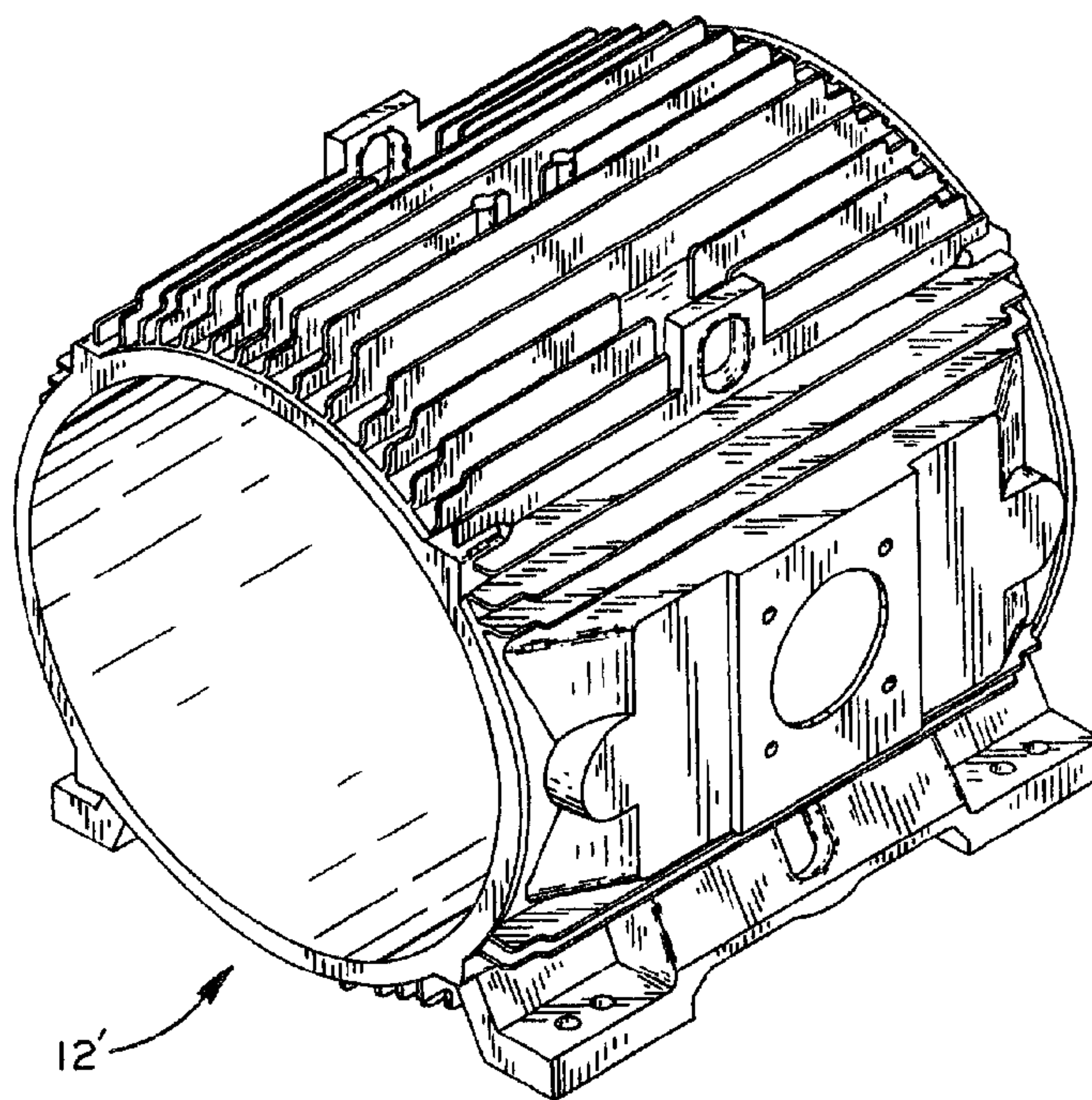




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(54) Titre : CARCASSE DE STATOR POUR MACHINE DYNAMO-ELECTRIQUE, ET METHODE DE FABRICATION DE LA CARCASSE
 (54) Title: STATOR FRAME FOR DYNAMOELECTRIC MACHINE AND METHOD FOR MAKING SAME



(57) **Abrégé/Abstract:**

A dynamoelectric machine having a stator housing or frame made by a lost foam process utilizing a unitary single-piece vaporizable pattern. The stator housing may be made from a molten metal such as cast iron or aluminum, and has an annular wall having integral outwardly extending longitudinal cooling fins, mounting foot pads and a conduit box support pad. The lost foam process enables the cooling fins to have a greater effective cooling fin height to housing diameter ratio and greater effective cooling fin height to housing annular wall thickness ratio than obtained with prior sand casting processes. The greater height fins provide increased strength and rigidity for the annular wall which can be made radially thinner than prior stator housing or frames of similar size. Substantial savings in material and labor are realized by the stator housing.

Abstract of the Disclosure

1 A dynamoelectric machine having a stator housing or
2 frame made by a lost foam process utilizing a unitary single-
3 piece vaporizable pattern. The stator housing may be made from
4 a molten metal such as cast iron or aluminum, and has an annular
5 wall having integral outwardly extending longitudinal cooling
6 fins, mounting foot pads and a conduit box support pad. The
7 lost foam process enables the cooling fins to have a greater
8 effective cooling fin height to housing diameter ratio and
9 greater effective cooling fin height to housing annular wall
10 thickness ratio than obtained with prior sand casting processes.
11 The greater height fins provide increased strength and rigidity
12 for the annular wall which can be made radially thinner than
13 prior stator housing or frames of similar size. Substantial
14 savings in material and labor are realized by the stator housing.

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**STATOR FRAME FOR DYNAMOELECTRIC MACHINE
AND METHOD FOR MAKING SAME**

Background of the Invention

1 The present invention relates generally to stator
2 housings or frames for dynamoelectric machines, and more
3 particularly to new and improved stator frames and methods of
4 making same.

5 It is conventional in the design of many types of
6 dynamoelectric machines, such as electric motors and generators,
7 to support a stator core within a stator housing or frame. The
8 stator housing or frame may have external support feet or pads
9 which facilitate mounting on a suitable support or apparatus such
10 as a machine tool in a predetermined orientation, and defines a
11 cavity to receive a stator core. Suitable bearings and support
12 structure are also provided to rotatably support a rotor in
13 coaxial relation within a bore in the stator core. In motors and
14 generators it is a common practice to provide generally
15 longitudinal fins on the external surface of the stator housing
16 or frame which enhance cooling by means of air passing over or
17 around the fins during operation.

18 In general, cast stator frames, housings or shells of
19 prior dynamoelectric machines have been made of cast iron by sand
20 casting processes. The sand casting process generally entails
21 burying a pattern with packed sand defining at least one parting
22 plane, removing the pattern to form a mold cavity, positioning
23 a separate core piece within the mold cavity to define the stator
24 core accommodating cavity of the stator frame, positioning
25 separate pieces that will define mounting feet or pads on the
26 casting and, if desired, one or more conduit box support pads to

1 finalize the mold cavity. Molten iron is then poured into the
2 mold cavity. After solidifying and cooling, the casting is
3 removed and cleaned, leaving a relatively rough surface casting.
4 This process is very time consuming, generally taking several
5 hours from start to finish.

6 A significant drawback in making stator housings by
7 such sand casting processes is that the sand-cast stator housing
8 must undergo substantial machining. For example, a common
9 technique for mounting an annular stator core within the stator
10 housing cavity is to cold-press the stator core into the bore.
11 In this procedure, the as-cast interior of the housing generally
12 requires significant machining to bring the dimensional
13 configuration thereof to a proper size and tolerance range. An
14 alternative procedure for mounting stator cores within stator
15 housings is by known heat shrink techniques. In this procedure,
16 the wall thickness of the stator frame must be relatively
17 precisely machined to have uniform thickness walls to insure
18 uniform and low stress shrinkage after heating the housing to
19 receive the stator core. Sand-cast stator housings thus
20 generally require substantial machining to prepare the as-cast
21 housing for assembly with a stator core by heat shrink methods.
22 Moreover, the heat shrink process generally takes two to three
23 hours to complete. Also, the opposite end surfaces on the stator
24 housing generally require significant machining to prepare them
25 for mating relation with rotor shaft bearing support frames,
26 commonly also referred to as end shields. Additional machining
27 may be necessary when a cooling fan cover or shroud is to be
28 supported on one end of the stator housing for directing fan-
29 driven air over the cooling fins.

30 Thus, conventional sand-cast stator frames are
31 associated with expensive and labor intensive machining
32 operations and generally result in significant material waste,
33 all of which adds to their cost of manufacture.

34 Another very significant drawback with sand-cast stator
35 housings or shells is that a sand mold imposes substantial
36 limitations on the stator housing design. For example, in larger
37 size motors and generators where heat transfer, i.e. cooling, is

1 a particularly important factor, sand casting characteristics
2 limit the relative height and thickness dimensions of the cooling
3 fins formed on the external surface of the stator housing. More
4 specifically, the rough sand surfaces defining the fin cavities
5 create a relatively high friction interface with the poured
6 molten metal, causing the molten metal to flow slowly at the
7 interface with the sand. If the fin height to fin thickness
8 ratio is relatively high, as desired to obtain optimum cooling,
9 the molten metal may solidify before it completely fills the fin
10 cavity, thereby resulting in an incomplete fin or a non-uniform
11 fin surface, either of which may result in a defective casting.
12 Further, casting material has a tendency to crack and break the
13 sand mold before the molten metal reaches the full depth of the
14 fin cavities. This phenomenon results in disadvantages that
15 practice of the present invention overcomes.

16 In an attempt to overcome the drawbacks associated with
17 sand cast stator housings, alternative techniques have included
18 consideration of making stator housings by lost foam casting
19 processes. This type of technique or process, which may also be
20 termed evaporative pattern or evaporative foam casting, generally
21 entails making one or more metallic tools or intermediate molds
22 which define a cavity substantially equal to the finished cast
23 product desired, or a portion of the finished product. The
24 intermediate mold cavity is filled with small polystyrene plastic
25 beads, and high temperature steam is injected into the plastic
26 beads to fuse them together. This creates a vaporizable
27 polymeric pattern which, after removal from the tool or mold, has
28 a configuration substantially identical to the corresponding
29 final product casting desired. The pattern is then given a thin
30 vapor-permeable film or coating.

