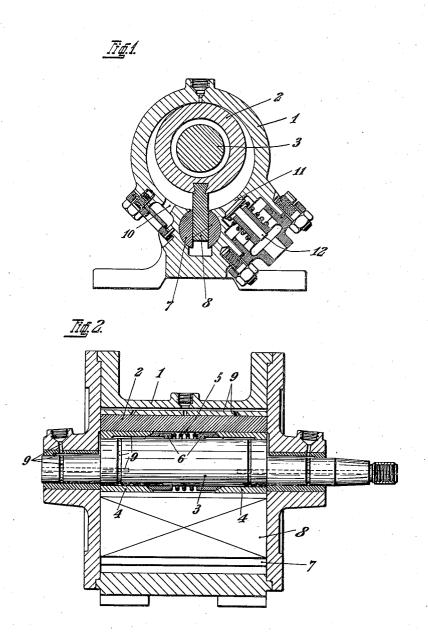
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ROTARY COMPRESSOR Filed Feb. 7, 1927



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ROTARY COMPRESSOR.

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This invention relates to high-pressure rotary compressors of the kind wherein a casing contains an eccentric rotor connected with a radial slide that is mounted to swing 5 in the casing, and divides the working chamber into a suction space and compression parts in both views. space, and has to withstand the working Referring to the d pressures.

One drawback met with in all compressors 10 of this kind and not yet obviated by any constructional modifications proposed heretofore was the defective joint between the rotor or slide and the casing. One cause of this defective joint is due to the fact that for 15 pivotally mounting the slide in the casing there was employed a one-piece boss or hub having a longitudinal groove entered by the slide. It has been found that when the slide was journaled in this manner there were 20 always losses of highly compressed gas through the groove as it is not possible in practise to make the groove so that the slide enters it without play so as to form a fluidtight joint. Another cause of leakage was 25 that owing to the high pressures the rotorshaft journaled at both ends in the casing became worn rapidly in its bearings and was consequently subject to play which made a tight joint between the rotor and inside face 30 of the casing impossible.

This invention has for its main object to obviate these defects and to provide an improved construction of compressor of the kind described.

According to the invention the boss or hub previously made in one piece comprises two segmental packing pieces rotatable in the casing, between which packing pieces the slide extends. For driving the rotor there serves an eccentric that is journaled in the rotor and has pivots situated coaxially in relation to the casing; bushings that are not rotatable in relation to the rotor are arranged between the latter and the eccentric; 45 also spring-pressed wedge-shaped pressure rings of very hard material are arranged between the bushings and the eccentric, which pressure rings prevent play between the eccentric and bushings even if any wear 50 is caused by friction.

One embodiment of the invention is illustrated diagrammatically and by way of example in the accompanying drawing,

Figure 1 is a cross-section through one

form of compressor according to the invention, and

Figure 2 is a vertical longitudinal section

thereof. Like reference numerals designate like 60

Referring to the drawing, a casing 1 has a rotor 2 arranged eccentrically in it. An eccentric 3 journaled centrically in the end walls of the casing 1 is rotatable in the rotor 65 At each end of the rotor 2 there is arranged between it and the eccentric 3 a bushing 4, for example of bronze, fixed against rotation in relation to the rotor, the inner ends of which bushings are conically 70 undercut. The middle of the eccentric is surrounded by a coil spring 5 which presses two wedge-shaped pressure rings 6 of hardened steel into the conical ends of the two bushings 4 and thereby prevents play be- 75 tween the bushings and the eccentric in the event of the bushes becoming worn.

In the lower part of the casing 1 two segmental packing pieces 7 are journaled, between which extends fluid-tight a slide 8 so mounted radially on the rotor. This slide is ground exactly in the packing pieces. Owing to the slide being pivotally mounted in this manner in the casing, if the slide and packing pieces are carefully lubricated play 85 between these members can be prevented and leakage at this place can be avoided.

Ducts or grooves 9 serve for lubricating all the rubbing parts of the compressor. The inlet port 10 for fresh gas and the outlet 90 port 11 for the compressed gas are controlled each by a valve 12, whereof only one is illustrated.

It has been found that with the described compressor pressures up to 10 atmospheres 95 can be obtained when working with a single stage.

I claim: 1. A high-pressure rotary compressor of the kind described, comprising in combina- 100 tion a cylindrical casing containing a working chamber, an eccentric in the latter with pivots journaled centrically in the end walls of the casing, a rotor rotatably carried by said eccentric and making fluid-tight con- 105 tact with the wall of said working chamber, a slide fixed at one end radially on said rotor and extending into a cylindrical cavity in said casing, packing means for the slide, journaled in said cavity, and packing means 110

axially adjustable on said eccentric and be tween the same and the rotor for preventing play between the same when wear of one of

these parts occurs.

2. A high-pressure rotary compressor according to claim 1, wherein said axially adjustable means comprise two bushings surrounding the eccentric and fixed against rotation in relation to the rotor, and two 10 spring-pressed pressure rings of very hard material surrounding said eccentric and ex-tending each between the adjacent end of a ciated bushing. said bushing and the eccentric.

3. A high-pressure rotary compressor as 1927.

15 set forth in claim 1, wherein said means comprise two bushings surrounding the ec-

centric and fixed against rotation in relation to the rotor and spaced apart in the direction of length of the eccentric, the inner ends of which bushings are conically undercut, two 20 pressure rings of very hard material spaced apart and surrounding said eccentric between the bushings and having each a wedgeshaped end extending between one of the conical ends of the bushings and the eccen- 25 tric, and a spring between said pressure

Signed at Berne, this 18th day of January

HERMANN WEBER.