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(54) **UPRIGHT VACUUM WITH REDUCED NOISE**

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(57) **ABSTRACT**

According to various embodiments, a method of reducing noise generated by a vacuum is described. The method comprises providing a vacuum comprising a motor including a motor shaft adapted to operate at a rotational speed of n; attaching a centripetal fan with t number of blades attached to the motor shaft; enclosing the fan between the motor and a motor housing end bracket disposed about the shaft, the motor housing end bracket defining an air input opposite an air output; wherein the variables of speed n and the number of blades t are selected so that a blade pass frequency (BPF) of the centripetal fan is greater than about 5000 Hertz, and the BPF is calculated as BPF=n*t/60. A vacuum implementing the method is also described.

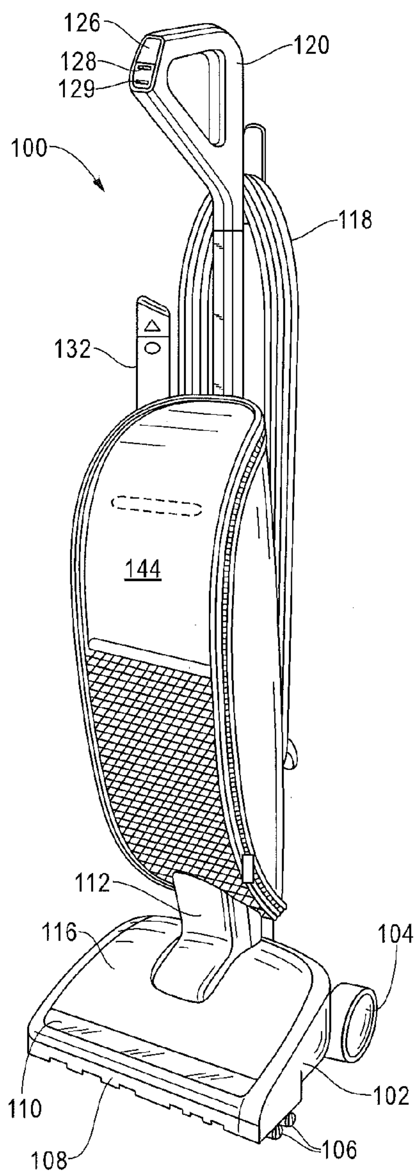
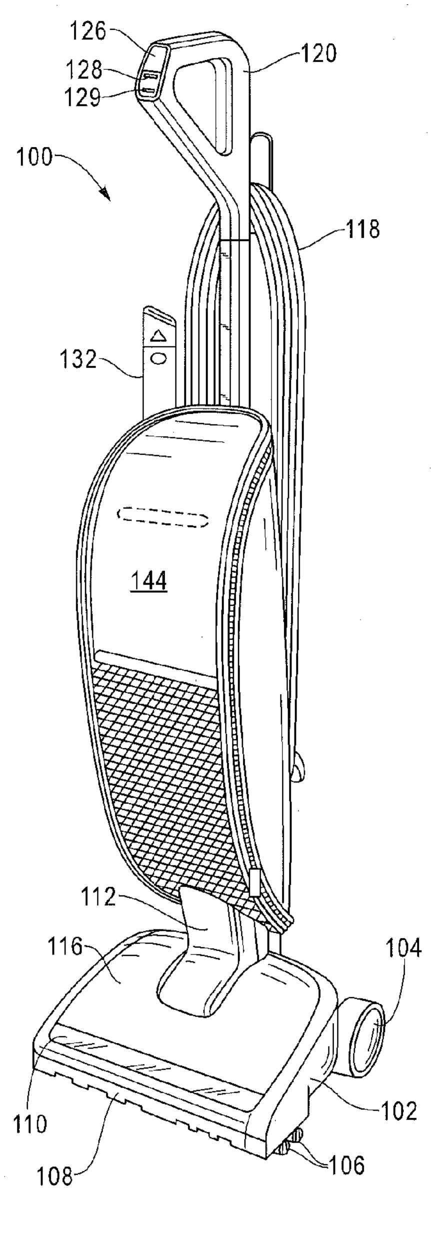


Fig. 1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128
129
120
100
118
132
144
112
116
104
110
102
108
106

Fig. 2

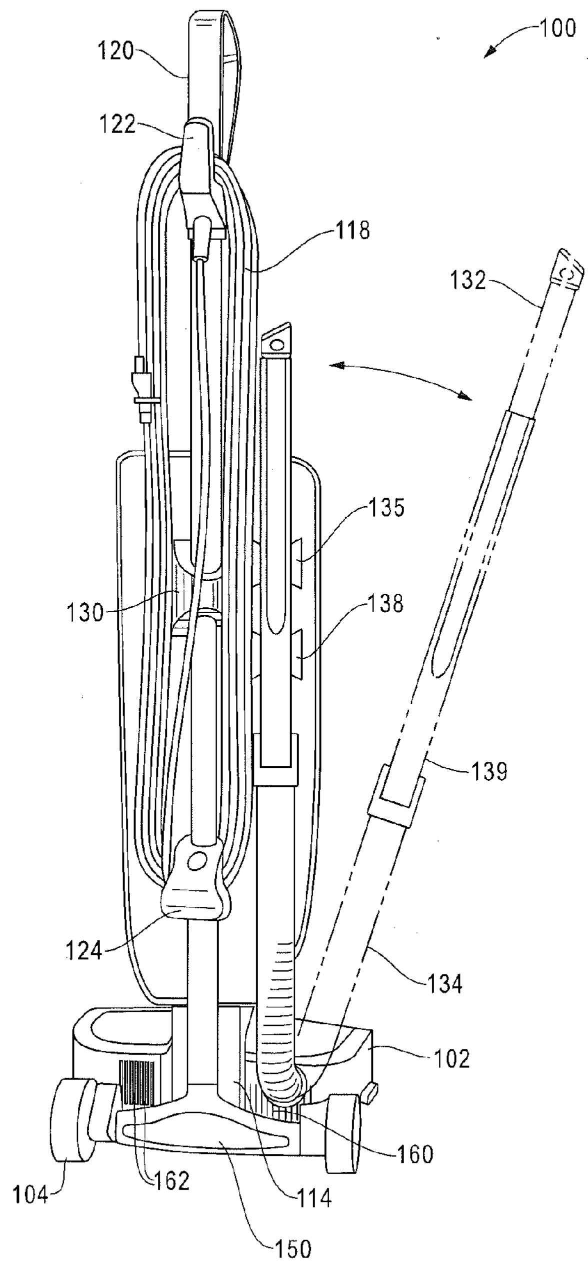
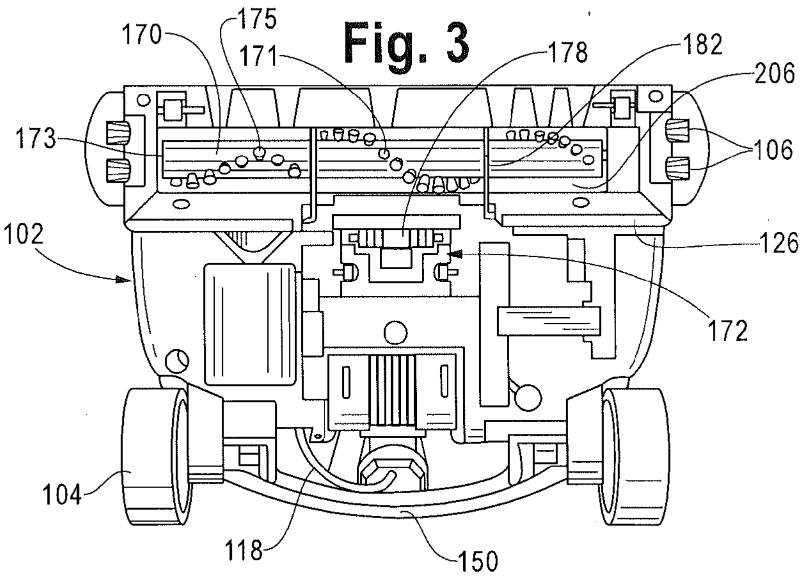
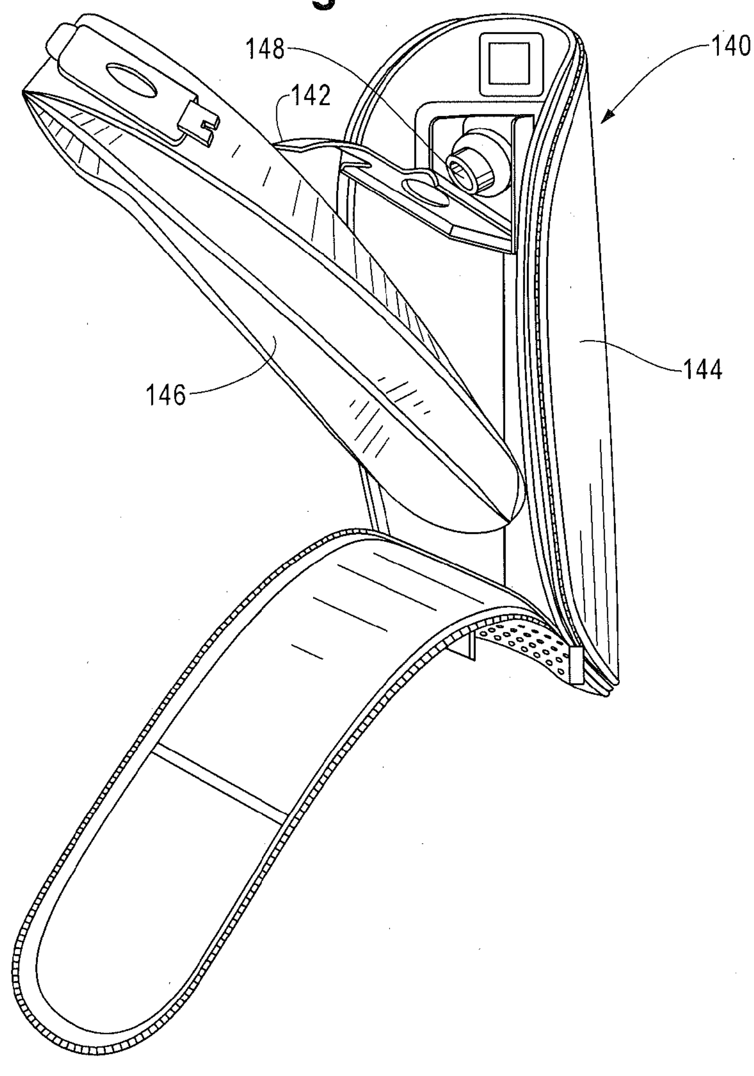

