maintaining the unworked sections of the strip in the datum path.

24 Claims, 12 Drawing Figures



[15] **3,704,614** [45] **Dec. 5, 1972**

3,704,614





3,704,614

SHEET 2 OF 5



95

3,704,614





SHEET 3 OF 5

3,704,614

SHEET 4 OF 5



FIG.10



SHEET 5 OF 5





ìоз

ADJUSTABLE STRIP CONDITIONER

This invention relates to an improved apparatus for conditioning strip metal such as steel and the like, to redistribute therein non-uniform internal stresses whereby to produce a straightened longitudinal flat 5 strip.

More particularly, the present invention relates to an improvement in a strip conditioning apparatus of the type disclosed in U. S. Pat. No. 3,006,401, issued Oct. 31, 1961, to James N. Wognum, Roy A. Moody, and ¹⁰ Emil Simich, and assigned to the assignee of the present invention. This prior strip conditioning apparatus included guide means defining a planar datum path for the strip, and a plurality of individually adjustable upper and lower work rolls spaced apart longitudinally ¹⁵ along substantially the entire length of the guide means for plastically working the strip.

It is a general object of the present invention to provide a strip conditioning apparatus wherein the work rolls are all confined to a central portion of the guide means, with the upper ones of the work rolls being carried by a movable work platen having the position thereof adjustable with respect to the guide means throughout a range range of strip working positions. 25

It is an important object of this invention to provide apparatus for conditioning an elongated metal strip comprising guide means for supporting a metal strip for longitudinal movement along a predetermined planar datum path, a work platen disposed adjacent to the 30 guide means and having the position thereof adjustable with respect to the guide means throughout a range of strip-working positions, and a plurality of deflecting members carried by the work platen and adjustable therewith in directions substantially normal to the 35 plane of the datum path, the deflecting members engaging the strip and cooperating with the guide means for deflecting the strip out of the plane of the datum path plastically to work the strip, adjustment in the position of the platen relative to the guide means serv- 40 ing to change the amount of deflection of the strip by the deflecting members for thereby adjusting the amount of work done on the strip.

It is another object of this invention to provide a conditioning apparatus of the type set forth, wherein the 45 guide means has a strip working section provided with a plurality of first deflecting members thereon, a plurality of second deflecting members being carried by the work platen and adjustable therewith, the first and second deflecting members respectively engaging the ⁵⁰ strip along the opposite planar surfaces thereof and cooperating to effect deflection thereof out of the plane of the datum path plastically to work the strip.

Another object of this invention is to provide a conditioning apparatus of the type set forth which further ⁵⁵ includes a drive assembly coupled to the work platen for effecting controlled movement thereof toward and away from the guide means to adjust the position of the work platen.

Another object of the invention is to provide a conditioning apparatus of the type set forth, wherein the work platen is generally rectangular in shape and extends longitudinally of the guide means, and further including four drive assemblies coupled to the work platen for respectively effecting controlled movement of the four corner sections thereof toward and away from the guide means to adjust the position of the work

platen, the deflecting members including a pair of edge deflecting members disposed adjacent to one end of the platen for deflecting and plastically working the edge sections of the strip, and a center deflecting member disposed adjacent to the other end of the work platen for deflecting and plastically working the center section of the strip, each of the drive assemblies being individually operable for adjusting the position of the associated corner of the work platen to effect different combinations of the deflections of the edge and center deflecting members.

In connection with the foregoing object, it is another object of this invention to provide a conditioning apparatus of the type set forth wherein a first assembly of edge and center deflecting members are carried by the strip working section of the guide means, and a corresponding second assembly of edge and center deflecting members are carried by the work platen, the first and second deflecting members cooperating to effect deflection of the strip alternately in opposite directions out of the plane of the datum path.

A still further object of this invention is to provide a conditioning apparatus of the type set forth, which further includes center guide means for holding the center longitudinal section of the strip in the datum path at points disposed transversely of the edge deflecting members, and edge guide means for holding the edge sections of the strip in the datum path at points disposed transversely of the center deflecting member.

Further features of the invention pertain to the particular arrangement of the parts of the strip conditioning apparatus whereby the above outlined and additional operating features thereof are attained.

The invention, both as to its organization and method of operation, together with further objects and advantages thereof will best be understood with reference to the following specification taken in connection with the accompanying drawings, in which:

FIG. 1 is a top plan view of a portion of a flat metal strip having a lateral curvature or camber therein;

FIG. 2 is a top plan view similar to FIG. 1, and showing the section of strip illustrated in FIG. 1 in the straightened condition after having had the camber removed therefrom by the strip conditioner of the present invention, and indicating the longitudinal lines along which the strip is worked by the conditioner of the present invention;

FIG. 3 is a top plan view of an adjustable strip conditioner constructed in accordance with and embodying the the features of the present invention;

FIG. 4 is a side elevational view of the adjustable strip conditioner illustrated in FIG. 3;

FIG. 5 is an enlarged fragmentary side elevational view in partial section of the work platen of the adjustable strip conditioner of this invention;

FIg. 6 is a further enlarged fragmentary top plan view of one of the drive assemblies for the work platen illustrated in FIG. 5;

FIG. 7 is an enlarged view in vertical section taken along the line 7—7 in FIG. 4 and illustrating the guide rolls and one of the carriage positioning assemblies of the present invention;

FIG. 8 is a further enlarged view in vertical section taken along the line 8—8 in FIG. 4 and illustrating the work roll assemblies of the present invention;

FIG. 9 is an enlarged fragmentary top plan view, in partial section of the work platen of the present invention, illustrating the position of the platen work roll assembly;

FIG. 10 is a view, similar to FIG. 9 but with the work 5 platen and the top plates of the upper carriage removed to show the position of the lower carriage work roll assembly:

FIG. 11 is a view in vertical section taken along the 10 line 11–11 in FIG. 9 and illustrating the cooperation of the upper and lower outboard edge working rolls of the present invention; and

FIG. 12 is a view in vertical section taken along the line 12-12 in FIG. 9, and illustrating the cooperation 15 of the upper and lower center working rolls of the present invention.