31 The coated vaporizable pattern thus produced, together
32 with suitable sprue and gate pieces which may also be made of a
33 coated vaporizable polymeric material, are then buried in a sand
34 container which may be vibrated to pack the sand about the
35 pattern, sprue and gate pieces. As molten metal, such as grey
36 iron or aluminum, is poured into the pattern, the polystyrene
37 pattern vaporizes and is replaced by the molten metal. After

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1 solidification and cooling, the resulting casting is removed from
2 the sand. In general, the lost foam process results in a casting
3 having substantially improved dimensional accuracy, stability and
4 surface finish over products of sand casting processes.

5 One attempt at making grey iron stator frames or shells
6 by the lost foam casting process has been undertaken in the prior
7 art by at least one foundry. The vaporizable patterns used by
8 such foundry in making grey iron stator frames are made in two
9 separate pieces or sections which are secured together, as by a
10 suitable adhesive, to form a completed pattern. It is believed
11 that this approach, however, has drawbacks in that the parting
12 line or parting planes of the pattern (that is, the interfacing
13 adhered surfaces) prevent the pattern from having the degree of
14 dimensional accuracy that would be necessary to fully realize the
15 benefits otherwise achievable with lost foam casting. Another
16 drawback in the stator frames made by the known prior techniques
17 using a lost foam process is that the external longitudinal
18 cooling fins on the stator frames appear to be of equal or
19 reduced height in comparison to the fin height of comparable size
20 stator frames made by conventional sand casting techniques.
21 Also, it is believed that the number of external cooling fins for
22 a given diameter stator frame has been reduced on prior lost foam
23 stator frames as compared to comparably sized sand cast stator
24 frames, thereby resulting in increased circumferential spacing
25 between fins on the lost foam stator frame.

26 The prior techniques and drawbacks described above
27 appear to represent conventional thinking of persons skilled in
28 the art. For example, when tool makers were approached by
29 applicant to make tools or molds for making vaporizable patterns
30 for lost foam stator housing castings having external cooling
31 fins, the conventional approach seemed to represent a bias toward
32 making the final product configuration such that it would be
33 easier and less costly to produce the vaporizable patterns from
34 the tools or molds. More particularly, it is apparently believed
35 that reducing the height and number of cooling fins on a motor
36 stator housing makes it much easier and less expensive to make
37 the metallic tools or molds, and to remove the vaporizable

1 patterns from the tools. Thus, efforts to increase the number
2 of fins (and length or height thereof) would appear to be
3 contrary to conventional wisdom in the art, although fewer and
4 shorter height fins would certainly seem to make it easier to
5 remove a pattern from the tool. It thus seems that prior
6 decisions relating to making castings to be used for motor
7 housings have been driven by casting process artisans. However,
8 it is believed that it would be more desirable to identify
9 desirable criteria for the functional performance of a finished
10 cast motor housing, and then refine or improve casting techniques
11 to provide finished housings having such characteristics. Such
12 characteristics are related to, among other things, the overall
13 strength and rigidity of the fins and stator housing; the total
14 amount of material needed to maintain a desired degree of
15 structural integrity; heat transfer qualities of a finished
16 housing; the amount of post casting machining that will be
17 required; and the type of process to be used for assembling the
18 stator and housing.

19 Practice of the present invention clearly contemplates
20 increasing fin height to add strength and rigidity to the stator
21 housing. The latter is particularly desirable to allow an
22 increase in the rigidity of the generally annular wall which
23 immediately surrounds the stator core, and enables the use of a
24 thinner annular wall. The thickness of this annular wall
25 significantly affects the heat transfer and cooling
26 characteristics of the stator frame, and thus improved heat
27 transfer characteristics as well as structural characteristics
28 and material utilization are achieved.

29 Summary of the Invention

30 A general object of the present invention is to provide
31 a new and improved dynamoelectric machine having a novel stator
32 housing and a method for making the stator housing.

33 A more particular object of the present invention is
34 to provide a novel stator housing made by a lost foam casting
35 process wherein the stator housing has improved strength and

1 structural integrity, and thermal characteristics, over prior
2 sand cast and lost foam cast stator housings.

3 Another object of the present invention is to provide
4 a novel stator housing made in accordance with a lost foam
5 process wherein the stator housing has longitudinal cooling fins
6 having increased ratio of effective cooling fin height to housing
7 diameter; and an increased ratio of fin height to annular housing
8 wall thickness when compared to prior stator housings made in
9 accordance with either sand casting or prior lost foam processes;
10 and preferably without reducing the spacing between cooling fins.

11 A further object of the present invention is to provide
12 a novel stator housing for use with dynamoelectric machines and
13 a method of making the stator housing by a lost foam process,
14 wherein the stator housing requires substantially reduced
15 machining and thereby has both reduced material waste and reduced
16 machining costs as compared to sand cast stator housings, and is
17 characterized by improved strength and structural integrity, and
18 heat transfer characteristics, while using less material than
19 required for prior sand cast stator housings of comparable motor
20 horsepower ratings.

21 A still further object of the present invention is to
22 provide a new and improved motor having a stator housing made by
23 a lost foam process, and wherein the housing has improved
24 structural and thermal characteristics so that desirable transfer
25 of heat from the motor is dissipated readily from the housing,
26 and so that less material is used in the overall manufacture of
27 such motor.

28 One feature of stator housings made in accordance with
29 the present invention lies in their ability to be manufactured
30 by a lost foam process which, in addition to providing economic
31 savings through reduced machining and material waste, is
32 substantially less labor intensive, thereby resulting in further
33 cost savings.

34 Another feature of stator housings made in accordance
35 with the present invention lies in the ability to manufacture the
36 stator housings by a lost foam process which provides significant
37 economic advantages and improved heat transfer characteristics

1 by enabling a greater ratio of effective cooling fin height to
2 diameter than heretofore obtainable, while also increasing the
3 strength and structural integrity of the stator housings.

4 Another feature of a preferred form of stator housing
5 and a preferred lost foam method of making the housing in
6 accordance with the present invention lies in the ability to make
7 the stator housing from iron or aluminum with the resulting
8 stator frame being substantially similar in appearance when
9 either of the two materials is used.

10 Further objects, features and advantages of the present
11 invention, together with the organization and manner of operation
12 thereof, will become apparent from the following detailed
13 description of the invention when taken in conjunction with the
14 accompanying drawings wherein like reference numerals designate
15 like elements throughout the several views.

16 Brief Description of the Drawings

17 FIG. 1 is a perspective view of a stator housing
18 constructed in accordance with the present invention for use with
19 dynamoelectric machines;

20 FIG. 2 is a perspective view of the stator housing of
21 FIG. 1 but viewed from the opposite side from FIG. 1;

22 FIG. 3 is a perspective view of a vaporizable pattern
23 as employed in the lost foam casting process for making the
24 stator housing illustrated in FIGS. 1 and 2;

25 FIG. 4 is side elevational view of the stator housing
26 of FIG. 1 but with portions broken away for purposes of clarity;

27 FIG. 5 is a partial end elevational view and partial
28 transverse sectional view taken substantially along line 5-5 of
29 FIG. 4;

30 FIG. 6 is a fragmentary transverse sectional view taken
31 substantially along line 6-6 of FIG. 4, but having portions of
32 a rotor and stator core mounted within the stator housing cavity;

33 FIG. 7 is a bottom view of a mounting foot pad taken
34 generally along line 7-7 of FIG. 4; and

35 FIG. 8 is a side elevational view of a dynamoelectric
36 machine employing a stator housing in accordance with the present

1 invention, shown in phantom, and having other parts of a
2 completed motor, including end frames and a rotor shaft.

3 Detailed Description

4 Referring now to the drawings, a dynamoelectric machine
5 constructed in accordance with the present invention is
6 illustrated in outline form and indicated generally at 10 in FIG.
7 8. As used herein, the term dynamoelectric machine is meant to
8 cover any electromechanical apparatus or machine which employs
9 a stator housing, such as electric motors and generators. The
10 dynamoelectric machine 10 includes a stator housing or frame 12
11 which, as illustrated in FIGS. 1, 2 and 4-6, has a substantially
12 annular wall 14 defining an axial cylindrical bore or cavity 16
13 of generally uniform diameter adapted to receive a stator core
14 assembly 18, as shown in FIG. 6. The axial bore 16 intersects
15 opposite ends 12a and 12b of the housing 12 which also define
16 annular end surfaces on the wall 14 and lie in planes
17 substantially transverse to the longitudinal axis of bore 16.

18 As will be described, the stator housing or frame 12
19 is made by a lost foam casting process using an evaporative
20 pattern which has a configuration substantially identical to the
21 stator housing so that minimal machining is required to prepare
22 the stator housing to receive a stator core of the stator
23 assembly 18, and end frames or end shields with associated rotor
24 shaft bearings, as indicated at 20a and 20b in FIG 8. The lost
25 foam casting process enables the stator housing 12 to be cast
26 from molten iron or aluminum with close dimensional tolerances
27 required for assembly with a stator core and end frames or the
28 like without substantial machining, thus leading to significantly
29 reduced material waste with attendant cost savings in both
30 material and labor.

31 The annular wall 14 of the stator housing 12 is of
32 generally uniform radial thickness along a major portion of its
33 length. As illustrated in FIG. 4, the annular wall 14 is formed
34 so as to have increased radial thickness at its opposite ends in
35 the form of outwardly tapered or frustoconical external surfaces
36 14a and 14b. For a stator housing 12 having a nominal length of

1 approximately seventeen inches and having a nominal axial bore
2 diameter of approximately fifteen inches, the outwardly tapered
3 end surfaces 14a and 14b extend from the opposite end surfaces
4 12a and 12b, longitudinally along the length of the housing a
5 distance of approximately two inches at which point the tapered
6 surfaces 14a and 14b merge with a smaller diameter outer surface
7 14c formed along an annular wall section of the annular wall 14
8 intermediate the outwardly tapered end surfaces 14a and 14b. For
9 purposes of description, the wall 14 and its outer end surfaces
10 14a and 14b and intermediate length surface 14c are described as
11 annular even though, as will be described, the wall 14 has
12 external longitudinal cooling fins formed integral therewith.
13 With the axial bore 16 of the stator frame housing 12 having a
14 nominal diameter of approximately fifteen inches, the radial
15 thickness of the intermediate wall portion 14c of the annular
16 wall 14 may be approximately .365 inch. The opposite ends 14a
17 and 14b of the annular wall 14 taper outwardly such that the
18 radial thickness of wall 14 at the opposite end surfaces 12a and
19 12b is approximately .390 inch. This results in substantial
20 material savings in comparison to a similar size stator frame
21 made in accordance with prior sand casting techniques wherein the
22 annular wall of the cast stator frame or housing had a nominal
23 radial thickness of approximately .625 inch throughout its length
24 before machining the bore to accommodate a stator core.

