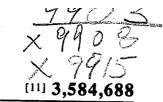
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| [21] | Appl. | No. 84 | eorge A. Utesch, Jr., Chicag 17,162 | zo, both of, Ill. | |
| [22] | Filed | | ug. 4, 1969 | | |
| | Paten | | ine 15, 1971 | | |
| [73] | Assigr | | eneral Fire Extinguisher Co | rporation | |
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| [54] METHOD FOR CONTROLLING FIRES 7 Claims, 9 Drawing Figs. | | | | | |
| [52] | U.S.C | l. | ••••••••••••••••••••••••••••••••••••••• | 169/2, | |
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| [50] | Field o | f Search | | 169/1, 2, 5, | |
| | | | 9, | 11, 19, 26, 20 | |
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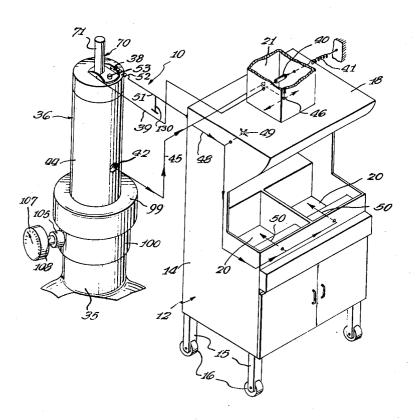
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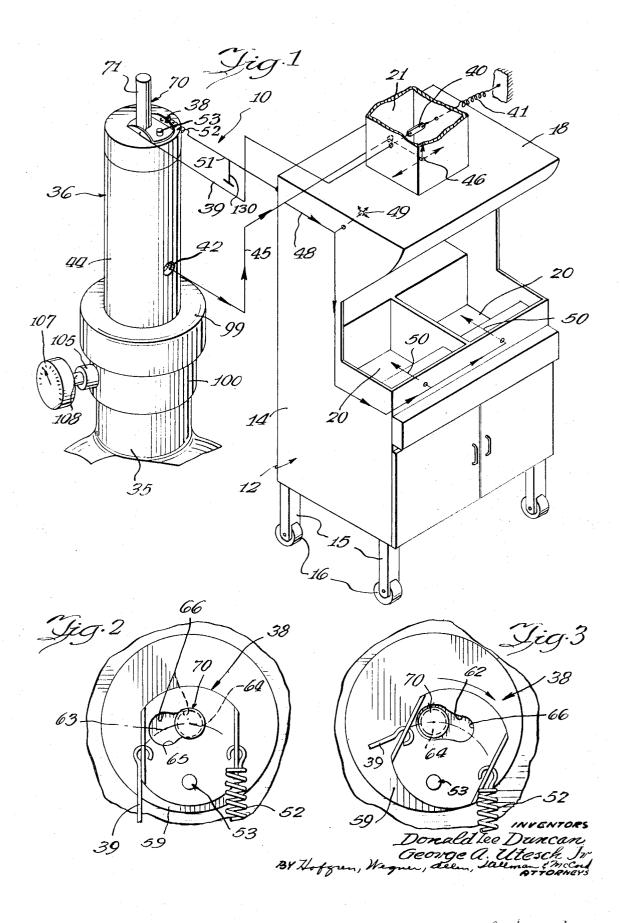
ABSTRACT: A method or system for controlling fires is provided particularly for use in areas where food is prepared using fats and greases. The method or system provides for detecting flame or excessive heat, the presence of which automatically releases a high volume flow of extinguishant over at least one of several critical areas of the fire so as to rapidly eliminate any flame and rapidly reduce the temperature in the critical areas. The method or system then provides for automatically reducing the rate of flow of the extinguishant from the same source of extinguishant and continues to flow the extinguishant at said reduced rate for a considerable period of time sufficient to reduce the temperature of the area below the flame auto-ignition point.