120
100
122
118
132
135
130
138
139
124
134
102
160
104
162
114
150

Fig. 3



175
170
171
178
182
206
173
106
102
126
172
104
118
150

Fig. 4



148
140
142
146
144

Fig. 5

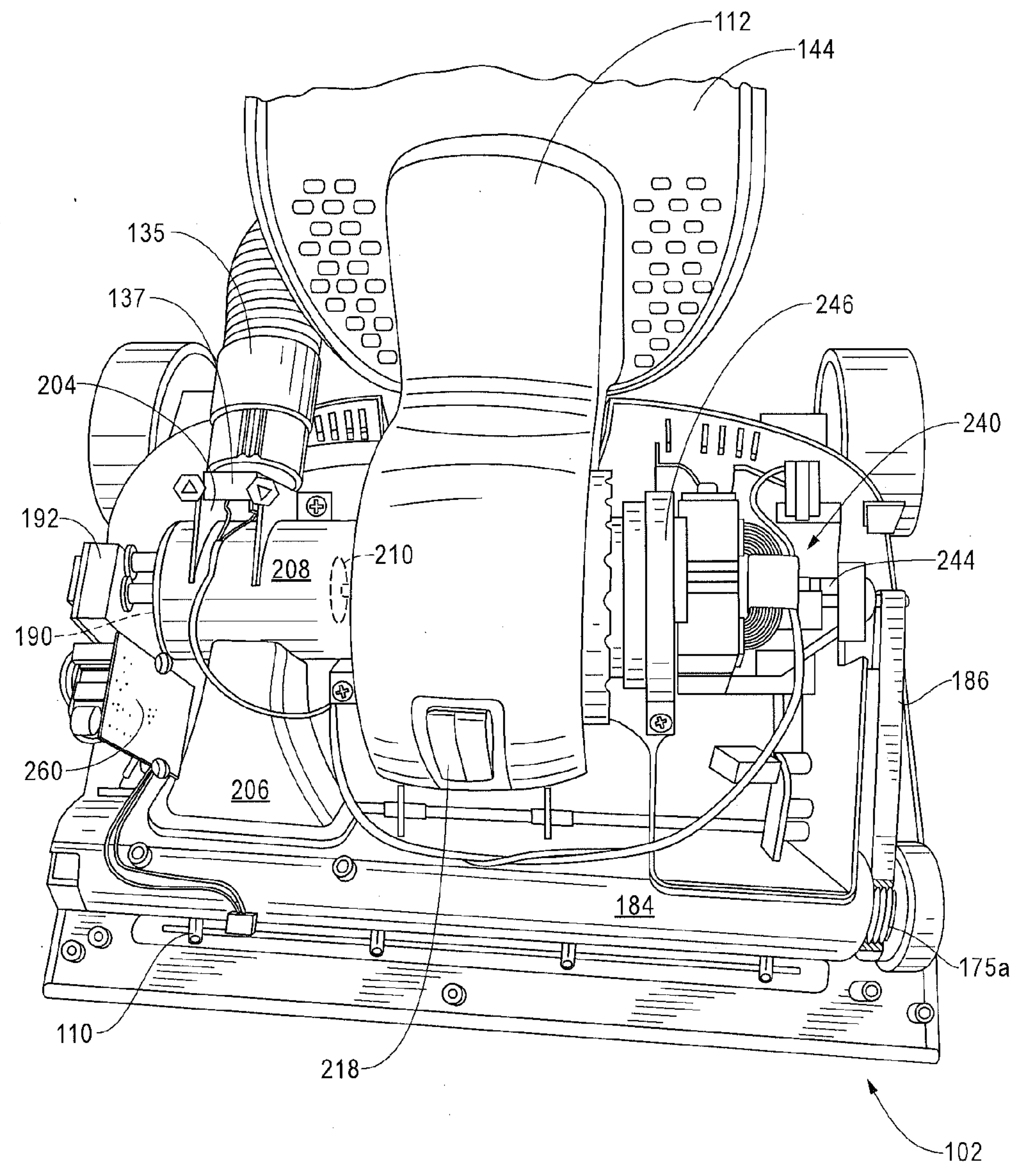
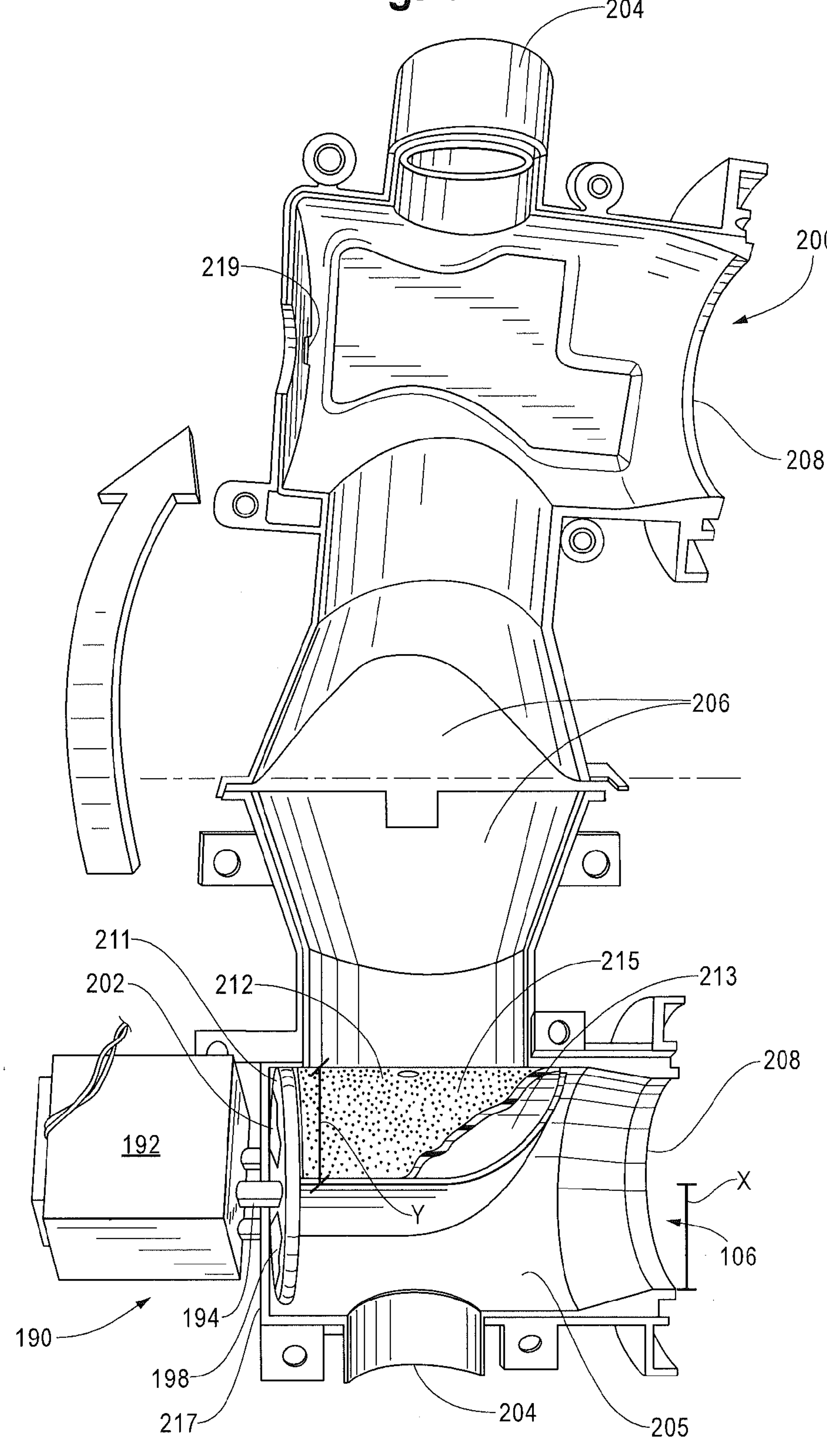

112
144
135
137
204
192
190
260
246
240
244
186
208
210
206
184
175a
110
218
102

Fig. 6


204
200
219
208
206
211
215
213
202
212
208
192
X
Y
106
190
194
198
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204
205