25 The stator frame housing 12 has a plurality of external
26 longitudinally extending cooling fins 24 formed integral with the
27 annular wall 14. In the illustrated embodiment, and referring
28 particularly to FIG. 5, the cooling fins 24 are formed about the
29 annular wall 14 so that a plurality of substantially equally
30 spaced cooling fins extend outwardly from each of four generally
31 equal arcuate segments or quadrants of the annular wall. For
32 example, an upper quadrant of the annular wall 14 is defined as
33 the arcuate portion of wall 14 between a pair of mutually
34 perpendicular planes, indicated by phantom lines 26a and 26b,
35 which intersect at the center axis 20 of the longitudinal bore
36 16 and form included 45° angles with a vertical plane containing
37 the center axis 20. In similar fashion, a lower arcuate quadrant

1 of wall 14 is defined between the intersection of planes 26a and
2 26b with the lower portion of the annular wall 14. Similarly,
3 the planes 26a and 26b establish opposite side quadrants of the
4 annular wall 14 which extend between the upper and lower
5 quadrants.

6 As illustrated in FIG. 5, the cooling fins 24 formed
7 integral with the upper quadrant of the annular wall 14 extend
8 vertically upwardly. The cooling fins 24 formed integral with
9 the bottom or lower quadrant of annular wall 14 extend vertically
10 downwardly. The longitudinal cooling fins 24 formed integral
11 with the opposite side quadrants of annular wall 14 extend
12 outwardly in substantially horizontal planes.

13 Substantially all of the cooling fins 24 of stator
14 frame 10, except the cooling fins extending downwardly from the
15 center portion of the lower quadrant of annular wall 14, are of
16 generally equal height. For purposes of description, the term
17 "height" refers to the distance the respective cooling fins
18 extend outwardly from the outer surface of the intermediate
19 length portion 14c of the annular wall 14, where the outer
20 diameter of the intermediate length wall portion 14c is measured
21 at the base of generally equal radius fillets formed between and
22 at the base of adjacent pairs of cooling fins, such as indicated
23 at 32 in FIG. 5. With the cooling fins 24 having substantially
24 equal height, particularly along the upper and opposite side
25 quadrants of the annular wall 14, the outer longitudinal edges
26 of the equal height fins lie generally near to a circle
27 concentric with the bore axis 20. The cooling fins 24 on the
28 lower quadrant of the annular wall 14 are made somewhat shorter
29 adjacent the central portion of the lower quadrant so that the
30 outer longitudinal edges of the lower cooling fins are spaced
31 above a plane containing the bottom coplanar surfaces of mounting
32 foot pads cast integral with the wall 14, as will be described.

33 In accordance with one feature of dynamoelectric
34 machines and stator housings that embody aspects of the present
35 invention, the cooling fins 24 have an effective cooling fin
36 height to stator housing diameter ratio of at least 4.0, and
37 preferably approximately 4.8. The effective cooling fin height

1 to diameter ratio is defined as the ratio of the sum of the
2 heights of all of the cooling fins 24 over the outer diameter of
3 the annular wall 14 at the longitudinal center of the stator
4 frame. If desired, this ratio can be increased to approximately
5 6.0. This ratio is significantly higher than the effective
6 cooling fin height to diameter ratio obtained with prior sand
7 cast stator housings or frames of similar axial bore size. With
8 the cooling fins 24 in the described embodiment having heights
9 of approximately 1.69 inch and mean thickness of approximately
10 .206 inch, and with the cooling fins on both the stator housing
11 12 and prior sand cast stator frames of comparable bore size
12 being spaced apart a nominal distance of approximately .75 inch,
13 the increased ratio of effective cooling fin height to diameter
14 for stator housing 12 results in an increase in cooling or heat
15 transfer surface area of approximately 20-25 percent over prior
16 sand cast stator housings of the same nominal axial bore size.
17 The increased heat transfer surface area due to an increased
18 effective ratio of cooling fin height to diameter ratio for
19 stator housing 12 has resulted in an increased cooling or heat
20 dissipation rate (watts/°C) of approximately 4-8 percent, as
21 tested per IEEE 112 B.

22 Referring to FIG. 4, the longitudinal cooling fins 24
23 extend along substantially the full longitudinal length of the
24 annular wall 14 of the stator housing 12. Preferably, a slightly
25 increased diameter or radial flange is formed at each end of the
26 annular wall 14, such as indicated at 34a and 34b. The flanges
27 34a and 34b are each comprised of four arcuate segments which
28 extend about the aforescribed corresponding upper, lower and
29 opposite side quadrants of the annular wall 14. The cooling fins
30 24 extend longitudinally between the end flanges 34a and 34b.
31 The opposite ends of each cooling fin 24 are recessed, such as
32 indicated at 36 in FIG. 4, to accommodate and pilot a generally
33 annular edge of a cooling fan shroud or shell (not shown) when
34 mounted on a completed fan-cooled motor employing the stator
35 housing 12. By providing substantially similar recessed surfaces
36 36 on opposite ends of each of the cooling fins, the cooling fan

1 shroud and associated fan blade assembly may be mounted at either
2 end of the stator housing 12.

3 The increased height longitudinal cooling fins 24
4 formed on the stator housing 12 enable the annular wall 14 to
5 have a thinner radial thickness along substantially its full
6 length then has heretofore been obtainable with similar size
7 stator housings having similar horsepower ratings and made by
8 prior sand casting processes. By forming the increased height
9 cooling fins integral with the annular wall 14, the cooling fins
10 add rigidity to the thinner annular wall throughout substantially
11 its full length so that the overall rigidity and strength of the
12 resulting housing 12 is equal to or greater than prior sand cast
13 stator housings or frames having substantially the same nominal
14 size stator core cavities or bores and horsepower rating. A
15 ratio of effective cooling fin height to annular wall thickness
16 of at least approximately 200, and in the range of approximately
17 200-350, is advantageous for a stator housing having the
18 aforementioned fin size and wall thickness and an axial stator
19 core bore of approximately fifteen inches diameter. The
20 effective ratio of cooling fin height to annular wall thickness
21 is defined as the ratio of the sum of the heights of all of the
22 fins 24 to the radial thickness of the annular wall 14 at the
23 longitudinal center of the stator housing 12. The thinner radial
24 thickness of annular wall 14 enabled by the greater height
25 cooling fins 24 also increases the cooling rate through the
26 annular wall as compared to thicker annular walls that would be
27 required for comparable size stator housings made in accordance
28 with prior sand casting processes. In the illustrated and
29 described embodiment, the stator frame or housing 12 is
30 representative of stator housings which are believed to be
31 particularly well suited for stator housings or frames
32 categorized as NEMA size 320 up to 449.

33 The stator housing or frame 12 has four mounting foot
34 pads 40a, 40b, 42a and 42b formed integral with the annular wall
35 14 in pairs adjacent opposite ends of the housing. The pairs of
36 mounting foot pads 40a,b and 42a,b are substantially identical
37 in configuration so that only the mounting foot pad 40a will be

1 described in detail. Referring to FIGS. 4-7, the mounting foot
2 pad 40a includes a generally rectangular pad 44 having a lower
3 or bottom planar surface 44a which is coplanar with the
4 corresponding bottom surfaces on the other mounting foot pads 40b
5 and 42a,b. The mounting pad 44 is formed integral with the
6 annular wall 14 through a support web 46 which forms an included
7 angle of approximately 20° with a plane perpendicular to the
8 bottom surface 44a and parallel to the longitudinal bore axis 20
9 of the housing 12.

10 A feature of the stator housing 12 lies in making the
11 connecting web 46 of substantially equal thickness to the radial
12 thickness of the annular wall 14. This is particularly desirable
13 in lost foam casting where substantially equal wall thickness for
14 various portions of the casting is desired. By making the
15 connecting web 46 of generally equal thickness to the annular
16 wall 14, and with the wall 14 being thinner than prior sand cast
17 stator frames of similar size, the support strength of the
18 associated mounting pad 44 would be slightly reduced over prior
19 similar size sand cast stator housings where the thickness of the
20 connecting webs integrally connecting the mounting foot pads to
21 the corresponding annular stator walls is generally greater than
22 the annular stator wall thickness. To overcome the reduced
23 strength of the thinner connecting web 46, a plurality of
24 reinforcing ribs, such as indicated at 48a-d in FIG. 7 and at 50
25 in FIG. 5, integrally interconnect the mounting foot pad 44 to
26 the web 46 and to the annular wall 14. The reinforcing ribs
27 48a-d lie in parallel spaced planes perpendicular to the web 46
28 and extend off the backside of web 46 between the mounting pad
29 44 and the annular wall 14. The web 50 is a single web
30 substantially perpendicular to web 46 and disposed at one end of
31 the mounting pad 44 so as to enable access to two mounting bolt
32 holes 52a and 52b formed in the mounting pad 44.

33 The mounting foot pads 40a,b and 42a,b have their lower
34 or bottom planar surfaces, such as indicated at 44a, lying in a
35 common plane perpendicular to a vertical plane containing the
36 center axis 20 of the annular wall 14. Depending upon the
37 particular envelope limitations on the stator housing in its

1 finalized motor or generator assembly application, the downwardly
2 extending cooling fins 24 on the lower arcuate quadrant of the
3 annular housing 14 may be of substantially identical height.
4 Alternatively, the downwardly extending longitudinal cooling fins
5 24 generally adjacent the center of the lower quadrant of the
6 annular wall 14 may be made of a shorter height such that none
7 of the downwardly extending cooling fins extend below the plane
8 of the coplanar bottom mounting surfaces on the mounting foot
9 pads 40a,b and 42a,b.