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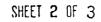
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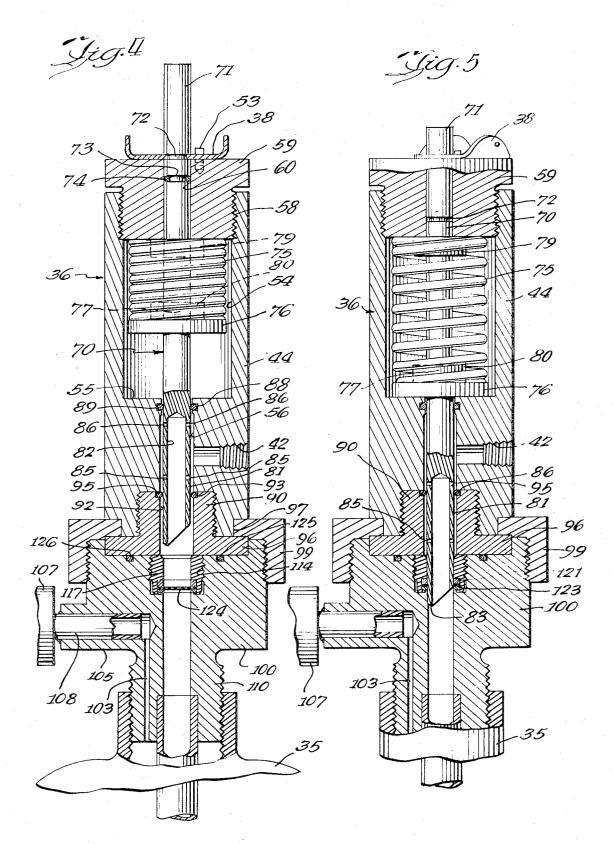




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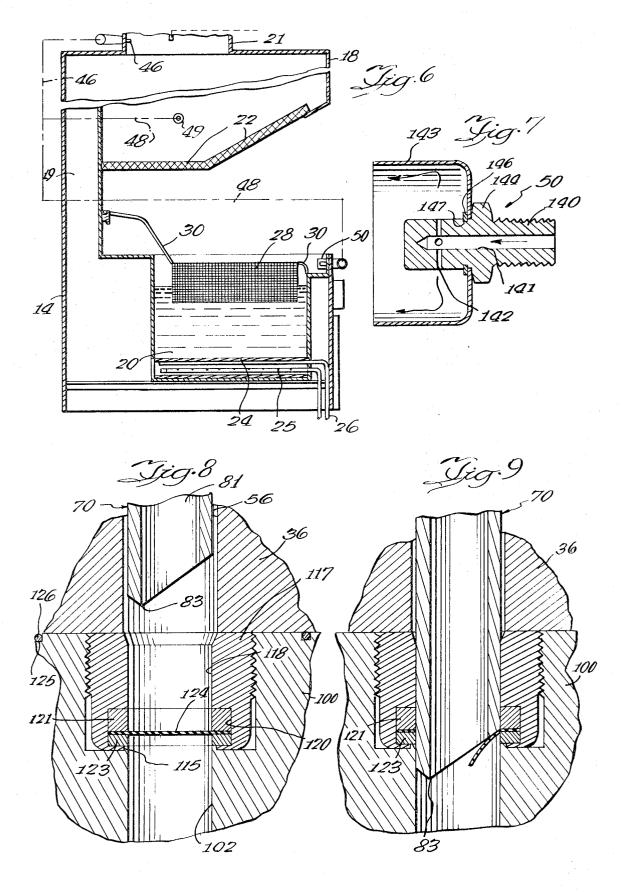




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METHOD FOR CONTROLLING FIRES

BACKGROUND OF THE INVENTION

1. Field of the Invention

This invention relates to a method or system for controlling fires and more particularly to a method or system for controlling fires in areas where food is prepared by the use of fats or grease.

2. Description of the Prior Art

In fire control, one of the hardest fires to put out and to keep out is a fire that develops in and around areas where considerable amounts of fat or grease are used in food preparation. This is particularly true in the equipment known as "deep 15 ments of the fire control apparatus; fryers" where potatoes, fish and other foodstuffs are immersed in a vat of fat or grease which is generally maintained at temperatures around 300° to 400° Fahrenheit.

