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2,821,254

FOAM-PLASTIC SKIVING MACHINE

Filed Oct. 4, 1954

4 Sheets-Sheet 1

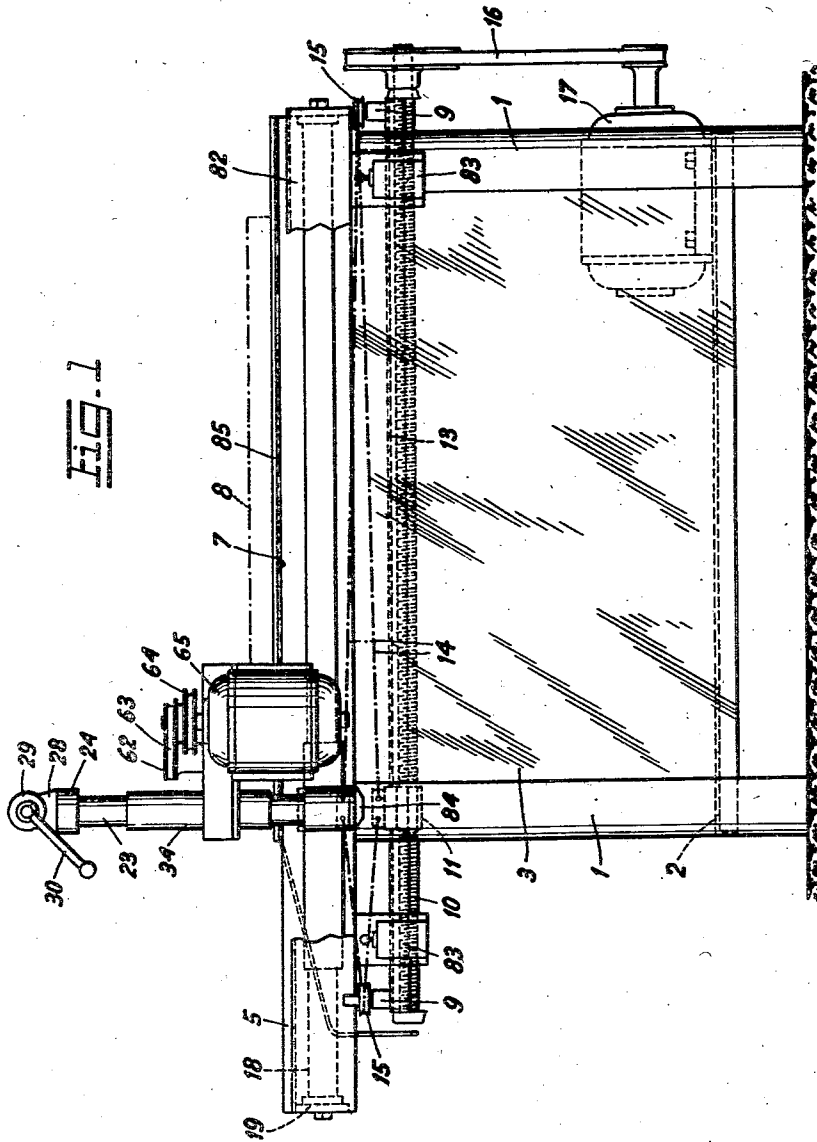


FIG. 1

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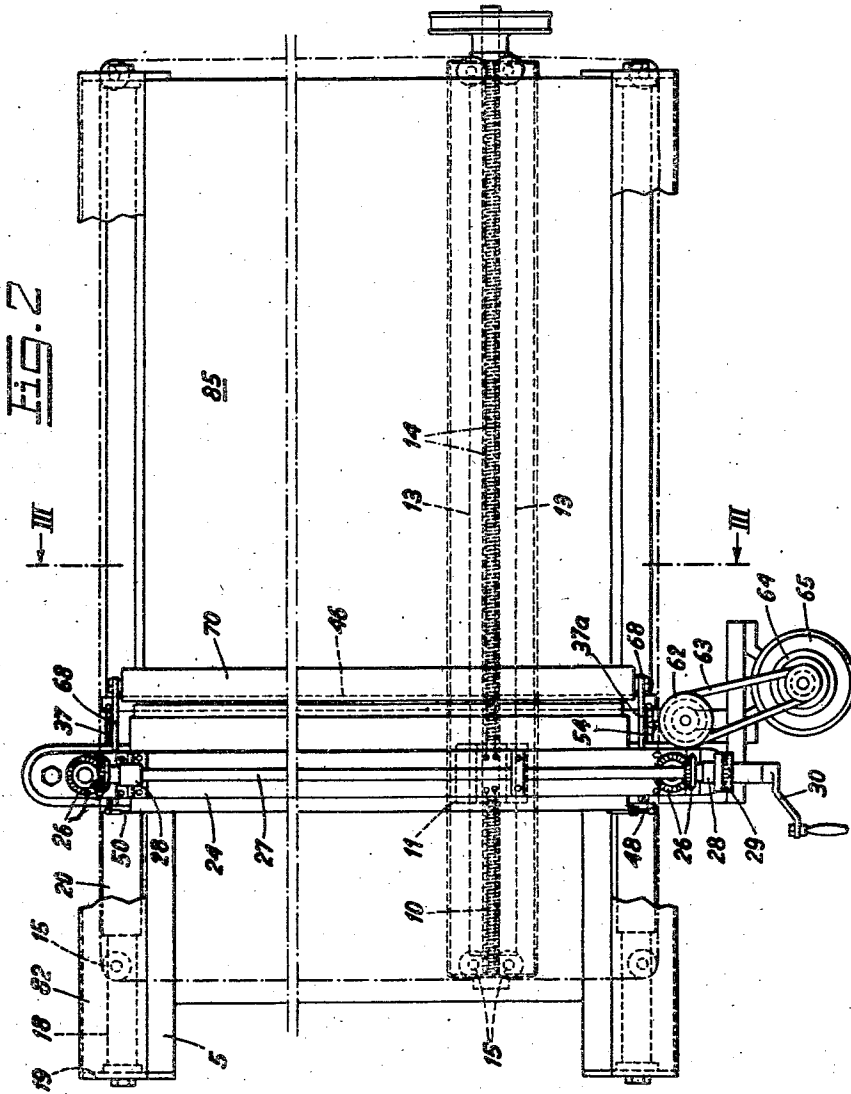