10 As briefly described, the stator housing or frame 12
11 is made by a lost foam casting process which enables precise
12 manufacture of the housing 12 in cast iron or aluminum. A
13 feature of making the stator housing 12 by the lost foam method
14 is that the finished stator housing is substantially similar in
15 appearance when made of either aluminum or cast iron. The lost
16 foam process is generally known and includes the various
17 aforescribed process steps. A significant advantage that can
18 be obtained from using a lost foam process as compared to prior
19 sand casting processes for manufacturing stator housings is
20 significant cost savings which may be realized through
21 substantially reduced material waste and through reduced
22 machining requirements. The applicant has determined that the
23 precise tolerance limits obtainable with the lost foam process
24 can lead to greatly reduced machining requirements and thus
25 substantially reduced labor costs associated with producing a
26 finished stator housing or frame. Of significant importance is
27 the fact that a lost foam process facilitates manufacture of the
28 stator housing 12 with a thinner annular wall 14 about the stator
29 core receiving bore than has been obtainable with prior sand cast
30 stator housings of comparable size and horsepower rating. The
31 thinner wall, coupled with the ability to provide longitudinal
32 cooling fins which establish a greater ratio of effective cooling
33 fin height to annular wall outer diameter, and a greater ratio
34 of effective cooling fin height to annular wall thickness than
35 heretofore obtained with sand cast stator housings or frames,
36 results in a lighter weight stator housing having substantially

1 equal or improved strength and rigidity over stator housings made
2 in accordance with prior sand casting processes.

3 It is emphasized that synergy between the cooling fins
4 24 and stator housing annular wall 14 permits the use of fins of
5 greater height (i.e., generally radial length) and a thinner
6 annular wall which results in a reduction in the total amount of
7 raw material used. Thus, were it not for the structural
8 reinforcement provided by the fins, a thicker annular wall would
9 be required for strength purposes. However, adding a small
10 amount of material to make the fins of greater height or outward
11 length permits an even greater reduction in material in the
12 annular wall, and the thinner annular wall and greater height
13 fins both contribute to improved thermal performance (as well as
14 reduced total material consumption).

15 FIG. 3 illustrates a pattern, indicated at 12', made
16 of a vaporizable polymeric material, such as fused small
17 polystyrene plastic beads having a non-vaporizable but gas
18 permeable coating. The pattern 12' is an integral single piece
19 pattern which contains no parting lines or parting planes as in
20 prior known patterns for making motor stator housings or frames.
21 A significant benefit of the unitary single-piece pattern 12' is
22 that it duplicates the aforescribed stator housing 12 in
23 substantially identical dimensional configuration so that the
24 resulting casting requires minimal machining while obtaining the
25 aforescribed radial wall thickness and higher effective cooling
26 fin height to diameter and annular wall thickness ratios. This
27 increases the heat transfer surface area for cooling purposes and
28 adds rigidity to the thinner annular wall 14 of the stator frame
29 housing 12. The lost foam process may be used for various size
30 stator housings or frames, but, as noted, finds particular
31 beneficial application with NEMA 320 up to 449 size frames. The
32 lost foam process also significantly reduces the time involved
33 from start-to-finish in making the stator housings or frames over
34 prior sand casting processes.

35 The integral single-piece pattern 12' is configured to
36 form an integral conduit box mounting or support pad on the
37 stator housing 12, as indicated at 60 in FIGS. 4-6. The conduit

1 box mounting pad 60 has peripheral horizontal and vertical
2 support walls 62a-d which are integral with the annular wall 14
3 of the stator frame housing 12 so that an internal cavity 64
4 within the conduit box mounting pad is in open communication with
5 the axial bore 16. The cavity 64 is bounded on its outer surface
6 by a generally vertically disposed wall 66 having a circular
7 opening 66a to facilitate entry of electrical conductors
8 internally of the mounting pad 60 for connection to internal
9 windings of a stator core. The wall 66 also has four holes 66b
10 spaced about the circular opening 66 to facilitate attachment of
11 a connector plate or conduit box or the like.

12 By making the stator housing or frame 12 by the lost
13 foam process, the peripheral walls of the conduit box mounting
14 pad 60 may be made thinner than previously obtainable with
15 conduit box support or mounting pads on sand cast stator housings
16 or frames, thereby providing greater flexibility with
17 substantially reduced possibility of cracking or other fatigue
18 stress which may result with sand cast stator housings.

19 The lost foam pattern 12' is also configured to form
20 cast upstanding lift bosses 70a and 70b on the stator housing 12
21 which extend upwardly from laterally opposite sides of the
22 housing intermediate its length. The lift bosses 70a,b are
23 formed integral with the annular wall 14 and have suitable
24 openings 72a and 72b, respectively, formed therein to receive
25 lift hooks or the like to facilitate lifting and handling of the
26 stator housing 12 as well as a completed dynamoelectric machine
27 10 such as depicted in FIG. 8.

28 As aforescribed, by making the stator housing 12 of
29 the dynamoelectric machine 10 by the lost foam process, the axial
30 bore 16 may be formed with close dimensional tolerances so as to
31 enable assembly with the stator core 18 with minimal machining
32 of the bore 16. Referring to FIG. 6, in the illustrated
33 embodiment the stator core 18 includes a wound core 78 of
34 conventional design. The core 78 has an outer cylindrical
35 surface 78a which facilitates cold-press assembly into the close
36 tolerance bore 16 so as to maximize heat transfer from the stator
37 core to the stator housing 12. A rotor 80 is rotatably supported

1 within an axial bore 78b in the wound core 78 by a rotor shaft
2 82 which in turn is rotatably supported within suitable bearings
3 (not shown) carried by the end frames or shields 20a and 20b
4 (FIG. 8) as is known.

5 While a preferred embodiment of a dynamoelectric
6 machine employing a stator housing or frame in accordance with
7 the present invention, and the lost foam process for making the
8 stator housing, have been illustrated and described, it will be
9 understood that changes and modifications may be made therein
10 without departing from the invention in its broader aspects.
11 Various features of the invention are defined in the following
12 claims.

What Is Claimed Is:

- 20 1. A method for making an integral single piece stator
frame for use in a dynamoelectric machine, wherein the
stator frame comprises a housing having a generally annular
wall defining an axial bore and having a plurality of
external longitudinally extending cooling fins integral with
25 the annular wall, said method comprising the steps of;
- a. forming an integral single piece vaporizable
pattern having a configuration substantially identical to
the stator frame and being capable of vaporization when
contacted by a molten metal;
 - 30 b. forming a coating on the full surface of the
pattern which is gas permeable but will not vaporize when
contacted by said molten metal,
 - c. burying the coated pattern in sand in a manner
enabling molten metal to be poured into contacting relation
35 with the pattern,
 - d. pouring a sufficient quantity of molten metal into
contacting relation with the pattern to vaporize the pattern
and fill the resulting void with molten metal so as to
define a molten metal annular wall and external longitudinal
40 cooling fins establishing a ratio of effective cooling fin
height over the outer diameter of said annular wall in the
range of approximately 4.0-6.0,
 - e. allowing the molten metal to solidify and cool to
thereby form a single piece stator frame casting, and
 - 45 f. removing the casting from the sand.
2. The method as defined in claim 1 wherein said
pattern is configured to form said cooling fins so that they

20 establish a ratio of effective cooling fin height to annular wall outer diameter of approximately 4.8.

25 3. The method as defined in claim 1 wherein said annular wall defines upper, lower and opposite side quadrants, said upper and lower quadrants having external longitudinal cooling fins lying in generally vertical substantially equally spaced planes, said opposite side quadrants having external longitudinal cooling fins lying in generally horizontal substantially equally spaced planes.

30 4. The method as defined in claim 3 wherein the cooling fins formed along at least said upper and opposite side quadrants have outer longitudinal fin edges lying on a circle concentric with the longitudinal axis of said axial bore.

35 5. The method as defined in claim 1 wherein said longitudinal cooling fins are substantially equally spaced circumferentially about said annular wall.

6. The method as defined in claim 1 wherein said pattern is configured to form a plurality of mounting foot pads integral with said annular wall.

40 7. The method as defined in claim 6 wherein said pattern is configured to form said foot pads in pairs at opposite ends of said housing, said foot pads having coplanar bottom surfaces enabling mounting of the stator frame in a predetermined orientation.

45 8. The method as defined in claim 7 wherein said pattern is configured to form integral interconnecting webs between each of said mounting foot pads and said annular wall, and a plurality of reinforcing ribs formed on opposite sides of said webs in generally perpendicular relation to said webs.

50 9. The method as defined in claim 8 wherein said

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interconnecting webs have a transverse thickness
20 substantially equal to the thickness of said annular wall.

10. The method as defined in claim 6 wherein said
pattern is configured to form at least one conduit box
support pad integral with said annular wall.

25 11. The method as defined in claim 1 wherein said
pattern is configured to form said annular wall with
outwardly tapered substantially annular opposite ends.

12. The method as defined in claim 11 wherein said
longitudinal cooling fins have opposite ends formed integral
with said outwardly tapered opposite ends.

30 13. A method for making an integral single piece
stator frame for use in a dynamoelectric machine, wherein
the stator frame comprises a housing having a generally
annular wall defining an axial bore and having a plurality
of external longitudinally extending cooling fins integral
35 with the annular wall, said method comprising the steps of;

a. forming an integral single piece vaporizable
pattern having a configuration substantially identical to
the stator frame and being capable of vaporization when
contacted by a molten metal;

40 b. forming a coating on the full surface of the
pattern which is gas permeable but will not vaporize when
contacted by said molten metal,

c. burying the coated pattern in sand in a manner
enabling molten metal to be poured into contacting relation
45 with the pattern,

d. pouring a sufficient quantity of molten metal into
contacting relation with the pattern so as to vaporize the
pattern and fully fill the resulting void,

e. allowing the molten metal to solidify and cool to
50 thereby form a stator frame casting, and

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f. removing the casting from the sand, said pattern
20 being configured to form said annular wall and said external
longitudinal cooling fins so that said fins establish a
ratio of effective cooling fin height over annular wall
outer diameter of at least approximately 4.0, and have an
effective cooling fin height over annular wall thickness
25 ratio of at least about 200.

14. A stator frame for use in a dynamoelectric
machine, said stator frame comprising an integral one-piece
cast housing having a generally annular wall of a
predetermined outer diameter and defining opposite ends
30 having end surfaces lying in planes generally transverse to
the longitudinal axis of said annular wall, said housing
having a generally cylindrical axial bore intersecting said
opposite ends and adapted to receive a stator core, and a
plurality of external longitudinally extending cooling fins
35 spaced about said annular wall and formed integral
therewith, said cooling fins being configured to
collectively establish a ratio of effective cooling fin
height to said annular wall outer diameter of at least 4.0
at approximately the longitudinal center of said frame.