Temperature controls are provided so that if the temperature of the material in the vat exceeds a predetermined 20 amount, the heat source for the vat is shut off. Occasionally, the controls fail and the temperature continues to rise until the auto-ignition point is reached.

Another source of fire around a cooking area is created by the accumulation of grease and fat in the hood or exhaust duct 25 area above the appliance where, when sufficient grease or fat has accumulated and the temperature rises above the auto-ignition point, a fire can develop from burning the fats or greases

It has generally not been too difficult to put out the flame of 30 either a vat fire or a hood and exhaust duct fire, but it has been difficult to keep the fire out because the flame will raise the temperature to around 800° F. so that, even though the flame is momentarily extinguished, the grease will immediately reignite as long as the temperature is above the auto-ignition 35 point.

In current use there are two systems operating on different principles for controlling the reignition of the flames. The first is to blanket the grease and fat area with powder to smother the flame and prevent reignition. The second approach is to cool the grease and fat of the surrounding area by an agent that has a cooling effect.

Of the two alternative systems on the market today, the first system, wherein powder is mixed with a propellant such as CO_2 and is spread as a blanket over the affected area, has the 45principal disadvantage that the equipment has to be carefully watched until it cools down below the auto-ignition point and then has to be completely cleaned before it can be reused. This means discarding large amounts of usable fats and/or grease and the like.

The second currently used system employs two cylinders of carbon dioxide (CO_2) separately piped to the cooking area, wherein the first cylinder applies the CO₂ at a fast rate to extinguish the flame and then the second cylinder is activated to propel CO₂ at a slow rate over the grease to cool the grease below the autoignition point. Carbon dioxide by its nature requires large volumes of material in order to effectively control the more common fires in deep fry units and the like.

SUMMARY OF THE INVENTION

In our invention, we provide an improved method or system of fire control using a single container of extinguishant and a valve arrangement which makes it possible to control the flame and cool the area effectively by a unique combination of 65 steps. The steps broadly include detecting the presence of excessive heat or a flame, automatically operating a valve for releasing a large volume of extinguishant into at least one of several areas for rapidly extinguishing any flame that may be present. The method or system then provides for automati- 70 cally reducing the rate of flow of extinguishant from the cylinder and continuing to flow the extinguishant at the reduced rate over the affected area for a considerable period of time, thereby cooling the area well below the autoignition point.

DESCRIPTION OF THE DRAWINGS

FIG. 1 is a perspective view of a deep-fry piece of equipment with a schematic showing the location of the improved fire control apparatus thereon;

FIGS. 2 and 3 are top plan views of two positions of a cam release mechanism of the invention;

FIG. 4 is a cross-sectional view taken along a plane passing vertically through the valve, showing the valve in cocked posi-10 tion;

FIG. 5 is a cross-sectional view similar to FIG. 4 showing the valve in the initial operating position;

FIG. 6 is a cross-sectional schematic view taken along a vertical plane of FIG. 1 showing the relationship of various ele-

FIG. 7 is an enlarged cross-sectional view of one of the improved fire control nozzle arrangements;

FIG. 8 is an enlarged cross-sectional view showing the puncturing member of the valve in the cocked position, and

FIG. 9 is an enlarged cross-sectional view similar to FIG. 8, only showing the valve puncturing member in puncturing position. Referring now to the drawings, there is shown in FIG. 1 and 6 a fire control apparatus 10 of the instant invention as applied to a piece of commercial deep-fry equipment 12 of the type in common use in restaurants offering fried foods, such as French fried potatoes, fried shrimp, fried fish and the like. It is intended that the fire control apparatus, method or system could be applied to other pieces of equipment where an initial high volume, short duration blast of extinguishant is required followed immediately by a lower volume but considerably longer duration flow of extinguishant, first to extinguish the flame and then to lower the temperature of burnable materials and the surrounding equipment.

