PATENT SPECIFICATION

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(54) MACHINE AND METHOD FOR FORMING A CONTINUOUS FLEXIBLE TUBE

We, AUTOMATION INDUSTRIES, Inc., a corporation organised and existing under the laws of the State of California, United States of America, of 1901 Building, Century City, Los Angeles, California, 90067, United States of America, do hereby declare the invention, for which we pray that a patent may be granted to us, and the method by which it is to be performed, to be parti-10 cularly described in and by the following statement: -

The present invention relates to a machine for and a method of forming a continuous flexible tube from a strip of flexible material, the tube being wound in helical fashion and

having interlocking edge portions.

In US Patent Specification No. 3,199,541 a machine and method of forming a continuous flexible tube is described. The crosssectional configuration of the strip from which the tube is made includes a longitudinally extending bead along one edge portion and a longitudinally extending downwardly facing groove along the other edge portion of the strip, the groove being configured to correspond with the shape of the bead to snugly interlock therewith. US Patent Specification No. 3,199,541 describes forming of the tubing by wrapping the strip of 30 flexible material helically about a mandrel with the plate bottom surface of the strip in contact with the mandrel.

In US Patent Specification No. 3,778,327 a flexible tubing is formed on a mandrel means which comprises a pair of rollers having their axes in askew relation, the mandrels being formed with annular grooves. US Patent Specification No. 3,778,327 discloses the wrapping of a strip of flexible material which is planar about the roller surfaces and at the same time feeding a wire onto a portion of the strip and in alignment with a groove in the mandrel so that the tension of the wire and strip cause the face of the strip in contact with the mandrel to conform to the mandrel groove formation. In US Patent Specification No.

3,778,327 the strips of material are overlapped and the wire serves to reinforce the tubing. In US Patent Specification No. 3,778,327 both rollers of the mandrel means include annular grooves and only one of the rollers is driven, the other roller being an idle roller.

According to the present invention a 55 machine for forming a continuous flexible tube from a strip of flexible material having a cross sectional configuration including interlocking means along one edge portion interlockable with corresponding interlocking means along the other edge portion when the strip is wound in helical fashion, the combination of a frame means; a pair of spaced rollers extending from said frame means with axes in non-parallel relation; means for mounting said rollers on said frame means for adjusting the space between axes of said rollers and the angular relation

of said axes; means on at least one of said rollers cooperable with edge portions of said strip of flexible material during helical winding of the strip; and means for rotating both of said rollers for winding said strip thereabout and for discharge of the tube formed thereby from free ends of said rollers.

Thus continuous tube of different diameter may be formed by adjusting the roller positions. The strip preferably has a cross sectional configuration similar to that shown in US Patent Specification No. 3,199,541 with groove means along one edge portion interlockable with longitudinal bead means along the other edge portion when the strip is wound helically. Driving means may be provided for rotating both of the rollers at a

uniform rotational speed.

At least one of the rollers may be provided with a plurality of radially outwardly extending annular fins spaced apart a distance related to the width of the strip of flexible material being fed to the mandrel means so that the strip of material wound upon the mandrel means will be received between adjacent fins and will be restrained

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from creeping movement relative to the axis of the rollers.

The method of forming a continuous flexible tube according to this invention comprises helically winding a strip of flexible material over a pair of spaced rollers having their axes in non-parallel relation so that interlocking means provided along one edge portion of the strip is interlocked dur-10 ing winding about said rollers with corresponding interlocking means along the other edge portion, in which the strip is guided onto the rollers so that the interlocking means engage and the tube formed on the rollers is axially guided along the rollers by means on at least one of the rollers cooperable with edge portions of the strip, the diameter of the tube being variable by means for adjusting the spacing between the roller axes and the angular relation of the axes.

Preferably external pressure is applied to the external interlocked edge portions of the strip at each roller as the strip is wound onto the roller whereby the interlocked edge portions of the strip are uniformly engaged and bonded together.

The rollers are preferably both positively driven for wrapping a strip of material in a synchronized manner and thereby forming a constant and dimensionally stable tube.