Fig. 7A

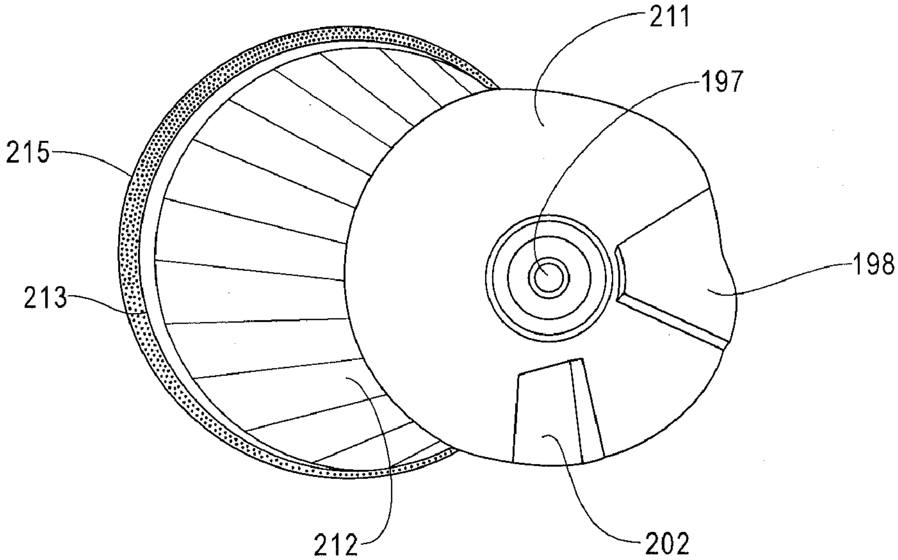


Fig. 7B

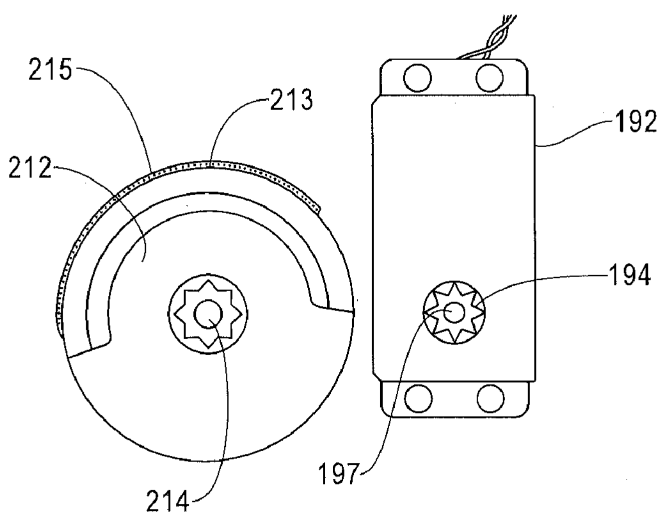


Fig. 8A

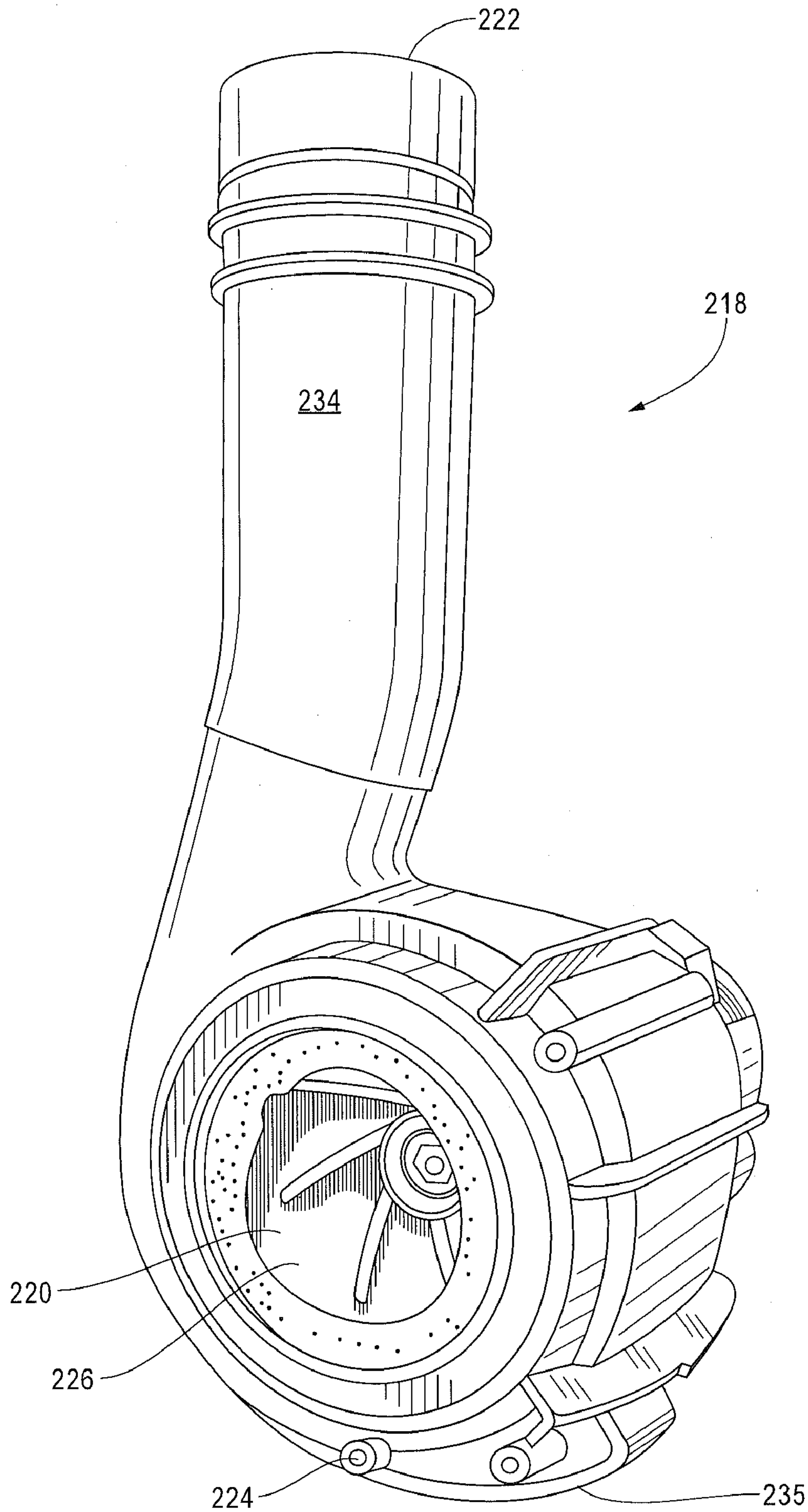


Fig. 8B

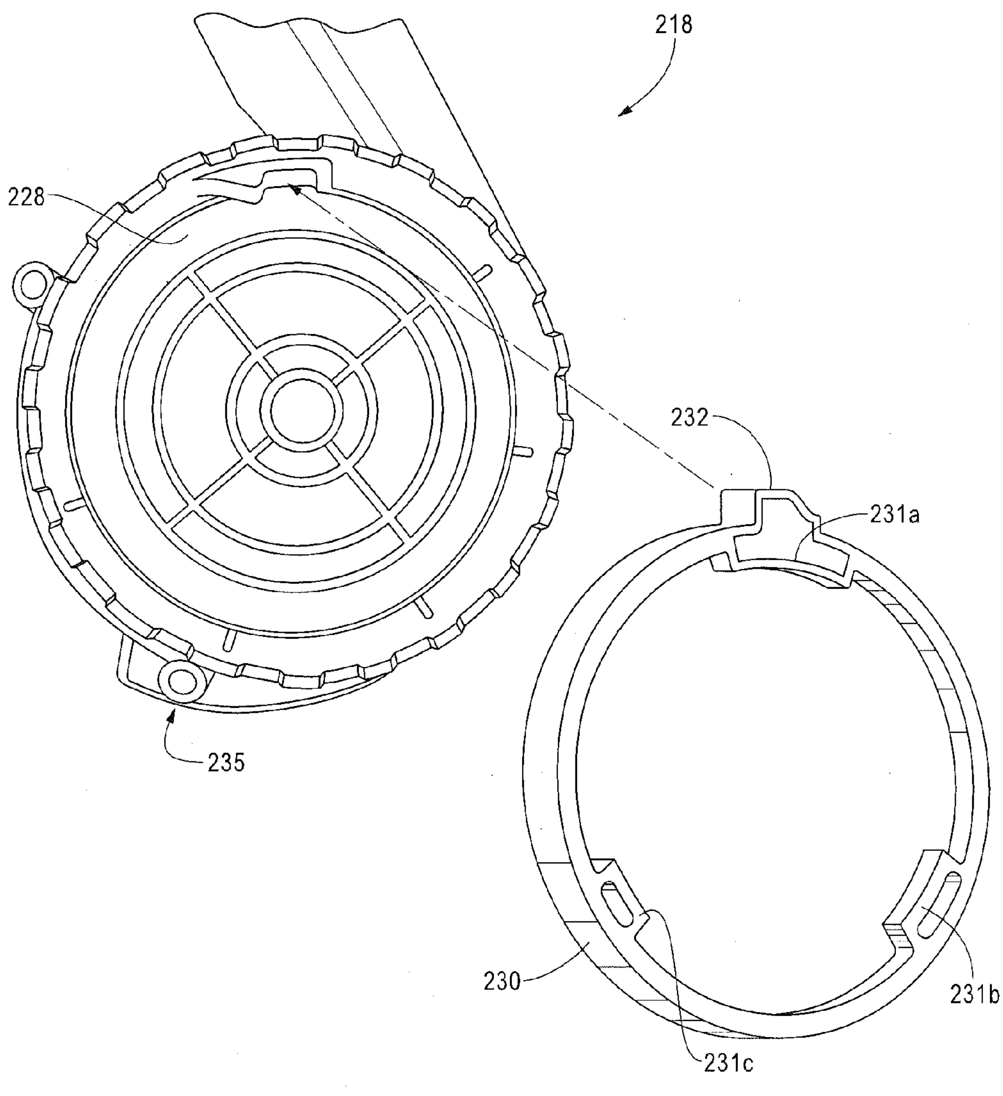


Fig. 9

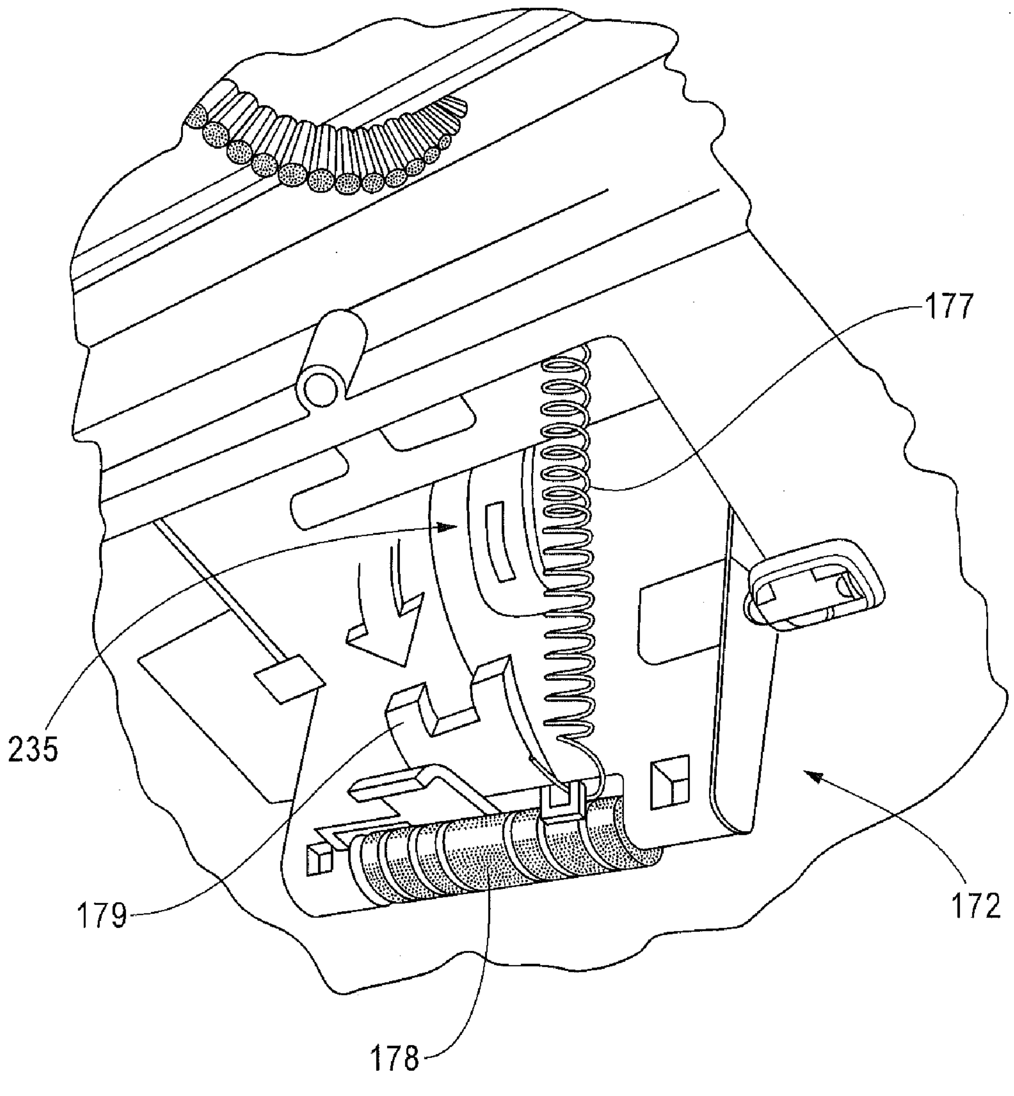
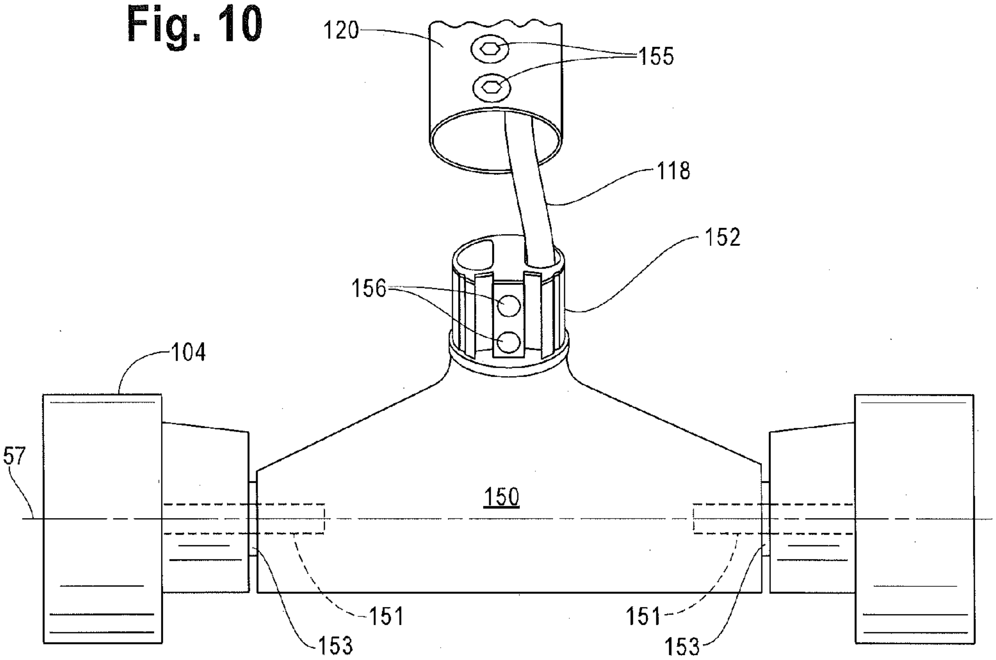


Fig. 10


120
155
118
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104
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153
153