The illustrated deep-fry equipment 12 is composed of a stainless steel or porcelain body 14 having legs 15 and casters 16 upon which the equipment can be moved if desired. Doors 17 are provided on the front of the body 14 for storage of cooking implements (not shown). A vent portion 19 extends upwardly from the rear of the body 14 and has a hood 18 pro-40 jecting forwardly in overhanging but vertically spaced relationship to a pair of open vats 20 formed in the body 14. A stack or exhaust duct 21 communicates with the inside of the hood 18 and is connected to an exhaust system for the restaurant. The hood 18 is provided with removable screens or filters 22 which are intended to filter the fumes and vapors and to collect droplets of fat or grease that are carried upward from the hot fat or grease in the vats 20 as is well known in the food preparation art.

As best shown in FIG. 6, the vats 20 have ducts 24 passing 50 through the lower portion thereof in which an appropriate heat source 25 is positioned to supply heat to the vats 20 for maintaining the fat or grease in the vat at a predetermined cooking temperature. In the illustrated form, the heat source 25 is a flame fed by a gas source through pipes 26 with the exhaust gases and products of combustion going through the ducts 24, vent portion 19 into the hood 18 and up the stack 21. The source of heat could be an electric unit or any other compatible system. An appropriate set of controls and ther-

60 mostats are provided in the vat 20 and in the piping 26 whereby the desired temperature of the fat or grease can be set and automatically maintained. In normal use of the equipment, the vat 20 is filled to the correct level with fat or grease and by means of the heat source 25 it is heated to the desired temperature. The potatoes or other food to be deep fried are placed in a basket 28 which is suspended in the hot fat or grease by means of appropriate suspension means 30. The vapors and odors from the cooking are drawn up through the filter 22 and on up the stack, by action of an exhaust fan.

Occasionally, due to a faulty heat control or thermostat or due to improper or poor maintenance of the equipment or due to inexperienced operating personnel, the cooking temperatures become elevated to a point that flames will auto-ignite either from the vat, in the hood or in the stack. The flame will 75 immediately raise the temperature of the fat or grease in the

vat, hood and stack to a point that even though the flame is momentarily extinguished, it will immediately reauto-ignite into flame again. Therefore, to effectively bring the fire under control not only must the flame be extinguished but also the surrounding area must be cooled to below the auto-ignition 5 point of the fats or greases. The invention is directed to a novel valve and nozzle in a fire control apparatus whereby the flame is extinguished and the temperature of the affected area is reduced to prevent reauto-ignition of the flame.

As shown schematically in FIGS. 1 and 6, the fire control 10 apparatus is comprised of a cylinder 35 containing an appropriate extinguishant under pressure. The cylinder 35 is held in a bracket on the sides or back of the body 14 or can be nested in a frame on the top of the hood 18. The exact location is primarily a matter of servicing convenience. The only requirement is that it be fixed relative to the stack or exhaust duct 21. The fire extinguisher head assembly 36 is removably connected to the cylinder 35, as will be described hereinafter, and has a release cam 38 pivotally mounted thereon which is 20 connected by a cable 39 to a fusible link 40 resiliently positioned in the stack or exhaust duct 21 by a spring 41. Flame or excessive heat in the stack or exhaust duct 21 will melt the link 40 releasing the cable 39 and release cam 38 to actuate the fire extinguisher and head assembly 36 as will be described in 25 detail hereinafter.

The head assembly 36 has an outlet 42 in the wall of the discharge valve body 44 through which outlet 42 extinguishant will flow. Appropriate piping 45 connects outlet 42 with a nozzle 46 in the stack or exhaust duct 21. The piping 45 30 valve stem. has a branch 48 which connects in one portion through body 14 to a nozzle 49 in the hood 18 above filters 22 therein and in another portion to a pair of nozzles 50 projecting into the area above the fat or grease in the vats 20. A cable 51 is connected to a spring 52 which in turn is connected to the release cam 38 35 for pivoting the cam 38 in a clockwise direction about a pivot 53. The spring 52 is not as strong as spring 41 so that spring 41 overcomes the loading of spring 52. The cable 51 and spring 52 are used to release the cam 38 when the frangible link 40 disintegrates due to a fire in the equipment. 40