An embodiment of the invention will now be described by way of example with reference to the accompanying drawings, in

Figure 1 is a schematic view of an apparatus for forming a continuous tube.

Figure 2 is an enlarged schematic perspective view of part of the apparatus shown in Figure 1 comprising a tube forming machine according to an embodiment of the invention.

Figure 3 is a side elevational view of the tube forming machine shown in Figure 2 on a larger scale, the view being partly in section to show the mounting of the end of the rollers forming mandrel means.

Figure 4 is an end view taken from the right of Figure 3.

FIGURE 5 is a fragmentary side elevational view of the mandrel means of the tube forming machine to a larger scale, the view being partly in section to show winding of the strip on the mandrel means, the approach of the strip to the bottom mandrel being schematicaly illustrated.

FIGURE 6 is an enlarged fragmentary top view of the strip at the bottom roller and the application of adhesive in the groove means on the strip.

FIGURE 7 is a fragmentary perspective view of the strip formed into a continuous tube by the tube forming machine of this invention, the strip including a section showing the cross sectional configuration of the strip.

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FIGURE 8 is an enlarged fragmentary sectional view of interlocked strip portions on one of the rollers of the mandrel means.

FIGURE 9 is a side view partly in section

showing a completed tubing.

Referring first to FIGURES 7, 8 and 9, a completed length of tubing generally indicated at 10 is formed from an extruded strip 11 having a flat main web 12 provided with a longitudinaly extending edge bead portion 13 extending arcuate one face of the web 12 and a longitudinally extending groove portion 14 extending from the other edge of web 12 and defining a longitudinally extending groove 15 configured to receive and mate with the longitudinal edge bead 13 of an adjacent turn of strip 11. The longitudinal bead 13 and groove arcuate portion 14 provide continuous interlocking strip portions when the strip is helically wound into tubular form. The strip may be made of a resilient self-supporting thermoplastics material such as polyethylene, polypropylene, polyvinylchloride, nylon, or other flexible materials suitable for providing a tube or hose having desirable physical and chemical characteristics. As shown in FIGURE 8 in exaggerated form, an elongated band of adhesive 16 may be deposited on internal surfaces of groove 15 for bonding with bead 13 when the interlocking portions of the strip are joined. It will be understood that the strip may include other reinforcing elements.

FIGURE 1 schematically shows an apparatus for continuous production of tube 10. 100 Pellets of suitable selected plastics material may be fed into a hopper 18 for supplying an extruder 19 of well known manufacture. Extruder 19 is provided with suitable dies to provide the cross sectional configuration 105 of the extruded strip 11, such extrudate being passed through a cooling water trough 20 by drive or pulling rolls 21. The formed strip 11 passes through an accumulator 22 of usual loop form, the number of loops 110 provided in the accumulator and the length thereof being controlled by a photocell type control 23 of well known manufacture. Extruded strip 11 may then be passed around a guide pulley 24 and through a strip ten- 115 sion control means 25 which may comprise a pair of pressure rollers engaging said strip so that as the strip is advanced to the tube forming machine 26, selected tension will be applied to the strip as it is wound 120 upon the mandrel 27 as more fully described later. In the approach to the mandrel 27, the arcuate portion 14 may be widened to permit the introduction of a continuous band of adhesive supplied by an adhesive 125

extruder 28. The strip is wound in helical fashion on the mandrel 27 and is discharged

from the end of the mandrel into a water

cooling trough 29 and is then advanced to

a tube cutting means 30 where selected 130

lengths of tubing 10 may be cut over an air blower plenum 31 having air jets along its length to facilitate handling of the cut tubing. A suitable console 32 is provided with necessary controls in well known manner for operation of the system generally described above. It will be understood that the finished tubing 10 may be wound into reels of selected length or may be cut to relatively short lengths depending upon the use for