Fig. 11

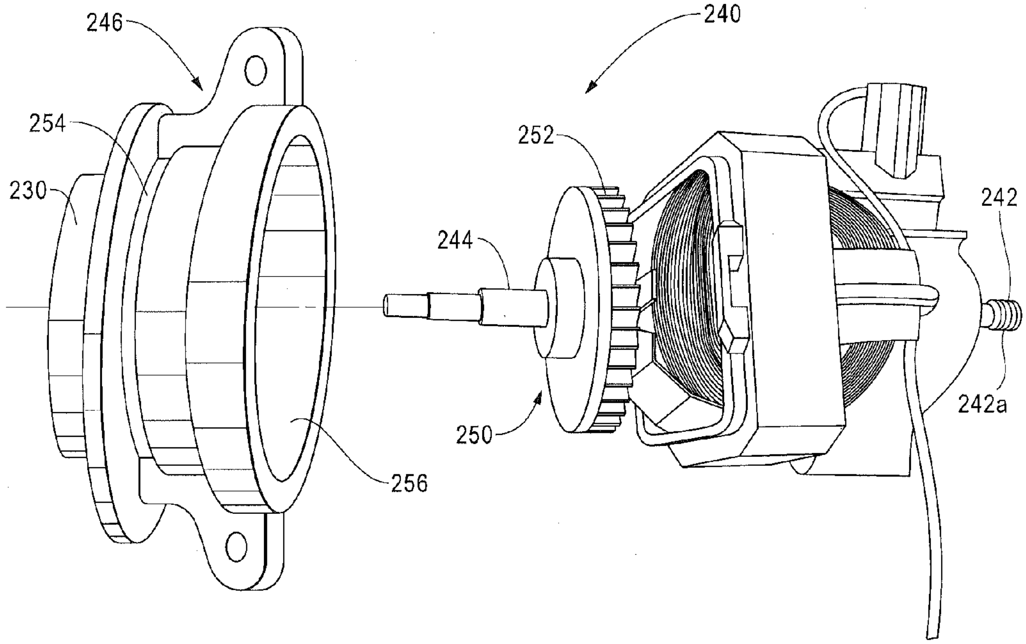
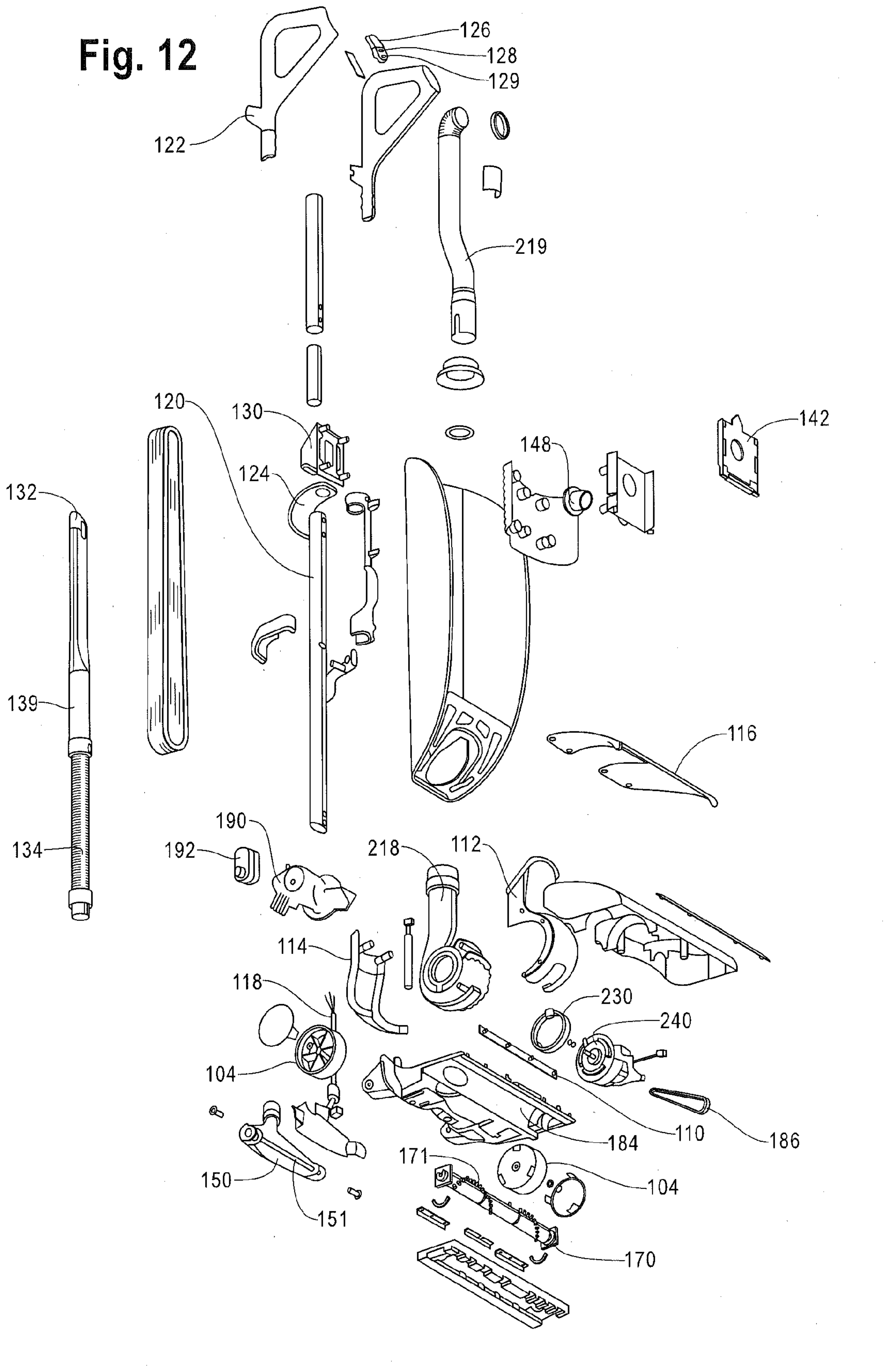

246
240
254
252
230
242
244
242a
250
256

Fig. 12


126
128
129
122
219
120
130
142
148
124
132
139
116
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240
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186
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184
171
150
151
104
170