The head assembly 36 as shown in FIGS. 4 and 5 has the discharge valve body 44 bored or recessed at 54, which bore extends from one end thereof throughout a major portion of the length of the body and terminates in a shoulder 55 through which shoulder an aperture 56 extends axially the remaining 45 length of the body. The mouth of the recess 54 is internally threaded at 58 into which is screwed a valve cap 59 which is apertured at 60 in axial alignment with aperture 56 in the body 44. The pivot pin 53 for the release cam 38 is seated in the external end face of the cap 59 about which pin the release cam 5038 pivots. The cam 38 has an arcuately shaped keyhole slot 62 therethrough (FIGS. 2, 3) with the centers 63, 64 respectively, of the large diameter end portion 65 and the small diameter end portion 66 lying on the same circle, which circle is cen-55 tered at the center of the pivot pin 53. The circle through the centers 63, 64 of the keyhole slot 62 also passes through the center of the aperture 60 in the valve cap 59.

An elongate valve stem 70 is slidably mounted in the aligned apertures 60 and 56 in the cap 59 and valve body 44, respectively, with one end portion 71 extending upwardly beyond the cap 59 and in which portion 71 is formed a pair of axially spaced annular grooves or recesses 72, 73. An O-ring 74 is seated in recess 73 for sealing the stem 70 with the cap 59. Within the valve body 44 and surrounding the valve stem 70 is 65 a compression spring 75 which is retained between the valve cap 59 and a valve retaining collar 76 by means of collar drive pin 77 passing through the collar 76 and the valve stem 70. The cap 59 has a spring centering member 79 fixed on the inner surface thereof such that the spring 75 will be centered 70 with respect to the valve stem 70 by the combined action of said member 79 and the stepped down shelf 80 on the retaining collar 76.

The lower end portion 81 of the valve stem 70 is hollowed

to form a puncturing point 83. At appropriate locations along the length of said end portion 81 is formed at least two pairs of axially spaced apart outlet openings 85, 86. Each pair of openings will preferably consist of one opening in one side and another opening diametrically opposite thereto but variations therefrom have been found to work successfully. In the wall of the opening 56 of the valve body 44 is an annular recess 88 in which is seated an O-ring 89 which is adapted to provide a seal between the valve stem 70 and the valve body 44. The recess 88 and O-ring 89 are located between the outlet 42 of the valve body 44 and the recess 54 in which the spring 75 is located to prevent extinguishant from leaking into said cavity 54 and to prevent loss of pressure in the system.

The valve stem 70 with the spring 75 trapped between the 15 valve cap 59 and the retaining collar 76 is assembled with the valve body 44 by threading the end portion 81 with the puncture point 83 through the opening 56 so that the threaded cap 59 can be drawn down tight in the valve body. An orifice shut off bushing 90 has an aperture 92 through the center thereof which aperture has a recess forming a shoulder 93 at the inner end thereof on which shoulder 93 an O-ring or gasket 95 is nested. The bushing 90 is inserted in the threaded lower end of the valve body 44 with the puncture point 83 on the lower end portion 81 of the valve stem passing into the aperture 92. The O-ring or gasket 95 sealingly engages the outer surface of the valve stem 70 as the bushing is threaded into nested relation in the valve body. With the bushing 90 drawn up tight the gasket 95 will form a pressure seal between the valve body and the

Trapped between a flange 96 on the bushing 90 and the walls of a recess 97 formed in the one edge of valve body 44 is the discharge valve retaining collar 99. The fit between the collar 99 and the valve body 44 will permit the collar to turn relative to the valve body. The collar 99 has an opening through its center for accommodating passage of the valve body 44 and is internally threaded in its cylindrical portion for attachment to the refillable cylinder or tank 35 of extinguishant.

Carried on the cylinder or tank 35 is a storage cylinder valve head 100 which is axially bored at 102. A bore 103 is disposed parallel to the axial bore 102 and intersects with a bore 104 in a threaded hub 105 projecting from the sidewall of said head 100. A standard pressure gauge 107 with a threaded stub 108 is threaded into the bore of hub 105 so that when the threaded nipple 110 on the valve head 100 is screwed into the cylinder or tank 35, the pressure in the cylinder will be recorded on the gauge 107. The valve head 100 has a syphon tube 112 connected in the nipple 110 and projects down into the interior of the cylinder or tank 35 in the usual manner for the most efficient method of discharging the extinguishant from the cylinder.

