# (12) UK Patent Application (19) GB (11)

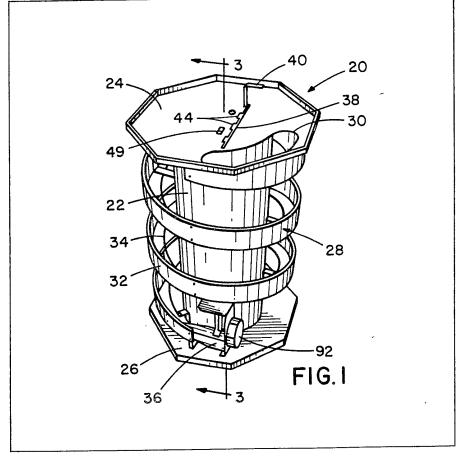
## 2053699 A

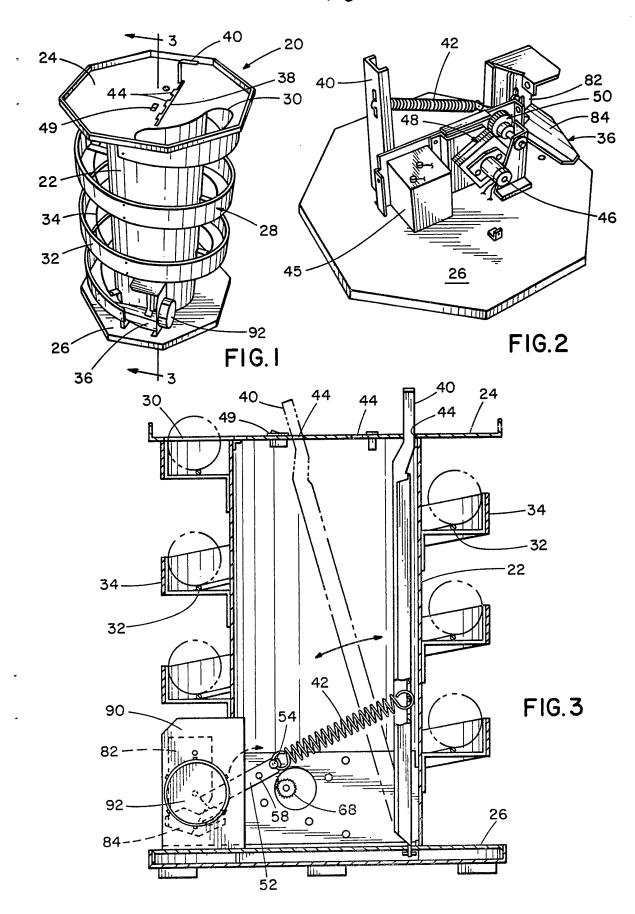
- (21) Application No 8022457
- (22) Date of filing 9 Jul 1980
- (30) Priority data
- (31) 59984
- (32) 23 Jul 1979
- (33) United States of America (US)
- (43) Application published 11 Feb 1981
- (51) INT CL<sup>3</sup>
  A63B 69/40
- (52) Domestic classification A6D 11G
- (56) Documents cited **GB 1310661**
- (58) Field of search A6D
- (71) Applicants
  Rallymaster, Inc.,
  2625 American Lane,
  Elk Grove Village,
  Illinois 60007,
  United States of America.
- (72) Inventors