Fig. 13A Prior Art

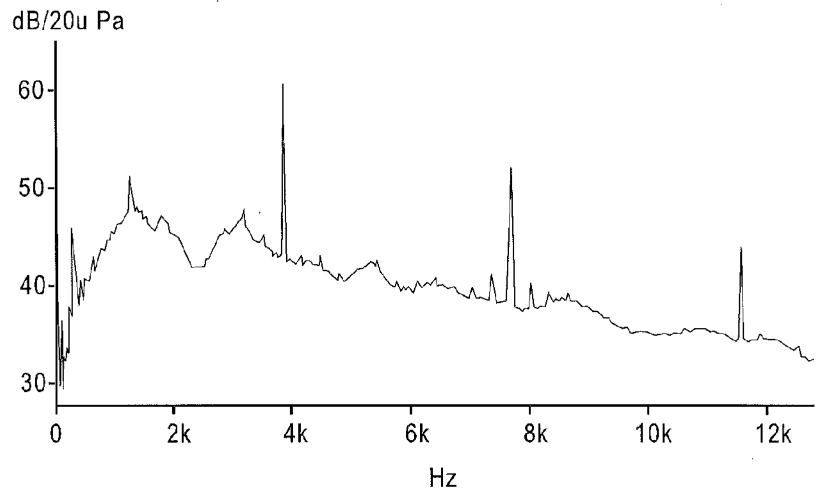


Fig. 13B

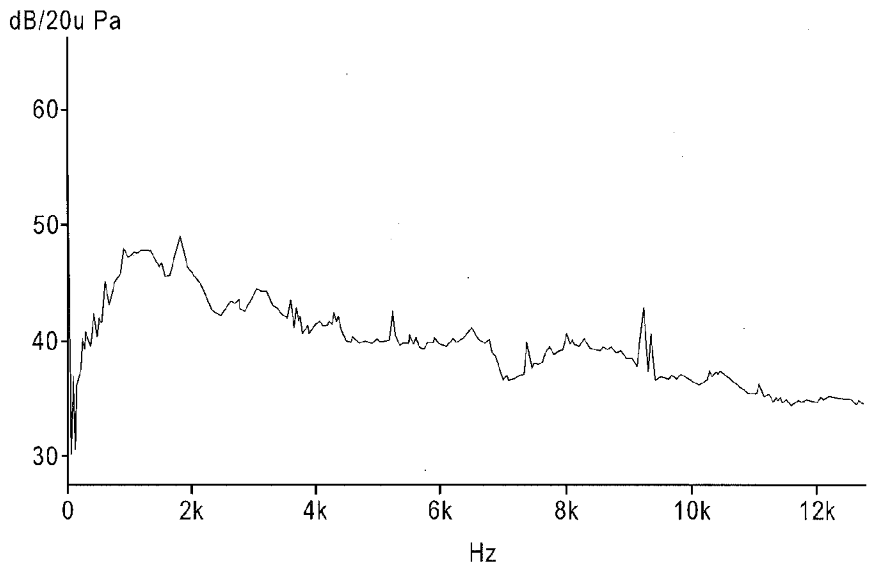


Fig. 14

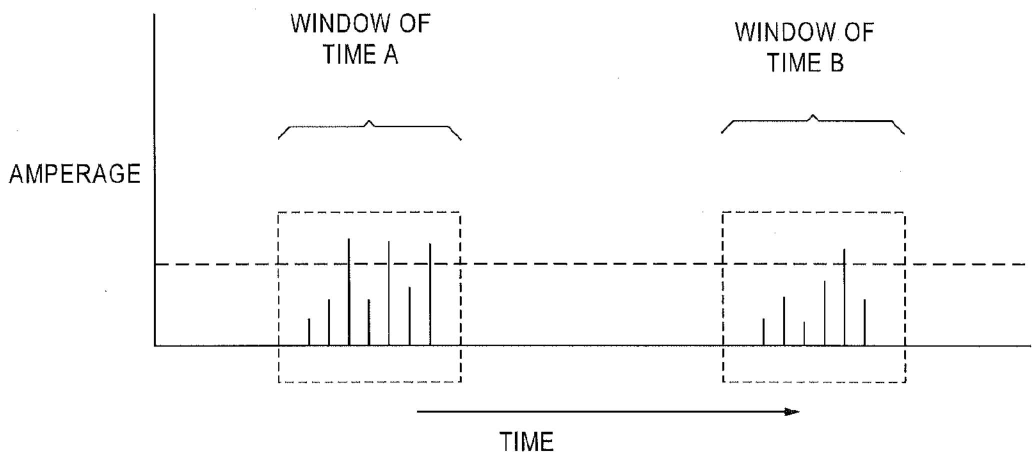

WINDOW OF
TIME A
WINDOW OF
TIME B
AMPERAGE
TIME

Fig. 15

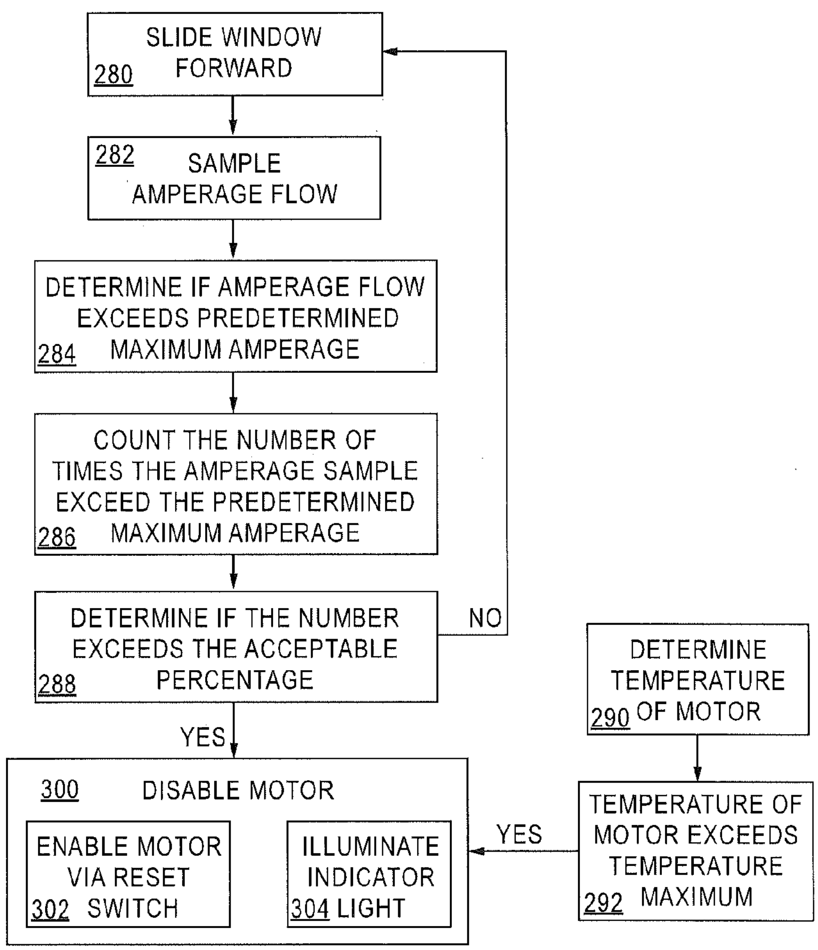
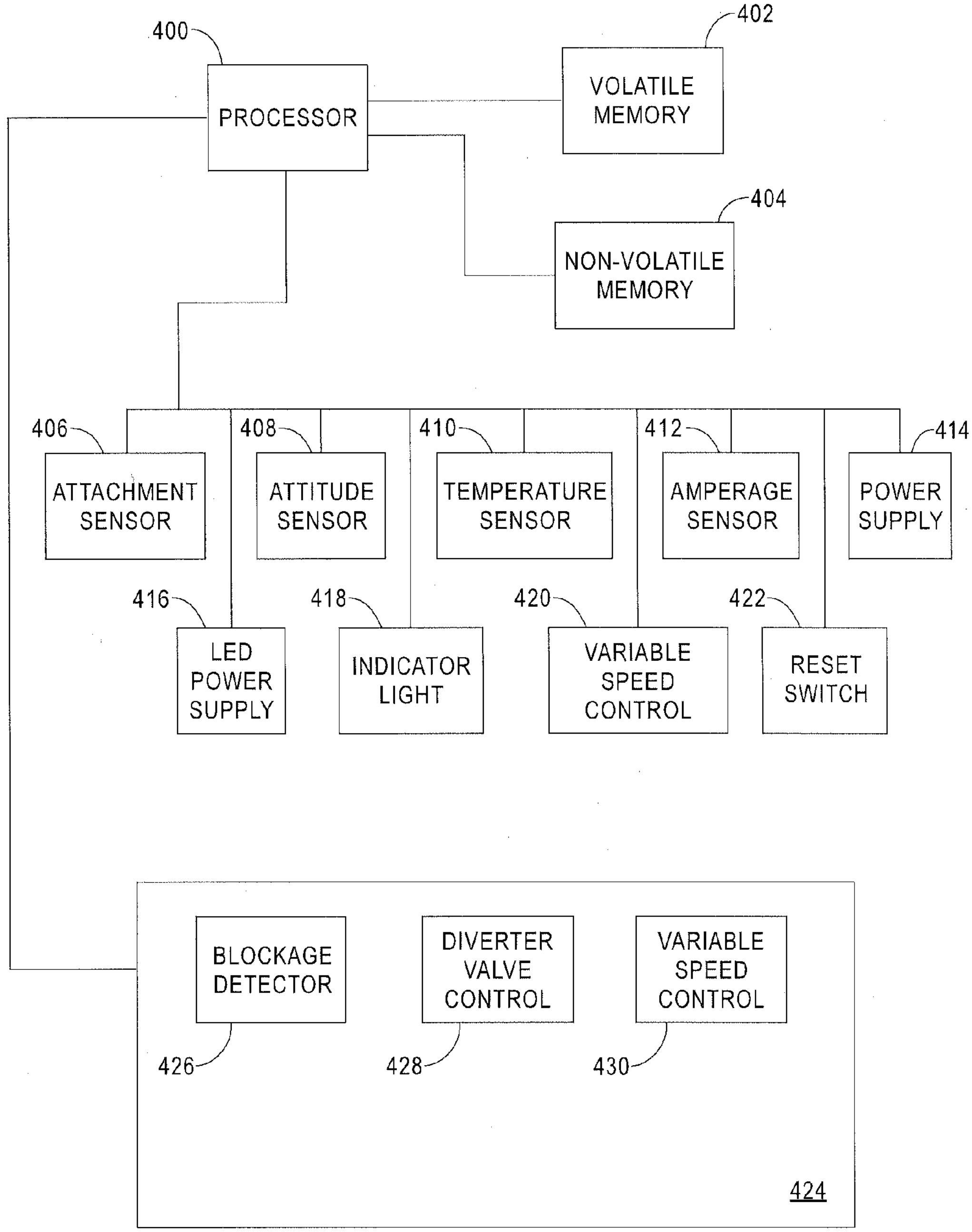

SLIDE WINDOW FORWARD
280
282
SAMPLE AMPERAGE FLOW
DETERMINE IF AMPERAGE FLOW EXCEEDS PREDETERMINED MAXIMUM AMPERAGE
284
COUNT THE NUMBER OF TIMES THE AMPERAGE SAMPLE EXCEED THE PREDETERMINED MAXIMUM AMPERAGE
286
DETERMINE IF THE NUMBER EXCEEDS THE ACCEPTABLE PERCENTAGE
288
NO
YES
DETERMINE TEMPERATURE OF MOTOR
290
300
DISABLE MOTOR
ENABLE MOTOR VIA RESET SWITCH
302
ILLUMINATE INDICATOR LIGHT
304
YES
TEMPERATURE OF MOTOR EXCEEDS TEMPERATURE MAXIMUM
292

Fig. 16


400
PROCESSOR
402
VOLATILE MEMORY
404
NON-VOLATILE MEMORY
406
ATTACHMENT SENSOR
408
ATTITUDE SENSOR
410
TEMPERATURE SENSOR
412
AMPERAGE SENSOR
414
POWER SUPPLY
416
LED POWER SUPPLY
418
INDICATOR LIGHT
420
VARIABLE SPEED CONTROL
422
RESET SWITCH
BLOCKAGE DETECTOR
426
DIVERTER VALVE CONTROL
428
VARIABLE SPEED CONTROL
430
424

UPRIGHT VACUUM WITH REDUCED NOISE

TECHNICAL FIELD

[0001] The present teachings are directed toward reducing noise generated vacuum cleaners. In particular, the disclosure relates to an upright vacuum cleaner that has a handle, motor and centrifugal fan that can provide reduced noise generation at certain frequencies in a vacuum's noise profile. The unique design of the centrifugal fan results in an increase in the amount of air moved, increased cooling capabilities of the vacuum, as well as a significant reduction in noise generated by the vacuum motor.

BACKGROUND

[0002] A need has been recognized in the vacuum cleaner industry for upright model vacuum cleaners that are easy, efficient to use, light weight, and have reduced manufacturing costs while providing superior cleaning abilities. The prior art powerful upright vacuum cleaners often have powerful motors and large fans for increasing air movement. However, these designs have many drawbacks. In vacuum cleaners where the largest motors and biggest fans are used, the amount of noise generated by the vacuum becomes a nuisance to the user. Testing of prior art vacuums reveals that many produce low frequency sounds between about 2000 to about 4000 Hertz (Hz). Sounds within the 2000 to 4000 range are often considered "irritating" noises. Long teen use of noisy appliances in general can result in hearing damage and loss. In attempts to reduce the noise generated by vacuum cleaners, manufactures have sacrificed motor power/size and fan size. In addition, prior art vacuum using more powerful motors and heavy, metal fans are more expensive to produce. However, these prior art vacuum cleaners have poor cleaning abilities.

[0003] The vacuum cleaners and methods of reducing noise in a vacuum described herein are efficient, light weight, and have reduced manufacturing costs. The upright vacuums of the instant invention utilize injection molded parts, for motor housings, as well as fans. The costs of injection molding is significantly lower than those of stamped metal. Another advantage is that the instant vacuum cleaners have larger motors without the additional weight associated with similar vacuum designs due to the use of lighter materials. Another advantage is that the instant vacuums are capable or producing high frequency sounds that are outside of a human's hearing. Thus, the user has a more pleasant vacuuming experience. The advantages over the prior art vacuums, such as the quieter, less expensive to produce, and unique fan design move more Cubic Feet Per Minute (CFM) of air with less irritating accompanying noise that other prior art vacuums.

SUMMARY

[0004] According to various embodiments, a method of reducing noise generated by a vacuum is described. The method comprising: providing a vacuum comprising a motor including a motor shaft adapted to operate at a rotational speed of n; attaching a centripetal fan with t number of blades attached to the motor shaft; enclosing the fan between the motor and a motor housing end bracket disposed about the shaft, the motor housing end bracket defining an air input opposite an air output; wherein the variables of speed n and the number of blades t are selected so that a blade pass frequency (BPF) of the centripetal fan is greater than about 5000 Hertz, and the BPF is calculated as $BPF=n*t/60$.

[0005] In some embodiments, the speed n is greater than 10,000 revolutions per minute (RPM). In some embodiments, the total area of the centripetal fan blade area generates greater than about 20 cubic feet per minute of airflow between the air input and the air output.

[0006] In some embodiments, the centripetal fan has 30 or more blades.

[0007] In some embodiments, the motor selectively operates at about 13000 RPM and about 18000 RPM.

[0008] In some embodiments, the blades of the centripetal fan have a slight backward curve relative to a rotational direction of the fan.

[0009] The method can be used for cooling components disposed in the vacuum base with the airflow exiting the air output. In some embodiments, the method comprises: determining the temperature of the motor and shutting off the motor when the temperature exceeds a predetermined temperature.

[0010] According to various embodiments, a vacuum is described. The vacuum comprises: a motor including a motor shaft to operate at a rotational speed of n; a motor housing end bracket disposed about the shaft, the motor housing end bracket defining an air input opposite an air output; and a centripetal fan with t number of blades attached to the motor shaft and enclosed between the motor and the motor housing end bracket. The variables of speed n and the number of blades t are selected so that a BPF of the centripetal fan is greater than about 5000 Hertz, and the BPF is calculated as $BPF=n*t/60$.

[0011] In some embodiments, the blades of the centripetal fan have a slight backward curve relative to a rotational direction of the fan. In some embodiments, the vacuum includes an air exhaust disposed in a vacuum base and an air intake in the vacuum base in the same plane the an air exhaust.

BRIEF DESCRIPTION OF THE DRAWINGS

[0012] The same reference number represents the same element on all drawings. It should be noted that the drawings are not necessarily to scale. The foregoing and other objects, aspects, and advantages are better understood from the following detailed description of a preferred embodiment of the invention with reference to the drawings, in which:

[0013] FIG. **1** illustrates a front prospective view of one embodiment of an upright vacuum cleaner;

[0014] FIG. **2** illustrates the rear view of one embodiment of an upright vacuum cleaner;

[0015] FIG. **3** illustrates the bottom of the base of an upright vacuum cleaner according to one embodiment;

[0016] FIG. **4** illustrates the bag assembly of a debris capturing device of an upright vacuum cleaner according to one embodiment;

[0017] FIG. **5** illustrates the interior of the base of an upright vacuum cleaner according to one embodiment;

[0018] FIG. **6** illustrates an automated diverter valve assembly of an upright vacuum cleaner according to one embodiment;

[0019] FIGS. **7**A and **7**B illustrate an automated diverter valve and motor assembly of an upright vacuum cleaner according to one embodiment;

[0020] FIGS. **8**A and **8**B illustrate one embodiment of a scroll of an upright vacuum cleaner according to one embodiment;

[0021] FIG. **9** illustrates a lifting assembly of an upright vacuum cleaner according to one embodiment;

[0022] FIG. **10** illustrates an exploded view of a yoke assembly of an upright vacuum cleaner according to one embodiment;

[0023] FIG. **11** illustrates an exploded view of a motor assembly of an upright vacuum cleaner according to one embodiment;

[0024] FIG. **12** illustrates an exploded view of an upright vacuum cleaner according to one embodiment;

[0025] FIG. **13**A illustrates sound data generated by a prior art cooling fan blade;

[0026] FIG. **13**B illustrates sound data generated by a cooling fan according to one embodiment;

[0027] FIG. **14** illustrates a graph of the amperage draw of a motor in a window of a selected duration according to one embodiment;

[0028] FIG. **15** illustrates a flow diagram indicating control mechanisms to shut down a motor according to one embodiment; and

[0029] FIG. **16** illustrates a logical view of a system to control and manage a vacuum cleaner according to one embodiment.

