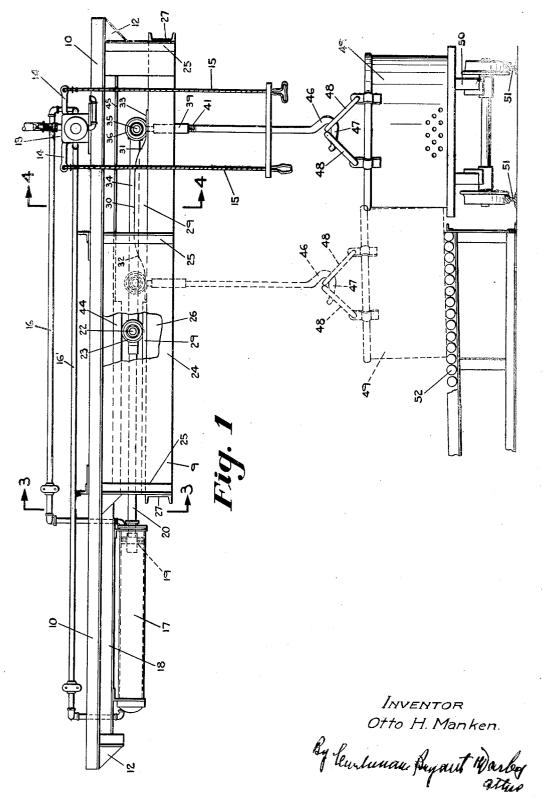
Oct. 17, 1933.

O. H. MANKEN

AIR LIFT TRANSFER DEVICE

Filed Oct. 17, 1930

2 Sheets-Sheet 1

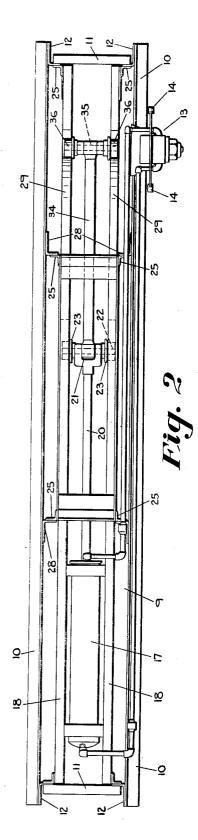


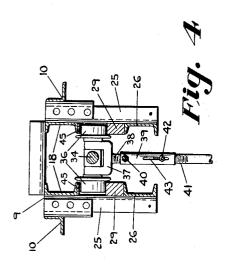
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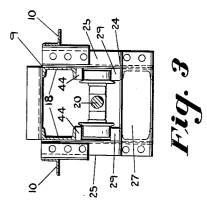
O. H. MANKEN AIR LIFT TRANSFER DEVICE Filed Oct. 17, 1930

1,930,608

2 Sheets-Sheet 2







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UNITED STATES PATENT OFFICE

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AIR LIFT TRANSFER DEVICE

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10 Claims. (Cl. 214-1)

This invention relates to new and useful improvements in air lift transfer devices primarily designed for lifting and transferring from one place to another, heavy objects which cannot be expeditiously handled by manual labor.

An important object of the invention is to provide means for transferring such heavy objects from one point to another and at the same time elevating and lowering such objects during 10 transit.

A still further object of the invention is to provide a load carrier associated with the transfer device in a manner which will expedite the association of an object representing the load as

15 well as the disconnection of such load therefrom.

A still further object of the invention is to provide a gravity conveyor in association with the discharge station of the air lift which utilizes

20 the weight of the load for propelling the same away from the unloading station and to provide means which will expedite the movement of the load up to the loading station of the air lift. Other objects and advantages of the invention

25 will become apparent during the course of the following description.

In the accompanying drawings forming a part of the description and wherein like numerals are employed to designate like parts:-

30 Figure 1 is a side elevation of the combined air lift and transfer device;

Figure 2 is a top plan of the same;

Figure 3 is a transverse section of the same taken on the line 3-3 of Figure 1; and

Figure 4 is a transverse section taken on the 35 line 4-4 of Figure 1.