An oversized bore 114 is concentrically formed with respect to the bore 102 in the valve head 100. The bore 114 is threaded and has a valve seat 115 formed in the base of the bore around the edge of the bore 102. A rupture disc housing 117 has an opening 118 through the center thereof and is threaded on its outer surface. The one end portion of the 60 opening 118 in the housing 117 is enlarged as at 120 for receiving a pair of washers 121, 123 between which is sandwiched a rupture disc 124. With the washers and disc 124 seated therein, the housing 117 is threaded into the bore 114 in the valve head 100 so that the washer 123 seats on the valve seat 115 for pressure sealing the rupture disc housing 117 to the valve head 100.

In the system, the cylinder or tank 35 is to be filled and sealed with the just described valve head 100 and rupture disc 124. The top wall of the valve head 100 has a circular recess 125 formed therein in which an O-ring or gasket 126 is nested which serves as a seal between the valve head 100 and the head assembly 36 when the two are assembled.

With a fire control system installed on a deep fry piece of equipment, the valve head assembly 36 is held firmly in a or tapped as at 82 with the end severed at an angle and dressed 75 housing or bracket either on the piece of equipment to be fire

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protected or close to it. The valve head assembly 36 which includes principally the valve body 44, valve stem 70, spring 75, collar 76, cap 59 and release cam 38 is cocked or loaded by manually or mechanically pulling the valve stem 70 upward out of the cap 59 until the recess 72 is aligned with the release cam 38 whereupon the release cam 38 is pivoted until the edges of the small diameter end portion 66 of the keyhole slot 62 nest in the recess 72. Release of the valve stem 70 will permit the coiled spring 75 to urge the top wall of the recess 72 against the release cam 38 to hold the valve system in the 10cocked position. The cable 39 extends from the release cam 38 through one wall of the stack or exhaust duct 21 and is connected to one side of the frangible link 40. The other side of the frangible link 40 is connected through the wall of the stack 21 to a spring 41 which is anchored to a fixed member. The cable 39 is resiliently connected to the release cam 38. The cam release spring 52 is connected at one end to a fixed point through the cable 51 and is stretched and connected at its other end to the release cam 38. The cable 39, frangible link 20 40 and spring 41 are loaded sufficiently to hold the release cam 38 against the tension of spring 52, with the wall 66 of the slot in the cam 38 nesting in the recess 72 in the valve stem 70 as shown in FIG. 2 which is the cocked position of the mechanism shown in FIG. 4. The spring 52 is loaded and at- 25 tempts to pull the release cam 38 in a clockwise direction to the position of FIG. 3 against the higher loading of spring 41 in cable 39. A hand release 130 is connected to the cable 51 and hangs in a position where it can be quickly grabbed and pulled to overcome the tension of the spring 41 to pivot the release 30 cam 38, thereby releasing the valve stem 70 and actuating the system.

The valve head assembly 36 is connected to a charged cylinder 35 by threading the collar 99 of the assembly 36 onto the storage cylinder valve head 100. The gasket 126 is com- 35 pressed between the head assembly 36 and the storage valve head 100 to effect a pressure seal therebetween.

It will be noted that the puncture point of the valve stem 70 is spaced from the rupture disc 124 as shown in FIG. 5.