DETAILED DESCRIPTION

[0030] The present teachings provide an upright vacuum cleaner including improved cleaning features. The essential structure of the vacuum comprises a handle, body, base, automated diverter valve and air duct including two input ports. An automated diverter valve assembly at the junction of the dirty air intake within the base extends the air duct within the base and connects to the main air duct of the vacuum to the beater bar input and an attachment input. The automated diverter valve causes the air intake of the vacuum to be drawn from either the beater bar (floor) air input or the attachment input. The main air duct is in air flow communication with a vacuum motor located in the body of the vacuum spaced from a distal end of the air duct with respect to the flow of air.

[0031] In some embodiments the vacuum cleaner comprises a servo assembly for moving the automated diverter from the beater bar input port to the attachment input port. In some embodiments the vacuum cleaner comprises a control board to operate the servo assembly in a desired rotational movement between the two input ports for a duration. In some embodiments the vacuum cleaner further comprises a signal from a user actuated switch, wherein the signal can be used by the control board to determine the valve position between the first input port and the second input port. In some embodiments the user actuated switch comprises a magnetic sensor disposed fixedly in the vacuum, and a magnet disposed in a rotatable portion of the vacuum, wherein placing the handle in a locked position rotates the rotatable portion, and disposes the magnet opposite the magnetic sensor. In some embodiments the diverter valve assembly comprises a vacuum attitude sensor, wherein a detection signal from the vacuum attitude sensor determines the valve position between the first input port and the second input port. In some embodiments the vacuum cleaner further comprises an attachment sensor signal to denote the absence of an attachment connected to the first input port, and the signal directs the control board to direct airflow from the second input port to the output port.

[0032] In some embodiments the servo assembly comprises a servo motor and a gear assembly, wherein the servo assembly is able to position the diverter as desired in two seconds or less. In some embodiments the diverter valve assembly includes detents to stop a movement of the automated diverter. In some embodiments, the rotatable scroll can be part of an upright vacuum cleaner in which the vacuum motor is located in the air path that contains dirt from a cleaning surface (sometimes referred to as a "dirty-air" type vacuum).

[0033] The result is an upright vacuum with significantly greater cleaning capability and ease of use. Since the diverter valve rotates between the beater bar input port and the attachment port automatically, an operator generally need not work as hard to utilize either the attachment or floor features of the vacuum. The diverter valve essentially seals the airflow path to direct air from only one input, thereby increasing the suction to any one input without suction loss from the other input port. Further, the vacuum cleaner need not shut the motor down when switching between beater bar and hand held use.

[0034] FIG. **1** is a perspective view of an exemplary embodiment of an upright vacuum cleaner **100**. A handle **120** can be connected to base **102** via yoke **150** (see FIG. **9**). Handle **120** can comprise aluminum. Wheels **104** can be disposed on yoke **150**. Ergonomic aluminum handle **120** can include control buttons, such as power button **126**, high speed setting button **128** and low speed setting button **129** for easy user controls of the vacuum cleaner. Bag assembly **144** can be connected to aluminum handle **120** via bag slide **130** (see FIG. **2**). Base **102** can include a fascia **116**. Further, fascia **116**, scroll top cover **112**, and scroll bottom cover **114** (see FIG. **2**) can be made of different designs, textures and patterns in order to appeal to a user's preference or to individualize vacuum cleaners. Fascia **116** can be secured to the base **102** using means known in the art, for example, tabs (not shown) and slots (not shown) to receive the tabs. In some embodiments, scroll top cover **112** and scroll bottom cover **114** can comprise a fascia. Base **102** can further comprise side brushes **106**, a bumper **108**, and a light emitting diode (LED) strip **110** for improved cleaning capabilities of the upright vacuum cleaner unit. Vacuum **100** can include a power cord **118** and an extendible crevice tool **132**.

[0035] FIG. **2** is a rear view of an exemplary embodiment of an upright vacuum cleaner **100**. Power cord **118** can be connected to handle **120** and stored by top cord hook **122** and bottom cord hook **124** for easy storage and management. Base **102** can further comprise intake vent **160** for proper and adequate ventilation of any interior air flow propulsion devices. In one aspect of this embodiment, an exhaust vent **162** can be positioned adjacent the rear wheels **104**. Accordingly, airflow drawn in from the intake vent **160** can be expelled from exhaust vent **162** and diffused over the surfaces of the rear wheel **104** as it leaves base **102**. The diffusion can reduce the velocity of the airflow and reduce the likelihood that the airflow will stir up particulates on the floor surface. Base **102** can further comprise attachment hose input **136** for a hand held attachment. For example, one embodiment of a hand held attachment includes a flexible hose **134**, a rigid hose **139** and an extendible crevice tool **132**. In some embodiments, hand held attachments can include, but are not limited to brushes, squeegees, beater bars, extension hoses, nozzles, etc. In one embodiment, the upright vacuum cleaner comprises a tool caddy **138** for easy and convenient storage of a hand held attachment, for example, extendible crevice tool **132**. A tool holder **135** can be disposed on bag assembly **144**. Tool holder **135** can friction fit around extendible crevice tool **132** for easy storage and management of flexible hose **134**, rigid hose **139** and extendible crevice tool **132**. Extendible crevice tool **132** can be removed from tool holder **135** for use.

[0036] FIG. **3** is a bottom view of an exemplary embodiment of an upright vacuum cleaner **100**. Base **102** is supported by wheels **104** and front wheel **178**. Base **102** generally hovers over a cleaning surface, such as a floor. Base **102** can contact a cleaning surface, for example, when the cleaning surface is a deep shag carpet. Agitation devices, such as a beater bar **170**, squeegees **126**, and side brushes **106** can provide agitation of cleaning surfaces in order to dislodge and direct debris into floor air intake port **206** (not shown). Beater bar **170** can be driven by a motor assembly **240** (see FIG. **5**) via a flexible belt **186** (see FIG. **5**) or other mechanism. Anti-ingestion bars **182** prevent large sized items from being drawn into the floor air intake. Beater bar **170** can include a spindle **175** and an arrangement of bristle tufts **171** that sweep the particulates into the air intake port **206** (see FIG. **3**). As seen in FIG. **5**, a belt receiver **175*a*** can be disposed on spindle **175**. Belt receiver **175*a*** can include grooves to receive corresponding grooves disposed in belt **186**. Bristle tufts **171** can be arranged on beater bar in many different orientations. The fibers of the bristles can be of substantially identical stiffness, diameter and geometry or of different stiffnesses, diameters and geometries as desired. The fibers of the bristles can be made of natural or synthetic materials, or combinations thereof, including but not limited to nylon, plastic, polymers, rubber, hair (e.g., boar's hair). In one embodiment, bristles can be arranged in a double helix pattern.

[0037] In a preferred embodiment, the bristle tufts can be arranged in a single helix or helical row. The single helical row can reverse its direction of rotation, e.g., at bristle tuft **173** in FIG. **3**. The single helical row can reverse its direction of rotation after about one and a half turns about spindle **175**. The average length of the fibers of the bristle tufts can be from about 0.300 inches to about 0.500 inches. The average diameter of the fibers of the bristle tufts can be from about 0.008 inches to about 0.015 inches. Additionally, the bristle tufts can be angled out or placed non-orthogonally from the spindle to maximize the "embedded dirt" movement characteristics of the vacuum. The bristle tufts can be offset from the centerline about 0.08 inches to about 0.15 inches. In a preferred embodiment, the bristle tufts can comprise filaments comprising Nylon 6-6. The mean diameter of each filament can be about 0.012 inches. The mean amplitude of each filament can be about 0.022 inches. The mean tuft length of each filament can be about 0.370 inches. The tuft offset from centerline can be about 0.120 inches. In some embodiments, a single helix brush can be advantageously used in high shag carpets as its rotational speed is not inhibited to the same degree as the rotational speed of double helix brushroll. In embedded dirt cleaning performance tests, a single helix brushroll as described above can remove about 15% more dirt than the prior art double helix brushroll.

[0038] FIG. **4** is a bag assembly **140** of an exemplary embodiment. A debris collection device **146** is disposed in outer bag **144**. Debris collection device **146** can be connected to dirty air inlet **146** to collect and trap and filter debris taken into the vacuum. In one embodiment, debris collection device **146** can be a disposable bag. In another embodiment debris collection device **146** can be a reusable bag. In another embodiment debris collection device can be a reusable canister or container. Bag assembly **140** can optionally further include a variety of filters for cleaning dirty air. Such filters can include one or more wire, mesh, carbon, activated charcoal, or HEPA filters.

[0039] FIG. **5** is an interior view of an exemplary embodiment of base **102**. Beater bar housing **184** can be connected to the dirty air path via a diverter valve assembly **190** at input port **206**. Automated diverter valve assembly can also contain a second input port **204**. A connector **135** can connect to input port **204**. A hose and attachments can be connected to connector **135**. Airflow can be directed from either input port **206** or input port **204** to output port **208**. Servo assembly **192** can rotationally direct an automated diverter or diverter valve **212** (see FIGS. **7**A and **7**B) into a scroll/volute **218** (only a small portion is visible in FIG. **5**). Airflow can be generated by motor assembly **240** which draws air in from either input port **206** or input port **204** and out through rotatable scroll **218** into bag assembly **144** where debris can be contained. An impeller **226** (see FIG. **8**A) is driven by the motor shaft and is housed in scroll **218**. Motor assembly **240** can drive beater bar **170** via a flexible belt **186**. In some embodiments, flexible belts of the instant invention can exceed the mean time between failure (MTBF) of the vacuum cleaner itself. Thus, flexible belts may never have to be replaced during the lifetime of the vacuum. In some embodiments, the belts are circular belts or serpentine belts. In some embodiments the belt can include a flat or length-wise grooved surface. If the belt includes a grooved surface, the surface can include 1, 2, 3, 4, 5 or more grooves. The belts can be made of materials known in the art, including, but not limited to rubber, nylon, plastics, and polymers such as polybutadiene, and polyamide, among others. In one embodiment, the belt can be provided by Hutchinson FTS of Troy Mich. Motor assembly **240** can comprise an end cap **246** that houses fan **250** (not shown) and motor **248**.

[0040] Circuit board **260** of FIG. **5** can provide electrical current to motor assembly **240**, an LED light assembly **110**, servo assembly **192**, and an attachment sensor **137**. Attachment sensor **137** can comprise a contact switch which is depressed when connector **135** is disposed about input port **204**. A signal from attachment sensor **137** can be used by circuit board **260** prior to positioning diverter valve assembly **190** to select input port **204**. In other words, if connector **135** is not in place, a user cannot inadvertently be injured by the suction created at input port **204**. Circuit board **260** can also provide electrical current to various other components of the vacuum cleaner, such as motorized beater bars, motorized handheld attachments, temperature sensors, attitude sensors, magnetic sensors, indicator lights, etc.