The machine 26 for forming the tube 10 is best shown in schematic view FIGURE 2 and FIGURES 3 and 4. Machine 26 comprises a frame 33 including spaced upstanding frame walls 34 and 35 suitably supported upon a base generally indicated at 36. Each frame wall 34 and 35 includes a lower wall portion 36 and 37 through which a fixed drive shaft 38 passes, said drive shaft being mounted in suitable bearings, not shown, in each of wall portions 36 and 37 and extending beyond wall 34 and supporting a drive pulley 39. Frame walls 34 and 35 each include upwardly extending columns 40 and 41 respectively spaced apart to receive vertically adjustable wall portions 42 and 43. Each of the movable wall portions 42 and 43 include edge guide means generally indicated at 44 and 45 of tongue and groove type. Each movable wall 42 and 43 is adjustably vertically positioned by pairs of vertically extending screw threaded rods 46 and 47 respectively having their lower ends fixed to walls 36, 37 through fixed supports 48, 49. The upper screw threaded portions of rods 46, 47 extend through upper threaded supports 50 and 51 carried by wall portions 42, 43 respectively. Nuts 50a, 51a respectively provide securement of portions 42, 43 in selected position.

Wall portions 42, 43 support a drive shaft 53 which extends through wall portion 42 to provide a shaft end portion 54 substantially parallel to the corresponding end 55 of shaft 38. Upper drive shaft 53 may be mounted in a self-aligning bearing 56 supported on the inner surface of wall portion 42. The wall portion 43 may be provided with a downwardly facing opening 57 through which drive shaft 53 extends, this end of the drive shaft being supported by bearing means 58 having a bearing mount 59 transversely adjustable through bolt and slot means 60 on wall portion 43.

Mandrel 27 comprises a bottom rotatable roll member 62 connected to drive shaft 38 through a suitable universal means 63. Mandrel means 27 also includes a top roll member 64 rotatably connected to top drive shaft 53 through a universal means 65. Bottom member 62 may have its axis horizontally positioned; and roll member 64 has its axis arranged skew to the axis of roll member 62 by means of the vertical adjustment of

wall portion 43, the transverse adjustment of the shaft 53 by bolt and slot means 60, and adjustment of the space between said axes by connecting means 67 at the forward ends of the roll members.

Connecting means 67 comprises lapped connecting members 68 and 69, each member 68, 69 being secured to a respective end portion of each roll member 64, 62 through a screw bolt 70 carried in suitable bearing means 71 in each connector element 68, 69 and screw threaded in an axial bore as at 72 in the respective roll member 64, 62. A suitable lateral set screw 73 may be turned to fixedly secure the screw bolt 70 in desired position. Connector elements 69 and 67 are secured together by a set of parallel slots 75 in connector element 67, the slots 75 receiving securing bolt screws 76 screw threaded in conector element 69. The slots permit adjustment of the spacing of the axes of roll members 62, 64 and together with the vertical and lateral edjustment of the wall members 42, 43, the position of roll member 64 may be varied relative to roll member 62 so that the mandrel surface may be enlarged or contracted to permit production of tubing of different diameter.

Bottom roll member 62 comprises a cylindrical body having a plurality of longitudinally or axially spaced annular outwardly extending means in the form of relatively narrow fins 78. Fins 78 are spaced apart the width of web 12 of the strip 11 so that the web 12 of the strip may be received between 100 the fins. Each annular fin 78 is shown as being of triangular cross section, the outermost edge 79 of each fin being formed by surfaces of the fin arranged at an acute angle. Thus, as shown in FIGURE 8, the 105 edge 79 is readily received between the opposing face 80 defining one edge of web 12 and the face 81 defining the opposite edge of web 12 as shown in FIGURE 8. The plurality of fins 78 on roll member 62 110 extend for substantially the length of roll member 62, that is, from the universal joint means 63 to the end of roll member 62.

The top roll member 64 is cylindrical having a uniform diameter throughout its length 115 and of the same diameter as roll member 62. Top roll member 64 may be provided with a single radially outwardly extending fin 82 of similar shape as fin 78 of roll member 62. Fin 82 is located approximately 120 centrally of the length of roll member 64 and in this example, is located approximately between the sixth and seventh of fins 78 of member 62. Fin 82 is thus offset axially from the opposed fins 78 to assist in provid- 125 ing and maintaining a helical configuration to the strip 11 during its formation on the mandrel means 27. A plurality of fins 82 may be provided on the roll member 64 if desired.