  Jacob Simon Haller
- (74) Agents Raworth, Moss & Cook

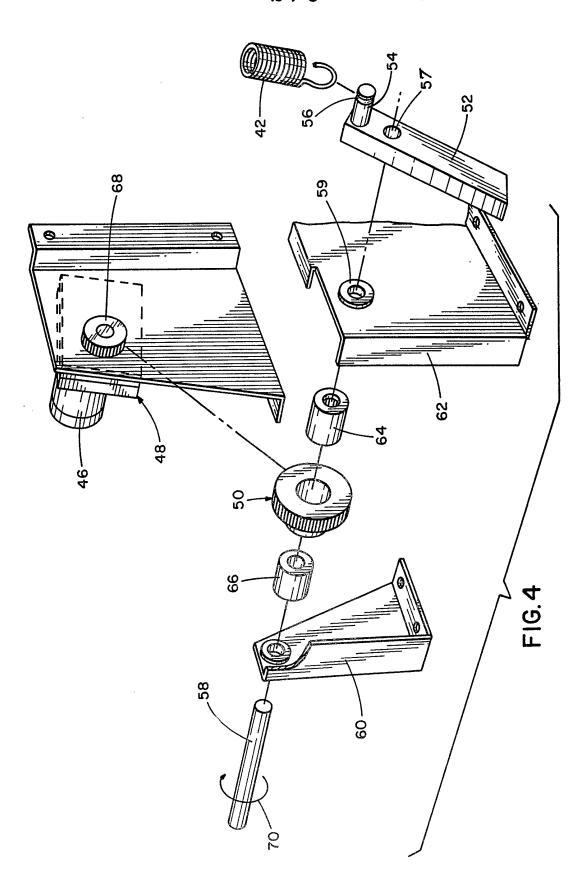
### (54) Portable ball throwing machine

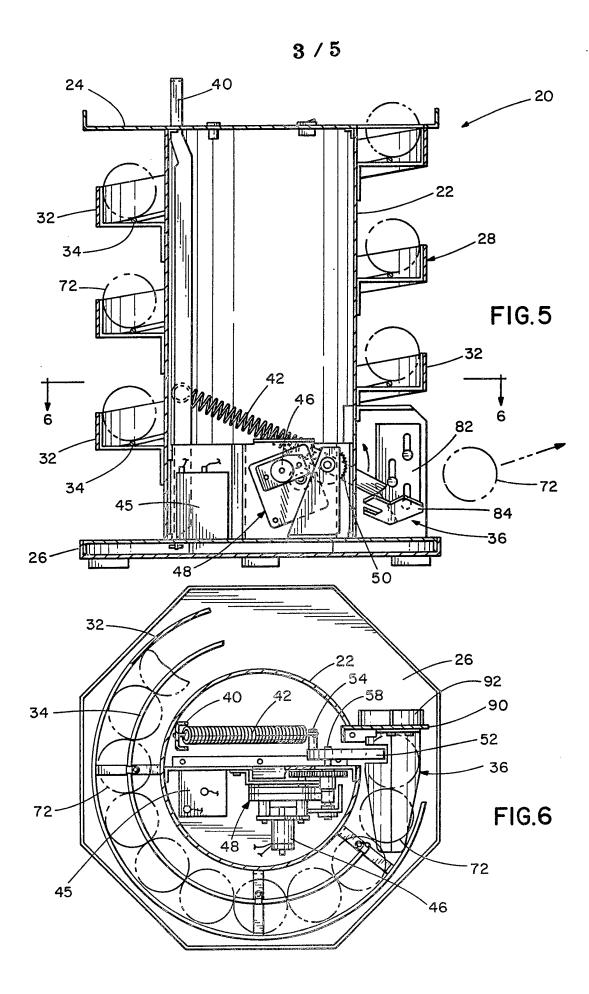
(57) A ball throwing machine is suitable for battery operation. Balls are supplied from a helical ball track to a firing track. A ball striker is cocked by a motor against the force of an adjustable spring. When the striker is positioned over-center the striker rapidly rotates striking the ball. A one-way clutch mechanism is included in the gear train between the motor and ball striker so that the kinetic energy remaining after the ball in launched is utilized to begin cocking the striker for the next cycle of operation. This results in significant reduction in the electrical energy required to operate the mechanism. An elevation control permits selective positioning of the firing track relative to the ball striker.