Referring now to the drawings and in particular to FIGS. 1 and 2 thereof, there is illustrated in FIG. 1 a section of metal strip 21 having a lateral curvature or 20 chamber therein, as illustrated at 22. The nature and causes of this lateral deformation are discussed in the aforementioned U. S. Pat. No. 3,006,401, the disclosure of which is incorporated herein by reference. In FIG. 2, there is illustrated the section of metal strip in a 25 structed, whereby the same reference numerals have straightened condition produced by removal of the chamber therefrom. This straightening operation entails plastic working of the strip along the opposite side edge sections thereof, as illustrated by lines 23 and 24, and along the center section thereof as illustrated by 30 line 25. This working of the center and edge sections of the strip is brought about by applying tensional forces greater than the elastic limit of the material to these sections of the strip, all as is explained in great detail in 35 the aforementioned U.S. Pat. No. 3,006,401.

Referring now to FIGS. 3 to 12 of the drawings, there is illustrated an adjustable strip conditioning machine, generally designated by the numeral 20, constructed in accordance with and embodying the features of the 40 present invention for plastically working the edge and center sections of a metal strip 21 in the manner described above. The conditioning machine 20 includes a lower carriage 30 and an upper carriage 70 assembled one above the other, the entire apparatus being mounted on a pair of base members 27, respectively disposed at the opposite ends of the machine 20. Each of the base members comprises a generally tubular structural member 27, substantially rectangular in transverse cross section, and secured by means of bolts 28 to the bottom of the machine 20. Each of the base members 27 is also provided with a swivel opening 29 therein for receiving therethrough a pivot pin, to facilitate the mounting of the machine 20 in a strip processing line between other pieces of strip processing 55 equipment. The lower carriage 30 includes a flat elongated, substantially horizontal bottom plate 31, and a pair of laterally spaced-apart upstanding side plates 32 and 34, respectively disposed substantially normal to 60 the bottom plate 31 adjacent to the opposite side edges thereof and secured thereto by means of bolts 33. Each of the side plates 32 and 34 is provided with a pair of recesses 35 respectively disposed in the upper edge thereof adjacent to the opposite ends thereof, and a 65 center recess 36 disposed in the upper edge thereof substantially midway between the ends thereof, the center recess 36 having a length approximately twice

the length of one of the end recesses 35. Respectively secured to the opposite ends of each of the side plates 32 and 34 by suitable fasteners (not shown) are four end retainers 38, each of the retainers 38 being in the form of an angle bracket having one leg disposed along the outer side surface of the associated side plate 32 or 34 and the other leg thereof disposed along the end surface of the associated side plate, each of the end retainers 38 extending upwardly a slight distance beyond the upper edge of the associated side plate 32 or 34.

The lower carriage 30 is provided with a plurality of edge-engaging guide roll assemblies 40 along one side thereof (the left-hand side as viewed in FIG. 7). For convenience, this side of the conditioning machine 20 will hereinafter be referred to as the outboard or operator's side, while the opposite side of the machine 20 will be referred to as the inboard side. Preferably, four of the edge-engaging guide roll assemblies 40 are provided, with one assembly being disposed adjacent to each of the recesses 35 and with two of the assemblies 40 being disposed side-by-side adjacent to the recess 36. The roll assemblies 40 are all identically conbeen applied to like parts of each. Each of the roll assemblies 40 includes a mounting block 42 secured to the outer surface of the side plate 32 by suitable bolt and nut assemblies 43. The block 42 is provided with a generally rectangular shaped recess 45 on the inner surface thereof adjacent to the upper end thereof, thereby defining an upper flange or lip 44 which extends inwardly above the associated recesses 35 or 36. Preferably, the lower edge of the recess 45 is substantially coplanar with the lower edge of the associated recess 35 or 36 in the side plate 32, and cooperates therewith to define a mounting recess for a guide roll 58, mounted for rotation about a vertically extending pivot pin 46 journaled in the block 42. The guide roll 48 is provided with a groove 49 extending circumferentially around the outer surface thereof for accommodating therein the adjacent edge of the metal strip 21 as it passes through the conditioning machine 20, as 45 is best illustrated in FIG. 7. It will be noted that the central recess 36 in the side plate 32 has a width sufficient to accommodate two of the roll assemblies 40 arranged side-by-side.

Arranged along the inboard side of the lower car-50 riage 30, directly opposite the end ones of the roll assemblies 40, are two edge-engaging roll assemblies 50, the roll assemblies 50 being identically constructed wherefor the same reference numerals have been applied to like parts of each. Each of the roll assemblies 50 includes an angle mounting bracket 51 having a horizontal flange 51a secured by a nut and bolt assembly 52 to the upper surface of the bottom plate 31, and a vertically extending flange 51b integral with the horizontal flange 51a and extending upwardly therefrom at the outer end thereof. An air motor 53 is secured by suitable means to the vertically extending flange 51b, the air motor 53 having a piston rod 54 extending substantially horizontally through a complementary opening in the vertically extending flange 51b and coupled at the inner end thereof to a mounting block 55. The mounting block 55 is substantially in the form of a clevis and is provided with a vertically extending pivot pin 56 journaled therein, a guide roll 58 being mounted for rotation about the pivot pin 56. The guide roll 58, identical in construction to the guide rolls 48, is provided with a groove 59 extending circumferentially about the outer surface thereof for receiving 5 therein the adjacent edge of the metal strip 21, as is best illustrated in FIG. 7. In use, the piston rod 54 of the air motor 53 is retracted to the right, as viewed in FIG. 7, for accommodating insertion of the metal strip 21 into the conditioning machine 20. Thereafter, the air motor 53 is actuated to move the piston rod 54 thereof to the left, as viewed in FIG. 7, thereby engaging the grooved outer surface of the guide roll 58 with the adjacent edge of the metal strip 21.