In the installation of this device, a framework 9 of structural iron may be suspended either from the structural work of the building in which

- 40 the apparatus is to be installed, or from any suitably auxiliary framework. This suspended frame 9 comprises essentially a pair of horizontal spaced parallel angle beams 10 connected at their ends by transverse braces 11 having angle
- 45 cleats 12 at their ends for attachment to the inner vertical flanges of the horizontal beams 10. At one end of this framework, an air control mechanism 13 is suitably supported by one of the horizontal beams 10 and is equipped with a
- 50 pair of valve handles 14 having operating cords 15 depending therefrom to a point within easy reach of an operator. This air control mechanism is of any well known type having a suitable inlet and outlet for the air whereby it may

55 be admitted at the will of the operator through

the pipes 16 or 16' to either end of an air cylinder 17 suspended from a pair of horizontal spaced parallel beams 18 which are suspended from the cross braces 11 and angle cleats 12 as shown in detail in Figure 2. This air lift cylinder 17 is of 60 the well known construction having reciprocably mounted therein, piston 19, the rod 20 of which projects through the inner head of the cylinder to be disposed horizontally as shown. The outer end of this piston rod 20 is forked as at 21 to 65 provide bearings for an axle 22 having wheels 23 rotatably mounted on its outer ends, constituting a cross-head.

In order to provide a track for these wheels, a track frame 24 is suspended from the horizon- 70. tal beams 10 by means of vertical angle bars 25. To the lower ends of these bars are secured a pair of horizontal spaced parallel side rails 26, which, rails at their ends, are connected by means of channel braces 27 to retain the rails 75. 26 spaced the same distance apart as the beams The angle bars 25 are suitably braced from 18. the beams 10 against angular deflection by means of the corner cleats 28. Mounted on the upper horizontal flange of the rails 26 are cam rails 29 80 forming tracks for the wheels 23. Beyond the range of movement of these wheels the rails 29 are provided with cam portions or inclined planes, the medial portion 30 of which is horizontal while the two ends thereof incline in opposite 85 directions as at 31 and 32, the former terminating in a horizontal portion 33 constituting the loading portion of the transfer device.

The load sustaining carriage includes a horizontally extending bar 34 pivotally connected at 90 its rearward end to the axle 22 in the bifurcation of the fork 21. The other end of this bar 34 is provided with a horizontal bearing for an axle 35 to the outer ends of which are secured wheels 36 95 adapted to ride up and down the cam portions of the track. Pivotally suspended from this axle is a fork 37, the bifurcation of which surrounds the forward end of the rod 34. This fork is provided with a threaded dependency 38 which is threaded into a sleeve coupling 39. The coupling 100is prevented from accidental turning movement on the dependency by means of a set screw 40 extending through the sleeve for engagement with the dependency 38. Into the lower end of this sleeve is threaded a shank 41 of a load carrying 105 hook. To prevent accidental rotation of the shank 41 in the sleeve, a set screw 42 extends from a threaded aperture in the shank through a vertical slot 43 provided in the coupling sleeve 39 110 as shown in Figure 4.

In order to prevent the vertical displacement of the wheels 23 from the tracks 29, horizontal guide strips 44 are secured to the lower horizontal flanges of the channel beams 18 to be arranged 5 directly above the treads of the wheels. Similar guide strips 45 are secured to the lower flanges of the channel beams 18 above the horizontal portion of the cam 30 whereby vertical displacement of the wheels 36 is likewise prevented during

10 movement of the carriage back and forth along the tracks. The load carrying shank 41 depends a substan-

tial distance below this overhead track and terminates in a hook 46 having a substantially hori-15 zontally disposed bill 47, over which the bales 48 of a container 49 are swung for supporting engagement therewith. The container 49 here illustrated is in the form of a circular perforated receptacle designed especially for the handling of

20 a large number of canned goods, the combined weight of which exceeds that which workmen can expeditiously handle.

These containers are conveyed up to the lift and transfer device by means of suitable trucks 50 preferably running on rails 51 up to the loading 25 station of the air lift device. Here, the hook 46 is swung upon its pivot 35 and the bales 48 of the container are lapped over the same as shown

- in Figure 1. Thereupon, the valve control cable $_{30}$ 15 is pulled by the workman to admit air under pressure into the right hand end of the cylinder 17 and to properly take care of the exhaust from the opposite end of the cylinder. The entrance of this medium into the cylinder causes the piston
- 35 rod 20 and associated carriage carrying the load supporting hook 46 to move toward the left of Figure 1. This movement causes the wheels 36 of the carriage to ride up the inclined plane 31 to elevate the container 49. During this move-
- 40 ment it will be noted that the bar 34 pivots upon the axle 22, while the perpendicular position of the hook is unaltered due to its pivotal suspension from the axle 35. Continued movement of the piston 19 to the left causes the wheels 36 to move
- over the horizontal portion 30 of the cam and then down the inclined portion 32 of the track 29 to lower the container 49 onto the conveyor 52. The bales 48 are then released from the load suspension hook 46 to release the container and
- 50 permit its movement by gravity down the conveyor 52 to a point of suitable discharge. Upon actuation of the other operating cable 15 of the valve mechanism 13, the valves are reversed to shift the carriage to the right of Figure 1 to re-
- trace its movements and come to a stop on the 55 horizontal portion 33 of the track in position to receive the next receptacle brought up by the truck 50. In such an arrangement, it will be evident that the load is not only moved vertically
- 60 by reason of the inclined planes 31 and 32 but is also caused to move horizontally to shift its load from the loading station to the discharge station above the gravity conveyor.