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4 Sheets-Sheet 2



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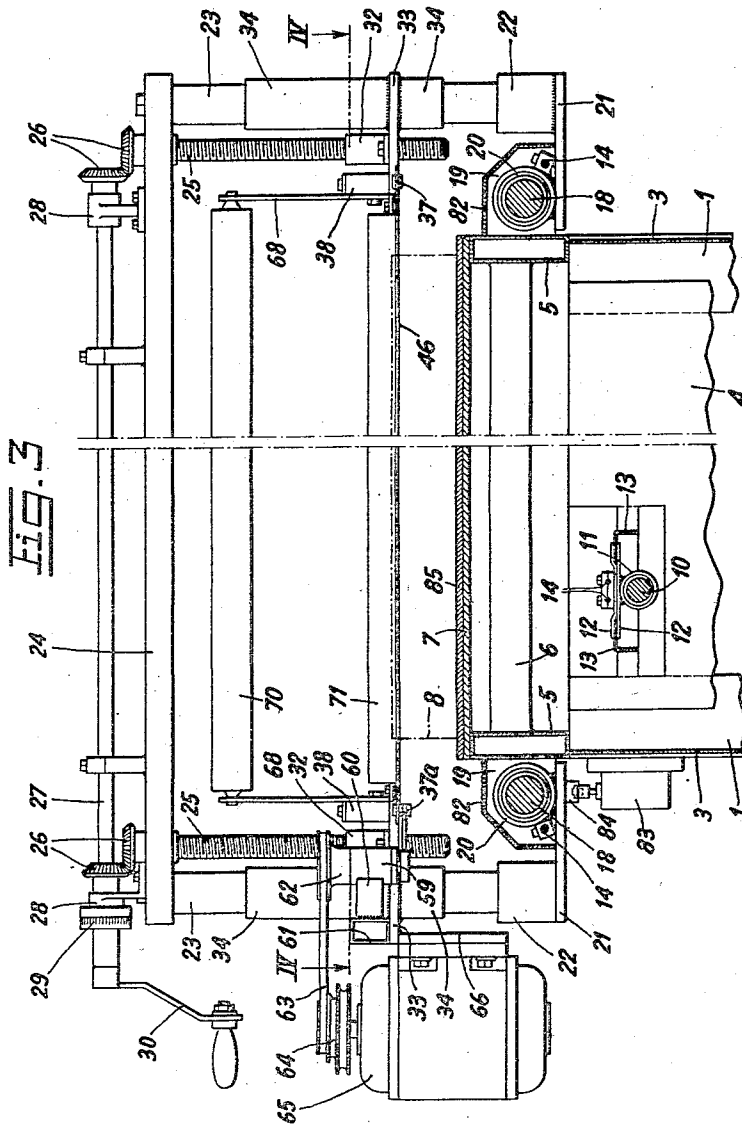
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FOAM-PLASTIC SKIVING MACHINE