15 Another edge engaging roll assembly, generally designated by the numeral 60, is disposed along the inboard side of the lower carriage 30 adjacent to the center recess 36 in the side plate 34. The roll assembly 60 includes an angle mounting bracket 61 having a $_{20}$ horizontally extending flange 61a secured by suitable means to the bottom plate 31, and a vertically extending flange 61b integral with the horizontally extending flange 6a and extending upwardly therefrom at the outer end thereof. A pair of bracing bolts are received 25 through complementary openings at the opposite ends of the vertically extending flange 61b adjacent to the upper edge thereof, the bolts 62 extending substantially horizontally and parallel to each other and having the inner ends thereof threadedly engaged with the side 30 plate 34. An air motor 63 is also secured to the vertically extending flange 61b by suitable means, the air motor 63 having a piston rod 64 extending substantially horizontally through a complementary opening in the vertically extending flange 61b and being coupled at ³⁵ the inner end thereof to a mounting block 65. The mounting block 65 is in the form of a clevis and has a width approximately twice the width of the mounting blocks 55. More particularly, the mounting block 65 in-40 cludes a substantially horizontal lower flange 67 generally trapezoidal in shape and a substantially rectangular horizontal upper flange 67 a. Extending vertically between the flanges 67 and 67 a and journaled therein are a pair of parallel pivot pins 66, each 45 of the pins 66 having rotatably mounted thereon a guide roll 68, the guide rolls 68 being identical in construction to the guide rolls and being disposed directly opposite the center two thereof. In operation, the roll assembly 60 is operated in the same manner as the roll 50 assemblies 50 described above. Thus, after the metal strip 21 has been inserted in the conditioning machine 20, the air motors 53 and 63 are all actuated to move the grooved guide rolls 58 and 68 into engagement with the inboard edge of the metal strip 21, and moving the 55 metal strip 21 laterally into engagement with the grooved guide rolls 48 on the outboard roll assemblies 40. Thus, the roll assemblies 40, 50 and 60 all cooperate to securely and accurately position the metal 60 strip 21 in a predetermined straight datum path, as will be described in greater detail below, the side edges of the strip 21 being received in the grooves 49, 59 and 69 of the guide rolls 48, 58 and 68, respectively, to prevent flattening of the side edges of the strip 21. 65

The upper carriage 70 is disposed immediately above the lower carriage 30 and includes a pair of substantially horizontally disposed rectangular top plates 71,

6

spaced apart longitudinally of the machine 20. A pair of side plates 72 and 74 are also provided, identical in construction to and respectively disposed in vertical alignment with the side plates 32 and 34 of the bottom carriage 30, each of the side plates 72 and 74 being secured to the top plates 71 by bolts 73. Each of the side plates 72 and 74 is provided with recesses 75 and 76 in the lower edge thereof, respectively identical to the recesses 35 and 36 in the side plates 32 and 34, and disposed in facing relationship therewith. Thus, the recesses 75 and 76 respectively cooperate with the recesses 35 and 36 to accommodate the guide rolls of the roll assemblies 40, 50 and 60. A plurality of spacers 77 are also provided along the side plates 32 and 34 of the bottom carriage 30, each of the spacers 77 being in the form of an angle bracket having a horizontal flange 78 overlying the upper edge of the associated side plate 32 or 34 and a vertical flange 79 extending downwardly from the outer end of the horizontal flange 78 along the outer surface of the associated side plate 32 or 34. Each of the spacers 77 is secured to the associated side plate 32 or 34 by a bolt 79a extending through a complementary opening in the vertical flange 79 and threadedly engaged with the associated side plate. Thus, in use, the spacers 77, which are preferably spaced apart longitudinally of the lower carriage 30, serve to separate the lower carriage 30 from the upper carriage 70 by a predetermined distance equal to the thickness of the horizontal flanges 78, all for a purpose to be described hereinafter. It will be noted that, when the upper carriage 70 is disposed in place on the spacers 77 above the lower carriage 30, the opposite ends of the upper carriage 70 are received within the upper ends of the end retainers 38 for preventing longitudinal or lateral movement of the upper carriage 70 with respect to the lower carriage 30.

Each of the carriages 30 and 70 is provided with a plurality of longitudinally spaced-apart guide roll assemblies, generally designated by the numeral 80. The guide roll assemblies 80 are so arranged that those in the upper carriage 70 are staggered with respect to those in the lower carriage 30. The guide roll assemblies 80 are all identically constructed, whereby the same reference numerals have been applied to like parts of each and only one will be described in detail. Referring, e.g., particularly to FIGS. 7 and 10 of the drawings, one of the guide roll assemblies 80 on the lower carriage 30 is illustrated, the assembly 80 including three spaced-apart clevis brackets 81 aligned transversely of the associated carriage 30 between the side plates 32 and 34 thereof, each of the clevis brackets 81 having the bight portion thereof secured to the inner surface of the bottom plate 31 by a suitable fastener. An axle shaft 83 extends transversely through complementary aligned openings in the legs of the three clevis brackets 81 and through complementary openings in the side plates 32 and 34, the axle shaft 83 being in the form of a bolt having a head disposed along the outer surface of the side plate 34 and being provided at the other end thereof with a nut 85 and a washer 85a along the outer surface of the side plate 32. Mounted for rotation about the axial shaft 83 are three guide rolls 85 respectively mounted between the legs of the clevis brackets 81, the guide rolls 85 all being identically constructed. The guide rolls 85 on the lower carriage 30

are all dimensioned and positioned so that the upper edges of the circumferential surfaces thereof all lie in a common horizontal plane; similarly, the guide rolls 85 on the upper carriage 70 are dimensioned and positioned so that the lower edges of the circumferential 5 surfaces thereof all lie in a common horizontal plane spaced above the horizontal plane of the bottom carriage guide rolls 85 by a distance substantially equal to the thickness or gauge of the metal strip 21 to be worked. Thus, the guide rolls of the upper and lower 10carriages 30 and 70 cooperate to define a substantially horizontal planar datum path for the metal strip 21.

