	EXCAVATION WITH CALLY GENERATED SHOCK
Inventor:	Jay R. Allgood, Camarillo, Calif.
Assignee:	The United States of America as represented by the Secretary of the Navy, Washington, D.C.
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	References Cited
UNI	TED STATES PATENTS
517 1/19	43 Konnerth 299/14
	ELECTRI WAVES Inventor: Assignee: Filed: Appl. No. U.S. Cl Int. Cl Field of Se

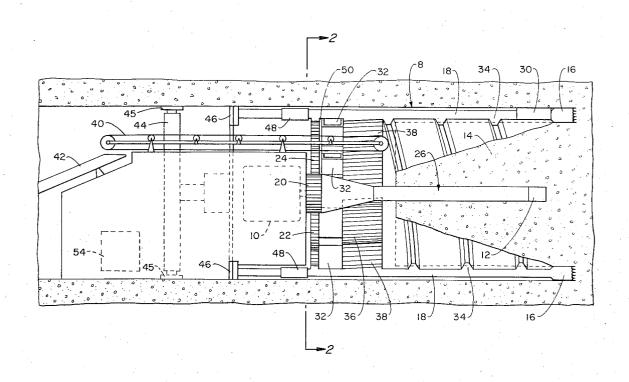
3,219,280	11/1965	Seldenrath et al 299/14 X
3,266,845	8/1966	Williamson et al 299/13 X
3,511,538	5/1970	Guenter 299/13

Primary Examiner—Ernest R. Purser Attorney, Agent, or Firm—Richard S. Sciascia; J. M. St. Amand; David O'Reilly

[57] ABSTRACT

An apparatus and method for excavating tunnels with electrically generated shock waves. The tunneling apparatus is comprised of a rotatable barrel for cutting a peripheral ring about a core and a bit which simultaneously bores a hole in the center of the core. The drive for the rock bit also contains electrodes which are pulsed to produce shock waves in the core to peel off a desired thickness of rock layer.

4 Claims, 4 Drawing Figures



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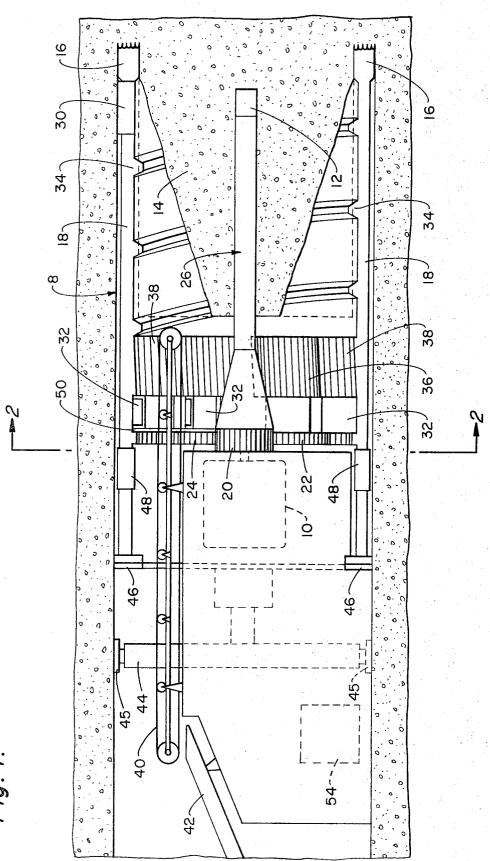
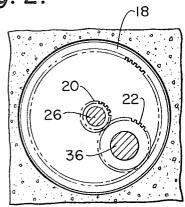
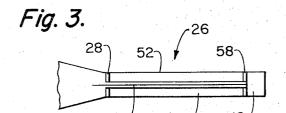


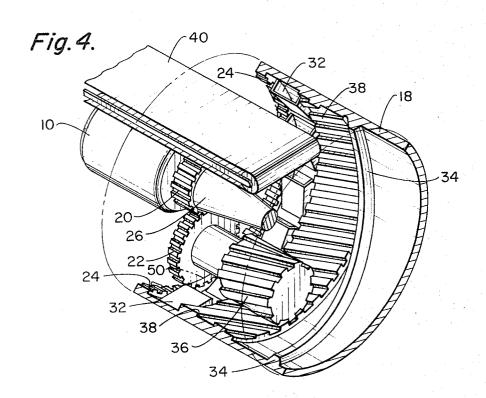
Fig. 1.

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Fig. 2.







TUNNEL EXCAVATION WITH ELECTRICALLY GENERATED SHOCK WAVES

BACKGROUND OF THE INVENTION

1. Field of the Invention

The present invention relates to earth boring devices and particularly to rock boring devices wherein stress waves are utilized for rock disintegration.