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4 Sheets-Sheet 3



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FOAM-PLASTIC SKIVING MACHINE

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4 Sheets-Sheet 4

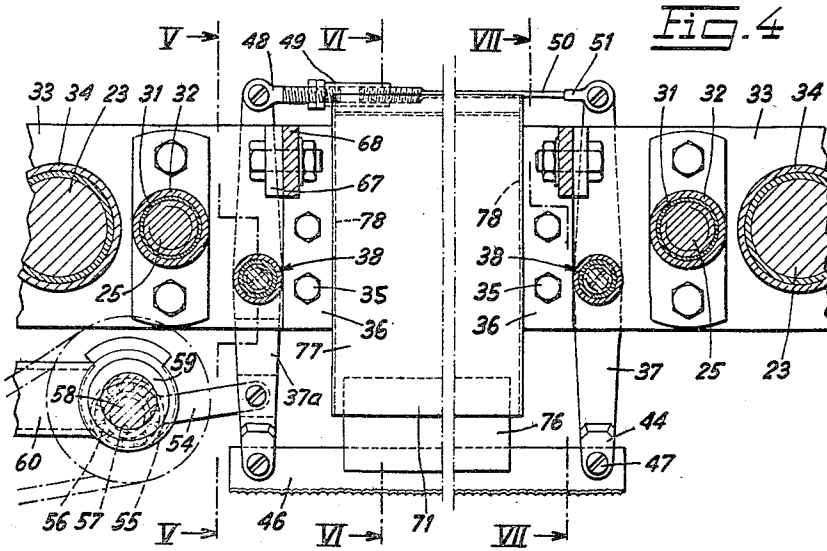


Fig. 4

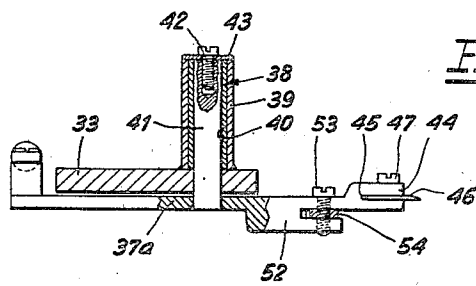


Fig. 5

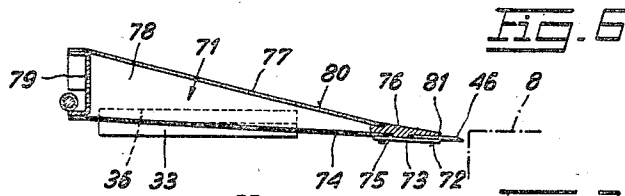


Fig. 6

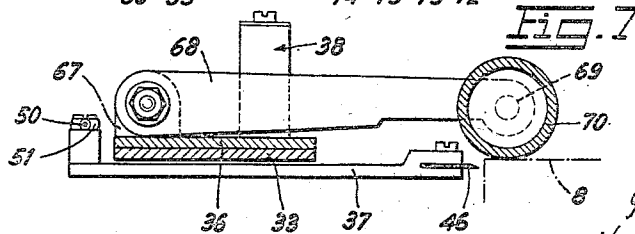


Fig. 7

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FOAM-PLASTIC SKIVING MACHINE

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Claims priority, application Switzerland October 10, 1953

9 Claims. (Cl. 164—75)

In the arts and industry there is a growing demand for foamed-plastic foils. Foamed plastic is rather expensive and is supplied by the manufacturers only in sheets of at least 3/8 inch thickness and mostly in thicknesses of one inch and more. So far it has been practically impossible to skive or cut up such sheets into foils, and in particular into foils having a smooth surface.

The main object of my present invention is to remedy such disadvantage. According to the method disclosed by my present invention, a skiver or knife having a serrated cutting edge is moved through the sheet and subjected to longitudinal oscillations with such frequency that the foamed plastic to be cut cannot resonate.

In the skiving machine disclosed herein, the longitudinally oscillatable skiver is a stretched spring-steel band of which one longitudinal edge forms the cutting edge, being ground towards the underside in serrated and pointed manner.