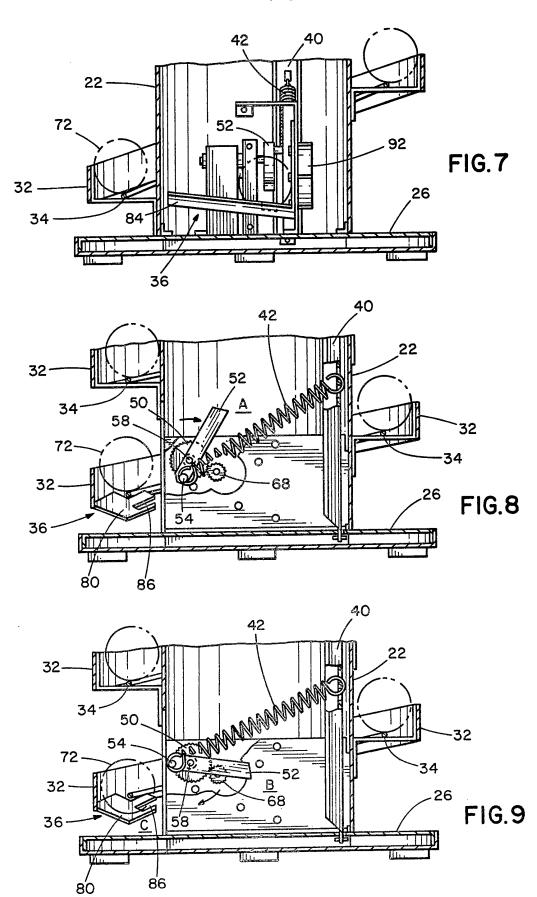


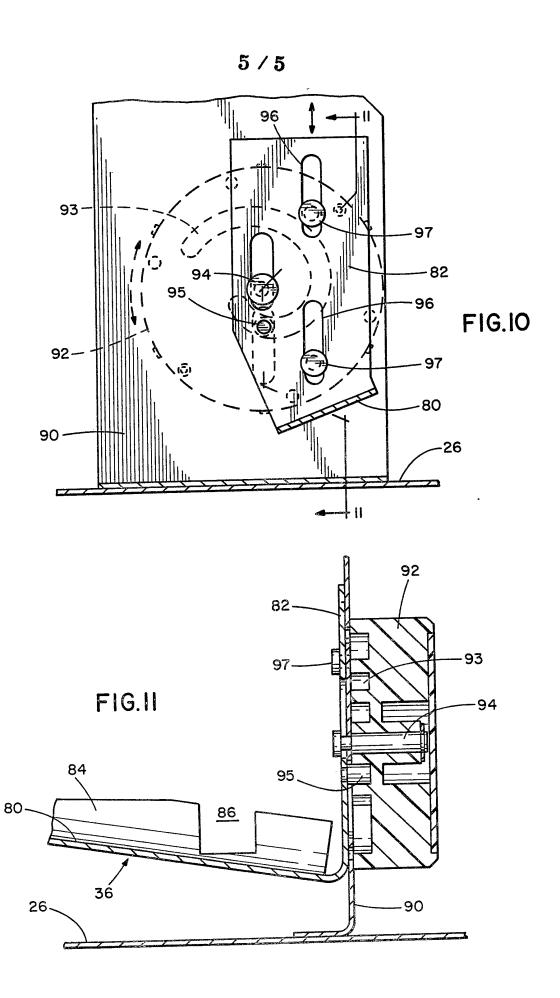






4/5





#### **SPECIFICATION**

#### Portable ball throwing machine

.5 This invention relates to ball throwing machines, for example, machines for projecting tennis balls and the like on a selected trajectory so that the players may practice.

Most such devices require a source of mains
10 voltage to operate the electric motor utilized for
driving the firing mechanism. This is because the
energy necessary to repeatedly cock and discharge
the striking mechanism will quickly wear down a
battery operated device. Accordingly, it has not been
15 possible to provide a ball throwing machine which
can be used away from a source of mains voltage.
Since tennis facilities are usually out of doors and
away from power receptacles this has limited the
wide use of tennis ball throwing machines by
20 players and instructors.

Although portable machines have been proposed, it has not heretofore been possible to provide a device which is portable, cordless and still capable of extended usage. Nor has it been possible to provide a cordless device which is able to project tennis balls at varying velocities and distances in a manner to simulate volleys from an opponent.

Embodiments of the present invention seek to provide a cordless portable ball throwing machine.

Embodiments of the invention also seek to provide a battery operated ball throwing device which efficiently utilizes the available electrical energy so as to provide prolonged periods of usage before battery replacement or recharge is required.

Embodiments of the invention further seek to provide a cordless portable throwing machine which is capable of a full range of velocity and distance adjustments so as to simulate the volleys of an opponent upon a tennis court.

40 Embodiments of the invention also seek to provide a cordless portable ball throwing machine in which the momentum remaining in the striker after a ball is struck is employed to assist in cocking the striking mechanism thereby to reduce the electrical energy 45 required for each cycle of operation.

According to the invention there is provided a portable ball throwing machine comprising means on which balls are positioned for launching, means for supplying balls to said launching means, means

- 50 for striking balls positioned on said launching means, said striking means including a ball striker rotatably mounted on a shaft, means for normally biasing said ball striker to a release position, electrical drive means for rotating said striker in a first
- 55 direction against the force of said biasing means to a fully cocked position, said striker, upon reaching said fully cocked position, rapidly rotating in said first direction toward said release position striking said balls, said drive means including clutch means for
- 60 permitting said striker to rotate on said shaft in said first direction only, whereby kinetic energy remaining in said striker after striking a ball is utilized to partially cock said striker for the next cycle of operation thereby reducing the energy required
  65 from said electrical drive means to rotate said striker

to said fully cocked position.