Each of the top plates 71 of the upper carriage 70 is also provided with a carriage positioning assembly, 15 generally designated by the numeral 90, the carriage positioning assemblies 90 cooperating to firmly hold the upper carriage 70 in position with respect to the lower carriage 30. The carriage positioning assemblies 90 are identically constructed, whereby the same 20 reference numerals have been applied to like parts of each. Referring in particular to FIGS. 3, 4 and 7 of the drawings, the carriage positioning assembly 90 includes four vertically extending support posts 91, with two of the posts 91 being arranged on the inboard side of the 25 cally received through a complementary opening in the machine 20 and two of the posts 91 being disposed on the outboard side of the machine 90. Each of the posts 91 is threadedly engaged at the lower end thereof with the adjacent side edge of the bottom plate 31, the upper ends of the posts 91 being respectively secured in $_{30}$ relationship therewith and secured thereto by means of complementary openings in the four corners of a substantially rectangular mounting plate 92. Mounted on the upper surface of the mounting plate 92 is an air motor 95 provided with a piston rod 94 extending vertically downwardly through a complementary opening in 35 the mounting plate 92, and threadedly engaged at the lower end thereof with the associated top plate 71 of the upper carriage 70.

A pair of suspension rods 96 are respectively the opposite sides of the mounting plate 92, approximately midway between the adjacent pair of support posts 91. Each of the suspension rods 96 is threadedly engaged at the lower end thereof with the associated top plate 71 of the upper carriage 70 and extends verti- 45 cally upwardly therefrom a substantial distance above the upper surface of the mounting plate 92 and is provided with a coil compression spring 97 disposed in surrounding relationship therewith. Each of the compression springs 97 is held under compression against the 50 upper surface of the mounting plate 92 by a washer 98 and a nut 98a. The compression springs 97 are so adjusted that they normally urge the suspension rods 96 upwardly for holding the upper carriage 70 suspended above the lower carriage 30 out of contact with the 55 spacers 77 for facilitating the replacement of the spacers 77 and the threading of the metal strip 21 into the conditioning machine 20. After the metal strip 21 has been positioned in the conditioning machine 20. 60 the air motors 90 are actuated to move the piston rods 94 thereof downwardly for thereby moving the upper carriage 70 downwardly into engagement with the spacers 77. The air motors 90 are held in this actuated condition during the operation of the conditioning 65 machine 20 for applying a high pressure to the upper carriage 70 for securely holding it in position with respect to the lower carriage 30.

The inner ends 101 of the top plates 71 are separated by a predetermined distance and define therebetween a work section 102 of the upper carriage 70, the work section 102 being coextensive with and immediately overlying a corresponding work section 103 of the lower carriage 30. Disposed between the inner ends 101 of the top plate 71 is a flat, substantially rectangular work platen 110, provided with four openings 111 respectively disposed therethrough adjacent to the four corners thereof. The work platen 110 is secured to the side plates 72 and 74 of the upper carriage 70 by means of four drive assemblies, generally designated by the numeral 115. The drive assemblies 115 are all identically constructed, whereby the same reference numerals have been applied to like parts of each. Each of the drive assemblies 115 includes a bolt 112 extending through a corresponding one of the openings 111 and threadedly engaged in a complementary threaded opening 112a in the upper edge of the associated side plate 72 or 74. Each of the bolts 112 has an enlarged head 113 having a diameter greater than the diameter of the opening 111, whereby the head 113 is supported on the upper surface of the work platen 110. Diametrihead **113** substantially normal to the longitudinal axes thereof is a lever pin 114 which serves as a handle for turning the bolt 112. The head 113 is also provided with an annular collar 116 disposed in surrounding set screws 116a, the collar 116 being provided with a flared flange or skirt 117 extending around approximately one half of the circumference thereof and disposed toward the center of the work platen 110. Each of the flanges 117 has a graduated scale thereon adapted for cooperation with an indicating pointer 118 on the upper surface of the work platen 110. Thus, as the bolt 112 is turned, the graduated scale rotates with respect to the pointer 118 and cooperates therewith to disposed through complementary openings adjacent to 40 indicate the angular position of the bolt with respect to a reference position. A compression spring 119 is also provided in a complementary recess 119a in the upper edge of the associated side plate 72 or 74 adjacent to each of the threaded bolt openings 112a, the upper end of the compression spring 119 being disposed in engagement with the lower surface of the work platen 110 for urging the adjacent corner thereof upwardly away from the side plates 72 and 74. Thus, the rotation of the bolts 112 cooperate with the associated compression springs 119 to provide a controlled upward or downward movement of the associated corner of the work platen 110, the relative vertical position of the associated corner of the work platen 110 with respect to the side plates 72 and 74 being indicated by the graduated scale on the flange 117.

> Mounted on the lower carriage 30 in the work section 103 thereof is a lower deflecting roll assembly, generally designated by the numeral 120, and comprising a plurality of deflecting or work rolls and guide rolls. More particularly, referring to FIG. 10 of the drawings, the deflecting roll assembly 120 comprises nine roll units of like construction, including an inboard edge front guide roll unit 121, an inboard edge work roll unit 122, an inboard edge rear guide roll unit 123, an outboard edge front guide roll unit 124, an outboard edge work roll unit 125, an outboard edge rear guide roll unit 126, a center front guide roll unit 127, a

10

15

center rear guide roll unit 128 and a center work roll unit 129. The roll units 121 to 129 are all arranged as illustrated in FIG. 10, with the roll units 121 to 123 being disposed in longitudinal alignment adjacent to the inboard edge section of the strip 21, with the roll units 5124 to 126 being disposed in longitudinal alignment adjacent to the outboard edge section of the strip 21 and with the roll units 127 to 129 being disposed in longitudinal alignment adjacent to the longitudinal center line of the strip 21. It will be noted that the edge-working roll units 122 and 125 are disposed adjacent to the front end of the work section 103, while the centerworking roll unit 129 is disposed adjacent to the rear end of the work section 103. Each of the work roll units 121 to 129 extends upwardly from the bottom plate 31 toward the planar datum path for the strip 21 defined by the guide roll assemblies 80, as described above.