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Guide and support means for the strip 11 as it is being wound upon the mandrel means 27 may comprise top and bottom guide members 84, 85 of similar construction. Each guide member 84, 85 includes a partially arcuate guide head 86 having on its inner arcuate face a continuous edge groove 87 having a width and depth adapted to substantially enclose with loose tolerances the external configuration of the interlocked joint formed by the groove means 14 and the bead 13 adjacent turns of strip 11 which have been helically wound on the mandrel. As best seen in FIGURE 4, guide head 86 defines an external arcuate path through an angle of less than one hundred and eighty degrees for the interlocked portions of strip 11. Tangent head portions 88 of head 86 permit guiding of the interlocked portions partially across the space between the top and bottom roll members 64, 62,

Means for adjusting the position of guide heads 86 into desired relation with the skew roller members 64, 62 may comprise a vertically extending shank 90 having a longitudinally extending slot 91 therein through which a screw bolt 92 extends for screw threaded engagement as at 93 with a forwardly projecting support member 94. Support member 94 is forwardly and rearwardly adjustable by a bolt 95 which extends through a slot 96 in a second support member 97 fixed at one end to a mounting plate 98 transversely adjustable on wall portion 43 35 by a set of slots 99 and bolts 100 extending therethrough and having screw threaded engagement with wall portion 43. The position of the guide heads 86 of both guide members 84, 85 may thus be three-dimensionally adjustable to accommodate variations in skew relation of the roll members; that is, adjustable along the length of the mandrel roll members, vertically adjustable depending upon the diameter of the tube to be formed, and transversely adjustable depending upon the relative transverse position of the roll member 64 with respect to roll member 62. In the example of the invention shown, the bottom guide support member 85 is aligned with the third fin from the rear portion of roll member 62 and the top guide member 84 is offset with respect thereto depending upon the pitch of the helical winding to be imparted to the turns of the

Means for rotatably driving the drive shafts 38 and 53 may comprise a suitable electric motor 102 mounted on base means 36. Drive pulley 103 connects the motor shaft 104 with the drive pulley 39 carried at the end 55 of drive shaft 38. Shaft end 55 and upper shaft end 54 each carry a drive collar 105 and 106 respectively provided with spaced annular grooves to receive under suitable tension drive pulleys 106 each of

circular cross section. The diameters of drive collars 105 are the same so that drive shaft 38 and 53 are rotated at the same speed.

In operation of the tube forming apparatus shown in FIGURE 1, part of which has been previously described, it will be understood that the strip extruder 19 may be continuously operated in order to provide a continuous length of strip 11 for feeding to the mandrel 27. It will also be understood that the extruder 19 may extrude a strip 11 which instead of being passed to an accumulator 22 may be wound upon suitable reels of selected length for later feeding to a mandrel to provide tubing of selected length. In many instances the continuous uninterrupted forming of the tubing 10 is preferable to satisfy existing demands for the tubing.

Strip 11 emerges from the tension control means 25 with the web 12 having the longitudinal bead 13 facing downwardly and the longitudinal groove 15 facing upwardly as best seen in FIGURE 5. As the strip 11 approaches the underside of mandrel roller member 62, a suitable groove spreader tool 108 is positioned in groove 15 to cause a slight spreading thereof as at 109 FIGURE 6. A nozzle 110 for applying a band of adhesive 111 to the bottom of the groove 15 is located immediately behind spreader tool 108 and into the spread apart groove for depositing band of adhesive 111. Adhesive nozzle 110 is continuously supplied by the adhesive extruding means 28.

As strip 11 moves into contact with the 100 cylindrical surface of bottom roller member 62, it is fed into position with one edge of web 12 guidably contacting a fin 78. Strip 11 is passed beneath member 62 then upwardly and over roll member 64 and then 105 downwardly to position the other edge of web 12 which has longitudinally bead 13 against the other side of fin 78. The next adjacent fin 78 is contacted by the edge of web 12 adjacent the groove means 14 since 110 the spacing of annular fins 78 is substantially the width of web 12. Thus, as the strip 11 is progressively turned around the roll members 62 and 64, the longitudinal groove 15 is moved over and into interlocking relation 115 with the longitudinal bead 13 of the immediately previous turn. Since the strip 11 is under selected tension as controlled by tension control means 25 and the speed of rotation of the mandrel, groove 14 is brought 120 into full interlocked relation with longitudinal bead 13 as the turns of the strip are passed around the mandrel.