It will be noted that the hook 46 is open toward $_{65}$ the delivery station and that the conveyor 52 is arranged to give the receptacle 49 a direction of travel in continuation of the movement of the receptacle in approaching the receiving station. Thus, as the receptacle is lowered onto the con-

70 veyor and the tension on the bale elements 48 released, the latter tend to move out of the hook so that the receptacle continues along conveyor 52 under an initial impulse imparted upon its deposit thereon.

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the construction and arrangement of parts may be resorted to without departing from the scope of the appended claims.

I claim:

1. A combined lift and transfer device com-80 prising a track having a cam portion, a wheeled reciprocable carriage movable upon the track and including a member pivotally connected at one end, the other end of said member having anti-friction means adapted to ride over said cam 85 portion, and a load carrier suspended from said member.

2. A combined lift and transfer device comprising a track having a cam portion, a wheeled reciprocable member movable along the track, a 90 bar pivotally connected to said member and equipped with wheels operable upon said track, and a load carrier pivotally suspended from the wheeled portion of said bar.

3. A combined lift and transfer device com-95 prising a track having a cam portion, a wheeled reciprocable carriage movable along the track and including a member pivotally connected at one end to said carriage, an axle mounted at the other end of said member, wheels on said 100 axle adapted to ride over said cam portion, and a load carrier pivotally suspended from said axle.

4. A combined lift and transfer device comprising a track having a cam portion, a reciprocable rod having a forked end carrying an axle 105 having wheels movable over said track, a bar pivotally connected at one end to said axle in the bifurcation of said reciprocable rod, an axle mounted in the other end of said bar, wheels mounted on said axle and movable over said 110 track, and a load carrier having a bifurcated end pivotally connected to said last named axle between said wheels and bar.

5. A combined lift and transfer device comprising a track, a piston rod having wheels to 115 run upon said track, a horizontal bar pivoted at one end to said rod, and having wheels at its other end to ride upon said track, a load carrier suspended from the wheeled end of said bar, and horizontal guide members above and in contact 120 with the wheels of said rod to prevent vertical displacement thereof.

6. A combined lift and transfer device comprising a carriage, a horizontally extending trackway on which said carriage travels, said track- 125 way having an upwardly offset portion, a fluid actuated piston articulated to said carriage for positively reciprocating it on said trackway, the carriage during its travel being guided to upper and lower positions by the trackway, and load 130 supporting means depending from said carriage.

7. A combined lift and transfer device comprising a carriage, a horizontally extending trackway on which said carriage travels, said trackway having depressed end portions and an in- 135 termediate elevated portion, a fluid actuated piston articulated to said carriage for positively reciprocating it from end to end of said trackway, the carriage during its travel being guided to upper and lower positions by the trackway, and 140 load supporting means depending from said carriage.

8. A combined lift and transfer device comprising a horizontally disposed cylinder and piston unit, a cross-head guiding the forward end 145 of the piston against vertical distortions, a rod pivoted to the forward end of the piston, a horizontally extending trackway in advance of the piston, said trackway having an upwardly offset It is to be understood that various changes in portion, means supporting the forward end of 150

displacements of the forward end of the rod upon its reciprocation by said piston, and load supporting means carried by said rod for ver-

5 tical and horizontal displacement thereby. 9. A combined lift and transfer device com-

prising a horizontally disposed cylinder and piston unit, a cross-head guiding the forward end of the piston against vertical distortions, a rod

10 pivoted to the forward end of the piston, a horizontally extending trackway in advance of the piston, said trackway having an upwardly offset portion, means supporting the forward end of said rod on said trackway to cause elevational livery station. 15 displacements of the forward end of the rod upon

its reciprocation by said piston, and load sup-

said rod on said trackway to cause elevational porting means suspended from the forward end of said rod.

10. Apparatus comprising a combined lift and transfer device, said device including a suspension hook movable between receiving and de-80 livery stations in a horizontal path having an intermediate portion elevated relative to its end portions, said hook being open toward the delivery station, a receptacle having suspension means engageable over the hook, and a conveyor 85 at the delivery station for receiving the receptacle and continuing the movement thereof in the same direction in which it approaches the de-90

OTTO H. MANKEN.

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