40 15. A stator frame as defined in claim 14 wherein said
cooling fins extend generally horizontally outwardly from
opposite arcuate side quadrants of the annular wall, and
extend generally vertically from upper and lower arcuate
quadrants of the annular wall, all of said cooling fins
45 extending generally horizontally from said side quadrants
and extending generally vertically from said upper quadrant
being of substantially equal height and thickness.

16. A stator frame as defined in claim 15 wherein said
horizontal and vertically extending cooling fins are spaced
50 apart substantially equal distances within their respective

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quadrants of said annular wall.

20 17. A stator frame as defined in claim 15 including a plurality of mounting foot pads formed integral with said annular wall in a manner to enable support of the stator frame in a predetermined orientation.

25 18. A stator frame as defined in claim 14 wherein said annular wall has at least one conduit box mounting pad formed integral with said annular wall so as to extend outwardly therefrom.

19. A stator frame as defined in claim 14 wherein said stator frame is made by a lost foam casting process.

30 20. A stator frame as defined in claim 19 wherein said stator frame is made from cast iron or aluminum.

35 21. A stator frame as defined in claim 14 wherein said annular wall has a predetermined radial thickness, said longitudinal cooling fins establishing a ratio of effective cooling fin height to annular wall thickness of at least approximately 200.

40 22. A stator frame as defined in claim 21 wherein said effective cooling fin height to annular wall thickness ratio is in the range of approximately 200 to 350 for a stator frame having a nominal axial bore diameter of approximately fifteen inches.

45 23. A stator frame as defined in claim 21 wherein said cooling fins further establish a ratio of effective cooling fin height to annular wall outer diameter of at least approximately 4.0.

24. A stator frame as defined in claim 23 wherein said effective cooling fin height to diameter ratio is approximately about 4.2.

50 25. A stator frame for use in a dynamoelectric machine, said stator frame including a generally annular

20 wall having opposite ends and defining a generally
cylindrical axial bore intersecting said opposite ends and
adapted to receive a stator core, said annular wall having
greater radial thickness adjacent said opposite ends than
the radial thickness of said annular wall intermediate said
25 opposite ends, and including a plurality of longitudinal
cooling fins formed externally longitudinally along and
integral with said annular wall so as to extend
substantially the full longitudinal length of said annular
wall, said cooling fins having greater height along said
intermediate longitudinal length of said wall than at said
30 increased thickness ends of said wall such that the rigidity
of the intermediate length of said annular wall is
substantially enhanced by the rigidity of fins tied to the
increased thickness opposite ends of said annular wall.

26. A stator frame as defined in claim 25 wherein said
35 stator frame is formed by a lost foam casting process.

27. A stator frame as defined in claim 25 wherein said
longitudinal cooling fins establish a ratio of effective
cooling fin height to annular wall outer diameter of at
least about 4.0.

28. A stator frame as defined in claim 27 wherein said
40 cooling fins have an effective cooling fin height to
diameter ratio in the range of approximately 4.0 to 6 along
the portions of said fins adjacent said intermediate length
of said annular wall.

29. A stator frame as defined in claim 26 wherein said
45 lost foam casting process forms said axial bore to receive a
stator core in a press-fit relation with minimal machining
of said bore.

30. A stator frame as defined in claim 25 wherein said
50 reduced thickness intermediate wall portion is uniform

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between said increased thickness opposite ends of said
20 annular wall.

31. A stator frame as defined in claim 30 wherein said
opposite ends of said annular wall are defined by outwardly
tapered external surfaces.

32. A stator frame for a dynamoelectric machine, said
25 stator frame being made by a lost foam process using an
integral single-piece vaporizable pattern operative to
produce a stator frame housing having a generally annular
wall defining an axial bore and having opposite ends lying
in planes normal to the axis of said bore, a plurality of
30 external cooling fins formed integral with and extending
longitudinally of said annular wall, and a plurality of
mounting foot pads formed integral with said annular wall
and defining coplanar bottom surfaces enabling the stator
frame to be supported on a support surface in a
35 predetermined orientation, said mounting foot pads each
being integrally interconnected to said annular wall through
a connecting web extending between the corresponding foot
pad and said annular wall and lying in a plane inclined to
the bottom surface of the corresponding foot pad, and
40 reinforcing ribs formed integral with and perpendicular to
each of said connecting webs and the associated mounting
foot pad, whereby said reinforcing ribs enable said
connecting webs to be made relatively thin without
diminishing the support strength of said mounting foot pads.

45 33. A stator frame as defined in claim 32 wherein said
reinforcing ribs extend outwardly from opposite sides of
their corresponding connecting webs in normal relation
thereto.

50 34. A stator frame as defined in claim 33 wherein each
of said connecting webs has a plurality of generally

parallel spaced reinforcing ribs formed integral therewith.

20 35. A stator frame as defined in claim 34 including four of said mounting foot pads integrally interconnected to said annular wall in pairs adjacent said opposite ends of said annular wall.

25 36. A dynamoelectric machine comprising in combination, a stator housing having a generally annular wall defining opposite ends and a longitudinal cavity, a stator core within said longitudinal cavity, a pair of end frames secured to the opposite ends of the stator housing, and a rotor rotatably supported by said end frames and
30 extending coaxially through said stator core, said stator housing having a plurality of cooling fins formed integral with and extending longitudinally along the exterior of said annular wall, said cooling fins and annular wall being
35 configured to establish a ratio of effective cooling fin height to annular wall outer diameter of at least 4.0 considered at substantially the longitudinal center of said stator housing.

40 37. A dynamoelectric machine as defined in claim 36 wherein said ratio of effective cooling fin height to annular wall outer diameter is in the range of approximately 4.0 to 6.0 along at least the intermediate lengths of said cooling fins.

45 38. A dynamoelectric machine as defined in claim 36 wherein said ratio of effective cooling fin height to annular wall outer diameter is approximately 4.8.

39. A dynamoelectric machine as defined in claim 36 wherein said stator housing is made of cast metal by a lost foam process.

50 40. A dynamoelectric machine as defined in claim 39 wherein said stator housing is made by a lost foam process

using an integral unitary vaporizable pattern.

20 41. A dynamoelectric machine as defined in claim 36
wherein said cooling fins extend along substantially the
full longitudinal length of said stator housing.

25 42. A dynamoelectric machine as defined in claim 41
wherein said cooling fins and said annular wall are
configured to establish a ratio of effective cooling fin
height to annular wall thickness of at least about 200.

30 43. A dynamoelectric machine as defined in claim 42
wherein said ratio of effective cooling fin height to
annular wall thickness is in the range of approximately 200
to 350 for a stator housing having a nominal axial bore
diameter of approximately fifteen inches.

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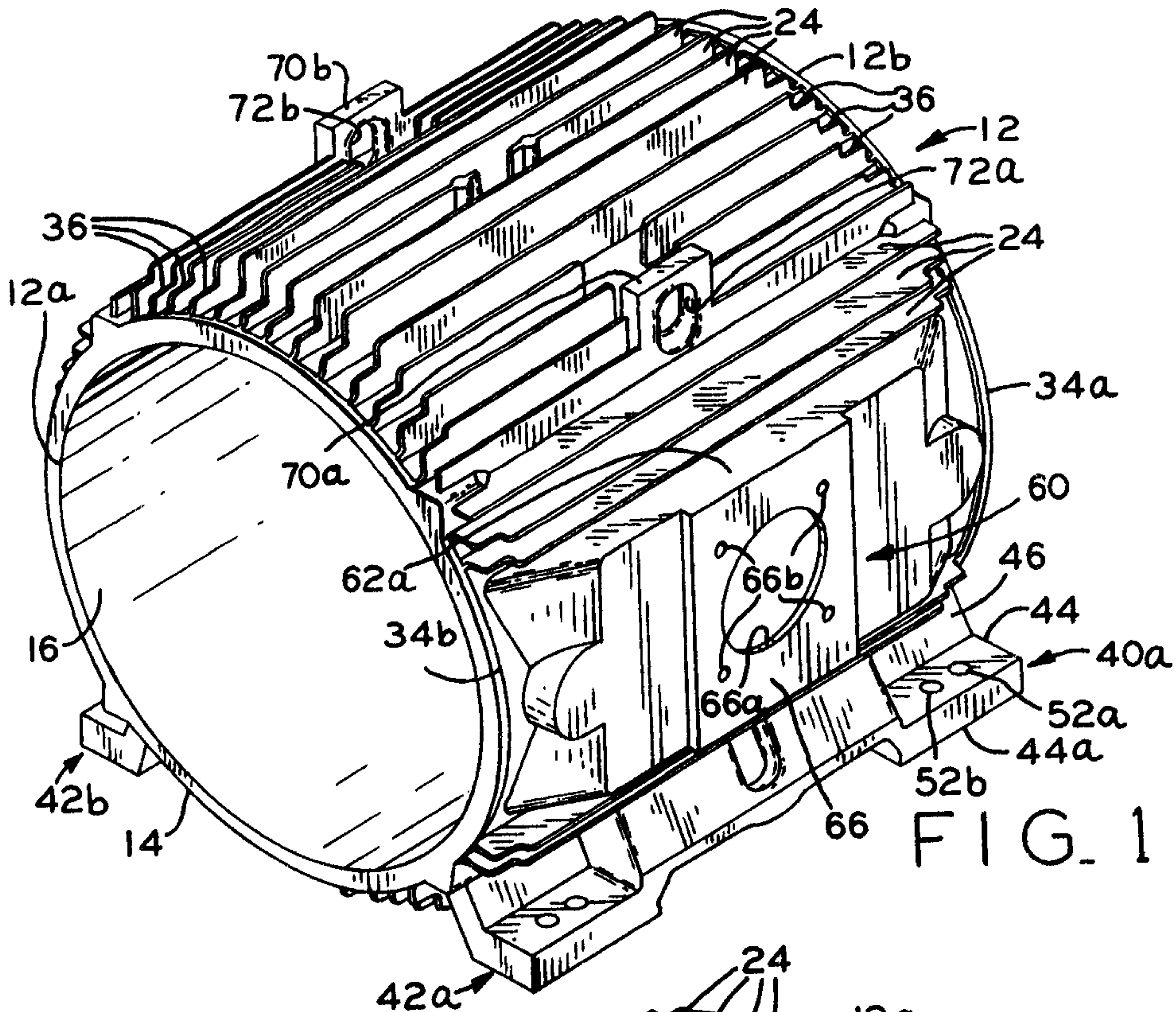