[0041] FIG. **6** is an interior view of an exemplary embodiment of diverter valve assembly **190**. Diverter valve assembly **190** can be assembled with assembly housing top **106** and assembly housing bottom **108**. When assembly housing top **106** and assembly housing bottom **108** are attached, the assembly can define input port **204**, input port **206** opposite input port **204**, and output port **208**. Servo assembly **192** can be disposed opposite output port **208**. A diverter valve **212** can be fixedly attached to servo assembly **192**. Airflow can be directed from either input port **206** or input port **204** by servo assembly **192** by rotating automated diverter valve **212** to block either input port **204** or input port **206**. Diverter valve assembly can comprise a cylindrical conduit **205** having a radius X that is slightly greater than a radius Y of automated diverter valve **212**. Automated diverter valve **212** can comprise a cylindrical portion.

[0042] In some embodiments automated diverter valve **192** includes detents to stop its movement. For example, diverter valve **212** can include diverter valve detents **198** and **202**, where a wall of diverter valve **212** forms a ridge. A wall **211**

of diverter valve **212** can be placed adjacent to a wall **217** of the diverter valve assembly against which servo assembly **192** is secured; this wall can a include bump-out **219** (see FIG. **6**) to stop the travel of diverter valve **212** against detents **198** and **202**. As such, detents **198** and **202** define a range of motion for diverter valve **212**.

[0043] In some embodiments, diverter valve **212** includes a low friction film **215** and a protective valve sheathing **213** deposed underneath. Protective valve sheathing **213** aids in sealing the diverter valve **212** over input port **206** or **204** as selected. Low friction film **215** allows diverter valve **212** to easily rotate between input port **206** and **204**. Protective valve sheathing **213** can be manufactured from, without limitation to, plastic, foam, felt, plastic or other suitable materials, or combinations therein. Low friction film **215** can be smooth film.

[0044] As seen in FIGS. **7**A and **7**B servo assembly **192** can drive diverter valve **212** through servo motor shaft **194** which can be fastened to diverter valve shaft aperture **214** by fastener **195**. The servo motor shaft **194** can be keyed to provide precision of movement. Servo assembly **192** can comprise a servo motor (not shown) and a gear assembly (not shown) that can rotate diverter valve into position using a desired speed and torque. Such speeds can include whole or fractions of a second. For example, the motor can be designed such that the diverter valve can be rotated from one input port to the other within or less than one-half, one, two, three, five or more seconds. Diverter valve **212** can comprise a shaft aperture **214** through which a fastener, for example, a screw, can be secured to a servo shaft aperture **197**.

[0045] FIG. **8**A is an illustration of an exemplary embodiment of a scroll **218**. Airflow for the upright vacuum can be generated via impeller **226**. Impeller **226** can be driven by motor assembly **240**. Impeller **226** draws air in from automated diverter valve assembly **190** via air intake **220**. The drawn air is sent via an air conduit **234** into air output **222**. Air output **222** can be connected via conduit **219** (see FIG. **12**) to bag assembly **144** where debris can be contained for discard. Conduit **219** can be removable to allow a user to remove air flow obstructions from conduit **219** and/or scroll **218**. Scroll **218** and air conduit **234** can include a cross-sectional area progression from dirty air intake **220** to the air output **222** that smoothly varies between the first cross-sectional area and the second cross-sectional area. Because the intake passage includes a smoothly varying area progression, turbulence within the intake passage may be reduced or inhibited, and noise generated by the airstream within the intake can be minimized. Scroll **218** can also comprise ramp **235**.

[0046] In some embodiments, scroll **218** comprises a magnet **224**. A magnetic sensor **210** (see FIG. **5**) can be disposed fixedly in vacuum base **102**. Magnet **224** is disposed opposite magnetic sensor **210** when scroll **218** is rotated to a predetermined position, for example, when handle **120** is placed in a locked position. In some embodiments magnetic sensor **210** can be located adjacent, e.g., below, diverter valve assembly **190**. Magnetic sensor can determine an attitude of vacuum base **102**, e.g., is the vacuum at rest, is the vacuum handle locked, or is the vacuum handle unlocked. Further, in some embodiments a signal generated from the magnetic sensor **210** can determine diverter valve **212** position between first input port **204** and second input port **206**. In one embodiment, magnetic sensor **210** is disposed beneath output port **208**. Magnetic sensor **210** is fixed to vacuum base **102**.

[0047] FIG. **8**B is an illustration of an exemplary embodiment of a scroll. Scroll **218** includes scroll ring receiving groove **228** to receive scroll ring **230**. When scroll ring **230** is disposed within scroll ring receiving groove **228**, scroll ring tab **232** clicks into place and locks scroll **218** into a locked upright position. Scroll **218** is locked in position by forming a friction fit of scroll ring tab **232** against an inner wall of scroll ring receiving groove **228** disposed in scroll **218**. When scroll **218** is locked, rotation of handle **218** about yoke axle **151** (see FIG. **10**) is also inhibited. In some embodiments, scroll ring **230** allows for a rotation of about 90 degrees to 120 degrees for scroll **218**. This translates into a similar rotation of about 90 degrees to 120 degrees about yoke axle **151** for handle **120**.

[0048] Scroll ring **230** is disposed about motor housing cap **246**. Key tabs **231***a*, **231***b*, and **231***c* are received by motor housing cap **246** to properly orient scroll ring **230** and scroll ring tab **232**. Motor assembly **240** is fixedly disposed in base **102**. As such, scroll ring **230** is fixedly disposed in base **102**, i.e., scroll ring **230** does not rotate. However, scroll **218** rotates about scroll ring **232** so that handle **120** can rotate. Rotation of scroll **218** causes bag slide (see FIG. **2**) to move up and down on handle **120** as needed.

[0049] FIG. **9** is an exemplary embodiment of a lifting mechanism. In some embodiments, when handle **120** is placed in a locked upright position, scroll **218** is rotated such that ramp **235** (see FIG. **8**A) contacts lift tabs **179** of lifting assembly **172**. When ramp **235** pushes against lift tabs **179**, lifting assembly **172** including front wheel **178** protrude out from base **102**. This causes base **102** to be raised off of a cleaning surface. In the absence of ramp **235** pushing on lift tab **177**, a biasing device **177**, e.g., a spring, keeps lifting assembly **172** pulled into base **102**. By pushing lifting base **102** against a cleaning surface the vacuum ceases to agitate the cleaning surface. This can prevent unnecessary dust and debris from being generated by the rotation of the beater bar **170**, side brushes **106** or squeegee **176**. Moreover, by raising the beater bar a load on the motor is reduced. This can reduce the wear and tear on the motor, the belt and the beater bar.

[0050] FIG. **10** is an exemplary embodiment of a yoke assembly. As seen in FIGS. **1** and **2**, yoke **150** and handle **120** are distinct from scroll **218** and bag assembly **144**. In one embodiment, yoke assembly **150** can be connected to handle **120**. In some embodiments, handle insert **158** is inserted into hollow handle **120**. Handle **120** can be secured to yoke **150** via fasteners (not shown). The fasteners can pass through fastener apertures **155** and be fastened to fastener receiving apertures **156**. Fasteners can include screws, tension clips, etc. Yoke assembly **150** can be divided by handle insert **152**. Handle insert **152** can includes two internal housings within yoke assembly for passing a power cord **118** therethrough. Advantageously, providing a distinct compartment and path for power cord **118** within yoke assembly **150** protects power cord from damage from with fasteners or handle **120**. Yoke assembly axles **151** and washers **157** can connect yoke **150** to wheels **104**. Advantageously, because yoke assembly **150** and handle **120** are distinct from base **102** and scroll **218**, yoke assembly **150** can provide a moment arm **157** anterior to base **102**. Moment arm **157** can be co-linear with yoke axle **151**. In some embodiments, yoke axle **151** can comprise a single rod secured to yoke **150**. In some embodiments, yoke axle **151** can comprise two rods secured to yoke **150**. Yoke axle **151** can be secured to yoke **150** via C-rings **153**. It is theorized that with an anterior moment arm, a force applied to handle **120**

causes yoke assembly **150** to be pushed towards a cleaning surface rather than pushing base **102** towards the cleaning surface. As such, any downward component of the force applied to handle **120** does not push base **102** down also. This reduces a frictional force of base **102** against the cleaning surface. The resulting reduction in friction provides a much easier vacuum to push and control for a user over a cleaning surface, and provides a "floating head."

[0051] FIG. **11** is an exemplary embodiment of a motor assembly. Motor assembly **240** can provide air flow for a vacuum cleaner. In one embodiment a shaft of motor assembly **240** can protrude from both ends of motor assembly **240**. Shaft portion **244** can rotate a fan (see FIG. **8**A), such as an impeller, housed within scroll **218** to generate air flow. Shaft portion **242** can turn drive belt **186** and rotate beater bar **170**. The outer surfaces of shaft portions **242** or **244** can be smooth, flat, textured, keyed or may include one, two, three or more grooves **242***a* as desired. Motor assembly cap **246**, located on the distal end of motor assembly **240**, can provide protection for fan **250**, while further defining an air inlet **245** and an air outlet **256**. The motor assembly cap **246** can propel air over motor assembly **240** disposed within base **102**. Advantageously, air flow generated by fan **250** exiting air outlet **256** can cool heat generated by motor assembly **240**, thereby allowing a vacuum to utilize a larger motor than found in prior art vacuums.

[0052] Base **102** can be an airtight chamber. As seen in FIG. **12**, base **102** can be assembled from base top **164** and base bottom **165**, which are held together by fasteners **166**. Base **102** can be sealed by gasket **167** situated between base top **164** and base bottom **165**. Gasket **167** can be made from any suitable material, including but not limited to paper, rubber, silicone, metal, cork, felt, neoprene, nitrile rubber, fiberglass, or a plastic polymer (such as polychlorotrifluoroethylene) or any combination thereof. Motor assembly **240** can draw air to cool the operating parts of the vacuum via air vent **160**. The drawn air can be exhausted via air vent **162**. Air vent **160** and air vent **162** can define an air path through base **102**. The air path can be a straight or convoluted path. The high volume of airflow produced by fan **250** allows the placement of a high powered motor in base **102**. The high CFM also permits cooling of components in the base even when no particular airflow path is defined within the base. For example, airflow generated by fan **250** can be circulated throughout base **102** by placing air intake vent **160** along the same wall as air vent **162**. Other configurations for disposing the air intake and air exhaust in the base can be used.

[0053] Centrifugal fan **250** can include multiple fan blades and a hub. Centrifugal fan blades can have a slight backward curve. Alternatively, the fan can be axial or squirrel cage fans, or other material handling fans. In some embodiments, fan **250** can be made of one or more of a combination of materials, including metals, such as aluminum or plastic. In some embodiments fan **250** can be a centrifugal fan with a slight backward curve including 30 blades made by injection molding. In some embodiments, fan **250** can generate a blade pass frequency (BPF) that is greater than the BPF of prior art fans. The fan BPF noise level intensity varies with the number of blades and the rotation speed and can be expressed as BPF=n*t/60, where BPF=Blade Pass Frequency (Hertz (Hz)), n=rotation velocity (rpm), and t=number of blades. In noise profiles of a fan, high-amplitude spikes are observed at the BPF and at the harmonics of the BPF. Humans perceive sound frequencies ranging from 20 to 15,000 Hz. Moreover, sounds between 2,000 to 4,000 Hz are often perceived as very irritating and annoying to humans.