An embodiment of the invention will now be described by way of example with reference to the accompanying drawings in which:-

70 Figure 1 is a perspective view of a ball throwing machine according to a preferred embodiment of the invention;

Figure 2 is a perspective view of the base and throwing mechanism of the ball throwing machine;

75 Figure 3 is a vertical section along the lines 3-3 of Figure 1;

Figure 4 is an exploded view of the ball throwing mechanism illustrating the details thereof;

Figure 5 is a view similar to Figure 3 looking in the opposite direction;

Figure 6 is a sectional view along the lines 6-6 of Figure 5;

Figure 7 is a partial vertical section illustrating the relationship between the ball track and the firing 85 track;

Figures 8 and 9 are partial sectional views illustrating the movement of the ball striker through a cycle of operation;

Figure 10 is a side elevational view of the adjusting 90 mechanism for the firing track;

Figure 11 is a sectional view along the lines 11-11 of Figure 10;

Referring to the drawings and particularly Figures 1 through 3, a ball throwing machine 20 is illustrated.

95 The ball throwing machine is ideally suited for projecting tennis balls and this description will refer to that use. It will be recognized, however, that the machine can be readily adapted to project balls utilized in other sports activities as, for example, racketball, squash, handball, and the like.

The ball throwing machine 20 includes a tubular housing 22 open at either end thereof. The top of the housing is covered by a top plate 24 while the bottom is secured to a base 26. Concentrically 105 positioned about the tubular housing is a ball track 28 arranged in a helix whereby balls are conveyed from the top plate, around the housing to the vicinity of the base 26. An opening 30 in the top plate permits access to the ball track by which balls can be 110 loaded thereon.

As seen in Figure 3, the ball track consists of an outer fence 32 in conjunction with a ball support rail 34. As the balls are placed into the opening in the top plate they travel helically, by force of gravity, around 115 the ball track until they reach a ball firing track 36 secured to the base 26. In the case of tennis balls, approximately 50 or more balls can be stored on the ball track in a typical embodiment of the invention. This permits long periods of operation without the 120 need to reload.

Projecting upwardly through an elongated slot 38 in the top plate is an adjusting lever 40 for controlling the tension applied to the striking mechanism by a tension spring 42. The lever 40 can be secured in a plurality of positions along the length of the slot 38 by means of cutouts 44 into which the lever can be captured. As indicated in Figure 3, movement of the lever along the length of the slot 38 is effective for increasing or decreasing the tension on spring 42 as desired.

Referring to Figures 2 and 4, the ball firing mechanism is illustrated. The mechanism is battery powered, preferably by means of rechargeable batteries. A representative battery is shown at 45 5 secured to the base 26. The battery may be electrically connected to a jack located in the top plate for periodic connection to an AC source whereby the battery may be recharged in a conventional manner. The battery is also connected to a small electric 10 motor 46. One side of the circuit between the battery and the motor is connected via a switch 49 mounted on the top plate 24 permitting selective energization of the motor. Electric motor 46 is connected to a gear reducer assembly 48, the output of which drives a 15 gear and clutch assembly 50 in a manner to be described.

The balls are projected from the machine by means of a ball striker 52 which, preferably, consists of a solid metal bar. Mounted at one end of the striker is a spring post 54 having a channel 56 thereon to receive and secure one end of the coil spring 42. The striker is provided with a circular bore 56 located inwardly of the spring post as illustrated in Figure 4.

The projecting means includes a shaft 58 mounted for rotation between a pair of frame members 60 and 62 secured to the base 26. One end of the shaft 58 passes through a sleeve 59 and is received in the striker bore hole 57 and secured thereto. This may be accomplished by means of a set screw, welding, pressfit, or other appropriate techniques. Concentrically mounted on the mid-portion of shaft 58 is the gear and one way clutch mechanism 50. Spacers 64 and 66 may be provided, as necessary, to correctly position the gear 50 relative to a drive gear 68 associated with the gear reducer mechanism 48.

In the assembled condition gear 68 drivingly engages gear 50 thereby rotating shaft 58 and striker 56. The ball striker 52 is positioned outside the frame 40 member 62 while the gear 50 and the spacers, if any, are positioned on the shaft between the frame members 60 and 62. The gear and one way clutch mechanism 50 is a type commercially available as, for example, part No. RCB-061014FS manufactured 45 by The Torrington Company of Connecticut, United States of America.