Engagement of edges of strip 11 with the annular fins 78 on bottom roll member 62 125 positively advances strip 11 with turns in interlocked relation along roll member 62. Annular fin 82 of top roll member 64, once turns of the strip have been interlocked on the mandrel means, serves to prevent axial 130

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displacement of the strip being wound relative to roll member 64. Slight variations in tension of strip 11 being wound on a smooth surfaced mandrel usually tends to produce axial displacement of the strip being wound relative to such mandrel. The outwardly extending annular fins 78 and 82 restrain such axial displacement because of the containment of a portion of the strip 11 between the spaced annular fins 78 and the partial penetration of fin 82 at a joint between adjacent strip turns as the strip is passed over upper mandrel roll member 64.

As mentioned above, roll members 62 and 15 64 are not parallel and are in skew relationship. In this example, bottom roll member 62 is positioned horizontally with its axis normal to walls 36, 37. The upper roll member 64 slopes downwardly with respect to 20 the horizontal bottom roll member 62 and is disposed at another angle offset 90 degrees with respect to the downward angle of the roll member 64. Such biangularity of the axis of the top roll member 64 with respect to the bottom roll member 62 causes linear movement of the hose being wound around the two roll members towards the free ends of the roll members. The formed hose moves off the free ends of the mandrel and has an internal diameter approximately but slightly less than the diameter of a circle including the outer surfaces of the two roll members 62, 64 about which turns of the strip are wrapped. It will be understood that in wrapping the turns of the strip about the mandrel roll member 62, 64, the strip may be slightly stretched because of the amount of tension imparted to the strip by the tension control means. Therefore, as the wound hose moves off the ends of the mandrel roll members there may be a slight contraction of the inner diameter of the hose and this factor must be considered in determining the mandrel circumference in order to control the inner diameter of the finished hose.

One of the advantages of the above described mandrel structure is that one set of mandrels may be used for manufacture of hoses having internal diameters of between 50 one to three inches. The change in diameter of the finished hose is readily made by vertical, horizontal and angular adjustment of the top mandrel roll member 64 relative to the bottom mandrel member as described above. Hose internal diameters of from three to six inches may be readily made by the apparatus 26 by using a set of mandrel roll members of greater outer diameter than those used for the smaller hose diameters. A second set of mandrel roll members may be used for producing hose having internal diameters of from three to six inches. It will be understood that while hose diameters are usually standardized, a specific hose diameter 65 may be readily made by this apparatus because the mandrel roll members may be readily adjusted relative to each other to provide any specific internal hose diameter within the range of diameters for which the roll members are designed.

The apparatus of this invention is thus quite versatile in the different types of products which may be made on the apparatus by varying the spacing and angles of the mandrel roll members, the design of the mandrel roll members with respect to pitch and length. It will be understood that because of the positive engagement of the fins on the two mandrel members with the turns of the strip being wrapped therearound, that the cross sectional shape of the strip being wrapped around the mandrel may be varied. In addition, the various reinforcing devices described in US Patent Specification No. 3,199,541 may also be incorporated in the 85 hose structure if desired.

It should be noted that the apparatus described above provides for continuous manufacture of hose by helically winding the strip about a readily adjustable mandrel. Finished hose may be produced at a rate of thirty lineal feet per minute. The finished hose produced by the apparatus and method described above is characterized by all of the advantages of the hose described in US Patent Specification No. 3,199,541 which includes flexibility, an interlocked seal along the helical joints against leaking, resistance to axial tension and radial forces because of the interlocking joint construction, the 100 prevention of distortion of the shape of the hose by torsional forces, the maintenance of desired shape, and the provision of the generally smooth internal cylindrical surface which minimizes frictional resistance to the 105 flow of fluids therebetween.