FIG. 1

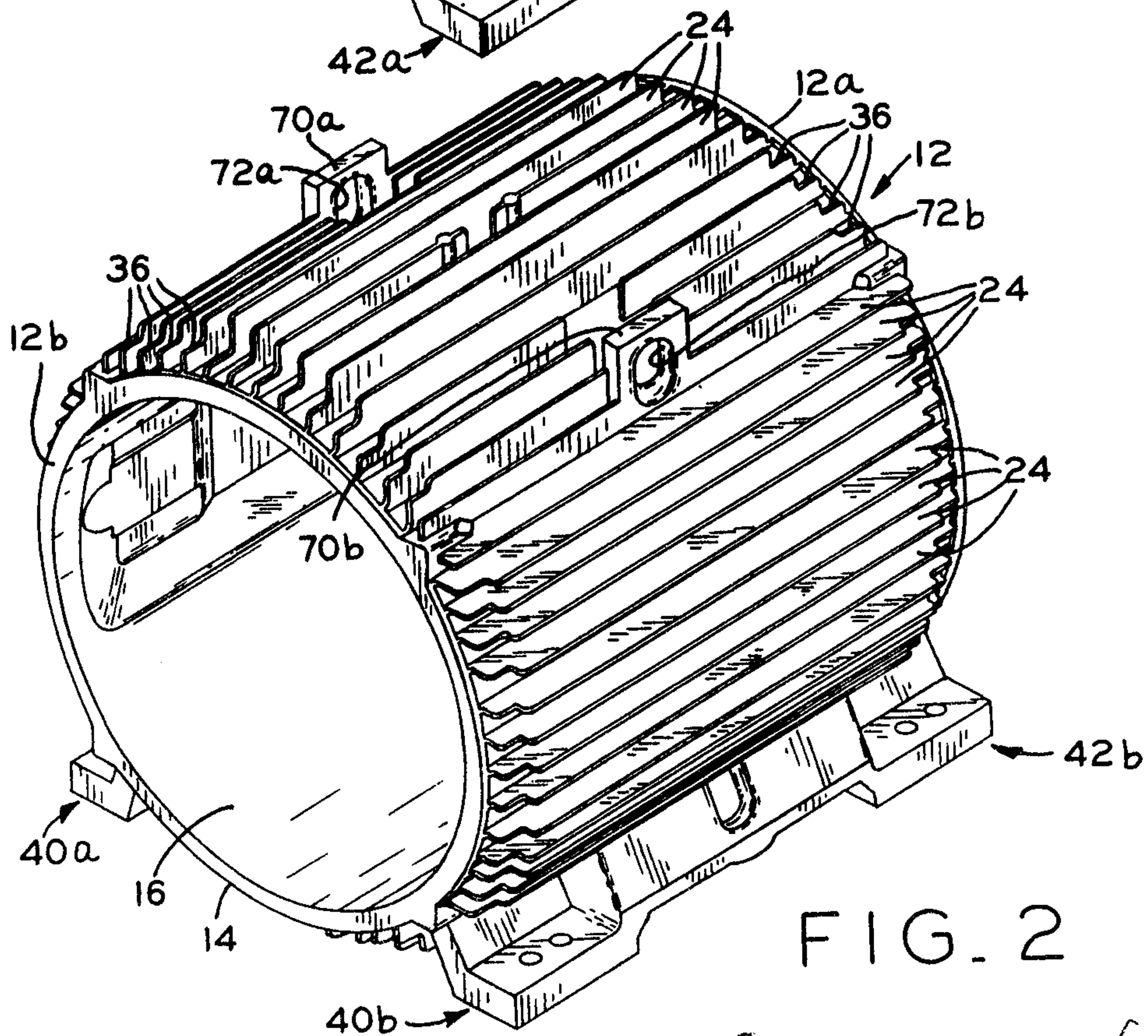


FIG. 2

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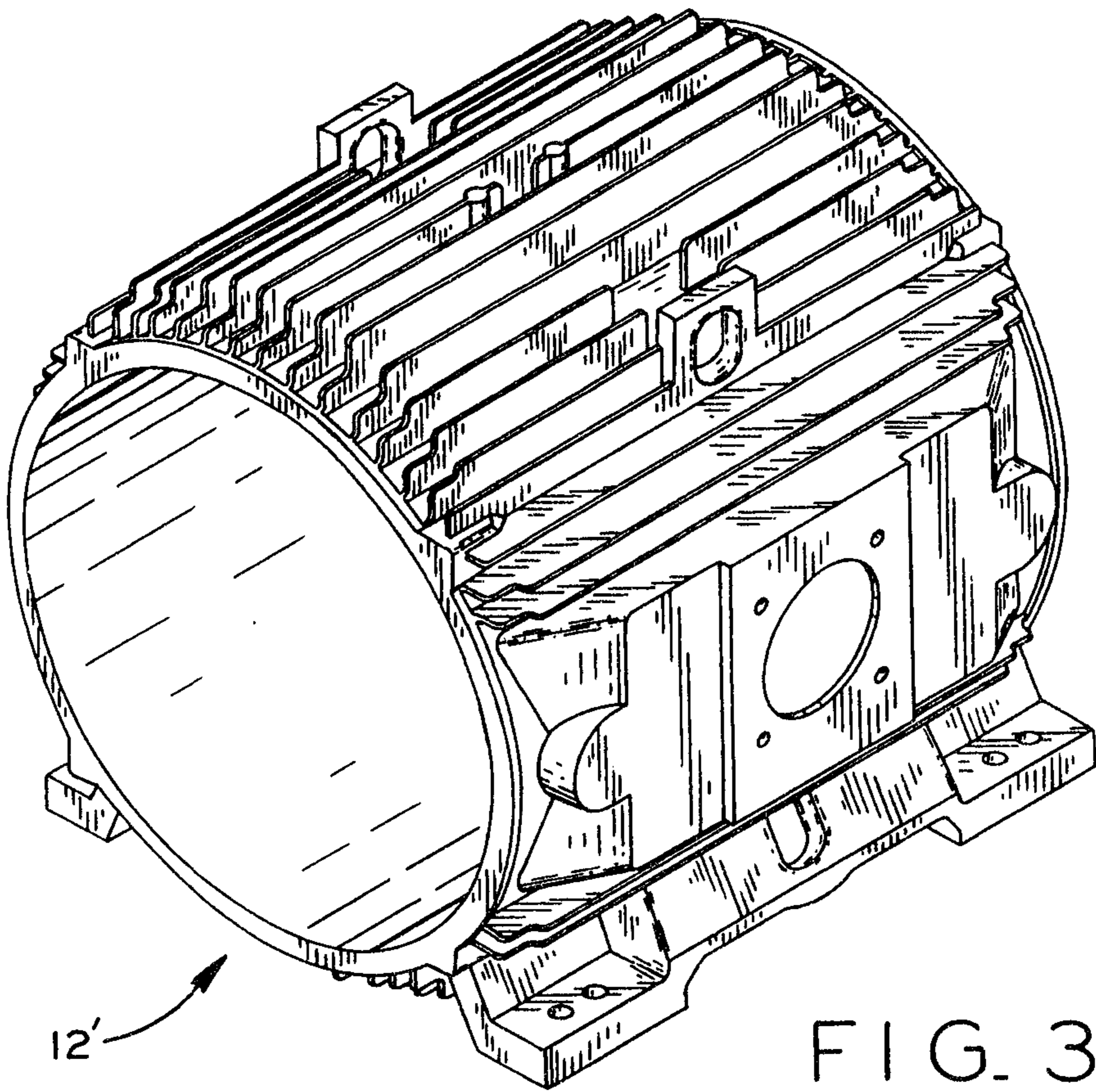
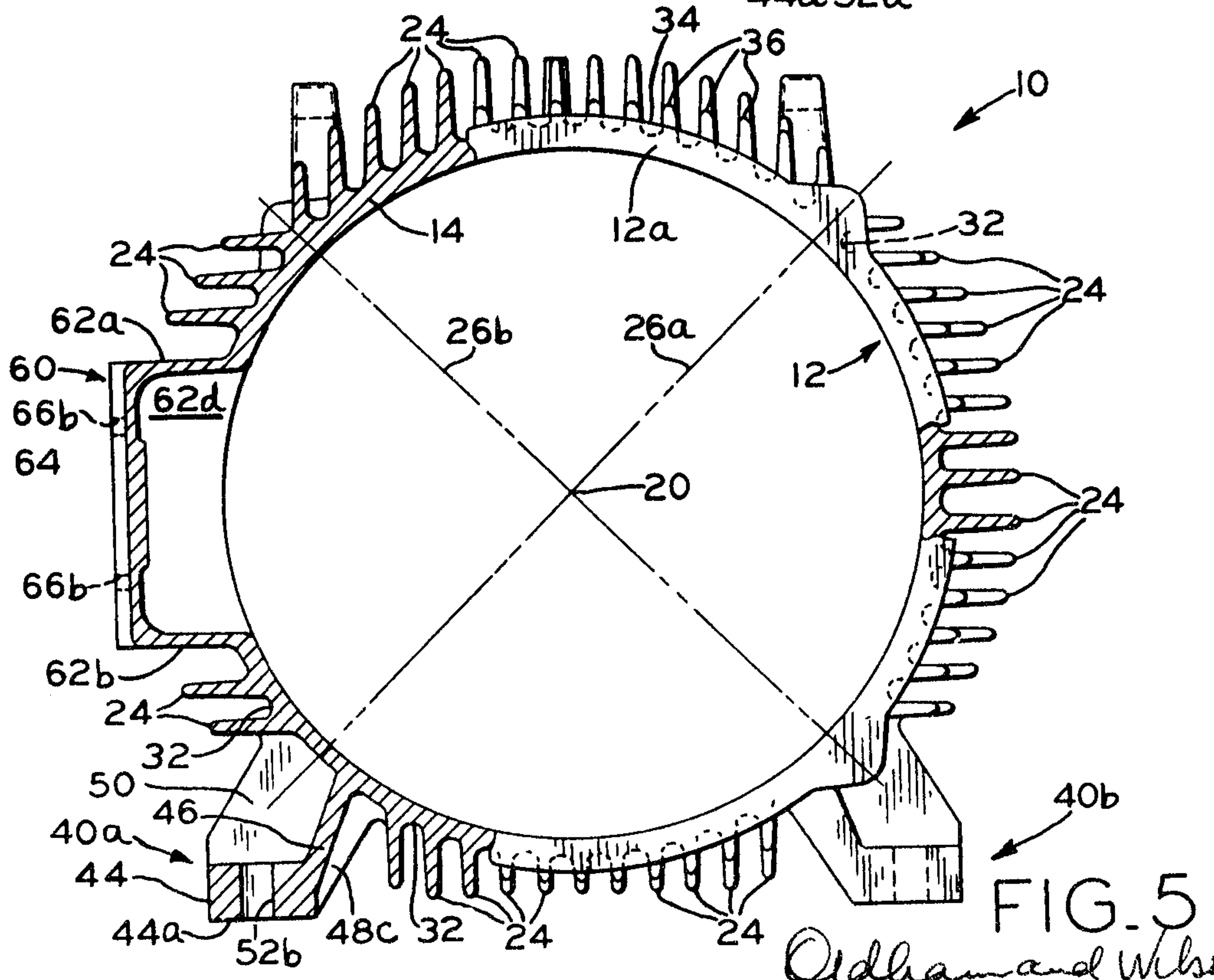
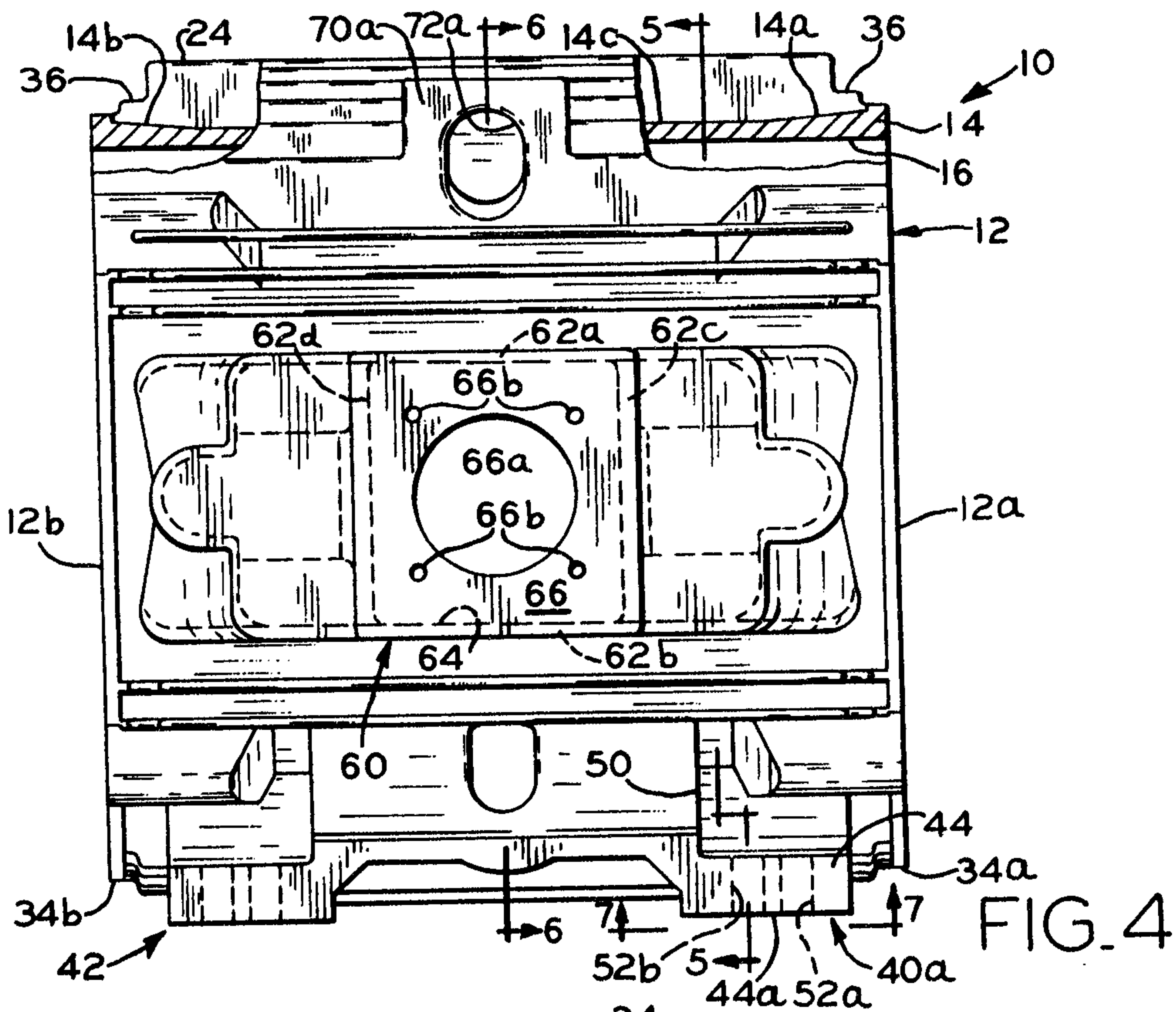