Similarly, the work platen 110 has mounted thereon an upper deflecting roll assembly, generally designated 20 by the numeral 130, and including a plurality of work roll units and guide roll units, respectively identical in construction to the roll units 121 to 1229 and arranged in a like pattern. However, the upper work roll assembly 130 is offset a slight distance rearwardly of the 25 lower work roll assembly 120 by a distance substantially equal to the radius of one of the roll units 121 to 129. More particularly, the upper work roll assembly 130 comprises nine roll units including an inboard edge work roll unit 131, an inboard edge front guide roll unit ³⁰ 132, an inboard edge rear guide roll unit 133, an outboard edge work roll unit 134, an outboard edge front guide roll unit 135, an outboard edge rear guide roll unit 136, a center front guide roll unit 137, a center 35 work roll unit 138 and a center rear guide roll unit 139. It will be noted that the arrangement of the work roll assembly 130 is substantially identical to the arrangement of the work roll assembly 120, with the exception that the positions of the inboard edge front guide roll 40 unit and work roll unit have been interchanged, the positions of the outboard edge front guide roll unit and work roll unit have been interchanged, and the position of the center front guide roll unit and work unit have been interchanged. Each of the roll units 131 to 139 is 45 rolls 132, 133, 135, 136, 137 and 139 are normally secured to the inner surface of the work platen 110 and extends downwardly therefrom toward the planar datum path of the strip 21 defined by the guide roll assemblies 80.

As indicated above, the roll units 121 to 129 and 131 50 to 139 are all of like construction, whereby only one will be described in detail. Referring, e.g., to FIGS. 8 of the drawings, it will be seen that the center front guide roll unit 137 includes a clevis mounting bracket 141 having the bight thereof secured by a bolt 142 to the 55 inner surface of the work platen 110. Extending between the arms of the clevis 141 and journaled in complementary openings therein is a stub shaft 143 having the axis thereof disposed substantially horizon-60 tally, transversely of the upper carriage 70. A guide roll 145 is mounted between the legs of the clevis 141 for rotation about the axis of the shaft 143, the guide roll 145 having a hub 146 disposed in surrounding relationship with and secured to the shaft 143. Secured to the 65 outer surface of the hub 146 is an annular inner race 147 cooperating with an annular outer race 148 for confining a circular array of ball bearings 149

therebetween. The rolls in the assemblies 120 and 130 disposed along one side edge of the strip 21 are, respectively, arranged coaxially with the corresponding rolls of the same assembly disposed along the opposite side edge of the strip 21.

Each of the guide roll units 121, 123, 124, 126, 127 and 128 is so arranged that the upper edge of the circumferential surface thereof lies in the horizontal plane defined by the upper edges of the lower guide rolls 85. On the other hand, each of the work roll units 122, 125 and 129 is so positioned that the upper edge of the circumferential surface thereof extends upwardly beyond the horizontal plane defined by the lower guide rolls 85, whereby the work rolls 122, 125 and 129 will serve to deflect the adjacent portions of the metal strip 121 upwardly out of the datum path thereof. In like manner, the work platen 110 is normally positioned so that the lower edges of the circumferential surfaces of the guide rolls 132, 133, 135, 136, 137 and 139 all lie in the horizontal plane defined by the upper guide rolls 85, while the lower edges of the circumferential surfaces of the work rolls 131, 134 and 138 extend downwardly below the horizontal surface defined by the upper guide rolls 85, thereby to deflect the adjacent portion of the metal strip 21 downwardly out of the datum path thereof. Thus, referring to FIGS. 11 and 12 of the drawings, it will be seen that the lower work rolls 122, 125 and 129 cooperate with the upper work rolls 131, 134 and 138 to deflect the adjacent edge and center sections of the strip 21 alternately upwardly and downwardly out of the datum path in a sinuous manner for plastically working these sections of the strip 21. The guide rolls 127 and 137 serve fundamentally to hold the center section of the strip 21 in the datum path, while the inboard and outboard edge sections thereof are being worked by the work rolls 122, 125, 131 and 134. Similarly, the guide rolls 123, 126, 133 and 136 serve fundamentally to hold the edge sections of the metal strip 21 in the datum path while the center section thereof is being worked by the work rolls 129 and 138.

It should be observed that, while the upper guide disposed as described above, whereby they cooperate to define the datum path of the metal strip 21, if the work platen 110 is moved downwardly from this normal position, the lower edges of these guide rolls will be disposed below the datum path, whereby these guide rolls will also deflect and work the adjacent sections of the metal strip 21, thereby enhancing the operation of the machine 20.

The operation of the conditioning machine 20 will now be described in detail. The conditioning machine 20 will normally be used in conjunction with other strip processing equipment, such as a strip leveler and a roll former. The assembled carriages 30 and 70 are inserted in position in the strip processing line with suitable mounting pins inserted through the openings 29 in the base members 27. Air conduits are then connected from a suitable source of pressurized air through a main supply valve and appropriate pressure gages to the various air motors 53, 63 and 90 for pneumatic operation thereof. With the air motors all deactuated, the upper carriage 70 is suspended by the suspending rods 96 above the lower carriage 30, as described above. The 3,704,614

12

proper size spacers 77 are then fixed to the side plates 32 and 34 of the lower carriage 30, a plurality of different size spacers normally being provided for use with different gauges of strip metal to be processed. At this -5 point, the air valve may be closed and the air motors actuated for checking the spacers 77 for proper setting of the upper and lower carriages 70 and 30 with respect to each other and for checking the air pressure in the air motors 53 and 63. After deactuating of the air mo-10 tors, the leading end of the metal strip 21 is then threaded between the upper and lower carriages 70 and 30 in the direction of the arrow in FIG. 3. The leading end of the metal strip 21 may then be threaded into the adjacent processing apparatus on the line, which 15 processing apparatus will normally be used to pull the metal strip 21 through the strip conditioner 20. At this point, the air valve is again closed, thereby actuating the air motor 53, 63 and 90 for laterally positioning the metal strip 21 between the edge guide rolls of the roll $_{20}$ assemblies 40, 50 and 60 and for positioning the upper and lower carriages 70 and 30 with respect to each other so that the guide rolls 85 thereof cooperate to define the planar datum path for the metal strip 21. Preferably, the drive assembly 115 of the work platen 25 110 will have been previously set to experimentally determined settings for the particular gauge and width of strip being conditioned. Normally, each of the drive assemblies 115 will be adjusted to the same setting so that the work platen 110 is perfectly level to effect the proper combination of edge section and center section working of the metal strip 21.