36 connected through piping 48 to the nozzles 46, 49 and 50. As illustrated in FIG. 7, the nozzle 50-which can be the same as nozzles 46 and 49-is of a novel design and includes a nipple 140 which is blind bored as at 141 with branch bores 142 radiating at right angles outwardly therefrom. A deflector 45 hood or cup 143 is trapped between the enlarged integral nut 144 and a snap ring 146 seated in a groove 147 in said nipple 140. The deflector hood 143 has cylinder sidewall 149 spaced outwardly and in overlapping relationship with respect to the bores 142. The extinguishant from the cylinder is propelled from the bores 142, strikes the wall 149 of the hood 143 and is deflected horizontally in a scrambled random pattern as a blanket onto the fat or grease in the vat 20. It is important to get the extinguishant out of the nozzle fast but not so fast as to 55 create excessive turbulence which can spread the flame to other surrounding areas. The hood 143 lays the extinguishant down as a blanket over the fire to extinguish the flame and cool the area. When the nozzle 50 is used in the hood or stack, the extinguishant flows over and around the inside of the hood 60 18 and screen 22 and into the stack or exhaust port 21. The nozzle 50 causes the extinguishant to engulf the affected areas so as to extinguish the flames and cool the equipment. The nozzle 50 is shown located on the front of the vat nearest the station occupied by an attendant so that the blast of extin- 65 tures can still be well above the auto-ignition temperature. guishant takes the flame away from the station as it extinguishes the flames and cools the equipment.

In operation, when a flame auto-ignites from the deep fry equipment, it will immediately ignite any grease condensed in the hood or stack, rapidly raising the temperature of all af- 70 fected areas. The frangible link 40 in the stack will disintegrate, releasing the cable 39 whereupon the spring 52 pivots the release cam 38 disengaging the small end 66 of the keyhole slot 62 from the recess 72 in the valve stem. The compressed spring 75 will drive the valve stem 70 through the en- 75 reach or exceed 900° F., no decomposition takes place and

larged end 65 of the keyhole slot 62 and puncture the rupture disc 124 with the piercing point 83. The pressure of the extinguishant in the cylinder or tank 35 will force the extinguishant up the bore 102, into the end of the hollow portion 81 of the valve stem 70 and drive the valve stem back up through the valve body 44 to recompress the spring 75. The extinguishant will rapidly flow through both pairs of openings 85, 86 and out the orifice 42 through the piping 45 to the nozzles 46, 49 and 50. After a short period of time of high volume flow of extinguishant, the pressure in the cylinder or tank 35 will drop a sufficient amount that the pressure of the spring 75 will overcome the pressure in the tank and the valve stem 70 will be forced down by the spring 75 until the ports or openings 85 pass below the O-ring or gasket 95 which will shut off the flow of extinguishant from said ports 85. The result of shutting off ports 85 will be to cut in half the flow rate of extinguishant which when coupled with the reduction in pressure in the cylinder 35 will reduce the flow of extinguishant from the orifice 42 to about one-half of the original flow rate. At the reduced rate of discharge the extinguishant will continue to flow for a considerable period of time until the cylinder 35 is exhausted.

The purpose of the high volume, short duration-low volume, long duration sequence of discharge of extinguishant is to make it possible to properly control fires of the type described hereinbefore. That is, the initial high volume blast of extinguishant flooding the vat, the hood area and the stack quickly puts out the flame. However, if at that point the extinguishant ceased flowing, the flame would reignite since the temperatures of the fat, grease and surrounding surfaces are usually well above the auto-ignition point. By automatically changing the flow rate to a substantially reduced amount of continuing at that reduced rate, the extinguishant acts as a coolant and lowers the temperature of the fat, grease and surrounding area so that after the passage of a predetermined period of lower volume discharge the possibility of further auto-ignition is eliminated.

The deep fry equipment can be cleaned up, a new frangible The piping 45 from the outlet 42 of the valve head assembly 40 link 40 installed, the valve head assembly 36 recocked and a new cylinder 35 assembled thereto ready for an immediate return of the equipment to service.

A specific operative system that has been successfully used will now be described. That is, a fire control system for protection of a specific piece of deep frying cooking equipment utilized a cylinder 35 holding approximately 20 pounds of Freon 1301 (a commercially available extinguishant) pressurized with nitrogen. The pairs of ports 85 and 86 in the valve stem 70 were designed to emit a part of the Freon 1301 in an initial blast. This initial blast was a high volume, short duration blast of extinguishant sufficient to extinguish the flames of a fire. The valve automatically cuts one pair of ports 85 out of the system and continues to discharge the extinguishant at a substantially reduced rate whereby the remaining Freon 1301 is discharged over an extended period of time until the cylinder is exhausted. This slower but continuous flow of extinguishant cools the equipment to below the auto-ignition point.