The finished hose is readily made by the method of this invention in that the internal surface structure of the strip is positively engaged by the mandrel means to limit 110 creeping, slipping and axial or circumferential displacement and also by the positive engagement of the external configuration of the joint by the top and bottom guide members which assure positive contact sealing of 115 the band of adhesive with the internal surfaces of the groove means and the external surfaces of the longitudinal bead in the interlocked joint.

It is understood that modifications and 120 changes which may be made in the apparatus and method described above and which fall within the scope of the appended claims are embraced thereby.

WHAT WE CLAIM IS:-

1. A machine for forming a continuous flexible tube from a strip of flexible material having a cross sectional configuration including interlocking means along one edge por- 130

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tion interlockable with corresponding interlocking means along the other edge portion when the strip is wound in helical fashion, the combination of:

a frame means;

a pair of spaced rollers extending from said frame means with axes in non-parallel relation;

means for mounting said rollers on said 10 frame means for adjusting the space between axes of said rollers and the angular relation of said axes:

means on at least one of said rollers cooperable with edge portions of said strip of flexible material during helical winding of the strip; and

means for rotating both of said rollers for winding said strip thereabout and for discharge of the tube formed thereby from free ends of said rollers.

2. A machine as claimed in claim 1 in which connecting means are provided at the free ends of the rollers to adjust the spacing between the free ends.

3. A machine according to claim 1 or claim 2 including driving means on said frame means for rotating both of said rollers at the same speed.

4. A machine according to claim 4, in which the driving means for rotating both of said rollers includes means for directly driving said one of said rollers; and means cooperably connecting said other roller to said one roller for rotating both of said 35 rollers at the same speed of rotation.

5. A machine according to any preceding claim, in which means for cooperating with edge portions of the strip are provided on both of the rollers.

6. A machine as claimed in claim 5, in which the means on said rollers cooperable with edge portions of the strip during winding comprise axially spaced outwardly extending ribs on one of the rollers spaced apart by the width of the strip and a single outwardly extending rib on the other roller.

7. A machine as claimed in any preceding claim, including means for applying adhesive to one of the edge portions of the strip before the strip is wound in interlocking relation on the pair of rollers.

8. A machine according to claim 7 including guide means cooperable with the external configuration of the edge interlocking portions for imparting guiding pressure thereto and for maintaining contact of said adhesive with both of said interlocking edge portions.

A method of forming a continuous flexible tube by helically winding a strip of flexible material over a pair of spaced rollers having their axes in non-parallel relation so that interlocking means provided along one edge portion of the strip is interlocked dur-65 ing winding about said rollers with corres-

ponding interlocking means along the other edge portion, in which the strip is guided onto the rollers so that the interlocking means engage and the tube formed on the rollers is axially guided along the rollers by means on at least one of the rollers cooperable with edge portions of the strip, the diameter of the tube being variable by means for adjusting the spacing between the roller axes and the angular relation of the axes.

10. A method according to claim 9 in which the interlocked edge portions of the strip are engaged along substantially the length of one of said rollers during winding.

11. A method according to claim 9 in which the interlocked edge portions provide a continuous joint along edges of said wound strip, the internal surface of said joint being engaged by at least one of the rollers on at least one turn of said strip being wound.

12. A method according to any of claims 9 to 11 in which a ribbon of adhesive is applied to one of the interlocking edge portions of the strip before interlocking with the other edge portion of the adjacent turn of the strip.

13. A method according to any of claims 9 to 12 in which the means for guiding the strip onto the rollers includes means for applying external guiding pressure at each roller to the interlocked edge portions of the strip as the strip is wound on the rollers.

14. A method according to any of claims 9 to 13 in which the strip of flexible material is continuously extruded and transported to 100 the rollers prior to being fed onto said pair of rollers for winding thereabout under selected tension.

15. A machine for forming a continuous flexible tube from a strip of flexible material 105 substantially as described with reference to the accompanying drawings.