However, it will be appreciated that in certain circumstances it may be desirable to vary the amount of 35 work done on the different sections of the strip 21. It is an important feature of the present invention that the work platen 110 is adapted for quick, easy and accurate adjustment of the combination of edge and center working to be done on the strip 21. For example, if it is 40 desired to increase the amount of work done on the center section of the strip 21, it is only necessary to rotate the rear ones of the bolts 112 to lower the rear end of the work platen 110 and, thereby, lower the center work roll 138 a sufficient distance to achieve the $_{45}$ desired additional deflection of the center section of the strip 21 to perform the desired additional work thereon. Similarly, the amount of center working done on the strip 21 may be decreased by raising the rear end of the work platen 110. In like manner, the amount of 50 edge working on the center strip 21 may be increased or decreased by respectively lowering or raising the front end of the work platen 110 for thereby altering the positions of the edge working rolls 131 and 134 with respect to the datum path.

Normally, when one end of the work platen 110 is adjusted, both of the drive assemblies 115 at that end of the work platen will be adjusted to the same setting. However, in rare circumstances, the operator may wish to work one edge of the strip more than the other edge in order to compensate for some unilateral curvature or camber in the strip 21. This can be accomplished by simply making the appropriate adjustment in the drive assemblies on the side of the machine 20 in question. For example, if it is desired to increase the work done on the inboard edge of the strip 21, it is only necessary to move the inboard side of the work platen

downwardly with respect to the datum path. It will be noted that in making such adjustments, it normally will be necessary to maintain the differential between the settings at the front and rear ends of the work platen **110**. Thus, the front and rear drive assemblies **115** on the side of the work platen affected should be adjusted by the same amount.

It will be noted that, in making any desired adjustment in the amount of work to be done on the strip 21, it is only necessary to rotate the bolts 112 of the drive assemblies 115 to the desired setting, an operation which can be performed by an operator in a matter of seconds. Further, it will be noted that the graduated scales on the flanges 117 of the drive assemblies 115 provide the operator with a ready reference for indicating the exact setting of the work platen 110. Thus, if the machine 20 is changed from a first setting to a second setting, and it is later desired to return to the first setting, this latter adjustment can readily be made, since the settings are all clearly indicated on the graduated scales. Furthermore, the effect of the setting adjustments is readily apparent to the operator, since they simply involve moving one end or the other of the work platen 110 upwardly or downwardly with respect to the datum path.

In addition, it will be appropriate that the centralized location of all of the work rolls of the present invention facilitates the removal and servicing thereof. Thus, ac-30 cess to all of the work rolls in the work roll assemblies 120 and 130 may be had by simply removing the work platen 110. This is an important advantage, since the work rolls are the ones which normally require the most frequent servicing and replacement, and this ser-35 vicing and replacement can thus be effected without having to disassemble the whole machine.

The exact degree of adjustment necessary to correct a specific condition of the strip may easily be experimentally determined. However, there is presented below a chart indicating, in general, the direction of adjustment which must be made in each of the drive assemblies 115 for correcting a number of commonly experienced undesirable conditions in the strip 21. In this chart, the front or edge-adjusting drive assemblies have been respectively designated "E.I." for the inboard drive assembly and "E.O." for the outboard drive assembly, and the rear or center-adjusting drive assemblies have been respectively designated as "C.I." for the inboard drive assembly and "L.O." for the outboard drive assembly. For each condition, the chart indicates the direction of change in the amount of work which must be done on each section of the strip to correct the indicated condition, it being appreciated that a 55 lowering of the work platen by the drive assemblies will effect an increase in the work done in the corresponding section of the strip while a raising of the work platen by the drive assemblies will effect a decrease in the amount of work done on the corresponding section of the strip.

Condition	EI	EO	CI	со
Bowing down	Less	Less	More	More
Bowing up	• • • • • • • • • • • • • • • • • • •		Less	Less
Bowing right	More	More		
Bowing left	More	More		
Right-hand twist	Less OR	More	More	More
Left-hand twist	More OR	Less	More	More

It will be appreciated that the conditioning machine 20 is usable for conditioning metal strips having widths which lie in a relatively narrow range of widths. If it is desired to condition the strips having widths outside of this narrow range, it will be necessary to utilize a different set of upper and lower carriages 30 and 70. Accordingly, there will preferably be provided a series of different size sets of upper and lower carriages 30 and 70 for use with different widths of metal strip 21. Thus, while three rows of work rolls are shown in the preferred embodiment of the invention, different numbers of rows of work rolls may be provided for use with different widths of metal strip.

From the foregoing, it will be seen that there has 15 been provided an improved strip conditioning machine providing readily adjustable work rolls centralized in a single location in the conditioning machine. More particularly, there has been provided an adjustable strip conditioner including upper and lower deflecting roll 20 assemblies, the upper deflecting roll assemblies being mounted on a movable platen having the position thereof easily adjustable.

There has also been provided novel drive assemblies for the work platen comprising screw-type mechanisms²⁵ respectively disposed adjacent to the corners of the work platen for effecting a simple and accurately controlled movement of the work platen with respect to the datum path of the strip to be conditioned. Finally, there has been provided a simplified adjustable strip³⁰ conditioner permitting quick and easy access to the work roll assemblies for facilitating the servicing or replacing thereof.

While there has been described what is at present 35 considered to be the preferred embodiment of the invention, it will be understood that various modifications may be made therein, and it is intended to cover in the appended claims all such modifications as fall within the true spirit and scope of the invention. 40

What is claimed is:

1. Apparatus for conditioning an elongated metal strip comprising guide means for supporting a metal strip for longitudinal movement along a predetermined 45 planar datum path, said guide means including a plurality of surface guide rolls spaced apart longitudinally of said datum path and engaging the opposite planar surfaces of the strip for vertically positioning the strip in the datum path and a plurality of edge guide rolls 50 spaced apart longitudinally of the datum path and engaging the opposite side edges of the strip for horizontally positioning the strip in the datum path, a work platen disposed adjacent to said guide means and having the position thereof adjustable with respect to said 55 guide means throughout a range of strip-working positions, and a plurality of deflecting members carried by said work platen and adjustable therewith in directions substantially normal to the plane of the datum path, 60 said deflecting members engaging the strip and cooperating with said guide means for deflecting the strip out of the plane of the datum path plastically to work the strip, adjustment in the position of said platen relative to said guide means serving to change the 65 amount of deflection of portions of the strip in registry with said work platen by deflecting members for thereby adjusting the amount of work done on the strip.