2. Description of the Prior Art

Typically, boring machines used to data consist of a 10 generator. rotating head on which is mounted a number of individual bits or cutters. Most of the advances in the art have depended on improvements in such bits and cutters. In effect, these cutters "chew" away the entire face of the opening being excavated. As a consequence, existing boring machines are unnecessarily limited in advance rate and by the maximum strength of rock which can economically be bored. Other investigators and inventors have recognized the deficiencies of existing machines and, with the motivation of a predicted multibillion dollar market for rock excavation in the next ten yeaars, are developing a variety of schemes for rapid excavation of rock. Most of these schemes are impractical because of their large operating energy requirements. What is needed is a machine which uses less energy and permits greater advance rates than present ones. The present invention achieves this goal.

SUMMARY OF THE INVENTION

This invention is principally concerned with the excavation of tunnels by the disintegration of a trepanned rock core with a train of stress waves. First, a small central hole is bored and a thin perimetrical slot is trepanned at the face of the tunnel. As these operations 35 progress, a train of stress waves is propagated from a pulser in the central hole. As the stress waves reflect from the exterior free surface of the core, successive layers of rock are "peeled" off.

Some of the primary advantages of this invention 40 where over prior art devices are:

- 1. Much less energy is required per unit of excavated
- 2. Only a fraction of the axial thrust is required.
- 3. Control of the particle size of the muck is possible. 45
- 4. Fracture of the walls from high contact pressure of mechanical thrusters is precluded.

These and other advantages accrue from incorporation in the machine of subsystems utilizing present 50 state-of-the-art techniques.

OBJECTS OF THE INVENTION

An object of this invention is to provide a means for rapid and economical excavation of rock.

Another object is to provide apparatus utilizing stress waves for disintegration of rock.

Another object is to provide a method of excavating rock in which apparatus utilizing stress waves is em-

Still another object is to provide stress wave apparatus for rock disintegration which expends only slight amounts of energy.

Other objects, advantages and novel features of the invention will become apparent from the following detailed description of the invention when considered in conjunction with the accompanying drawings.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a schematic view of a device incorporating the present invention showing the relation of the central boring device, cutters and stress wave mechanism.

FIG. 2 is a transverse sectional view along the line 2—2 of FIG. 1.

FIG. 3 is an enlarged schematic view of the shock

FIG.. 4 is a partial sectional view of the barrel and interior thereof showing details of the rock crushing means.

DESCRIPTION OF THE PREFERRED **EMBODIMENT**

Referring to FIG. 1, the operation and structure of the invention is as follows. The power unit 10 drives the rotary bit 12, which bores a hole in the center of the 20 rock core 14.

A variety of power sources are suitable for use, such as a Synchrogear Motor, manufactured by Emerson Electric Company.

Simultaneously, the core is trepanned by the cutters 25 16, attached to the barrel 18, driven through gears 20 and 22, the latter of which meshes with a rack 24, fastened to the barrel intrados. As the rock core 14 is trepanned, periodic cylindrical compression waves of high intensity and very short duration are propagated outward in the core by a pulser 26. These waves reflect from the outer boundary as tensile waves which exceed the tensile strength of the rock and "peel off" a layer of rock. The thickness of the layer of rock removed is governed by the duration of the stress wave.

An approximate relation for the thickness of rock layer removed from the extrados of the core is expressed by the following equation:

$$t = \eta \ \mathbf{C} f'_{td} / 2\sigma \tau$$

η = shape factor

C = seismic velocity of the rock

 f'_{td} = unconfined tensile strength of the rock

 σ = amplitude of the compressional wave as it reaches the extrados

τ = effective duration of the stress wave

From the foregoing equation, it is apparent that the thickness of the layer removed is directly dependent on the duration of the stress pulse. This duration must be of the order of a few microseconds to keep the thickness small enough for easy handling.

As the barrel 18 progresses in the excavation operation, a suitable concrete or other liner is cast around the rock intrados to ensure safety of the equipment.

Grindings which pass outside of the barrel 18 are picked up by bucket 30 and dumped inside through the hole beneath it as the bucket 30 reaches the crown position. Pieces of rock peeled from the core fall to the bottom of the barrel and are transported toward the multiplicity of pick-up buckets 32 by an inwardly extending helical-shaped vane 34. Enroute the material passes a crusher 36 where any large pieces of rock are broken to a small enough size to be transported in the buckets 32. In the band of the barrel 18, which passes under the rock crusher, there are short segments of 3

long pitch helixes 38 (FIG. 4). These segments aid in crushing the rock and in the rapid transport of the broken material into the buckets 32 and, thence, to the conveyor 40.