FIG. 3

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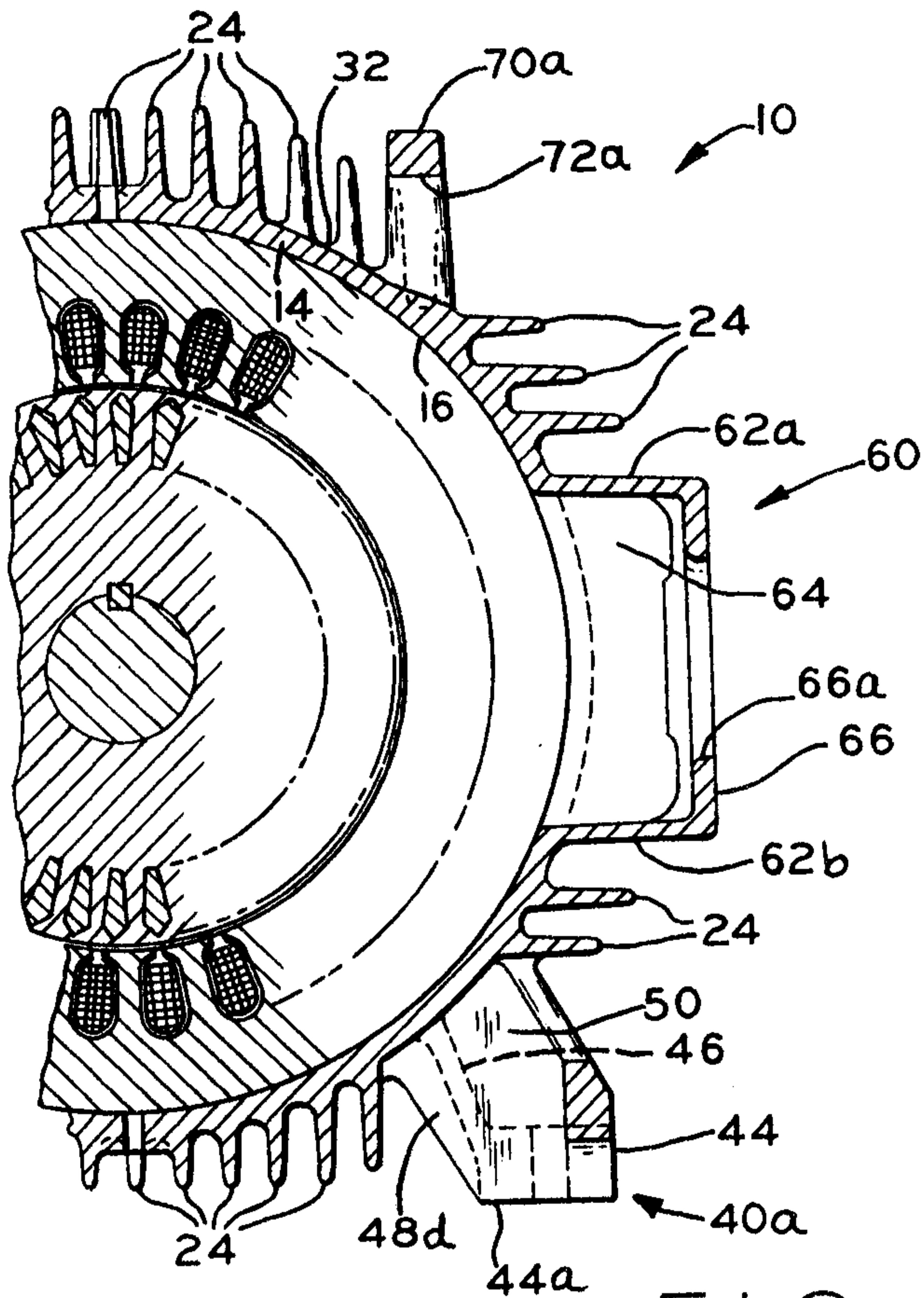