One form of my present invention is shown, by way of example, in the accompanying drawing which also serves to illustrate the method disclosed by the invention. In the drawings:

Fig. 1 shows the machine in elevation,

Fig. 2 is a top plan view thereof,

Fig. 3 is a cross-section on the line III—III of Fig. 2,

Fig. 4 is a horizontal section on the line IV—IV of Fig. 3, and shows, in an enlarged scale, details of the bearing and driving arrangement of the knife, and

Figs. 5 to 7 are cross-sections on the lines V—V, VI—VI and VII—VII of Fig. 4 respectively.

As shown in Fig. 1, the skiving machine comprises a stand which in the main is composed of corner posts 1, a lower frame 2, side sheets 3 and 4, two horizontal and parallel top box-girders 5, and bracing tubes 6 interconnecting the latter. To the girders 5 is secured a plate 7, e. g. by welding, which serves a table. The foamed-plastic sheet 8 to be cut into foils, either is secured to said table by adhesive means or simply laid thereon.

As shown in Fig. 3, a screw spindle 10 is rotatably but axially immovably mounted in two bearings 9 disposed on transverse members of the stand outside of the side sheets 4. Said spindle coacts with a nut 11 which by means of lateral forked wings 12 is guided on two rails 13 secured to the sheets 4. To nut 11 are secured the ends of two steel cables 14 trained over idling discs 15 mounted on the stand. An electric motor 17 mounted on frame 2 rotates spindle 10 through a belt drive 16 and, thus, moves nut 11 and the two cables 14 in one or the other sense. Outside the girders 5 and parallel thereto are arranged guide rods 18 of which the end faces through gussets 19 are connected to said girders. On rod 18 is movably mounted a sleeve 20 to which is welded a horizontal plate 21. The opposite ends of the cables are fixed to the sleeves 20, as shown in Figs. 2 and 3. Thus, when the nut is moved, the sleeves 20 and the parts carried thereby are moved. For example, when the nut 11, as shown in Figs. 1 and 2, is moved to the left, the sleeves 20 and the parts carried thereby are moved to the right. In a ring 22 welded to

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the latter is rigidly mounted a guide column 23. The two columns 23 are rigidly interconnected on top through a horizontal transverse member 24 in which two screw spindles 25 are mounted rotatably but axially immovably. The spindles 25 through two bevel gears 26 are drivably connected to a shaft 27 which is rotatable in bearings 28 fixed to member 24 and on one end of which is provided a graduated disc 29 and a crank 30. Each of the spindles 25 coacts with a threaded bush 31 rigidly inserted in a holder 32. The latter is screwed to a lateral lug 33 of a sleeve 34 which is longitudinally movable on the associated column 23, as shown in Figs. 3 and 4.

As shown in Figs. 3 and 4, lugs 36 of a knife beam 71 are inserted between the lugs 33 and secured thereto by screws 35. Lugs 33 and 36 together form a rigid carrier for two similar two-arm oscillating levers 37 and 37a of which the bearings are designated by 38. Each of said levers comprises a piece of pipe 39 welded to one of the lugs 33, a sliding bush 40 inserted in pipe 39, a bearing pin 41 welded to said lever, a screw 42 mounted and secured in pin 41, and a disc 43 supported on the upper end face of bush 40 for taking up the small axial forces. The front forked ends 44 of the levers 37, 37a are provided with a horizontal slot 45 in which is inserted one or the other end of a skiver or knife blade 46, the latter being mounted on a hardened screw 47. The rear ends of the two oscillating levers are interconnected through a stretching device comprising a threaded member 48, a threaded wire-receiving sleeve 49, a stretching wire 50 and a member 51. Lever 37a has a lug 52 to which is pivoted, on a screw 53, a bar 54 of an eccentric ring 55. The latter by means of balls 56 is mounted on the eccentric pin 57 of a shaft 58 which is rotatable in a bearing 59 which through parts 60, 61 is fixed to the adjacent sleeve 34. Shaft 58 through a belt drive 62 to 64 is connected to an electric motor 65 which is screwed down on a plate 66 welded to part 61.

As shown in Figs. 4 and 7, arms 68 are pivotally mounted in bosses 67 of the lugs 33. In bores of the latter are rotatably mounted pins 69 of a loading roller 70 which is parallel to the cutting edge of knife blade 46 and slightly in front of the latter. Roller 70 serves to press down the plastic sheet 8 on to a coating 85 of table 7, which coating increases the friction. Roller 70 in an extreme case may be situated only slightly lower with respect to knife 46 than shown in the drawing, as the arms 68 then rest on member 36.