As is known by those skilled in the art, such a clutch mechanism permits free movement of a shaft as, for example, shaft 58, in one direction only. 50 During the operation of the ball throwing mechanism the motor 46 is operated for the purpose of driving the shaft 58 in the direction in which the clutch permits. With reference to Figure 4, arrow 70 indicates the direction of rotation of the shaft. 55 Because of the presence of the one way clutch, shaft 58 is free to rotate in the indicated direction driven by the motor 46 or from the energy of the ball striker when the spring tension is released in a manner to be described. The clutch mechanism will not permit 60 movement of the shaft and, consequently, the ball striker attached thereto in the reverse direction and this results in an advantageous mode of operation wherein the energy requirements of the invention are significantly reduced.

With reference to Figures 8 and 9, the advan-

tageous operation of the throwing mechanism will be explained. Assuming that the motor is switched on with the striker in the position A illustrated in Figure 8, it can be seen that the battery operated 70 motor 46 must move the striker in the direction indicated by the arrow against the tension created by the coil spring 42. It will be recalled that one end of the spring is secured to a spring post 54 while the other end of the spring is secured to the tension 75 lever 40. As the rotation of the striker proceeds the tension of the spring increases until the striker reaches a position approximately as indicated at B in Figure 9. At this point in the cycle the spring post 54 is "over-center" relative to the shaft 58 on which the striker 52 is mounted. This causes the tension in spring 42 to rapidly rotate the free end of the striker through an arc which passes through the location at which a ball to be struck is located. (The ball position is indicated in phantom at 72 in Figure 9).

During this portion of the operating cycle the spring tension drives the shaft 58 at an angular velocity greatly in excess of the speed at which the motor 46 is driving the gear 50. The shaft, however, is free of any drag from the motor by virtue of the
 one way gear and clutch mechanism 50.

After the striker mechanism has contacted the ball (at point C) and caused ejection thereof a substantial amount of kinetic energy remains in the striker. This kinetic energy is advantageously used to reduce the 95 amount of electrical energy required to cock the striker in preparation for the next ball ejectment. As the striker rotates past the impact point it begins to tension the spring 42. Tensioning of spring 42 quickly reduces the kinetic energy in the striker to zero and in the absence of the one-way clutch mechanism 50 damped oscillating movement of the striker would result until an equilibrium position were reached. In the present invention, however, the one-way clutch mechanism prevents any reverse 105 movement of the striker (counterclockwise as viewed in Figures 8 and 9). Thus, the striker is maintained at the furthest point it has reached after "over-center" release. This point is approximately located at A in Figure 8. The striker is held at point A 110 until the motor "catches up" and causes the gear 50 to again rotate the shaft moving the striker from position A to position B for the next "over-center" release.

As will be appreciated by those skilled in the art,

this arrangement makes maximum utilization of the
energy provided by the motor. Energy from the
motor is needed to tension the spring only during
movement of the striker between points A and B.
The potential energy in the spring is then used to

cause the striker to eject balls from the machine. The
remaining kinetic energy after striking is utilized to
begin cocking the striker for the next cycle of
operation. This arrangement is very efficient as
compared with existing devices and significantly
reduces energy waste.

This highly efficient arrangement results in the ability to provide portable battery operation of the device for extended periods of time. In use, up to five hours of operation can be obtained without recharging the batteries. Because of the low energy usage it

is even possible to employ a solar panel (mounted on the top plate) in conjunction with the device in which case the operating period can be extended indefinitely, solely as a function of the available • 5 sunlight.

Referring now to Figures 6, 7, 10 and 11, the construction and operation of the ball firing track will be described. The firing track 36 consists of a ramp portion 80 connected to a vertical flange 82. The rear 10 portion of the ramp includes an upwardly deflected section 84 having a notch or opening 86 therealong. As can be seen in Figure 6, the notch permits the ball striker 52 to impact a ball positioned on the firing track. The ramp 80 terminates at a point adjacent the 15 end of the ball track and thus balls roll from the ball track onto the ramp serially as each ball is ejected by the striker. The length dimension of the ramp permits two balls at a time to be positioned thereon.

The height of the ramp 80, relative to the base and 20 striker path, affects the trajectory of the ball. In order to permit selection of ball trajectory an elevation control is provided.