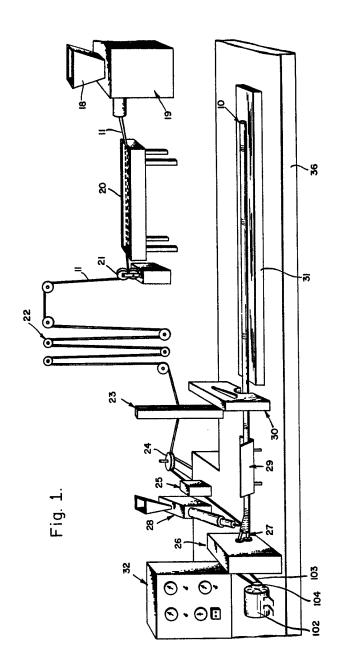
16. Apparatus for forming a continuous flexible tube including a tube forming machine as claimed in any one of claims 1 110 to 8 and claim 15, the apparatus further comprising extruding means for continuously extruding a strip of flexible material having interlocking means on its opposite edge portions, tension control means for advanc- 115 ing the strip to the tube forming machine at a desired tension, guide means for guiding the strip onto the tube forming machine, cooling means for cooling the formed tube as it leaves the tube forming machine, cut- 120 ting means for cutting the tube to desired lengths, and manually operable control means for operating the apparatus.

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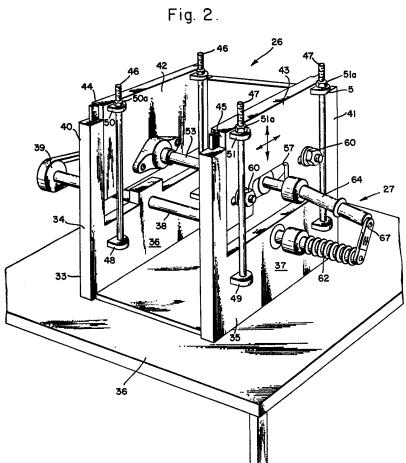
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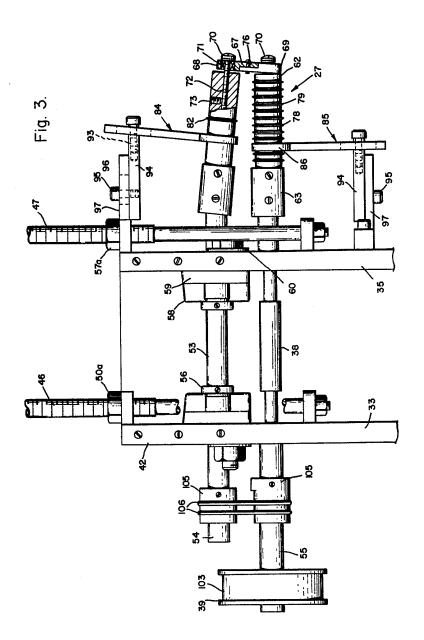




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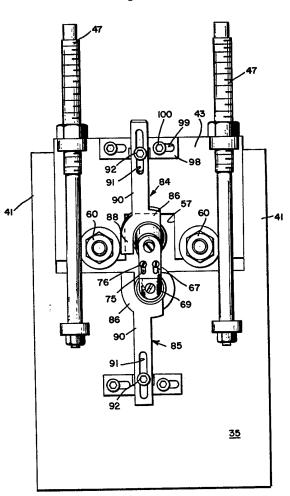
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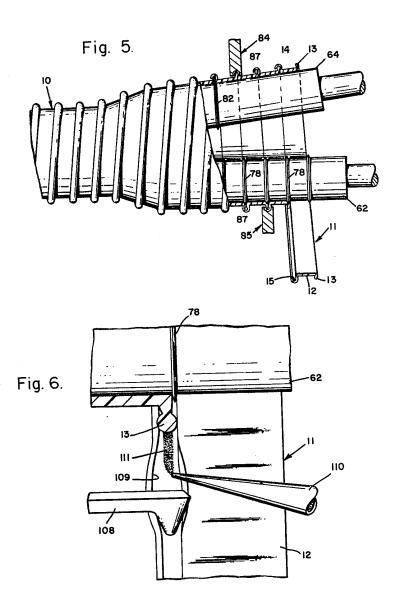




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