[0054] Prior art fans for motors used in vacuums generally use a stamped radial fan blade, a fan with blades extending out from the center along radii, usually comprising 2-12 blades. For example, in the prior art a vacuum motor having a 12-blade fan and operating at about 20,000 RPM would have a calculated BPF of about 4000 Hz. As can be seen in FIG. **13**A, the noise data profile for this prior art cooling fan produced decibel spikes over 50 dB/20 u Pa at approximately 4,000 Hz. At 50 dB/20 u Pa, the prior art fan's noise profile spike is about 20 dB greater than the noise observed immediately around the 4000 Hz spike frequency. The spike at about 4000 Hz is within the annoying and irritating noise range for humans. Furthermore, harmonic frequencies of the BPF within a human's average hearing range, e.g., 8000 and 12000 Hz, also produce large noise peaks.

[0055] By using a fan with a greater number of blades, the BPF can be manipulated to fall outside a desired sound frequency band. For example, the fan can comprise 20, 22, 24, 26, 28, 30, 32, 34, 36, 38, 40 or more blades. A further advantage is that the unique design of motor assembly **240** and blade **250** includes a bigger blade surface area. Furthermore, this increase in blade area coupled with the greater number of blades in the fan can generate a greater airflow. The greater airflow can by generated by a motor assembly cap having the same or less volume than a motor assembly cap housing of prior art. By manipulating the number of blades and the RPMs of the fan, the BPF can be adjusted to spike at a frequency greater than about 5000, 5500, 6000, 6500, 7000, 7500, 8000, 8500, 9000, 9500, 10,000 or more Hz. A change in the blade pass frequency of the fan provides a reduction in perceived motor and fan noise. In some embodiments, the noise spikes generated by the fan is selected such that a BPF spike is outside a human ear's irritation noise range. Further in some embodiments, a BPF spike is generated outside of a human ear's audible noise range. In some embodiments motor assembly **240** can operate at about 10,000 to about 20,000 rotations per minute (RPM). In some embodiments assembly **240** can operate at about 10,000 or about 20,000 RPM. In some embodiments assembly **240** can operate at about 13,000 or about 18,000 RPM.

[0056] As seen in FIG. **13**B, the BPF of fan **250** of the present vacuum is about 9000 Hz, when the fan is rotated at about 18000. Furthermore, a switch to centrifugal fans from the radial fans of the produce reduces the amplitude of the spike at the BPF. The spike at 9000 Hz is only about 4 dB/20 u Pa greater than the noise observed immediately around the 9000 Hz spike frequency. The use of the centrifugal also lowers the acoustic characteristic of noise at the BPF by an order of 5.

[0057] Vacuum cleaner **100** can be capable of detecting blockage along an airpath of vacuum **100** by determining the amperage flow of the electrical current, and detecting blockage along an airpath by sampling the amperage flow of the electrical current and counting how many times the sampled amperage draw exceeds a threshold amperage within a window of time. When the samples sampled exceeds the percent threshold determined, power to motor assembly **240** is terminated. Optionally, an indicator light can be illuminated when power is shut-off. After receiving a reset signal the current flow to the motor can be restored.

[0058] FIG. **14** illustrates a graph of the amperage draw of a motor in a window of a selected duration of an upright vacuum cleaner. Circuit board **260** can provide electrical current to motor assembly **240**. Measurements of current drawn by vacuum motor can determine whether there is blockage with the vacuum air duct or beater bar. Depending upon the severity of the blockage, circuit board **260** can shut off power to motor assembly **240**. For example, circuit board **260** can comprise an amperage flow sensor (not shown) to determine or measure the electrical current draw of motor assembly **240**. Circuit board **260** can also comprise a blockage determiner **262** to sample the electrical current draw with the amperage flow sensor and count the number of times the sampled electrical current draw exceeds a threshold amperage within a sliding window of time. As seen in FIG. **14**, the sliding window of time period or duration A illustrates that circuit board **260** counted three (3) instances or samples out of seven (7) instances where the current draw of the motor exceeded a threshold amperage (shown as the dashed line parallel to the horizontal axis). As such, during time period A about 43% (3/7*100) of samples exceeded the threshold amperage. In contrast, circuit board **260** counted only one (1) instance out of seven (7) for time period B where the current draw of the motor exceeded the threshold amperage. Windows A and B can overlap along the time (horizontal) axis. In some embodiments the blockage determiner can signal that upright vacuum cleaner **100** is experiencing blockage when the count exceeds a desired percentage of samples sampled in the window of time. In some embodiments, the desired percentage is at least 10, 20, 30, 40, 50 or more of the samples sampled in the window of time. In some embodiments, blockage determiner **262** samples the amperage draw 15, 30, 60, or 90 times a second or more. In some embodiments the sliding window of time **264** is greater than or equal to 5, 10, 15, 20, 30, 45, 60, 90, or 120 seconds.

[0059] Vacuum cleaner **100** and circuit board **260** can comprise multiple sensors and switches. In a broad sense, a "sensor" as used herein, is a device capable of receiving a signal or stimulus (electrical, temperature, time, etc.) and responds to it in a specific manner (opens or closes a circuit, etc.). A "switch," as used herein, can be a mechanical or electrical device for making or breaking or changing the connections in a circuit. In some embodiments sensors can be switches. In other embodiments the sensors are connected to indicator lights or the like to inform a user of a malfunction or the need to perform a necessary function. Vacuum cleaner **100** or circuit board **260** can comprise flow blockage, light, temperature, "bag full" sensors, and handle attitude sensors. Signals from these sensors can aid the user in using and assessing various states of the vacuum. Sensors can comprise electric, magnetic, optical, gravity, etc., sensors, as known in the art. Vacuum cleaner **100** or circuit board **260** can further comprise a "deadman" or "kill" switch which is capable of terminating power to the vacuum should the user become incapacitated. A temperature sensor **266** can determine the temperature of motor assembly **240**, base **102**, or other parts. Circuit board **260** can turn on an indicator light and/or terminate power to vacuum **100**. Further, vacuum cleaner **100** or circuit board **260** can include a reset switch which is capable of resetting power to vacuum cleaner **100** or circuit board **260**.

[0060] As shown in FIG. **15**, control mechanisms to shut down a vacuum motor are described. At step **280**, the window of time slides or moves forward. At step **282**, a samples of the amperage drawn by the motor is measured or determined. At step **284**, the control determines if the amperage flow exceeds a predetermined maximum or threshold amperage. At step **286**, the control counts the number of time the amperage samples exceeded the predetermined maximum amperage. The control determines if the number from step **286** exceeded the acceptable percentage within the single window of time at step **288**. If the percentage of samples that exceeded the threshold is acceptable, the control repeats the process and begins at step **280** again. If the percentage of samples that exceeded the threshold is not acceptable, then the control turns off the current to the motor and shuts down the motor at step **300**. The disablement of the motor can trigger the illumination of an indicator light at step **304**. The motor can be enabled by the user via manually activating a reset switch at step **302**.

[0061] In some embodiments, vacuum cleaner **100** includes a temperature sensor **266** that is capable of determining the operating temperature of motor assembly **240** at step **290**. When the operating temperature exceeds a predetermined high temperature threshold at step **292**, power to motor assembly **240** is shut off at step **300**. Optionally, an indicator light is illuminated at step **304** to notify the user of the temperature exceeded error condition. The current flow to the motor can be restored after receiving a reset signal at step **302**. In some embodiments, the reset can be automatic if the operating temperature comes down to be within a temperature operating range. In some embodiments, the threshold temperature can be greater than 150, 175, 200 degree Celsius.

[0062] FIG. **16** is a logical diagram of a system **400** to control and manage a vacuum cleaner. System **400** comprises a processor **402**, a volatile memory **404** to store operating variables and a non-volatile memory to store computer instructions and data necessary to operate system **400**. System **400** can include an attachment sensor **406** that can signal the presence or absence of an attachment, e.g., hose **135** (see FIG. **5**). Attachment sensor **406** can include sensor **137** seen in FIG. **5**. System **400** can include an attitude sensor **408** that can signal whether a vacuum handle is in a locked or unlocked position. Attachment sensor **406** can include sensor **210** seen in FIG. **5**. System **400** can include a temperature sensor **410** that can signal an operating temperature of a motor. In some embodiments, an amperage sensor **412** can determine or measure the current being drawn by the vacuum and/or its various components. In a preferred embodiment, the current being drawn by a motor is determined or measured by amperage sensor **412**. In some embodiments, system **400** includes one or more of a power supply **414**, an LED power supply **414**, an indicator light **418**, a variable speed control **420**, and a reset switch **422**. In some embodiments, system **400** includes instructions **424**. Instructions **424** can include a blockage determiner module **426** to implement the method of FIGS. **12** and **13**. In some embodiments, instructions **424** include a diverter valve control **430**. In some embodiments, a variable speed control **430** is included in system **400**.

[0063] The various embodiments described above are provided by way of illustration only and should not be constructed to limit the invention. Those skilled in the art will readily recognize the various modifications and changes which may be made to the present invention without strictly following the exemplary embodiments illustrated and described herein, and without departing from the true spirit and scope of the present invention, which is set forth in the following claims.

What is claimed is:

1. A method of reducing noise generated by a vacuum, the method comprising:

providing a vacuum comprising a motor including a motor shaft adapted to operate at a rotational speed of n;

attaching a centripetal fan with t number of blades attached to the motor shaft;

enclosing the fan between the motor and a motor housing end bracket disposed about the shaft, the motor housing end bracket defining an air input opposite an air output;

wherein the variables of speed n and the number of blades t are selected so that a blade pass frequency (BPF) of the centripetal fan is greater than about 5000 Hertz, and the BPF is calculated as BPF=n*t/60.

2. The method of claim **1**, wherein the speed n is greater than 10,000 RPM.

3. The method of claim **1**, wherein the total area of the centripetal fan blade area generates greater than about 20 cubic feet per minute of airflow between the air input and the air output.

4. The method of claim **1**, wherein the centripetal fan has 30 or more blades.

5. The method of claim **1**, wherein the BPF is greater than about 6000 Hertz.

6. The method of claim **1**, wherein the BPF is greater than about 9000 Hertz.

7. The method of claim **1**, wherein the BPF is outside a human ear's irritation noise range.

9. The method of claim **1**, wherein the motor selectively operates at about 13000 RPM and about 18000 RPM.

10. The method of claim **1**, wherein the blades of the centripetal fan have a slight backward curve relative to a rotational direction of the fan.

11. The method of claim **1**, further comprising cooling components disposed in the vacuum base with the airflow exiting the air output.

12. The method of claim **1**, further comprising disposing an air exhaust in a vacuum base in the same plane as an air intake.

13. The method of claim **11**, further comprising determining the temperature of the motor; and shutting off the motor when the temperature exceeds a predetermined temperature.

14. A vacuum comprising:

a motor including a motor shaft to operate at a rotational speed of n;

a motor housing end bracket disposed about the shaft, the motor housing end bracket defining an air input opposite an air output; and

a centripetal fan with t number of blades attached to the motor shaft and enclosed between the motor and the motor housing end bracket;

wherein the variables of speed n and the number of blades t are selected so that a blade pass frequency (BPF) of the centripetal fan is greater than about 5000 Hertz, and the BPF is calculated as BPF=n*t/60.

15. The vacuum of claim **14**, wherein the speed n is greater than 10,000 RPM.

16. The vacuum of claim **14**, wherein the total area of the centripetal fan blade area generates greater than about 20 cubic feet per minute of airflow between the air input and the air output.

17. The vacuum of claim **14**, wherein the centripetal fan has 30 or more blades.

18. The vacuum of claim **14**, wherein the BPF is greater than about 6000 Hertz.

19. The vacuum of claim **14**, wherein the BPF is greater than about 9000 Hertz.

20. The vacuum of claim **14**, wherein the motor selectively operates at about 13000 RPM and about 18000 RPM.

21. The vacuum of claim **14**, wherein the blades of the centripetal fan have a slight backward curve relative to a rotational direction of the fan.

22. The vacuum of claim **14**, further comprising:

an air exhaust disposed in a vacuum base; and

an air intake in the vacuum base in the same plane the an air exhaust.

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