Knife 46 is reciprocated at least 800 times per minute by motor 65 through the eccentric drive 55, 57 and lever 37a, and is made of a spring-steel band of which the longitudinal edge forming the cutting edge is serrated and ground sharply towards the underside in a manner similar to that of bread knives. The rear longitudinal edge of the knife moves in a guide formed on knife beam 71 of which the underside 72 is parallel to the knife underside, the spacing not exceeding the knife thickness. Such spacing corresponds to the thickness of a sheet 73 which at 75 is riveted to another horizontal sheet 74. The rear and top portion of said guide is formed on a solid constructional member 76 of triangular cross-section. To stepped surfaces at the rear of member 76 are brazed the lower and upper sheets 74 and 77 respectively, of a box-like structure to the somewhat thicker sidewall sheets 78 of which are welded the lugs 36. Said structure at the rear is braced by a sheet-iron member 79 of U-shaped cross-section. Fig. 6 further shows that the knife-beam upperside 80 rises rearwardly from a front edge 81 which is spaced from knife-beam upperside 80 by one to two knife thicknesses, first only at a slight slope and then somewhat steeper.

The carriage by the sleeves 20 is guided on the rods 18 which over their entire length are protected from impuri-

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ties through cover plates 82 which are secured to the plates 19. For the operation of cutting off foils from the sheet 8 stuck to table 7, the carriage is run back until the cutting edge of knife 46 is situated in front of sheet 8 (Figs. 6 and 7). By actuating crank 30, the foil thickness is set, whereupon a foil may be cut off when motor 65 is operating.

At the end of the forward and rearward runs of the carriage, motor 17 is reversed or stopped through limit switches 83 (Fig. 3) secured to the stand and through cams 84 secured to the carriage.

Tests have shown that with the machine disclosed, foils of very good surface property and minimal thickness of approximately $\frac{1}{32}$ of an inch may be cut off from foam-plastic sheets up to $3\frac{3}{4}$ inches thick, provided that the frequency of the knife oscillations is sufficiently higher than the frequency at which the surface of sheet 8 could resonate. A frequency of 800 per minute is quite sufficient in most cases. In order to diminish the foil surface roughness, it is advisable to make the tooth back somewhat rounded, i. e. to avoid burrs on the ground face of the knife serrations. The machine shown and described is only an example which in many details could be varied without leaving the scope of the invention.

What I claim as new and desire to secure by Letters Patent is:

1. A method of cutting off foils from a foam-plastic sheet, comprising the steps of supporting the sheet in a flat superposed position on a firm supporting surface, advancing a knife having a serrated cutting edge longitudinally of the surface and through said sheet and of simultaneously subjecting said knife to longitudinal oscillations with a frequency of such rate that the sheet to be cut cannot resonate.

2. A method as set out in claim 1, in which the frequency of said longitudinal oscillations is at least 800 per minute.

3. A method as set out in claim 1, in which the sheet portion situated in front of the cutting edge is held down by a slight, non-deforming pressure on the surface.

4. A machine for cutting off foils from a foam-plastic sheet comprising a horizontal supporting table on which the sheet is laid flat, a vertically adjustable frame transversely overlying the table, a spring steel cutting knife carried by the frame and positioned transverse of the sheet, said knife having a longitudinal cutting edge facing the sheet, means for longitudinally oscillating the knife,

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guide means in which the opposite longitudinal side edge of the knife is guided as the knife is longitudinally oscillated and means for moving the frame longitudinally of the table to advance the knife through the sheet.

5. A machine as claimed in claim 4, wherein the cutting edge is serrated and sharply ground towards the knife underside.

6. A machine as claimed in claim 4, wherein the guide is carried by the frame and has a slot in which the opposite longitudinal edge of the knife slides, the guide consisting of a knife beam having a flat underside which is spaced from the underside of the knife by an amount less than the thickness of the knife.

7. A machine as claimed in claim 4, wherein the guide is a knife beam which has a slot in which the opposite longitudinal edge of the knife slides, the knife beam having an upper surface which slopes rearwardly from the knife, the front edge of the beam, in which the slot is formed, being a distance of from one to two knife thicknesses above the knife.

8. A machine as claimed in claim 4, wherein said means for longitudinally oscillating the knife includes a pair of spaced apart, parallel horizontal arms, disposed transversely of the ends of the knife, means connecting the ends of the knife to one of the ends of the arms, axially adjustable stretching means interconnecting the opposite ends of the arms to stretch the knife, means mounting the arms, intermediate their ends, for pivotable movement about vertical axes, and power means connected to one of the arms for oscillating the arms.

9. A machine as claimed in claim 8, wherein said last means includes a drive motor, an eccentric driven thereby and a link connection between the eccentric and one of the arms.

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