2. The apparatus set forth in claim 1, wherein said work platen is disposed substantially parallel to the plane of the datum path intermediate the ends of said guide means.

3. The apparatus set forth in claim 1, wherein each of said deflecting members comprises a roll mounted for rotation about an axis substantially parallel to the plane of the datum path, the circumferential surface of the roll extending into the datum path for deflecting the strip therefrom.

4. The apparatus set forth in claim 1, wherein said work platen is disposed above and substantially parallel to the plane of the datum path, each of said deflecting members extending downwardly into the datum path for deflecting the strip downwardly out of the datum path.

5. Apparatus for conditioning an elongated metal strip comprising guide means for supporting a metal strip for longitudinal movement along a predetermined planar datum path and having a strip-working section, a plurality of first deflecting members carried by the strip-working section of said guide means, a work platen disposed adjacent to the strip-working section of said guide means and having portions thereof independently adjustable with respect to said guide means throughout a range of strip-working positions, and a plurality of second deflecting members carried by said work platen and adjustable therewith in directions substantially normal to the plane of the datum path, said first and second deflecting members respectively engaging the strip along the opposite planar surfaces thereof and cooperating to effect deflection thereof out of the plane of the datum path plastically to work the strip, adjustment of the position of the portions of said platen relative to said guide means serving to change the amount of deflection of the strip in registry with said work platen by said deflecting members for thereby adjusting the amount of work done on the strip.

6. The apparatus set forth in claim 5, wherein each of said first and second deflecting members comprises a roll mounted for rotation about an axis substantially parallel to the plane of the datum path.

7. The apparatus set forth in claim 5, wherein each of said first and second deflecting members comprises a roll mounted for rotation about an axis substantially parallel to the plane of the datum path, said first deflecting members being equal in number to said second deflecting members and respectively disposed adjacent to corresponding ones thereof, each of said first deflecting members being disposed above and slightly forwardly of the corresponding one of said second deflecting members and cooperating therewith to deflect the strip alternately upwardly and downwardly out of the datum path.

8. The apparatus set forth in claim 5, wherein each of said first and second deflecting members comprises a roll mounted for rotation about an axis substantially parallel to the plane of the datum path, the axes of rotation of said first deflecting rolls being disposed above the datum path and the axes of rotation of said second deflecting rolls being disposed below the datum path.

9. The apparatus set forth in claim 5, wherein each of said first deflecting members is fixedly secured to the guide means and fixedly positioned on the guide means with respect to the datum path.

10. Apparatus for conditioning an elongated metal strip comprising guide means for supporting a metal strip for longitudinal movement along a predetermined planar datum path, a work platen disposed adjacent to 5 said guide means and having portions thereof independently adjustable with respect to said guide means throughout a range of strip-working positions, a drive assembly coupled to said work platen for effecting controlled and independent movement of the portions 10 thereof toward and away from said guide means to adjust the position of said work platen, and a plurality of deflecting members carried by said work platen and adjustable therewith in directions substantially normal to the plane of the datum path, said deflecting members 15 engaging the strip and cooperating with said guide means for deflecting the strip out of the plane of the datum path plastically to work the strip, adjustment of the position of the portions of said platen relative to said guide means serving to change the amount of 20 deflection of the strip by said deflecting members for thereby adjusting the amount of work done on the strip.

11. The apparatus set forth in claim 10, wherein said drive assembly includes a screw coupling said work platen to said guide means, rotation of said screw in 25 one direction effecting movement of said work platen toward said guide means and rotation of said screw in the opposite direction effecting movement of said work platen away from said guide means.

12. The apparatus set forth in claim 10, wherein said ³⁰ drive assembly includes bias means coupled to said guide means and to said work platen for urging said work platen away from said guide means, and a screw coupling said work platen to said guide means, rotation of said screw in one direction causing movement of said ³⁵ work platen toward said guide means and rotation of said screw in the opposite direction accommodating movement of said work platen away from said guide means.

13. The apparatus set forth in claim 12, and further including an indicating gauge coupled to said screw and responsive to rotation thereof for indicating the position of said work platen relative to a predetermined reference position.

14. Apparatus for conditioning an elongated metal strip comprising guide means for supporting a metal strip for longitudinal movement along a predetermined planar datum path, a generally rectangular work platen disposed adjacent to said guide means and extending 50 longitudinally thereof and having the position thereof adjustable with respect to said guide means throughout a range of strip-working positions, four drive assemblies coupled to said work platen for respectively effecting independent and controlled movement of the 55 four corner sections thereof toward and away from said guide means to adjust the position of said work platen, a pair of edge deflecting members carried by said work platen adjacent to one end thereof and adjustable therewith in directions substantially normal to the 60 plane of the datum path, said edge deflecting members respectively engaging the strip adjacent to the opposite side edges thereof and cooperating with said guide means for deflecting the strip out of the plane of the 65 datum path plastically to work the edge sections of the strip, and a center deflecting member carried by said work platen adjacent to the other end thereof and ad-

justable therewith in directions substantially normal to the plane of the datum path, said center deflecting member engaging the strip adjacent to the center thereof for deflecting the center section of the strip out of the plane of the datum path plastically to work the center section of the strip, each of said drive assemblies being independently operable for adjusting the position of the associated corner of the work platen to effect different combinations of deflections of the edge and center deflecting members for thereby adjusting the amount of work done on the edge and center sections of the strip.

15. The apparatus set forth in claim 14, wherein each of said edge and center deflecting members comprises a roll mounted for rotation about an axis disposed substantially parallel to the plane of the datum path.

16. The apparatus set forth in claim 14, wherein each of said drive assemblies includes a screw coupling the associated corner of said work platen to said guide means, rotation of said screw in one direction effecting movement of the associated corner section of said work platen toward said guide means, rotation of said screw in the opposite direction effecting movement of the associated corner section of said work platen away from said guide means.