The rock crusher 36 is more clearly illustrated in 5 FIG. 4. The rock crusher 36 may be any conventional type but the roll type, shown in FIG. 4, manufactured by Universal Crusher Company, Cedar Rapids, Iowa, is preferred. As the rock peeled from the core passes from the helical-shaped vane 34 (FIG. 1) to the long pitch helixes 38, any large rocks are crushed into smaller pieces by the meshing of the gears of rock crusher 36. The crushed rock is then rapidly transported by the long pitch helixes 38 into open ended buckets 32, which scoop them up as the barrel turns 15 and deposits them on conveyor 40. Conveyor 40 dumps into chute 42 (FIG. 1) which may empty into a disposal conveyor system or into mucking cars.

Item 44 is one of two reaction and thruster elements that take the longitudinal load and cause the machine 20 to advance. The thrust is applied through thrust bearings 46 to the barrel 18. Rollers 48 maintain the alignment and prevent the extrados of the barrel from wearing away as it rotates. The annular rings 50 prevent crushed material from getting under the drive unit.

Several types of thrusters could be used with the Universal excavator. One such thruster is manufactured by the Jarva Inc., Solon, Ohio. Legs 45 (only two of which are shown for the sake of clarity) are clamped and a thrust is applied to thrust bearing 46 by hydraulic 30 thrusters (not shown). After the hydraulic thrusters are fully extended, legs 45 are unclamped and the drive portion (not shown) of the thruster is moved forward. Somewhat different thrusters are available from other manufacturers but they operate on the same principle. 35 The disadvantage of these mechanical and hydraulic thrusters is that they can damage the tunnel walls. Therefore, a polytoroidal thruster, which is the subject of Patent application Ser. No. 353,282, filed Apr. 23, 1973, by Dr. Howard Gaberson, is highly preferred. 40 The polytoroidal thruster operates on the principle of vermicular motion. That is, a series of inflatable bags in the form of a cylinder, which are sequentially inflated and deflated, provide the thrust.

The pulser 26 is comprised of a rock bit 12 and drive 45 56 for advancing the entire unit. Pulser 26 also contains a controlled pulse shock wave generator, shown in FIG. 3, which operates by creating an electrical discharge between a cathode 28 and an anode 58 through a fluid 60. An ideal electrohydraulic pulser unit is manufac- 50 tured by Environment-One (E-1) Company. The general principle of operation is described in a paper entitled "Electrohydraulics" by W. R. Brown, M. Allen and E. C. Schron, Science Journal, March 1968. In the present concept, the load is transmitted to the rock 55 ing: through a flexible cylindrical diaphragm 52. The electronics of the system are housed at 54 in the rear of the boring machine 8. Air is forced down through drive 56 and returns via a parallel path with the grindings from the bore hole produced by rock bit 12.

Thus, it is clear that only small quantities of the rock face are removed by the bit 12 and the cutters 16; hence, only a small amount of axial thrust is needed. Furthermore, because the rock is removed in relatively

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large pieces, considerably less energy is required.

Obviously, many modifications and variations of the present invention are possible in the light of the above teachings. It is therefore to be understood that within the scope of the appended claims the invention may be practiced otherwise than as specifically described.

I claim:

 A stress wave rock boring device comprising: a hollow, rotatable barrel having a plurality of cutters positioned peripherally at the forward end of said barrel for trepanning a core;

means for boring a hole in the center of said core; means for simultaneously driving the barrel and center hole boring means;

said center hole boring means including a controlledpulse shock wave generator comprising:

a cathode and anode with a fluid positioned there-

means for creating an electrical discharge between the cathode and anode, creating a controlledpulse shock wave which travels radially outward and reflects from the boundary of the core, thereby peeling off a desired thickness of rock layer.

2. The stress wave boring device of claim 1 wherein the approximate thickness of rock layer removed from the entrados of the core is defined by the following equation:

 $t = \eta \ C \ f'_{td}/2\sigma \ \tau$

where

 η = shape factor

C = seismic velocity of the rock

 f_{td} = unconfined tensile strength of the rock σ = amplitude of compressional wave as it reaches the extrados

 τ = effective duration of stress wave

whereby the thickness of the layer or rock removed is directed dependent on the duration of the stress pulse.

- 3. The stress wave rock boring device as defined in claim 1 further including a helical-shaped vane mounted on the interior wall of said barrel to force the pieces of rock peeled from said core toward the aft end of said barrel;
 - a series of spaced buckets mounted near the aft end of said barrel and adapted to transport said rock out of the barrel; and
 - a conveyor means upon which said rock is placed for further removal of said rock.
- **4.** A stress wave method of removing rock comprising:

cutting a peripheral ring about a core of rock; boring a center hole in said core;

generating shock waves by applying electrical pulses to electrodes in said center hole to simultaneously produce stress waves of controlled amplitude and duration which travel outward and reflect from the boundary of the core, thereby peeling off a desired thickness of rock layer.

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