FIG. 6

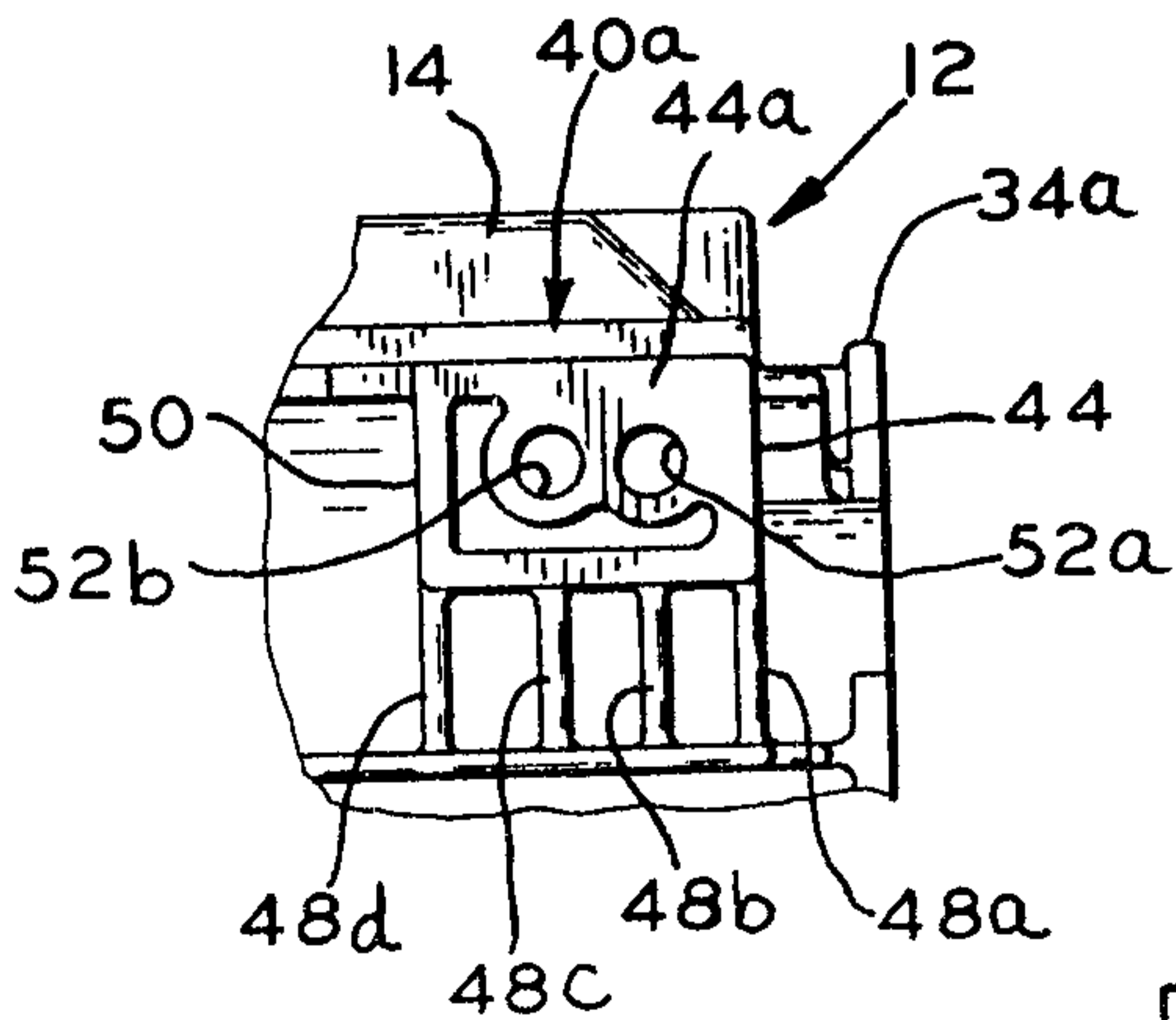


FIG. 7

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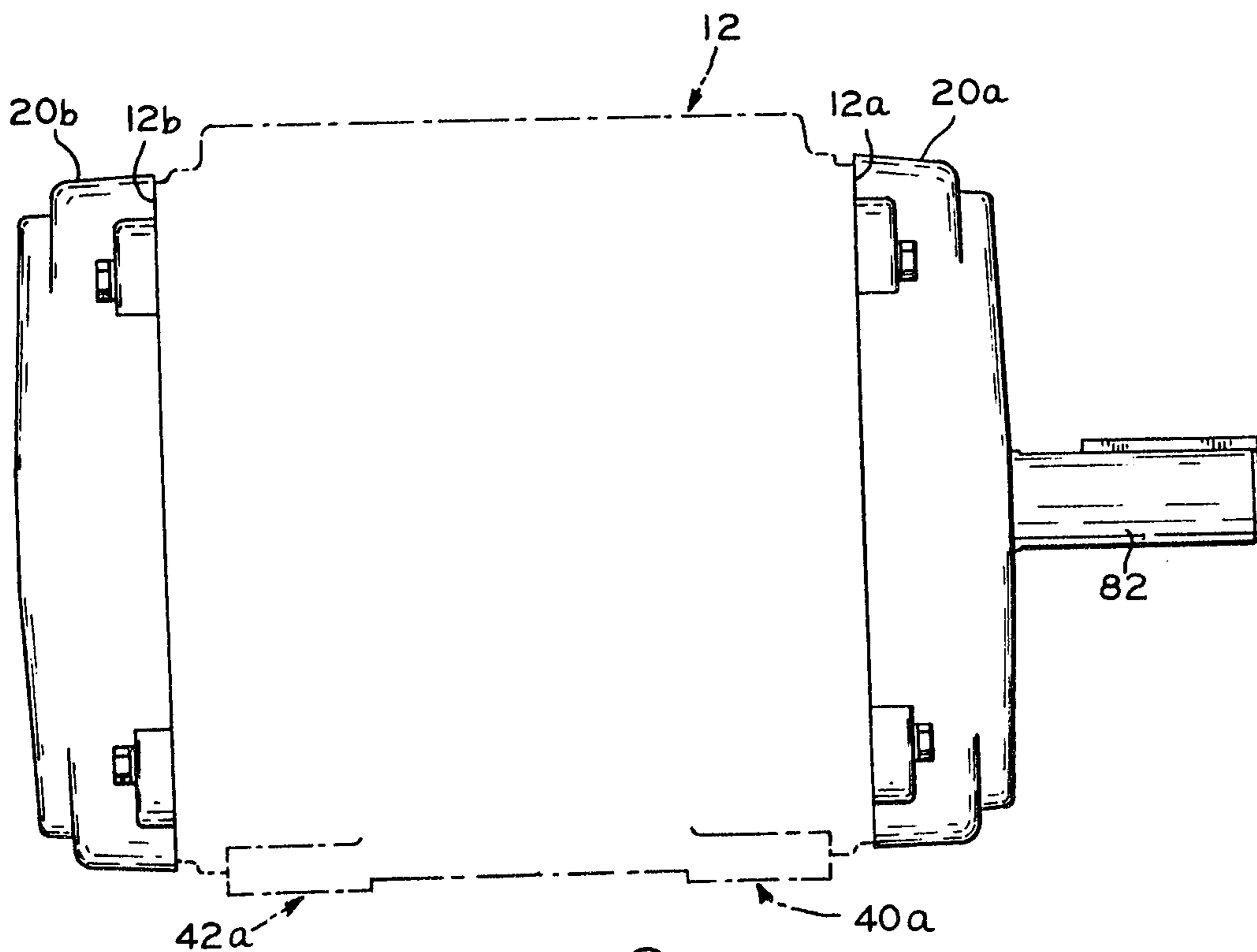
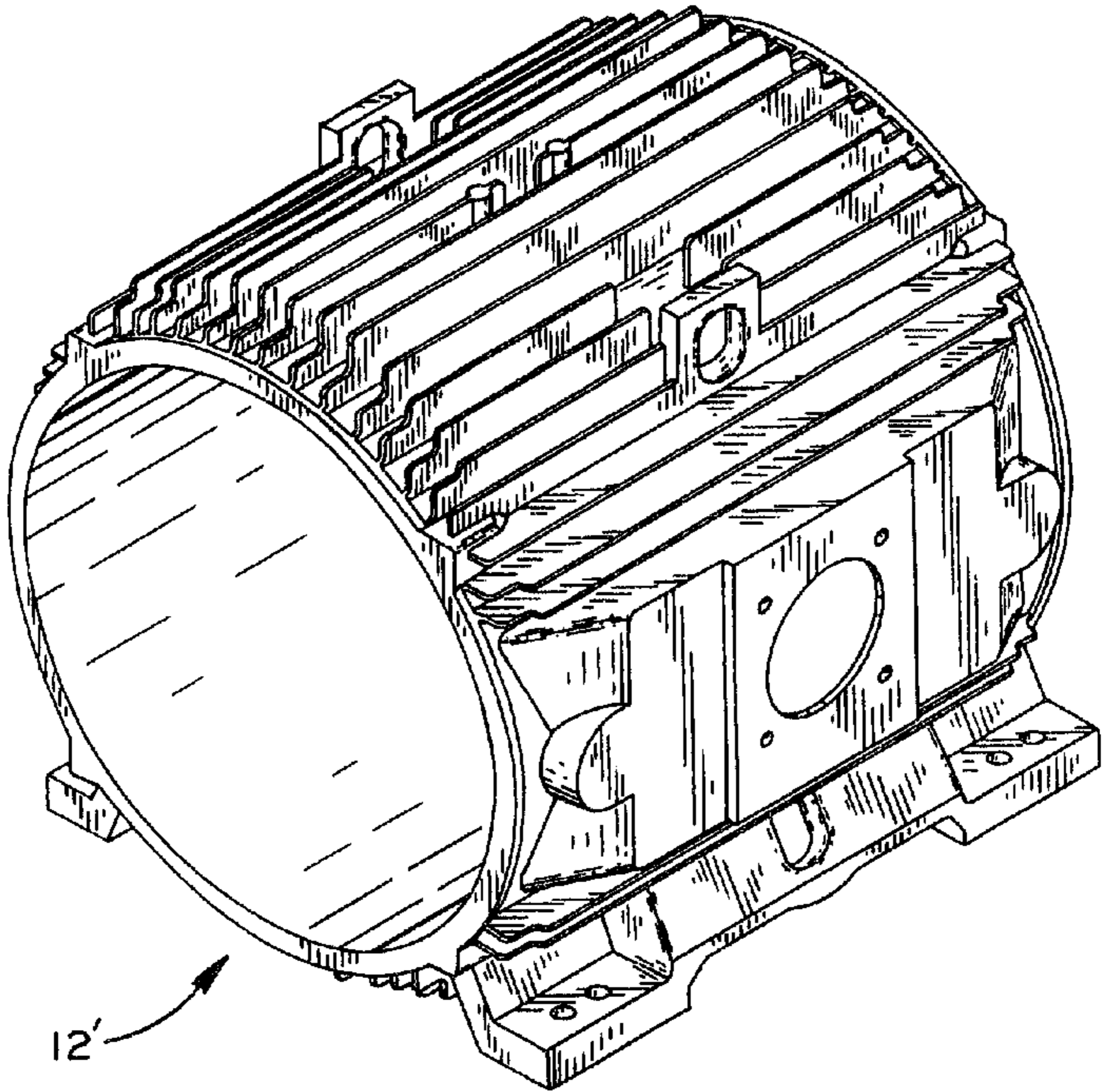


FIG. 8

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