The elevation of the ball track is controlled by selective positioning of the vertical flange 82 relative 25 to the frame 90. This can be accomplished in a number of ways as, for example, by latching the flange in selected apertures in the frame at a desired position. Alternatively, as illustrated in Figures 10 and 11, an elevation knob 92 containing a spiral cam 30 channel 93 can be employed whereby rotation of the knob moves a cam 95 attached to the vertical flange 82 in a desired direction to raise or lower the firing track. Slots 96 in the flange co-operate with retaining members 97 to permit vertical movement while 35 maintaining the flange in abutting relation with

From the foregoing, the operation of the machine 20 should be readily apparent. For completeness, however, a brief operating summary will now be 40 provided. After the device is charged it is disconnected from a source of AC current and taken to the location where it is to be utilized. It is then loaded with balls through the top plate opening 30 until the ball track is filled to capacity or as desired. Balls pass 45 from the ball track onto the firing track 36. The elevation of the firing track is set for a desired trajectory and the tension on the spring 42 is set to a desired position to adjust the impact of the ball striker. The machine is then switched on and opera-

50 tion proceeds in an automatic manner.
The DC motor drives the striker to the over-center position B at which point the spring 42 causes a snap action in which the striker impacts the ball ejecting it in the desired manner. The striker then continues on
55 past the point of impact, initiating cocking of the spring for the next cycle of operation. When the kinetic energy in the striker is exhausted, the one-way clutch assembly prevents reverse movement of the spring and holds the striker in the partially
60 cocked position A, Figure 8, until the motor drive engages and completes cocking the spring to the over-center release point B. During each cycle of operation, as a ball is ejected from the firing track,

succeeding balls roll off of the ball track and into

65 position over the aperture 86 in the firing track 36.

After a complete set of balls have been fired from the device it is switched off and reloaded as desired.

#### CLAIMS

70

- A portable ball throwing machine comprising means on which balls are positioned for launching, means for supplying balls to said launching means, means for striking balls positioned on said launching 75 means, said striking means including a ball striker rotatably mounted on a shaft, means for normally biasing said ball striker to a release position, electrical drive means for rotating said striker in a first direction against the force of said biasing means to a 80 fully cocked position, said striker, upon reaching said fully cocked position, rapidly rotating in said first direction toward said release position striking said balls, said drive means including clutch means for permitting said striker to rotate on said shaft in said 85 first direction only, whereby kinetic energy remaining in said striker after striking a ball is utilized to partially cock said striker for the next cycle of operation thereby reducing the energy required from said electrical drive means to rotate said striker 90 to said fully cocked position.
  - 2. A portable ball throwing machine according to Claim 1 wherein said launching means is a firing track including means for raising and lowering said track relative to the rotational path of said striker.
- 95 3. A portable ball throwing machine according to Claim 2 wherein said firing track has an opening permitting said striker to pass therethrough to strike a ball positioned on said track over said opening.
- 4. A portable ball throwing machine according to 100 any one of Claims 1 to 3 wherein said supplying means is a helical ball track on which balls roll from the top to the bottom thereof, said launching means being positioned adjacent said bottom to receive balls therefrom.
- 105 5. A portable ball throwing machine according to any one of Claims 1 to 4, wherein said ball striker is an elongated member including means for securing said striker to said shaft, said securing means being located near one end of said striker, the other end of 110 said striker being utilized to strike balls.
  - A portable ball throwing machine according to Claim 5 wherein said striker has an attaching means located at said one end, said biasing means being attached thereto.
- 115 7. A portable ball throwing machine according to any one of Claims 1 to 6 wherein said biasing means includes a spring which is tensioned by the rotation of the striker from the release position to the cocked position.
- 8. A portable ball throwing machine according to Claim 7 wherein said biasing means further includes means for adjusting the tension in said spring whereby the force imparted to said striker is selectable.
- 125 9. A portable ball throwing machine according to any preceding Claim wherein said drive means includes an electric motor drivingly connected to said shaft via said clutch means.
- 10. A portable ball throwing machine according130 to Claim 9 wherein said motor is battery operated.

- A portable ball throwing machine according to any preceding Claim, wherein said drive means includes an electric motor, a gear train connecting said motor to said shaft for rotation of the latter, said 5 gear train including said clutch means.
- 12. A portable ball throwing machine according to any preceding Claim wherein said clutch means is a one-way clutch permitting both free rotation and driven rotation of said shaft in said first direction
  10 while preventing any rotation of said shaft in the direction opposite said first direction.
  - 13. A portable ball throwing machine substantially as hereinbefore described with reference to the accompanying drawings.

Printed for Her Majesty's Stationery Office by Croydon Printing Company Limited, Croydon, Surrey, 1981. Published by The Patent Office, 25 Southampton Buildings, London, WC2A 1AY, from which copies may be obtained.