Typical deep fry equipment cooks at between 300° F. and 400° F. Failure of a control will permit the temperature to go up until at around 640° F. the fat or grease can auto-ignite into flame. The flames generate temperatures of around 800° F. such that when the initial blast of extinguishant floods the area, the flame is extinguished, but the surrounding tempera-The reduced flow rate over the longer period of time cools the surrounding area to a safe temperature well below the auto-ignition temperature.

The manner that Freon 1301 operates to extinguish a flame and to cool burning material is within the skill of a person in the art and will not be repeated here. Any other extinguishant having similar physical characteristics such as density, cooling capacity, heat absorption and the like is usable in the system.

By effectively controlling the fire before the temperatures

since the Freon 1301 is nontoxic, it will not taint the cooking equipment. Accordingly, after a fire the equipment can be returned to service without extensive cleaning and reconditioning being required.

We claim:

1. A method for controlling fires, comprising the steps of rupturing a frangible element by exposure to excessive heat, releasing a high pressure and high volume flow of extinguishant from a storage container into a system of fire control piping, directing the high pressure and high volume flow of 10 extinguishment over the burning area, automatically reducing the pressure and volume of flow of extinguishment from the same source of extinguishment and over the same area, whereby the large volume flow will extinguish any flame and the reduced flow will cool the area below the rekindling point. 15

2. The method for controlling fires as claimed in claim 1 and wherein the high volume and high pressure flow of extinguishment is maintained for a relatively short period of time and the reduced volume flow is maintained for a comparatively long period of time.

3. The method for controlling fires as claimed in claim 1 wherein the ratio of the duration of the reduced volume flow to the large volume flow is of the order of 18 to 1.

4. A method for controlling grease fires, comprising the steps of initially directing a large volume flow of extinguish-25 ment over the burning area, continuing said large volume stream of extinguishment for a short period of time, automatically reducing the volume of the flow of extinguishment from the same source of extinguishment and over the same area, continuing the reduced volume of flow of extinguishment for a 30 considerably longer period of time whereby the large volume flow will extinguish any flame and the reduced volume flow over a longer period of time will cool the area below the rekindling point.

5. A method for controlling grease fires in a vat having a 35 chimney vented hood thereover comprising the steps of initially directing a high pressure, high volume, diffused stream

of extinguishment generally horizontally over and across the surface of the vat, flooding said high pressure, high volume, extinguishment into the open area of the hood, and directing said high pressure, high volume, extinguishment in the chimney, all for extinguishing any flame therein, automatically reducing the pressure and the volume of the extinguishment from the same source of extinguishment and directing the same over and across said vat, into said area in the hood and into the chimney, said lower pressure, lower volume stream of extinguishment serving to cool the grease and surrounding surfaces to below the rekindling point thereof so as to prevent rekindling of said grease fire.

6. The fire control method of claim 5 wherein the duration of said high pressure, high volume, stream of extinguishment is relatively short when compared with the longer duration, lower pressure, lower volume flow of extinguishment.

7. A method for controlling fires in a control zone using fire control equipment having a source of pressurized extinguishment, a valve assembly operatively connected to said source 20 of extinguishment and having an outlet port, means connecting said outlet port with a nozzle directed toward said control zone, and means associated with said control zone for detecting the presence of flame or excessive heat, said method comprising the steps of detecting flame or excessive heat, said method comprising the steps of detecting flame or excessive heat by said last named means, valve means in said valve assembly activated by said detecting means for releasing the pressurized extinguishment, the pressure of said extinguishment opening said valve means for high volume flow of extinguishment to said nozzle and from the nozzle over said control zone, reduction of pressure of extinguishment in said source permits the valve means to reduce the volume of flow to a lower level, flowing said lower volume flow of extinguishment over the control zone for a period of time considerably longer than the high volume flow.

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