17. The apparatus set forth in claim 16, wherein each of said drive assemblies includes an indicator gauge responsive to rotation of said screw for indicating the position of the associated corner section of said work platen with respect to a predetermined reference position.

18. Apparatus for conditioning an elongated metal strip comprising guide means for supporting a metal strip for longitudinal movement along a predetermined planar datum path and having a strip working section, a pair of first edge deflecting members carried by the strip-working section of said guide means adjacent to one end thereof and respectively engaging one planar surface of the strip adjacent to the opposite side edges thereof, a first center deflecting member carried by the strip-working section of the guide means adjacent to the other end thereof and engaging said one planar surface of the strip adjacent to the center thereof, a generally rectangular work platen disposed adjacent to the strip-working section of said guide means and substantially coterminous therewith and having the position thereof adjustable with respect to said guide means throughout a range of strip-working positions, four drive assemblies coupled to said work platen for respectively effecting controlled movement of the four corner sections thereof toward and away from said guide means to adjust the position of said work platen, a pair of second edge deflecting members carried by said work platen adjacent to said first edge deflecting members and adjustable with said work platen in directions substantially normal to the plane of the datum path, said second edge deflecting members respectively engaging the other planar surface of the strip adjacent to the opposite side edges thereof and cooperating with said first edge deflecting members to effect deflection of the strip alternately in opposite directions out of the plane of the datum path plastically to work the edge sections of the strip, and a second center deflecting member carried by said work platen adjacent to said first center deflecting member and adjustable with said work platen in directions substantially normal to the plane of the datum path, said second center deflecting member engaging the other planar surface of the strip adjacent to the center thereof and cooperating with said first center deflecting 5 member to effect deflection of the strip alternately in opposite directions out of the plane of the datum path plastically to work the center section of the strip, each of said drive assemblies being independently operable for adjusting the position of the associated corner of 10 the work platen to effect different combinations of deflections of the edge and center deflecting members for thereby adjusting the amount of work done on the edge and center sections of the strip.

19. The apparatus set forth in claim 18, wherein each 15 of said first and second edge and center deflection members comprises a roll mounted for rotation about an axis disposed substantially parallel to the plane of the datum path, said first deflecting rolls being equal in number to said second deflecting rolls and respectively 20 disposed adjacent to corresponding ones thereof, each of said first deflecting rolls being offset longitudinally a slight distance from the corresponding one of said second deflecting rolls and cooperating therewith to alternately deflect the adjacent section of the strip in first 25 and second directions out of the datum path.

20. The apparatus set forth in claim 18, wherein each of said first and second edge and center deflecting members comprises a roll mounted for rotation about an axis disposed substantially parallel to the plane of 30 the datum path, the axis of rotation of said first guide rolls are disposed above the plane of the datum path and the axes of rotation of said second guide rolls are disposed below the plane of the datum path, said first and second guide rolls cooperating to deflect the ad- 35 jacent sections of the strap alternately upwardly and downwardly out of the datum path.

21. Apparatus for conditioning an elongated metal strip for longitudinal movement along a predetermined 40 deflecting the strip out of the plane of the datum path strip comprising guide means for supporting a metal planar datum path, a work platen disposed adjacent to said guide means and extending longitudinally thereof and having the position thereof adjustable with respect to said guide means throughout a range of strip-working positions, a pair of edge deflecting members carried 45 member carried by said work platen adjacent to the by said work platen adjacent to one end thereof and adjustable therewith in directions substantially normal to the plane of the datum path, said edge deflecting members respectively engaging the strip adjacent to the opposite side edges thereof and cooperating with said 50 guide means for deflecting the strip out of the plane of the datum path plastically to work the edge sections of the strip, center guide means for holding the center longitudinal section of the strip in the datum path at points disposed transversely of said edge deflecting members, 55 a center deflecting member carried by said work platen adjacent to the other end thereof and adjustable therewith in directions substantially normal to the plane of the datum path, said center deflecting member engaging the strip adjacent to the center thereof for 60 deflecting the center section of the strip out of the

plane of the datum path plastically to work the center section of the strip, and edge guide means for holding the edge sections of the strip in the datum path at points disposed transversely of said center deflecting member, adjustment in the position of said platen relative to said guide means serving to change the amount of deflection of the strip by said deflecting members for thereby adjusting the amount of work done on the strip.

22. The apparatus set forth in claim 21, wherein each of said center guide means and said edge guide means includes a pair of rolls respectively disposed on opposite sides of the plane of the datum path, each pair of guide rolls cooperating fundamentally to vertically position the adjacent section of the strip therebetween in the datum path.

23. The apparatus set forth in claim 21, and further including a plurality of auxiliary guide members respectively disposed adjacent to said edge and center deflecting members substantially in longitudinal alignment therewith for facilitating the working of the strip by the deflecting members.

24. Apparatus for conditioning an elongated metal strip comprising guide means for supporting a metal strip for longitudinal movement along a predetermined planar datum path, a generally rectangular work platen disposed adjacent to said guide means and extending longitudinally thereof and having the position thereof adjustable with respect to said guide means throughout a range of strip-working positions, four drive assemblies coupled to said work platen for respectively effecting controlled movement of the four corner sections thereof toward and away from said guide means to adjust the position of said work platen, a pair of edge deflecting members carried by said work platen adjacent to one end thereof and adjustable therewith in directions substantially normal to the plane of the datum path, said edge deflecting members respectively engaging the strip adjacent to the opposite side edges thereof and cooperating with said guide means for plastically to work the edge sections of the strip, means for holding the center longitudinal section of the strip in the datum path at points disposed transversely of said edge deflecting members, a center deflecting other end thereof and adjustable therewith in directions substantially normal to the plane of the datum path, said center deflecting member engaging the strip adjacent to the center thereof for deflecting the center section of the strip out of the plane of the datum path plastically to work the center section of the strip, and means for holding the edge sections of the strip in the datum path at points disposed transversely of said center deflecting member, each of said drive assemblies being independently operable for adjusting the position of the associated corner of the work platen to effect different combinations of deflections of the edge and center deflecting members for thereby adjusting the amount of work done on the edge and center sections of the strip.

65